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# **Prediction and Verification of Creep Behavior in Metallic Materials and Components for the Space Shuttle Thermal Protection System**

## **VOLUME III**

### **Phase III — Full Size Heat Shield Data Correlation and Design Criteria**

**AUGUST 1975**

*Prepared By* **B. A. Cramer and J. W. Davis**

**MCDONNELL DOUGLAS ASTRONAUTICS COMPANY - EAST**

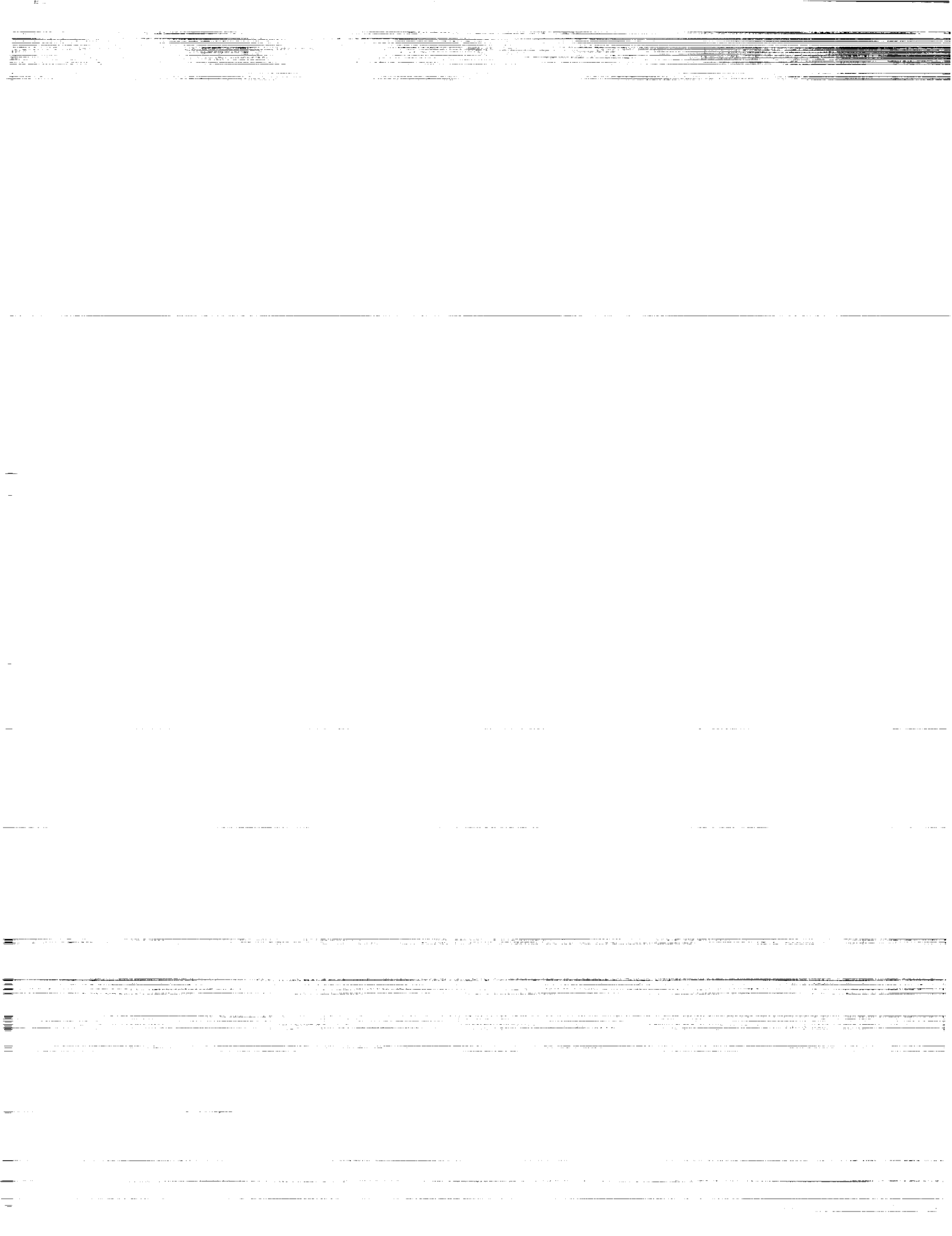
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**Phase III — Full Size Heat Shield  
Data Correlation and Design Criteria  
AUGUST 1975**

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**Prepared under Contract NAS 1-11774  
Prepared by McDonnell Douglas Astronautics Company-East  
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for National Aeronautics and Space Administration  
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ABSTRACT

Analysis methods, developed in Phase II, for predicting cyclic creep deflection in stiffened panel structures, were applied to full size panels. Results were compared with measured deflections from cyclic tests of thin gage L605, Rene' 41, and TDNiCr full size corrugation stiffened panels for which data were available in the literature. Empirical equations used in the analysis were developed for each material based on correlation with tensile cyclic creep data during Phase I of the program.

Based on results from the study, a design criteria is formulated for metallic TPS panels subjected to creep. This criteria addresses TPS design considerations, data requirements for creep analysis, and creep deflection analysis. Also included in this report are the users' information and listing for the TPSC (Thermal Protection System Creep) Computer Program developed to calculate creep deflections.

## FORWARD

This report was prepared by McDonnell Douglas Astronautics Company - East under Contract NAS 1-11774 for the National Aeronautics and Space Administration, Langley Research Center, Hampton, Virginia. It was administered under the direction of the Materials Division, Materials Research Branch, with Mr. D. R. Rummler acting as the technical representative of the contracting office. The McDonnell Douglas program manager was Mr. J. W. Davis. Mr. B. A. Cramer was responsible for structural considerations, analytical methods, and data analysis.

The TPSC Computer Program analysis approach was initiated in the MDAC-E Advanced Structural Technology Group by Mr. O. R. Otto and Mr. J. K. Lehman. Mr. B. A. Cramer was responsible for development of the TPSC program under the contract. Mr. M. B. Gedera assisted in programming the TPSC program.

This report covers the period from December 1974 to April 1975.

i. SUMMARY

Presented in this report are the results of both the Phase III and Phase IV contract phases. Phase III was directed at correlating results of full size panel cyclic testing based on material cyclic creep response behavior determined in Phase I studies (Reference 1) and analysis capability developed in Phase II (Reference 2). Full size panel data for this effort, were obtained from the literature. Phase IV effort was directed at summarizing program results into a TPS panel design criteria. Phase III is presented in Sections 2 through 4 of the report and Phase IV is presented in Section 5. The Users information and listing for the TPSC (Thermal Protection system Creep) Computer Program developed during Phase II are presented in Appendices B and C respectively.

Comparisons of predicted and test deflections are presented for L605 panels and Rene' 41 panels tested at McDonnell Douglas Corporation and for Haynes 25 (L605) and TDNiCr panels tested at Grumman Aerospace Corporation.

Resulting predictions for the L605 and Rene' 41 panels provided good correlation with test results. For both materials there was approximately a factor of two difference between test deflection results for two identical panels tested simultaneously. No explanation for this difference could be determined. Predictions for these panels were made both at the center, where temperatures were highest and at the panel transverse edges where temperatures were somewhat lower. For the L605 panels the predicted center deflections were approximately 20% less than the lowest panel test deflection agreed closely with the average test deflections. For the Rene' 41 panels the predicted deflection was very close to the higher of the test deflections at the panel center. Sensitivity of predicted deflections to variations in material gage and test temperature was demonstrated for the Rene' 41.

Predictions for the Haynes 25 panel and TDNiCr panel were both low in comparison to test deflections although considerable variation was evident in the data measured for four spans on the TDNiCr panel and the prediction was within the data range. This variation in test data could not be accounted for by temperature variations in the panels. The trend in the prediction relative to test deflections as a function of cycle for TDNiCr was shown to be consistent with prediction capability of the empirical cyclic creep equation. Test deflections for the Haynes 25 panel were two times higher than predictions.

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1.0 INTRODUCTION

One of the design requirements of reentry vehicle metallic thermal protection systems (TPS) is that deflections, occurring during ascent and entry mission phases, due to differential pressure and thermal loading, do not exceed design limits. These deflection limitations are established to maintain the aerodynamic shape, minimize localized aerodynamic heating and to minimize the need for panel refurbishment. Because deflections include permanent deformation due to creep, methods for predicting these deformations are needed.

Several arrangements of metallic TPS components have been investigated in the courses of previous spacecraft studies. The baseline design used in the McDonnell Douglas Phase B Shuttle Study Program is shown in Figure 1-1. Radiative metallic panels form the outer moldline. These panels are backed by fibrous insulation blankets. Differential air pressure loads on the panels are transmitted by beam bending to transverse support beams located at approximately a 50 cm ( $\approx$ 20 inches) spacing. Retaining straps are attached to the transverse support beams and retaining straps. Longitudinal joints between panels provide normal-to-panel shear continuity between adjacent panels, preventing joint gapping by forcing adjacent panels to deflect simultaneously under applied loads. Transverse support beams transmit loads to support struts which carry the loads to primary load carrying structure. Drag links, spanning diagonally between transverse support beams and primary structure, provide support structure system stability and carry longitudinal loads.

During Phase I (Reference 1) of this program, the influence of cyclic entry conditions on creep response was investigated for four material alloys: Ti-6Al-4V, Rene' 41, L605, and TDNiCr. Analysis of tensile creep test data during this phase resulted in empirical equations, for each material, which describe cyclic creep

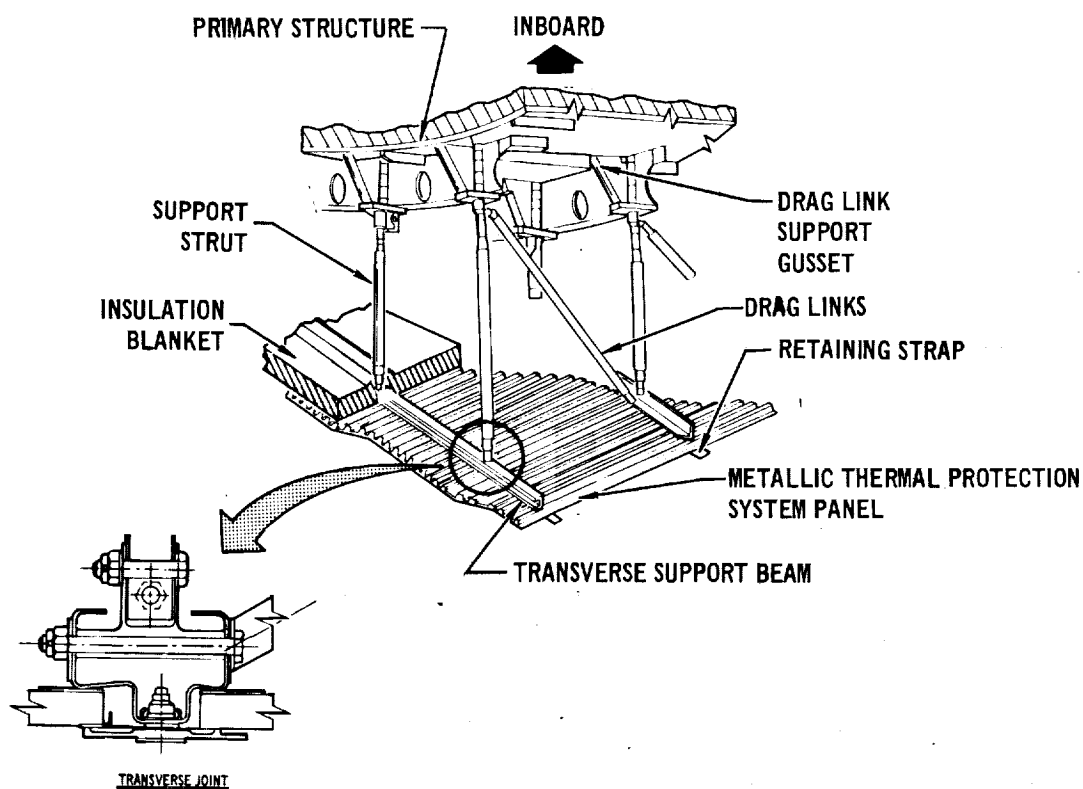


FIGURE 1-1 TYPICAL METALLIC THERMAL PROTECTION SYSTEM STRUCTURE

response characteristics as a function of stress, temperature, and time. These equations were used in conjunction with the time and strain hardening theories of creep accumulation to investigate creep prediction capability for cyclic trajectory stress and temperature profiles.

Phase II (Reference 2) was directed toward developing and verifying capability for prediction of creep deflection in metallic heat shields subjected to cyclic entry environments.

A computer program, Thermal Protection System Creep (TPSC) was developed for predicting beam creep deflections (Appendix B and C). This program offers an approach to creep predictions through application of iterative techniques and numerical integration. In the analysis, panel length is divided into segments over which bending moments are assumed constant and panel depth is divided into increments over which stresses and strains are assumed constant. Using a linear strain assumption, beam rotations are iteratively determined, based on either the time hardening or strain hardening theories of creep accumulation. Material cyclic creep response properties were defined by empirical equations developed from tests of tensile creep specimens in Phase I. The TPSC program capability includes definition of temperature as a function of beam length and depth, application of either the strain hardening or time hardening theory of creep accumulation, and the application of bending distributions for a full size panel based on the edge stiffness and the longitudinal and transverse panel stiffness. Program output includes definition of both elastic and creep deflected shapes, residual stresses, and creep strains as a function of cycle.

Seventeen subsize panel specimens, 6.35 cm by 3.05 cm, were tested to provide creep deflection data for verification of prediction capability. Corrugation cross section specimens were fabricated for test using thin gage ( $\sim 0.025$  cm) L605, Rene' 41, Ti-6Al-4V, and TDNiCr sheet material. Rib cross section specimens were also

fabricated for test using thicker gage ( $\sim 0.060$  cm) L605 and Ti-6Al-4V sheet material. These materials were procured for use both in Phase I and II. Each test consisted of cycling the panel for up to 100 entry thermal and bending load profiles representative of Shuttle entry missions. Testing was conducted in a vacuum furnace, using a load mechanism specifically designed to apply a two-point panel load that would be independent of panel deflection. Permanent deflections, due to creep, were measured as a function of cycle.

Comparisons of subsize panel creep deflection predictions with test results were made. Generally, good correlation was obtained between predicted and test deflections.

The objective of the program Phase III effort was to analyze full size panel creep data obtained from available test programs and to compare test results with prediction using methods of analyses developed during Phase I and II. Every effort was made to include all possible variations that could be ascertained from the documentation that might affect creep response so that as much confidence as possible could be associated with the comparison of predicted and test results. To this extent, the documented test data and results are summarized in this report. In addition, loads, temperatures, and panel geometry data required for creep analysis are reported.

The international system of units (SI) are used in this report. U.S. Customary Units are also generally provided. Applicable conversion factors are presented in Appendix A.

## 2.0 Background from Phase I and II

During Phase I and II of this program, the influence of cyclic entry conditions on creep response of L605, Rene' 41, TDNiCr, and Ti-6Al-4V were investigated and prediction capability for TPS panel creep deflections was developed. These cyclic creep data and analysis methods have been applied in the evaluation of full size panel test data in Phase III. Presented in this section are discussions of Phase I and Phase II results as they apply to the Phase III evaluation of full size L605, Rene' 41, and TDNiCr TPS panel test data.

### 2.1 PHASE I - CYCLIC TENSILE TESTING

In Phase I, thin gage tensile specimens were tested under cyclic loads and temperatures. Initially, creep response data were generated in what was designated as series of basic cyclic tests. These tests were conducted using the stress and temperature profiles shown in Figure 2-1 where the time per cycle was twenty minutes and the time between cycles (required for heat up and cool down portions of the profile) was 35 minutes. For each material, tests were conducted at three stress levels at each of four temperatures covering the range of temperature applicability for the respective materials.

Test temperature ranges were 978K (1300°F) to 1255 (1800°F) for L605, 1033K (1400°F) to 1155K (1620°F) for Rene' 41, and 1089K (1500°F) to 1478K (2200°F) for TDNiCr. Stress levels were selected at each temperature to yield 100 cycle creep strains of up to approximately 1%, the range of interest in analysis of TPS panels.

Analysis of cyclic tensile test data for each material resulted in empirical equations describing cyclic creep response characteristics as a function of stress, temperature, and time. These equations are presented in Table 2-1. Each equation represents a fit of cyclic data based on regression analysis. For each material

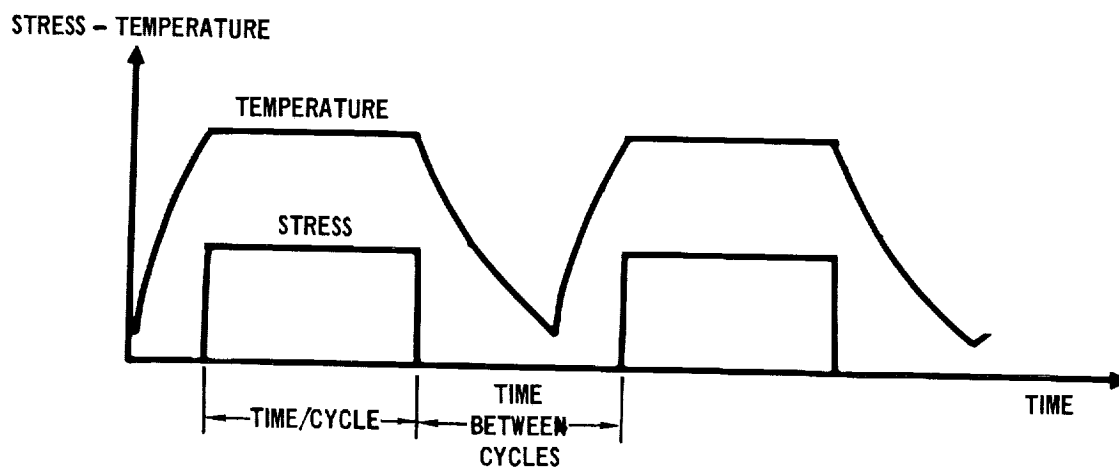


FIGURE 2-1 STRESS AND TEMPERATURE PROFILES FOR PHASE I TENSILE CYCLIC CREEP TESTS

TABLE 2-1 CYCLIC CREEP EQUATIONS DEVELOPED FOR  
PHASE I TENSILE CREEP DATA

MATERIAL	EQUATION	$\left\{ \begin{array}{l} t = \text{time, hours} \\ \sigma = \text{stress, MPa} \\ T = \text{Temperature, K/1000} \end{array} \right\}$	APPLICABLE TEMPERATURE MAXIMUM
L605	$\ln \epsilon = -2.89413 - .01743t + .54892 \ln t + 1.31015 \ln \sigma - 6.66548 (1/T) + .19131 \sigma \ln T + .00021 (T_{\text{tot}}).$		1255 K (1800°F)
RENE'41	$\ln \epsilon = -39.55860 + 29.13646T + .71922 \ln t + .92125 (\ln \sigma - 1.931) - .000016\sigma^2 + .08183 (\ln \sigma - 1.931)^3 - .000125 t-T + .0000105t^3$		1155 K (1620°F)
TDNiCr	$\ln \epsilon = -3.48443 - 10.37282 \left(\frac{1}{T}\right) + .28314 \ln t + 2.00118 \ln \sigma$		1478 K (2200°F)

NOTE: The equation developed for Ti-6Al-4V can be found in References 1 and 2.

considerable effort was directed toward determining appropriate equation forms, including stress, time and temperature interaction terms, to provide a "best fit" over the entire range of data resulting in the different equation forms shown. Typical comparisons of the tensile cyclic data and empirical equation predictions for each material are shown in Figures 2-2 through 2-4. Generally, the TDNiCr specimens failed at creep strains below .15%.

Because the empirical equations presented in Table 2-1 were derived from 100 cycle testing for 20 minutes at load per cycle, the total time of applicability of each of the equations is 33.3 hours. In the analysis of TPS panels subjected to mission load and temperature profiles, the profiles are "idealized" by dividing them into steps of constant load and temperature. To investigate the applicability of the empirical equations to these profiles, tests were also conducted for each material using the profile shown in Figure 2-1 with a ten-minute per cycle time at load and peak temperature. Results of these comparisons for each of the 10 minutes per cycle and 20 minutes per cycle materials are shown in Figure 2-5 for equal total time at load. Because close agreement between these test data was obtained, the equations are considered to be applicable in analysis of idealized mission profiles where smaller time steps are used. Also, this total time of equation applicability of 33.3 hours will be important in the analysis of test data where longer times of the stress and temperature profiles may result in a reduction of the number of cycles over which the equations are applicable.

Cyclic tensile tests were also conducted where stress was varied as a function of cycle (stepped stress tests) and where stress and temperature were varied within each cycle (mission profile tests). These test data were used to evaluate the applicability of the time and strain hardening theories of creep accumulation. Comparison of predicted creep strains using these hardening theories in conjunction with the empirical equations indicated that neither theory consistently provided

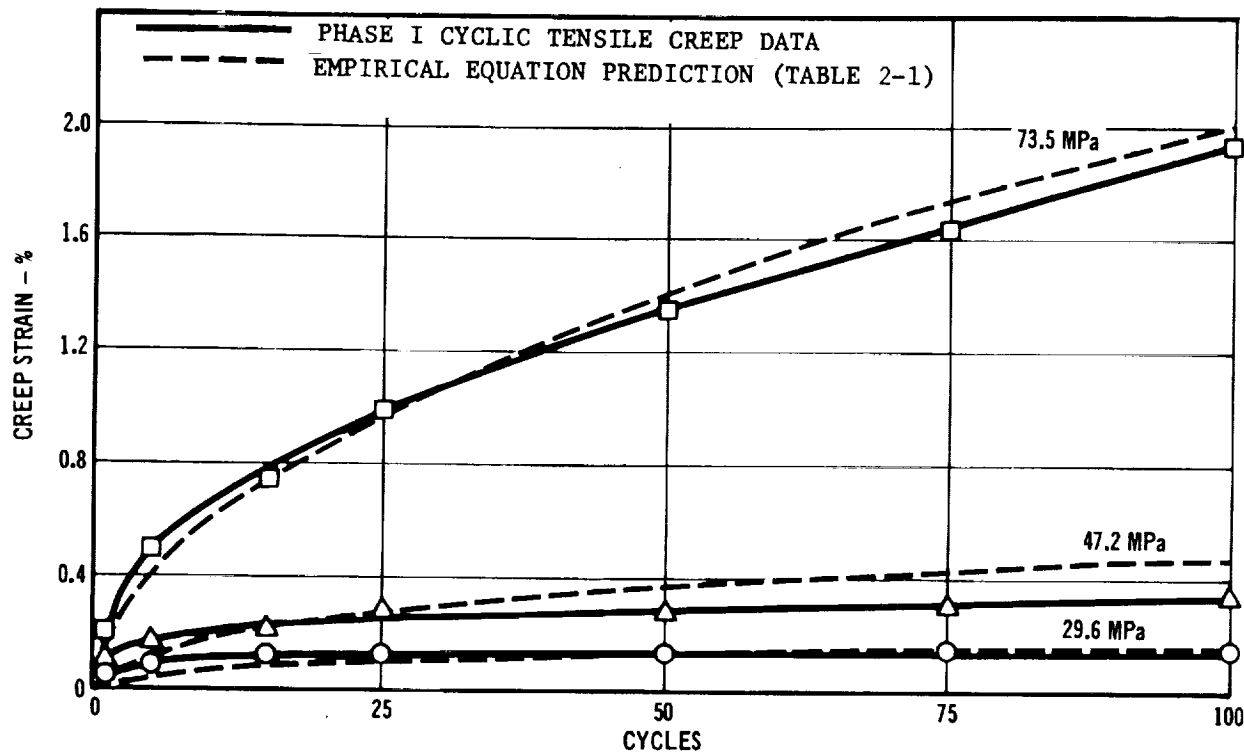


FIGURE 2-2 COMPARISON OF L605 PREDICTED AND CYCLIC TEST CREEP STRAINS AT 1144 K

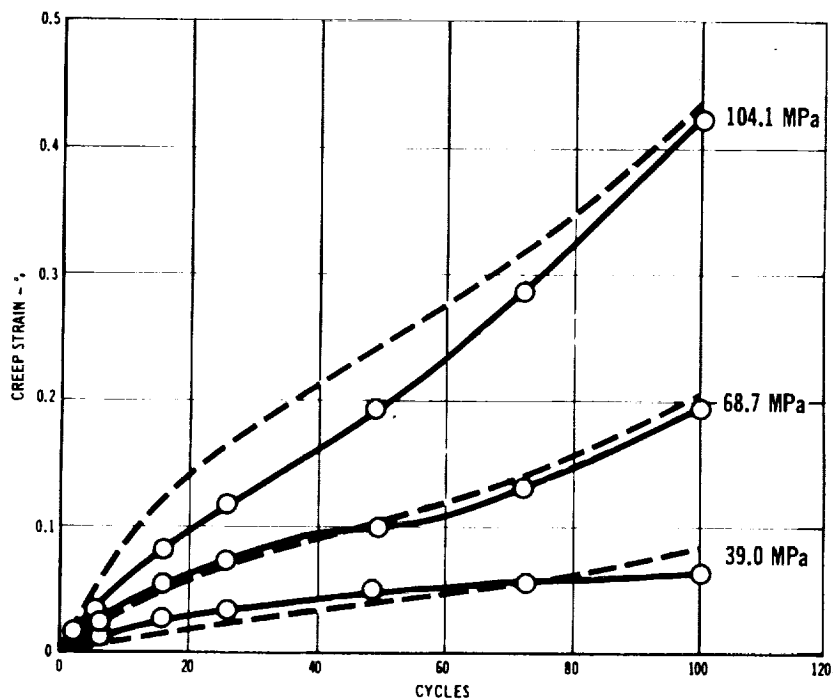


FIGURE 2-3 COMPARISON OF RENE 41 PREDICTED AND CYCLIC TEST CREEP STRAINS AT 1111 K

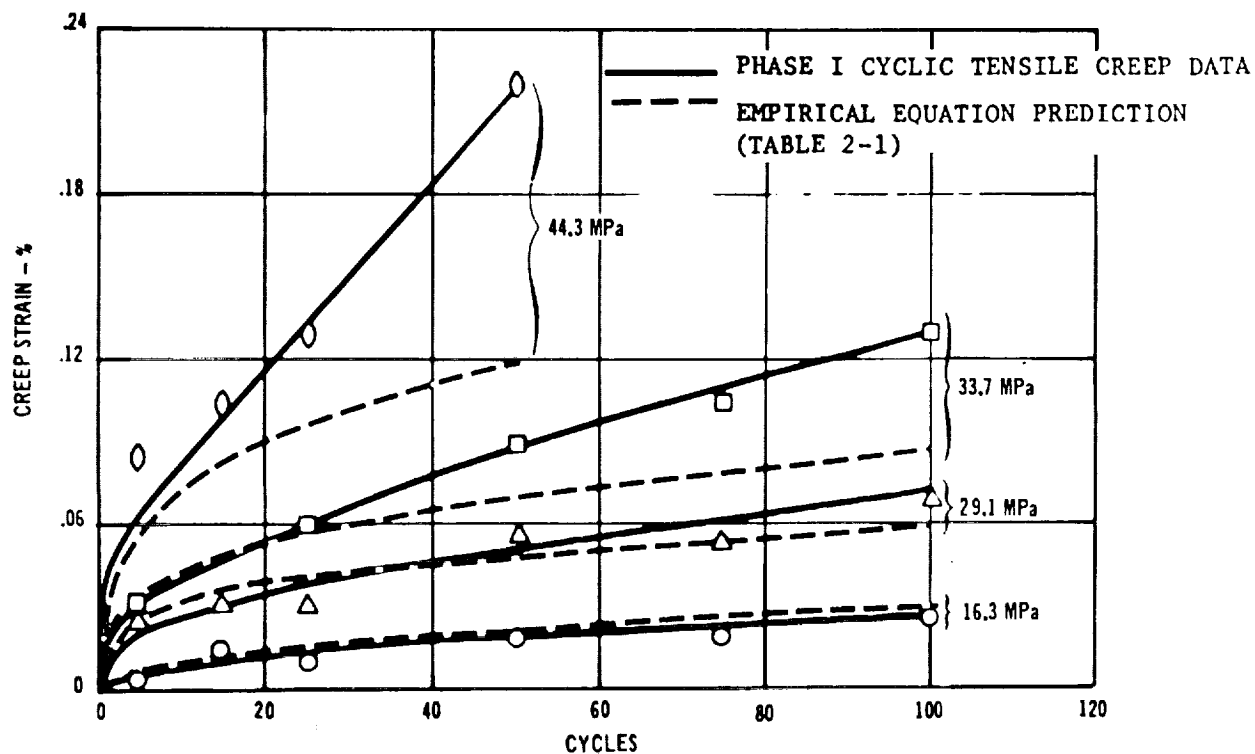
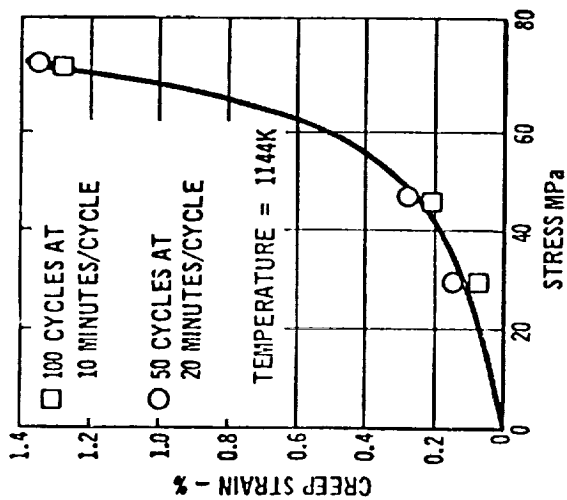
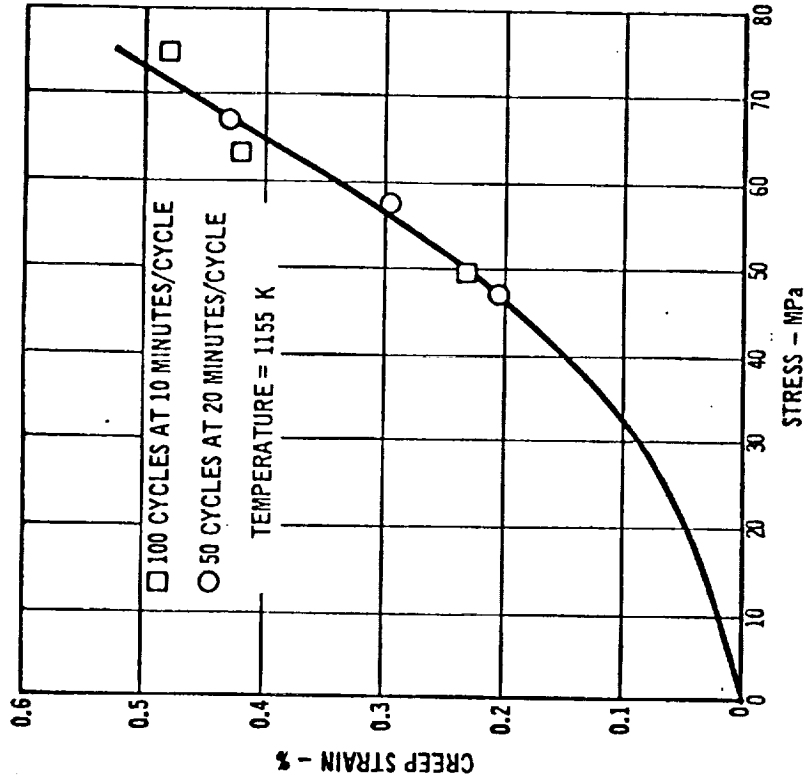


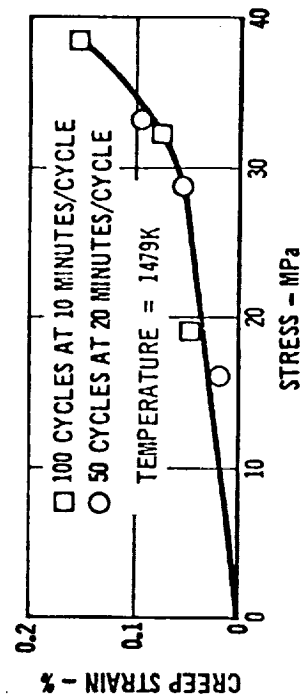
FIGURE 2-4 COMPARISON OF TDNiCr PREDICTED AND CYCLIC TEST CREEP STRAINS AT 1478 K



(a) L605



(b) RENE '41



(c) TDNiCr

FIGURE 2-5 CYCLIC TENSILE CREEP STRAINS AT DIFFERENT TIME  
PER CYCLE COMPARED AT EQUAL TIME AT LOAD

good predictions. Comparison of strain hardening and time hardening predictions with L605 cyclic tensile data showed that the strain hardening theory provided the best predictions for tests where stress was continually decreased as a function of cycle and the time hardening theory provided the best prediction for tests where stress was continually increased as a function of cycle. Therefore, an approach was proposed where both strain hardening and time hardening theories were used at each analysis step depending on whether the creep strain rate decreased or increased, respectively. Although this improved predictions for the L605 mission profile trajectory tests, it did not improve prediction for the other materials.

For Rene' 41, predictions based on the time hardening theory of creep accumulation were generally considered best. Predictions based on the strain hardening theory of creep accumulation were found to be approximately the same as for time hardening in predicting strains for testing where the stress was continuously increased as a function of cycle. Both predictions were close to test values. For specimens where stress was continually decreased, the time hardening predictions were up to 30% higher than test strains. However, predictions based on strain hardening were even higher, up to 75% higher than the time hardening predictions.

Predictions of creep strains for TDNiCr trajectory profile tensile tests using the cyclic creep equation, were found to be from 30% to 70% of test strains at 100 cycles. The applicability of hardening theories used in panel analysis will significantly effect prediction capability.

Another variable considered in Phase I tensile creep testing was the possible effect of recovery phenomena. To investigate this, tests were conducted where the stress profile was maintained for a longer period of time while temperature was being decreased rapidly, as shown in Figure 2-6(a). These comparative tests were conducted on L605 and Rene' 41 tensile specimens. Test results, shown in Figure 2-6(b), indicated that no variation in creep strains could be determined for the

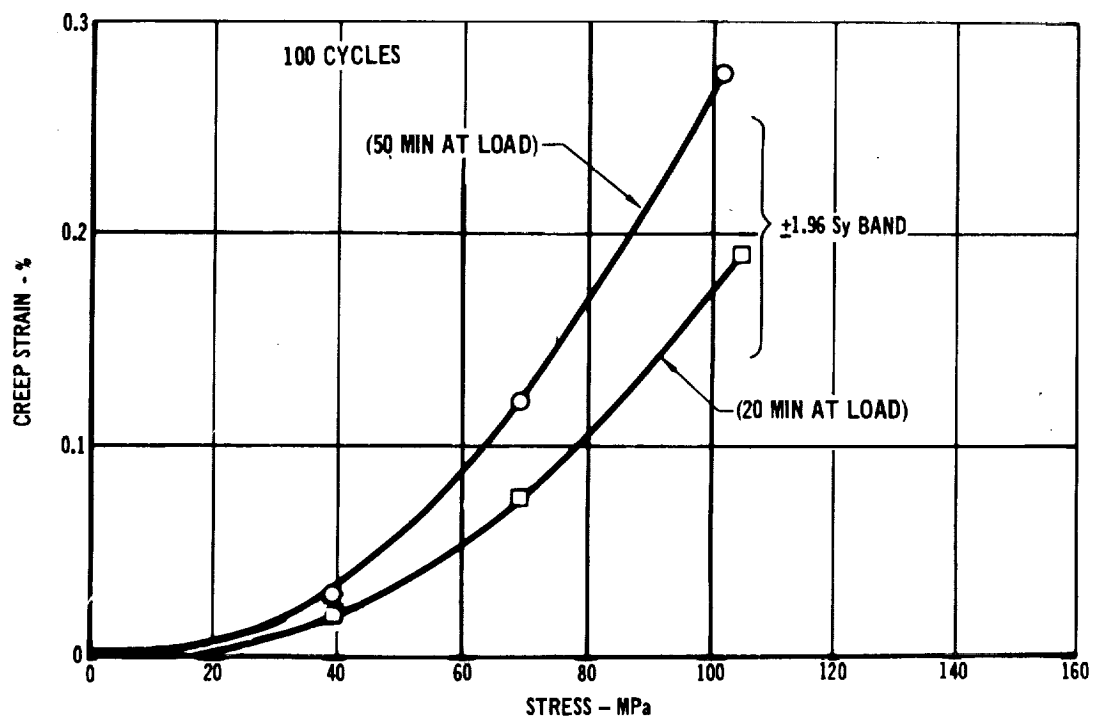
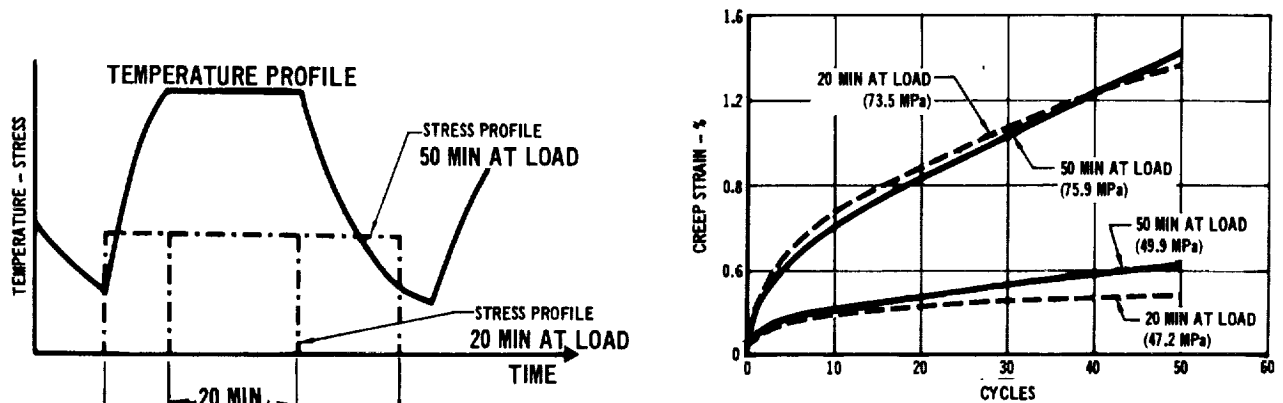


FIGURE 2-6 EFFECT OF MAINTAINING LOAD DURING HEAT UP AND COOL DOWN PORTIONS OF THE CYCLE PROFILE

L605 specimens, as indicated by the strain-time data plots. However, Rene' 41 creep strain results were consistently higher for each of three specimens tested. These results, plotted as a function of the stress levels, at 100 cycles are shown in Figure 2-6(c). This variation in creep strains for Rene' 41 was greater than expected based on data scatter as determined in the development of empirical equations. From these results it is difficult to draw conclusions as to the differences between the mission profile test results and predictions based on empirical equations developed for the twenty minute per cycle stress level. However, it has been demonstrated that an effect, due to possibly a material recovery phenomena, may exist to different degrees in the different materials, which may effect panel deflection predictions.

The empirical cyclic creep strain equations developed in Phase I were based on tests conducted on the thin gage sheet specimens ( $\sim .025$  cm) for each material. Initially in Phase I, steady state tests were conducted on both these thin gage materials and also on specimens from a sheet thickness of .064 cm. An effect of gage on creep response (thin gages creep faster) was noted in the L605 steady state tests and also in steady state data obtained from the literature. This effect is attributed to a change in material processing at about  $t = .064$  cm. This type of effect is discussed here to point out possible effects from sheet to sheet and due to thickness which may effect prediction capability in applying empirical equations developed on this program to panel tests from other programs.

For each of the alloys, cyclic tensile creep tests were conducted in Phase I to obtain data for the assessment of possible effects of atmosphere pressure on creep response. To provide these data, replicate tests were conducted, using idealized mission stress and temperature profiles. However, atmospheric pressure was held constant at 1.33 Pa, in one of the tests while a mission pressure profile was

applied in the other test. In each case variations in creep strain results were relatively small and were considered within the range of scatter for replicate tests. Therefore, no effect was attributed to atmospheric pressure.

## 2.2 PHASE II - PREDICTION AND VERIFICATION OF PANEL CREEP DEFLECTIONS

Phase II was directed toward developing and verifying capability for prediction of creep deflections in metallic heat shields subjected to cyclic entry environments.

A computer program, Thermal Protection System Creep (TPSC) was developed for predicting beam creep deflections and was used to predict results of subsize panel testing. Details of this work have been reported in the Phase II Summary report (Reference 2) and in the TPSC Program User Manual (Appendix B). This program offers an approach to creep predictions through application of iterative techniques and numerical integration. In the analysis, panel length is divided into segments over which bending moments are assumed constant and panel depth is divided into increments over which stresses and strains are assumed constant. The single skin corrugation TPS configuration with a skin bead, is automatically idealized through appropriate geometry input to the program. All of the full size panels analyzed in Phase III were of this configuration.

Using a linear strain assumption, beam rotations are iteratively determined, based on either the time hardening or strain hardening theories of creep accumulation. Material cyclic creep response properties developed from tests of tensile creep specimens in Phase I were used in the analysis and because the time hardening approach provided more consistently the best predictions of subsize panel data in Phase II, it was used for analysis purposes in Phase III. The TPSC program capability also includes definition of temperature as a function of beam length and depth, and the application of bending distributions for a full size panel based on the edge stiffness and the longitudinal and transverse panel stiffness. The

capability of including temperature as a function of panel length was utilized in analysis of the Reference 3 studies (Section 3.1) where temperature distributions were known.

Moment distributions are internally defined based on uniform pressure loads and simple panel supports. In addition, the moment distribution can be automatically calculated as a function of panel edge stiffness and the ratio of panel stiffness in the longitudinal and transverse directions. This option is based on combining solutions for an isotropic plate with two sides simply supported and two sides elastically supported as offered by Timoshenko (Reference 4) and the solution for an orthotropic plate with four sides simply supported as offered by Lekhnitskii (Reference 5). This option provides a first order approach to account for Poisson's effects in orthotropic plate structures. However, this option was not applied to the analysis of full size panel data in Phase III because of the large ratio of longitudinal to transverse stiffness for corrugated panels analyzed and because of the general lack of edge stiffness in the test panels. Edge stiffness was generally minimized in the full size panel tests to simulate as closely as possible actual entry vehicle panel conditions.

Both pressure and temperature load inputs are based on idealization of the test profiles into discrete time steps. During Phase I, cyclic tensile tests were conducted for both mission profiles and idealized profiles. Comparison of test results indicated that a minimum number of steps (4 steps used in Phase I testing) provided good correlation of results.

The following basic assumptions are made in the analysis:

- a) Only bending stresses are considered in the analysis. Deflections due to shear are assumed negligible.

- b) Total creep strain plus elastic strain distributions through the panel depth are linear.
- c) Load and temperature distributions and calculated deflections are assumed symmetrical with respect to the panel centerline
- d) Creep response equations, defined by the user, are assumed to be applicable for both tensile and compressive stresses. In addition, the equations developed based on Phase I cyclic testing are assumed to be applicable for the sheet material used in fabrication of the full size panels.
- e) Stress distributions are assumed uniform in the horizontal thin gage sections of the panel cross sections. In particular, the thin gage skin, loaded in compression, is assumed to carry load uniformly (except as altered by the  $M_y/I$  distribution in the bead) across the pitch length.

It is difficult to determine how much each of these assumptions might influence the predictive capability of the TPSC program. The first three of these assumptions probably are the most applicable to the analysis. The last two assumptions are of most concern as to applicability. Certainly the scatter documented in literature for sheet to sheet variations in creep response as well as variations in TPS panel skin stress distributions evidenced through strain gage data (References 3 and 6) and the occurrence of skin buckling (References 2 and 3) noted in TPS panel testing will effect creep deflections. Applicability of the hardening theories to the real material response will also be an unknown in the full size panel analysis.

Even with all of these assumptions considered, the TPSC computer program has been demonstrated to provide needed capability for prediction of permanent deflections, due to creep, in entry vehicle metallic thermal protection system panels subjected to complex cyclic loading conditions. The TPSC program is written in Fortran IV and is operational on the CDC 6600.

Four L605, three Rene' 41, four Ti-6Al-4V, and two TDNiCr subsize (6.35 cm by 30.5 cm) corrugation stiffened TPS panels were tested to provide creep deflection data for verification of prediction capability in Phase II. These specimens were fabricated using thin gage (approximately .025 cm) sheet material, however, the section geometry was more compact (pitch = 1.91 cm and depth = 1.27 cm) than those found in the full size panel testing. In addition, no skin bead was used in the subsize panels. Rib cross section specimens were also fabricated for test using thicker gage (.064 cm) L605 and Ti-6Al-4V sheet material.

Testing of subsize panels was conducted in a vacuum furnace, using a load mechanism specifically designed to apply panel bending loads that would be independent of panel deflection. Permanent deflections, due to creep, were measured as a function of cycle. Each test consisted of cycling the panel for up to 100 temperature and bending load profiles representative of Shuttle entry missions. Two types of cyclic profiles were used. The first consisted of a constant load and temperature applied for twenty minutes, with heat up and cool down periods at zero load yielding total cycle time of fifty five minutes. Two L605, two Rene' 41, three Ti-6Al-4V, and one TDNiCr subsize panels were tested to this type profile. The second type of profile consisted of mission temperature and load profiles for the same total cycle time as for the constant load cycles. The remainder of the seventeen panels (four L605, one Rene' 41, three Ti-6Al-4V, and one TDNiCr) were tested to these mission profiles.

Comparisons of the subsize panel creep deflection predictions with test results were made in Phase II (Reference 2). Predicted deflections, as a function of cycle, for the L605 subsize panels tended to be lower than test values for approximately 15 cycles and then increase to higher than test values by the conclusion of the test. This same trend was noted in the comparison of tensile

creep data and empirical equation predictions. Predicted panel creep deflections obtained using the time hardening theory of creep accumulation were found to generally yield the best predictions. Predicted deflections for the Rene' 41 panels generally were not as close to test values as had been demonstrated in the case of L605. Again, the time hardening theory of creep accumulation provided the best deflection predictions, although these predictions were lower than test data for the mission profile and higher than test data for the constant load and temperature profiles. Predictions for the Ti-6Al-4V panels generally agreed with test results. The shape of the predicted deflection curve as a function of time (or cycle) was in good agreement with the test data which is consistent with the prediction capability of the empirical equation for Ti-6Al-4V. Predictions for the TDNiCr subsize panels were a factor of two high in one test and a factor of two low in the other test. No rationale was determined for this apparent inconsistency although all deflections were small ( $\sim .05$  cm).



### 3.0 ANALYSIS OF FULL SIZE PANEL DATA

The Phase III effort consists of evaluation and analysis of full size panel data. In each case, analysis consists of the idealization of test load and temperature profiles and calculation of the TPS panel deflections using the material cyclic creep properties developed in Phase I and creep deflection prediction methods developed in Phase II.

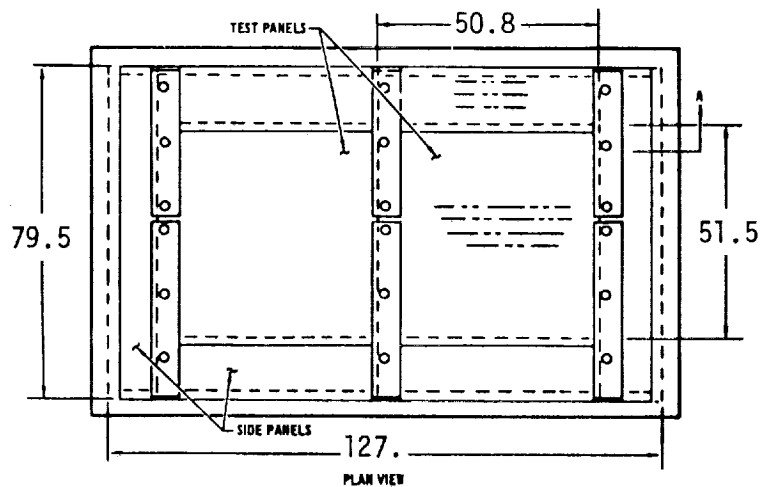
Four sets of panel data were evaluated in this phase. These data were for L605 panels and Rene' 41 panels tested at McDonnell Douglas Corporation (Reference 3) and Haynes 25 (L605) and TDNiCr panels tested at Grumman Aerospace Corporation (References 6, 7 and 8). Although it was desirable to evaluate data on panels for each of the four materials studied in Phases I and II, no test data on testing of full size titanium TPS panels were found.

#### 3.1 SSTP PROGRAM L605 AND RENE' 41 PANELS

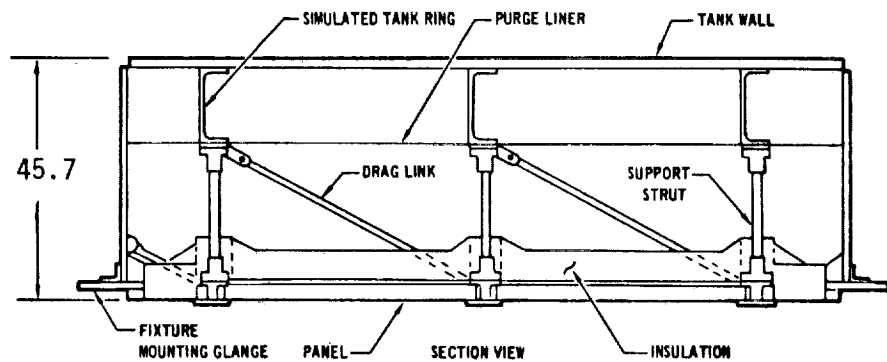
The SSTP program (Supplementary Structural Test Program) was a supplement to the Space Shuttle System Program Definition, conducted at McDonnell Douglas Astronautics Company - East. This program consisted of designing, fabricating, and testing of Space Shuttle primary structure and thermal protection systems. Purposes of the program were to verify feasibility of design concept, provide design data, demonstrate producibility, demonstrate reusability and verify unit weight predictions. Included in this program was the testing of L605 and Rene' 41 full scale metallic TPS panels.

Each test assembly consisted of two test panels, smaller side panels to provide proper boundary conditions, support beams and struts, and a section of simulated tank structure. Figure 3-1 shows the test assembly. The two primary test panels are each 50.8 cm. x 50.8 cm (20" x 20"). The 12.7 cm (5 in.) wide

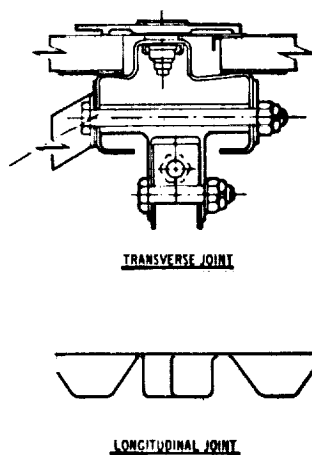
(a) FULL SCALE  
SSTP PANELS  
IN TEST FIXTURE



(b) SSTP  
SUPPORT STRUCTURE  
AND INSULATION  
LARGE PANEL  
TEST



(c) SSTP  
LARGE PANEL  
DETAILS



(d) PANEL  
DESIGN DETAIL

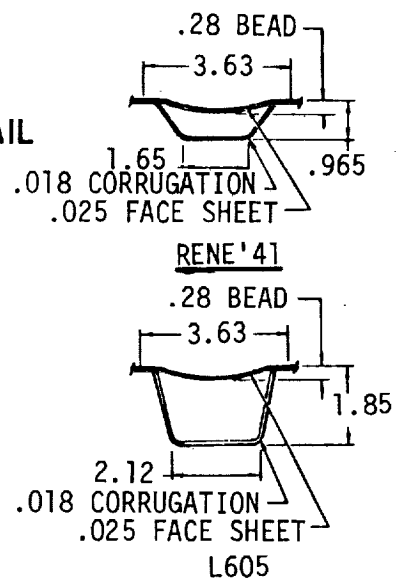
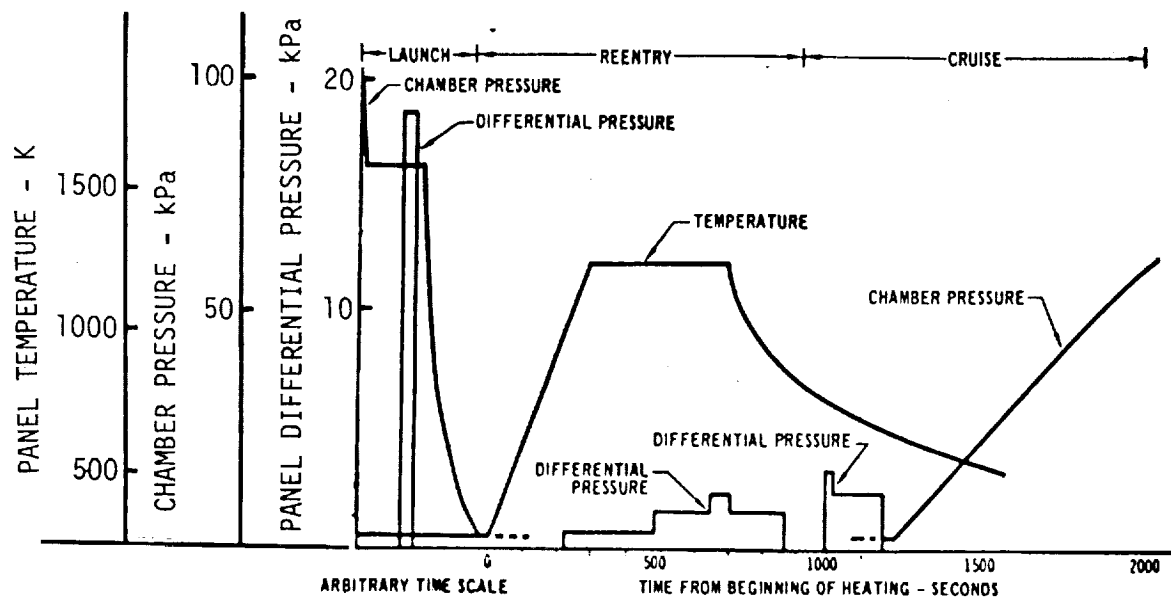


FIGURE 3-1 SSTP PANEL TEST GEOMETRY

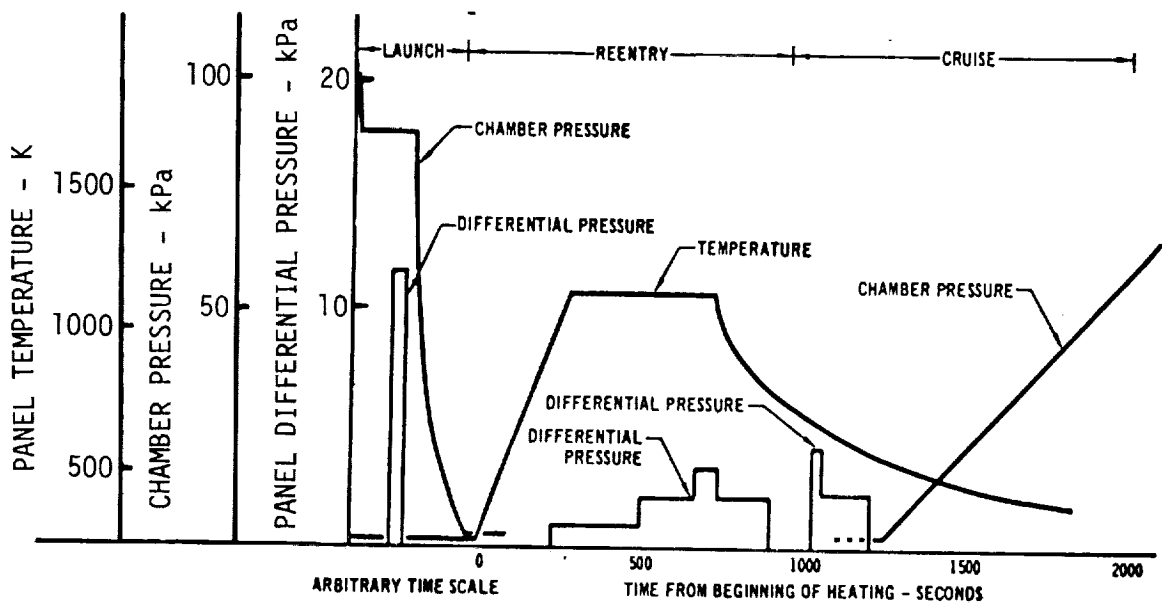
side panels simulated the boundary conditions by isolating the primary panels from the water cooled test fixture. The support beams for the panels were 50.8 cm (20 in.) apart. The panels were supported by "hat section beams and retaining straps with sufficient clearance to permit free expansion. The beams in turn were supported by tubular struts and drag links. Figures 3-1(a), 3-1(b), and 3-1(c) show the details of this construction. The beam also supported the insulation packages. To accommodate thermal expansion, the panels were mounted with slip fit joints. For the analysis, the panels were assumed to have simple supports at the edges because end fixity and friction were difficult to define.

The panels of both materials were the same basic design; single faced, corrugation stiffened, with beaded face skins and reinforcing doublers on the ends of the corrugations. The doublers served a dual purpose. They not only stiffened the corrugation ends, but they were also made thick enough so that after assembly a light machine cut could be taken across the doublers to make a close tolerance uniform thickness panel. Panels differed in corrugation depth as shown in Figure 3-1(d). The shallow beads in the face skins were designed to relieve the stresses caused by the thermal gradients. Heat treatment of the L605 and Rene' 41 panels were performed in two phases. For the first phase the panels were heated in air, without any restraining fixtures, so that a high emittance oxide coating would form on all surfaces. The panels were then clamped and heated a second time. These coating and straightening operations were incorporated into the normal heat treating sequence for the materials.

Testing consisted of exposing the structures to repeated cycles of simulated mission environment. Each cycle consisted of exposure to ascent pressure, entry pressure and temperature, and cruise pressure, as shown in Figures 3-2(a) and (b) for L605 and Rene' 41 panels respectively. Blocks of these cycles were followed by blocks of acoustic test cycles.



(a) L605 Panel



(b) Rene' 41 Panel

FIGURE 3-2 SIMULATED TEST MISSIONS USED IN SSTP PROGRAM



The programmed TPS differential pressure levels were met during the entry phase of the mission. Launch differential pressure levels were usually 35% low and cruise differential pressures were 50% low due to high TPS panel leakage. The Rene' 41 TPS panel was subjected to the complete 100 mission program, while the L605 panel was exposed to 30 temperature-pressure tests and 100 acoustic tests. Throughout each cycle the panel temperatures, not influenced by the joint, were controlled during heating. The cooling portion of the mission, however, was uncontrolled and in all cases cooled somewhat faster than expected. The lamp array used for heating the panels was made in sections creating an unavoidable gradient in the panels. Thermocouples were positioned so this gradient could be measured.

As part of the periodic inspections, the surface of the panels were mapped to detect any warpage or permanent set caused by the simulated missions. The aft panel on each of these assemblies deflected more than the forward panel. There was no obvious explanation for this difference, although it was probably caused by some phenomena associated with the test set-up. It is possible that the time at temperature for one panel was consistently slightly longer than the other.

The following sections describe the effort in evaluating data and in providing cyclic creep deflection predictions for these L605 and Rene' 41 TPS panels.

#### 3.1.1 SSTP PROGRAM L605 PANEL

SSTP L605 temperature data (Reference 3) were reviewed to define actual panel temperature distributions which would be applicable to the panel analysis. The data points for several representative cycles are used in developing the temperature variations shown in Figure 3-3. As indicated in the referenced report, the influence of the lamp bank splice was approximately 30K (~50°F). Variation is noted between the panel edge and center temperature levels, however, no significant

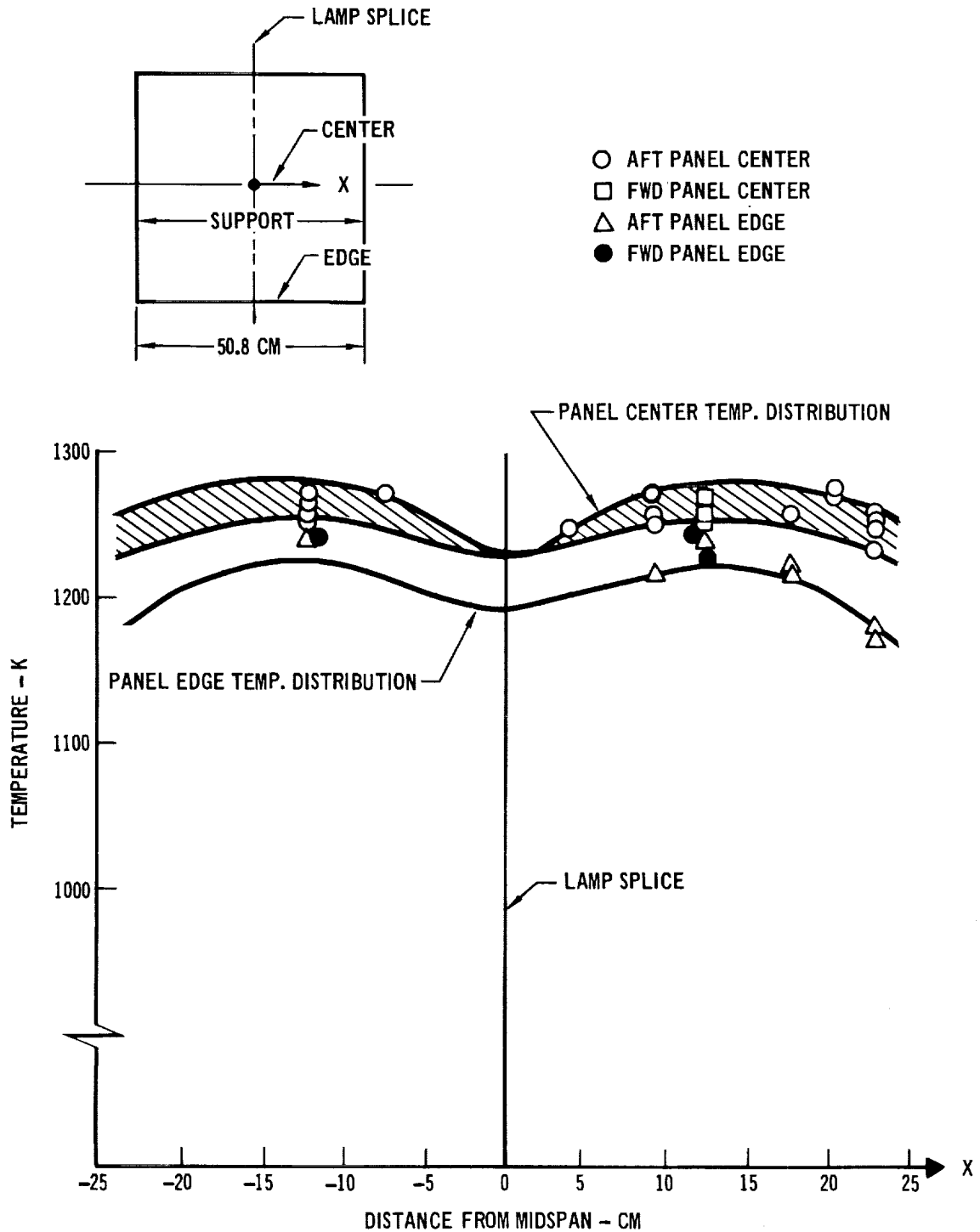


FIGURE 3-3 L605 TPS PANEL MEASURED TEMPERATURE DISTRIBUTIONS

difference can be detected between the forward and aft panels at either the center or edge locations.

Analysis of the panels was conducted for the reentry portion of the mission profile (Figure 3-2(a)). This profile was idealized into five constant temperature and differential pressure steps as indicated in Figure 3-4. Analysis was based on using the panel temperature distributions from Figure 3-3 defined as a function of time by the profiles in Figure 3-4.

Comparisons of creep deflection predictions with measured deflection data are presented in Figure 3-5. Considerable effort was given to evaluation of the creep deflection test data. A significant amount of variation was noted as indicated in the plots of Figure 3-5. Therefore, it is difficult to draw conclusions as to the prediction capability. A careful review of the reference 4 data did not reveal any differences which would account for the variation in deflection between the forward and aft panels as indicated in Figure 3-5(a). The prediction shown is based on the maximum of the temperature range shown in Figure 3-3 at the panel center. Analysis conducted using the minimum value of the temperature range resulted in approximately a 10% lower creep deflection prediction. The prediction shown for the panel edge in Figure 3-5(b) is based on the corresponding temperature distribution in Figure 3-3. Predicted deflections are based on the same unsupported span length (45.7 cm) and referenced measurement length (41.3 cm) as used in the SSTP Program. Data used in the analysis are presented in Tables 3-1(a), (b), and (c).

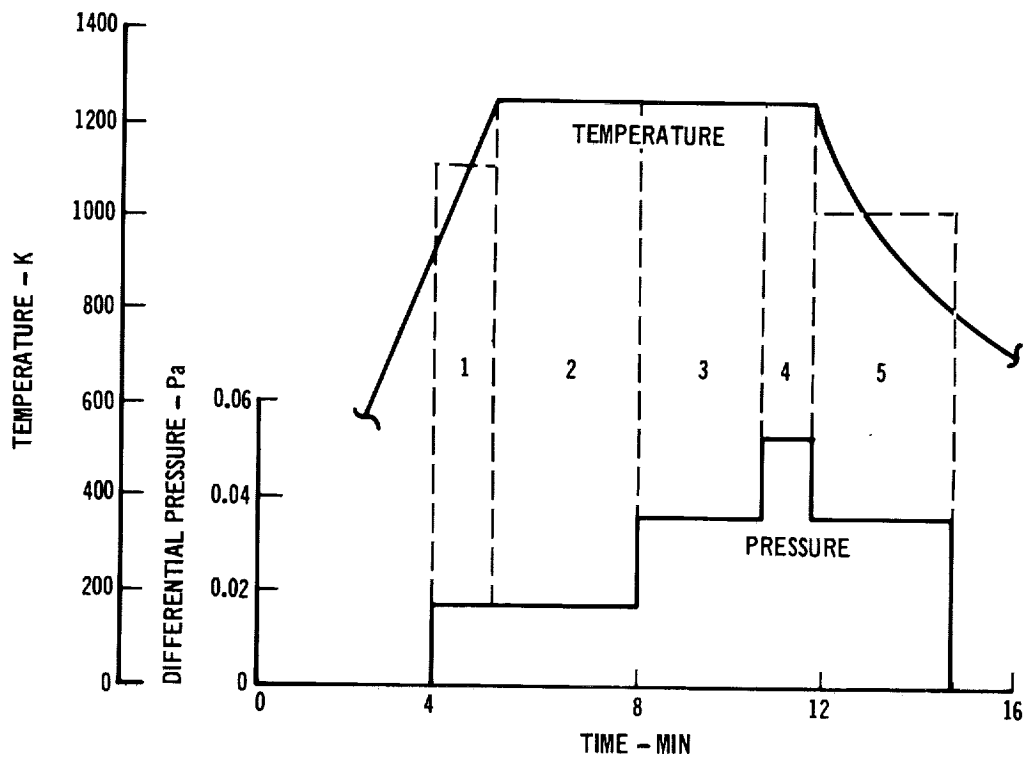
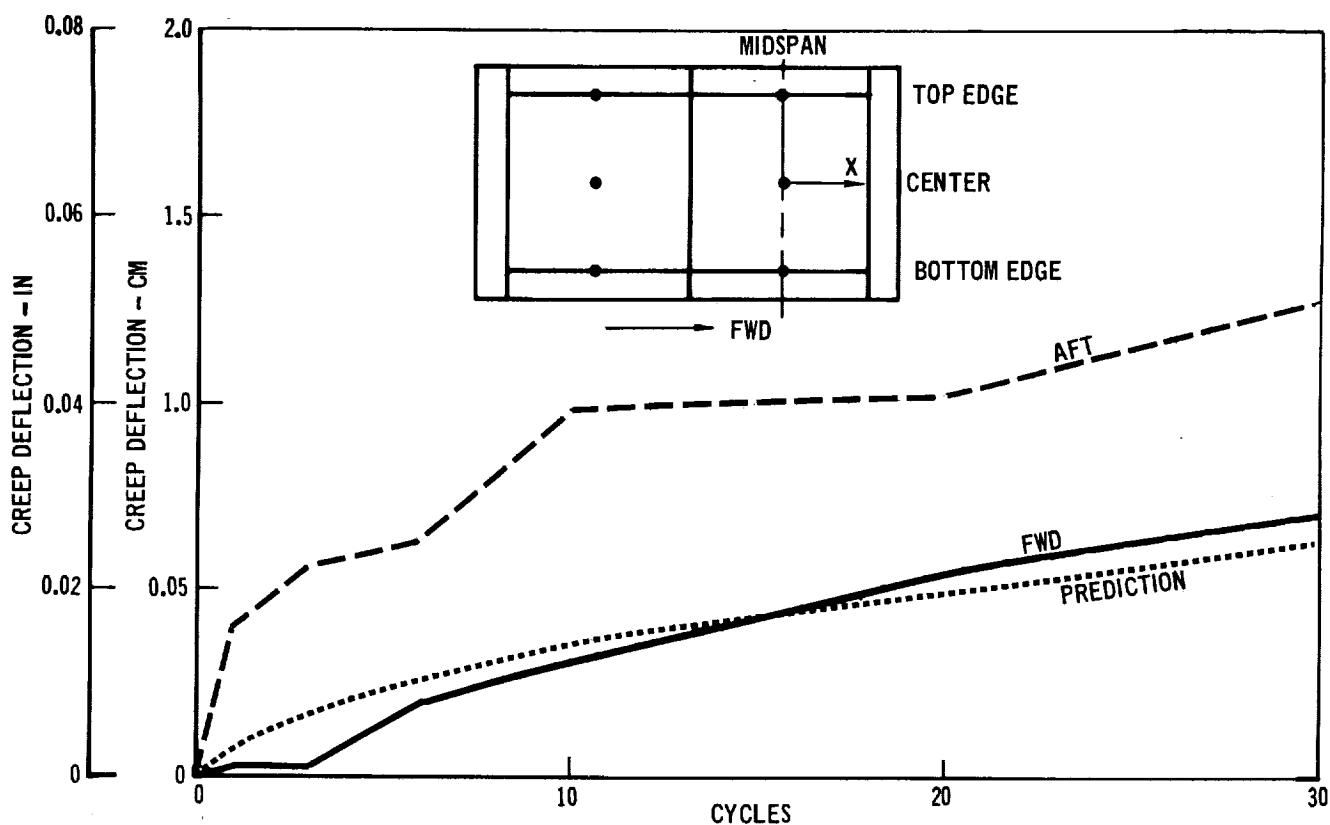
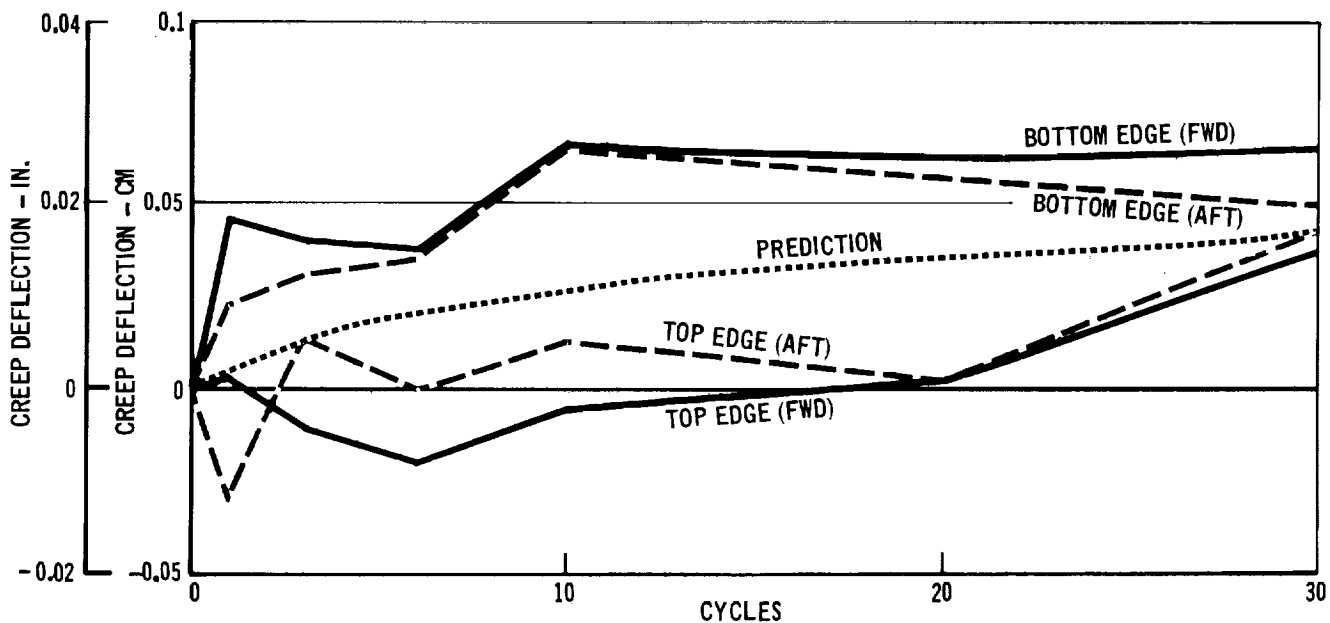


FIGURE 3-4 L605 ENTRY PROFILE USED IN CREEP PREDICTION ANALYSIS

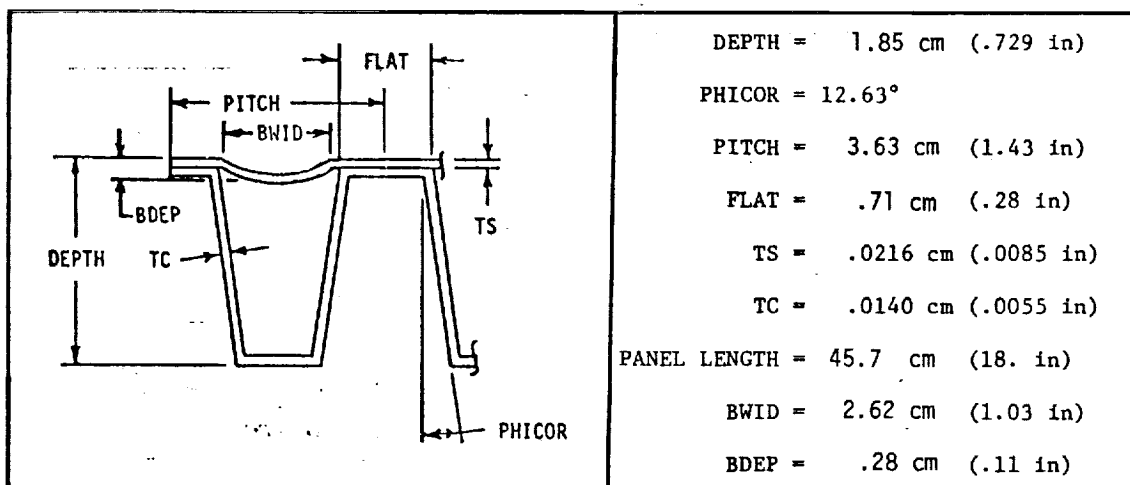


(a) Panel Midspan Deflections Along Center Span



(b) Panel Midspan Deflections Along Edge

FIGURE 3-5 COMPARISON OF L605 PANEL TEST AND PREDICTED CREEP DEFLECTIONS


**(a) L605 Panel Geometry**

TIME (MIN)	PRESSURE KPa (PSI)	PANEL TEMPERATURE - K (°F)					
		X=0	X=5.1	X=10.2	X=15.2	X=20.3	X=22.9
0 - 1.33	.76 (.11)	1149 (1609)	1162 (1631)	1167 (1640)	1159 (1627)	1129 (1573)	1122 (1560)
1.33 - 4.67	.76 (.11)	1261 (1810)	1275 (1835)	1281 (1845)	1272 (1830)	1239 (1770)	1231 (1755)
4.67 - 7.50	1.65 (.24)	1261 (1810)	1275 (1835)	1281 (1845)	1272 (1830)	1239 (1770)	1231 (1755)
7.50 - 8.50	2.48 (.36)	1256 (1800)	1269 (1825)	1275 (1835)	1267 (1820)	1233 (1760)	1225 (1745)
8.50 - 11.33	1.65 (.24)	1038 (1408)	1048 (1427)	1053 (1435)	1046 (1423)	1021 (1377)	1014 (1365)

**(b) Temperatures and Pressures Along Panel Center**

TIME (MIN)	PRESSURE KPa (PSI)	PANEL TEMPERATURE - K (°F)					
		X=0	X=5.1	X=10.2	X=15.2	X=20.3	X=22.9
0 - 1.33	.76 (.11)	1076 (1476)	1108 (1534)	1115 (1547)	1108 (1534)	1090 (1502)	1088 (1498)
1.33 - 4.67	.76 (.11)	1178 (1660)	1214 (1725)	1222 (1740)	1214 (1725)	1194 (1690)	1192 (1685)
4.67 - 7.50	1.65 (.24)	1178 (1660)	1214 (1725)	1222 (1740)	1214 (1725)	1194 (1690)	1192 (1685)
7.50 - 8.50	2.48 (.36)	1173 (1651)	1209 (1716)	1217 (1731)	1209 (1716)	1194 (1681)	1187 (1676)
8.50 - 11.33	1.65 (.24)	973 (1291)	1001 (1341)	1007 (1353)	1001 (1341)	986 (1314)	983 (1310)

**(c) Temperatures and Pressures Along Panel Edge**
**TABLE 3-1. GEOMETRY AND LOADING DATA USED IN L605 PANEL ANALYSIS**

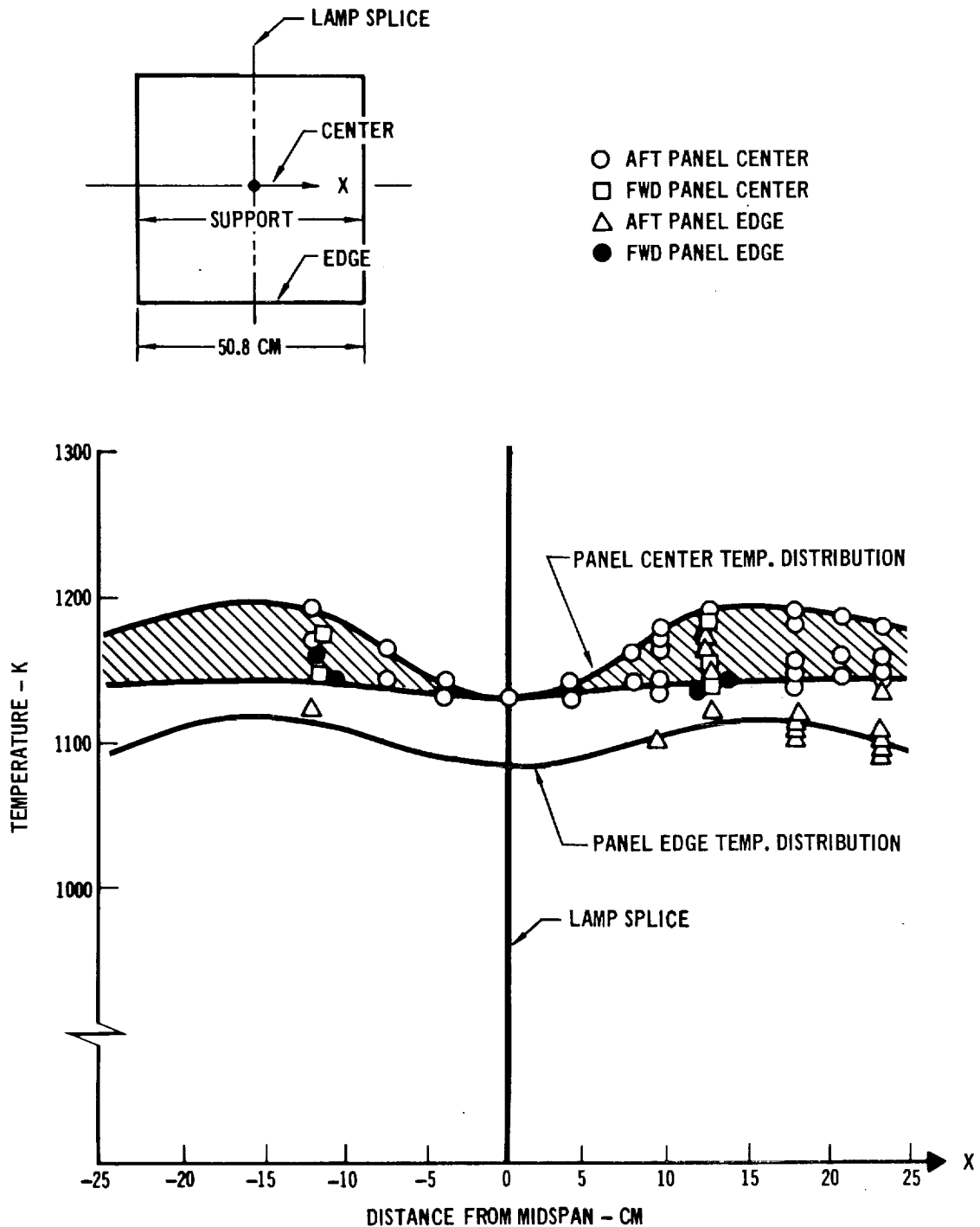


FIGURE 3-6 RENE'41 TPS PANEL MEASURED TEMPERATURE DISTRIBUTIONS



Initial analysis using a five time step idealization of the temperature and load profiles (reference Figure 3-4 for L605) indicated that less than 2 percent of the creep deflection occurred during the first and last time steps due to lower temperatures than in the other three steps. Therefore, subsequent analysis for Rene' 41 panels was conducted using the three step idealization of the entry test profiles as shown in Figure 3-7. The first step at constant peak temperature was extended in this case by approximately one half minute to compensate for the deleted two steps.

Comparisons of predicted panel deflections with test data are provided in Figure 3-8. Three predicted deflection curves are shown for the panel midspan (Figure 3-8(a)). The two curves of highest predicted creep deflection (designated A and B) are based on constant panel temperatures of 1144K (1600°F) and 1128K (1570°F) corresponding to the trajectory profile (Figure 3-7) temperature and the minimum panel center temperature distribution (Figure 3-6), respectively. These two analyses show the effect of this temperature variation on the predicted creep deflections. Both of these predicted curves are based on skin and corrugation gages of .0216 cm (.0085 in) and .0140 cm (.0055 in), respectively.

In an effort to demonstrate the effect of gage effects on the creep deflection, a third analysis was conducted (curve C) using the skin and corrugation gages of .0254 cm (.0100 in) and .0178 cm (.0070 in). The constant temperature of 1128K (1570°F) was applied, allowing comparison with the corresponding predicted deflection presented for the thinner gages (curve B). Again considerable variation was evident in the deflection data for the forward and aft panels tested. Shown in Figure 3-8(b) is the predicted creep deflection based on the panel edge temperature distribution defined in Figure 3-6. The panel dimensions, loads, and temperatures used in the analysis are defined in Table 3-2(a), (b), and (c).

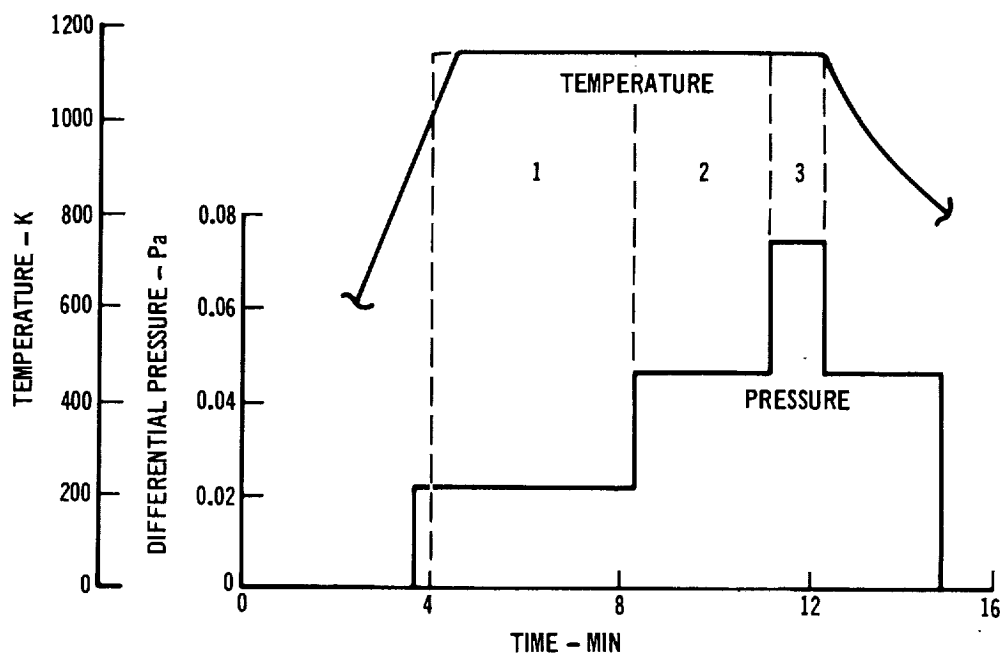
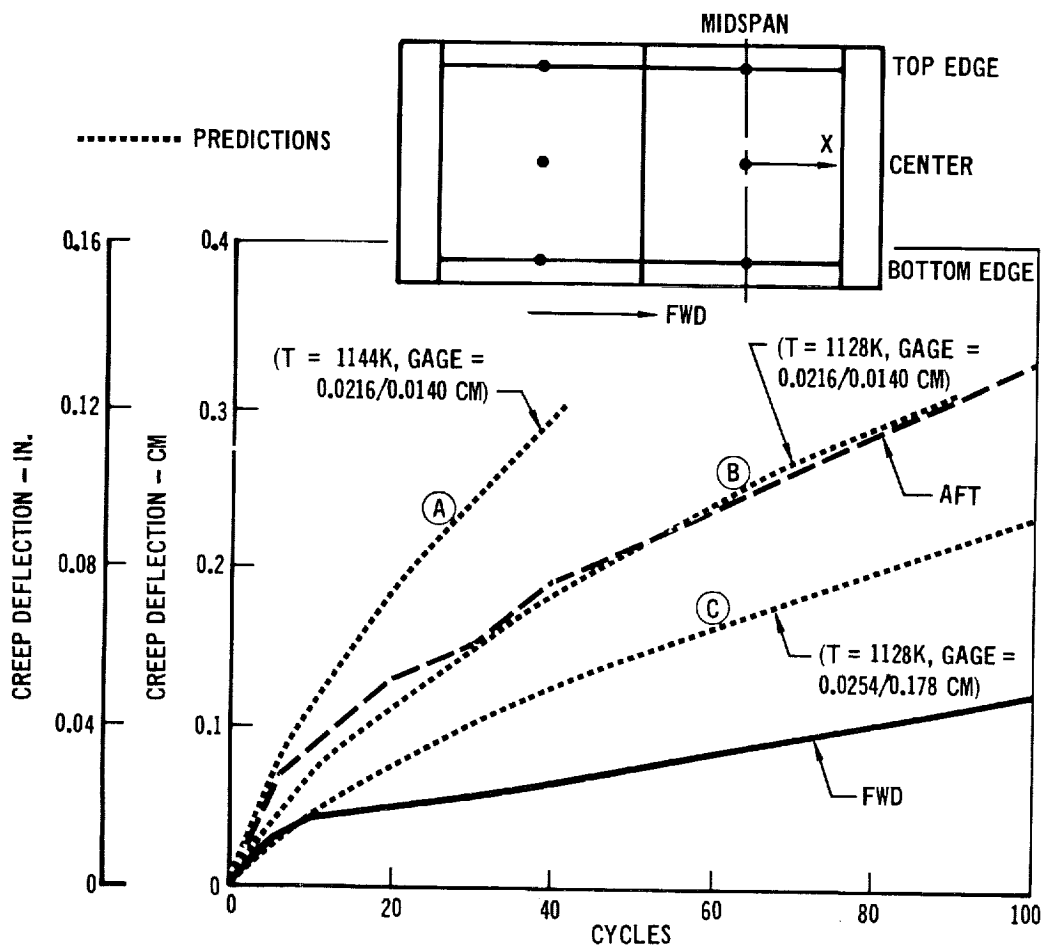
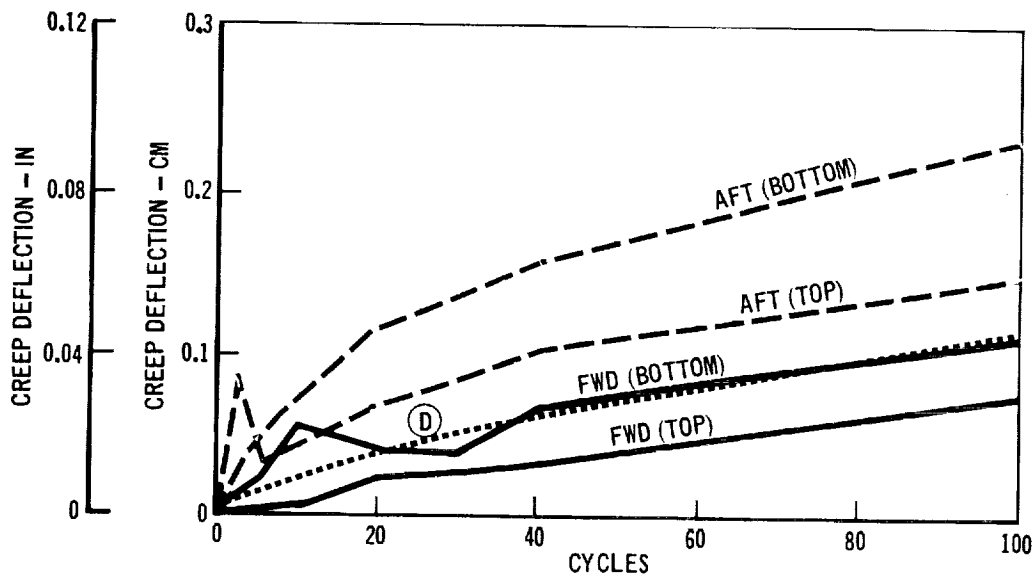


FIGURE 3-7 RENE'41 ENTRY PROFILE USED IN CREEP PREDICTION ANALYSIS

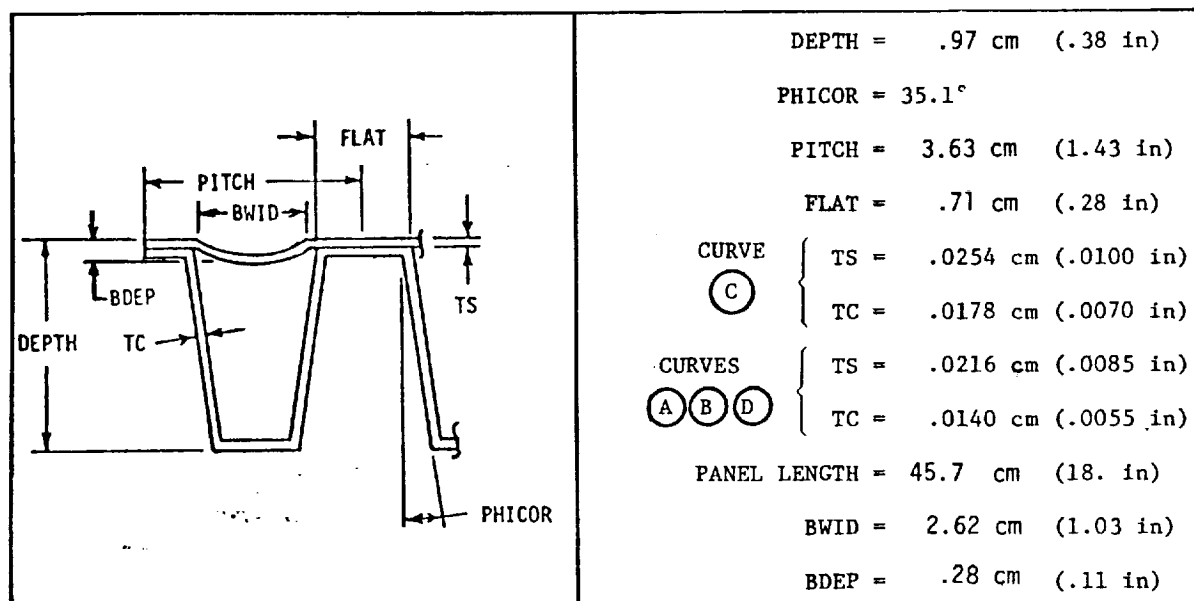


(a) Midspan Deflections Along Center Span



(b) Panel Midspan Deflections Along Edge

FIGURE 3-8 COMPARISON OF RENE'41 PANEL TEST AND PREDICTED CREEP DEFLECTIONS



(a) Rene' 41 Panel Geometry

TIME (MIN)	PRESSURE KPa (PSI)	TEMPERATURE (CONSTANT ALONG LENGTH)		
		CURVE A (FIG. 3-8)	CURVE B (FIG. 3-8)	CURVE C (FIG. 3-8)
0 - 4.2	1.03 (.15)	1144 K (1600 °F)	1128 K (1570°F)	1128 K (1570°F)
4.2 - 7.0	2.20 (.32)			
7.0 - 8.0	3.45 (.50)			

(b) Temperatures and Pressures Along Panel Center

TIME (MIN)	PRESSURE KPa (PSI)	TEMPERATURE - K (°F)					
		X=0	X=5.1	X=10.2	X=15.2	X=20.3	X=22.9
0 - 4.2	1.03 (.15)	1100 (1520)	1111 (1540)	1111 (1540)	1097 (1515)	1083 (1490)	1081 (1485)
4.2 - 7.0	2.20 (.32)						
7.0 - 8.0	3.45 (.50)						

(c) Temperatures and Pressures Along Panel Edge

TABLE 3-2. GEOMETRY AND LOADING DATA USED IN RENE' 41 PANEL ANALYSIS

### 3.2 GRUMMAN TDNiCr PANEL

Evaluation of TDNiCr, from the standpoint of creep deflections in TPS panels, represents a different case than the other TPS materials because relatively little creep is evident in this material before failure occurs. Because of these low creep strains and resulting low test panel deflections, the data have tended to exhibit a greater amount of scatter.

The TDNiCr panel data evaluated in this section were obtained from Reference 8. The TPS panel tested consisted of a corrugation stiffened TDNi-20 Cr metallic heat shield backed by a flexible fibrous quartz and radiative shield insulation system. The test article represents the intersection of two 50.8 cm (20 inch) square panels as shown in Figure 3-9. Each panel consists of a beaded 0.025 cm (.010 inch) skin and corrugation. Detail dimensions of the corrugation cross section, used in analysis for panel deflections, are presented in Figure 3-10.

These panels were tested to 90 cycles of combined pressure and temperature loading, simulating critical heating and aerodynamic pressure environments expected during repeated missions of a reentry vehicle. Prior to these 90 cycles, the panels were subjected to 10 cycles of heating conditions only. Typical thermal distributions determined during these cycles are included in Figure 3-9 at various locations on the panels.

Entry test profiles (Reference 8) and idealizations used in the analysis are shown in Figure 3-11. For purposes of analysis these profiles were idealized into the three constant load and temperature steps shown. These temperatures were assumed to be constant along a panel length of 47. cm (18.5 in.). Analysis of the corrugation panel geometry under the idealized loads and temperature profiles was conducted using the analytical methods developed in Phase II (Reference 2). The empirical creep strain equation (Table 2-1) developed in Phase I, was used in the

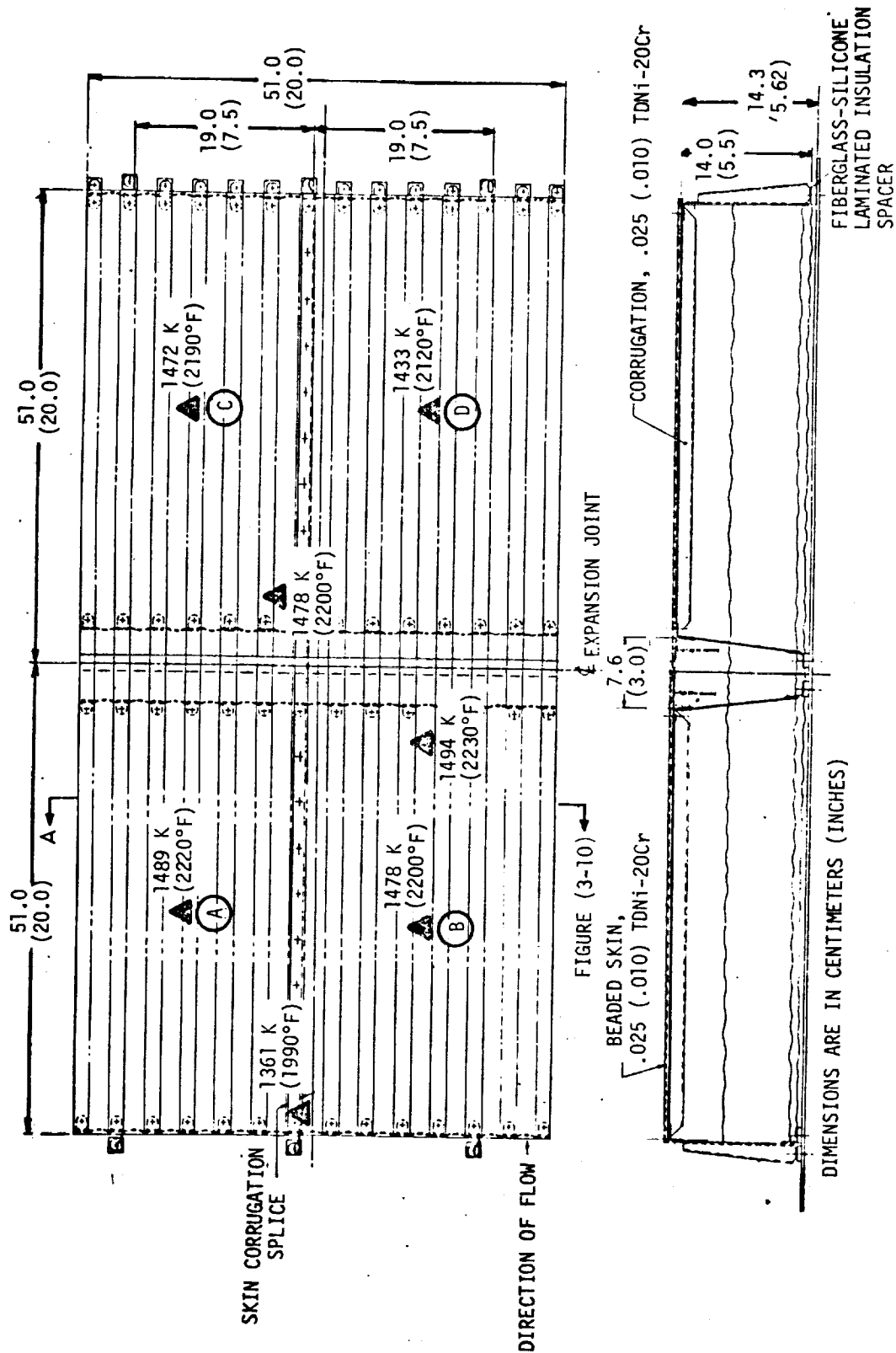
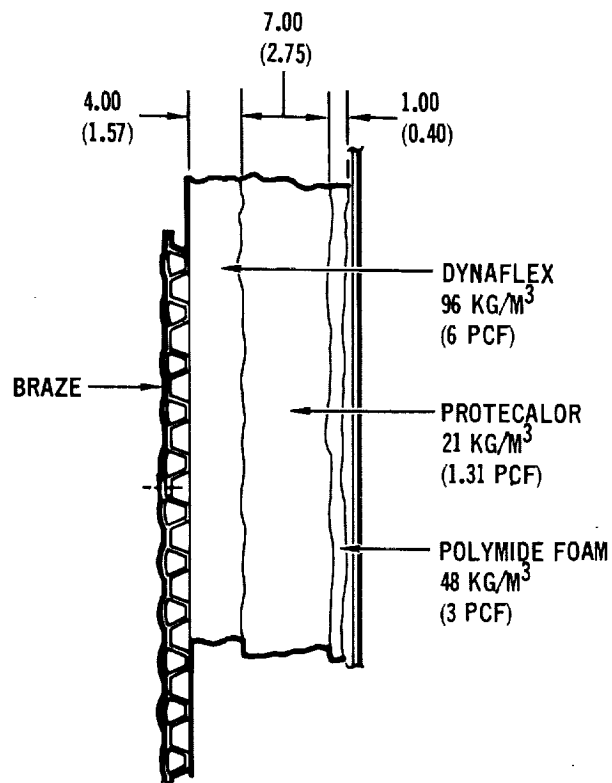


FIGURE 3-9 GRUMMAN TD NiCr TEST PANEL SETUP



Section A - A (Figure 3 -9)

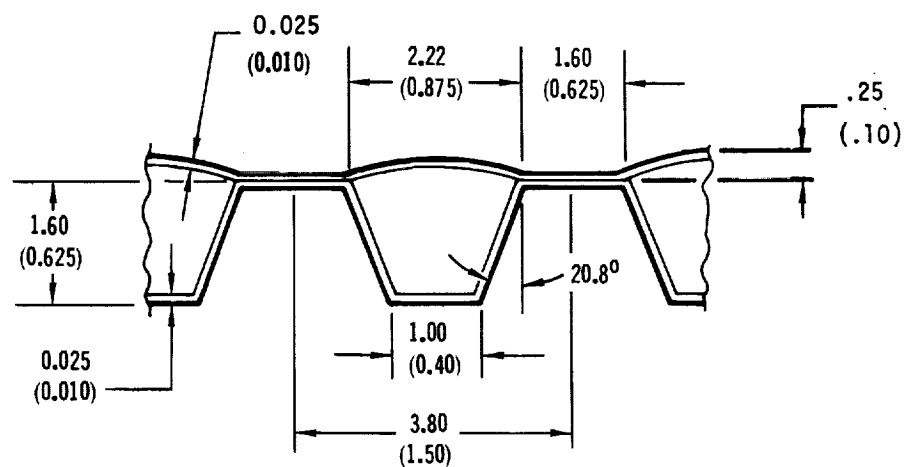


FIGURE 3-10 TDNi Cr TEST PANEL CORRUGATION DEFINITION

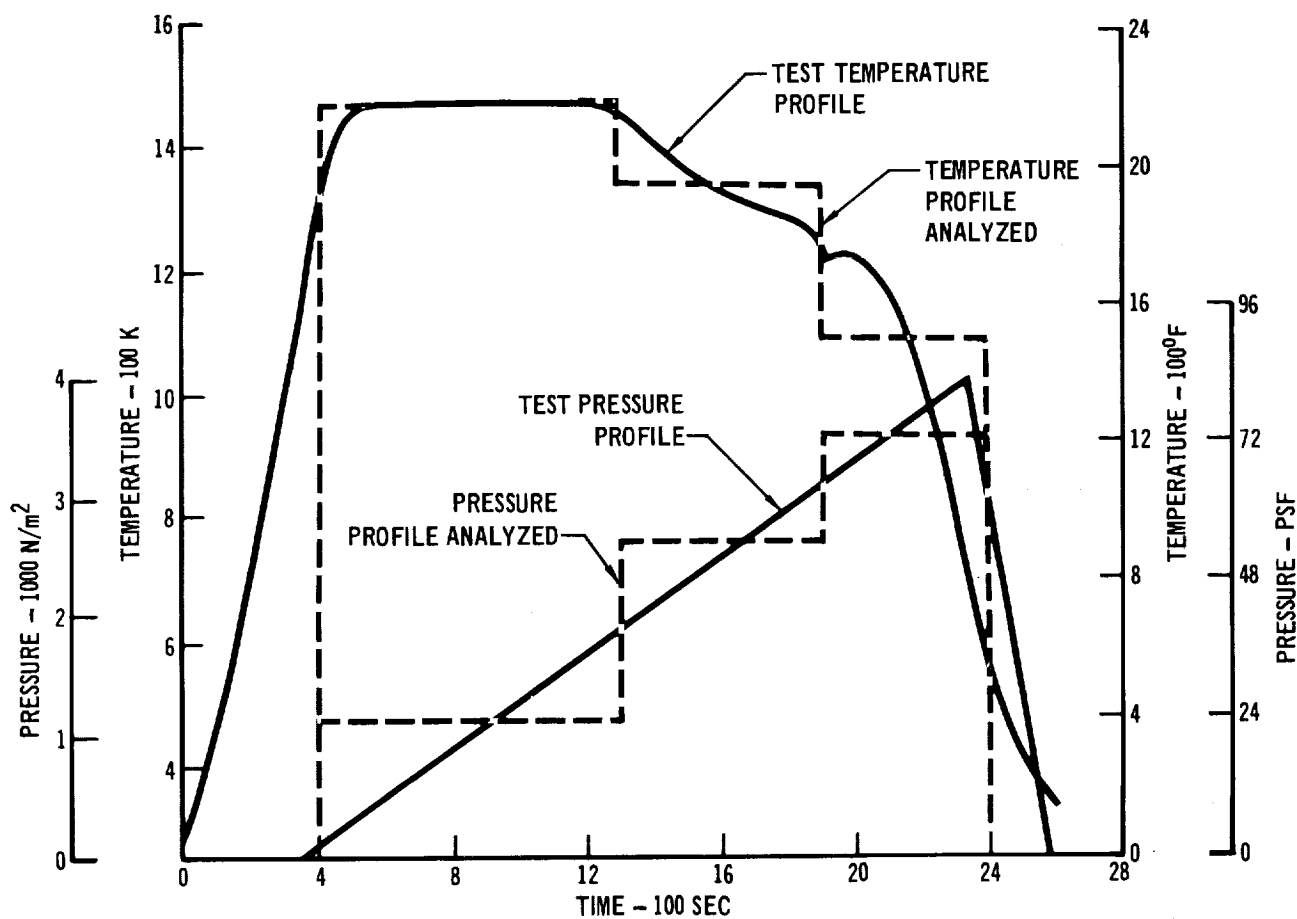


FIGURE 3-11 TEST AND IDEALIZED LOADING AND TEMPERATURE PROFILES

analysis to represent the material creep strain response and the time hardening theory of creep accumulation was applied.

Shown in Figure 3-12 are comparisons of the predicted creep deflections with measured permanent test deflections. The test deflections are plotted from initiation of the combined load and temperature cycles for four midspan locations referenced to the panel geometry in Figure 3-9. A significant variation is noted in these test data. In the reference 8 report the variation was attributed to the slightly higher temperatures observed at locations A and B. In addition it was noted that there was a significant increase in permanent deflection at locations C and D, between cycles 1 and 9. This was attributed to residual stresses, built into the panel during manufacture and assembly as well as thermally induced loads. Therefore, there remains some question as to the true amount of creep occurring between cycles 1 and 9.

The predicted creep deflections indicate a lower rate of creep than observed in testing. This is attributed to the empirical equation in the stress and temperature range applied. The analysis showed that approximately 75% of the creep occurred during the first load-temperature step in the profile (Figure 3-11). Temperature during this step was 1478K (2200°F) and calculated corrugation outer fiber stress was approximately 23 MPa (3300 psi). Comparison of Phase I cyclic tensile creep data with the empirical equation predictions shown in Figure 2-4 also indicate the lower slope of the predicted strains. The wide variation in creep deflections cannot be predicted, based on the temperature data.

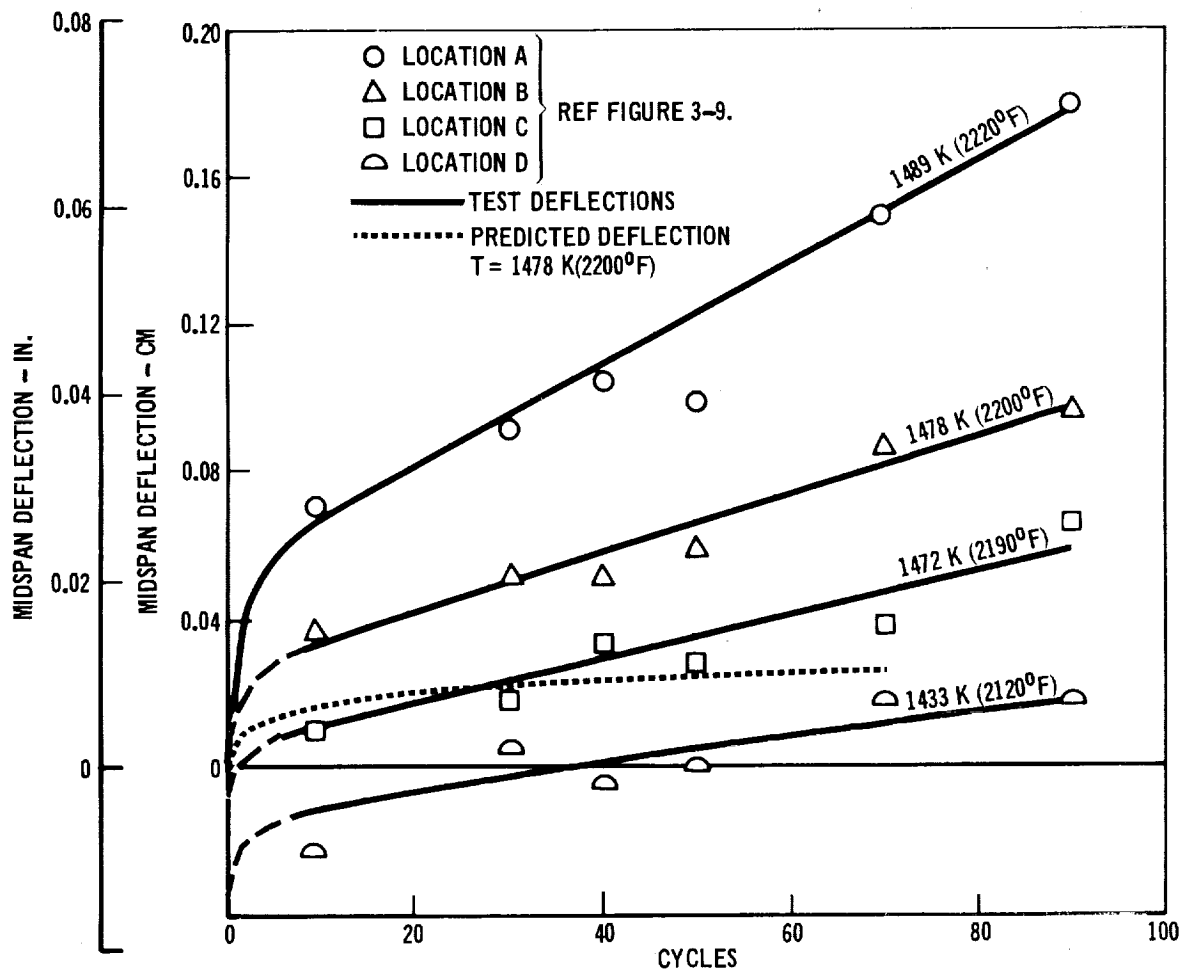


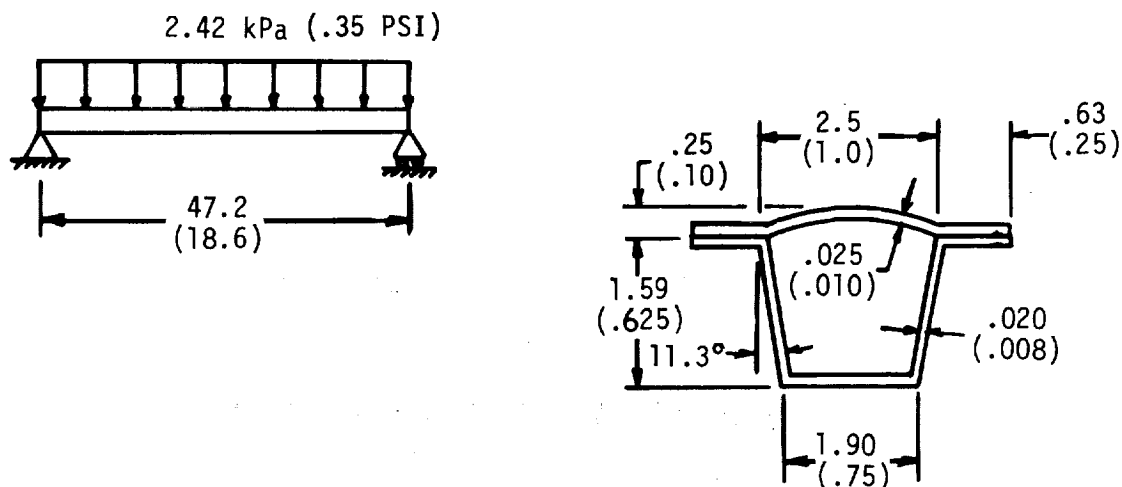
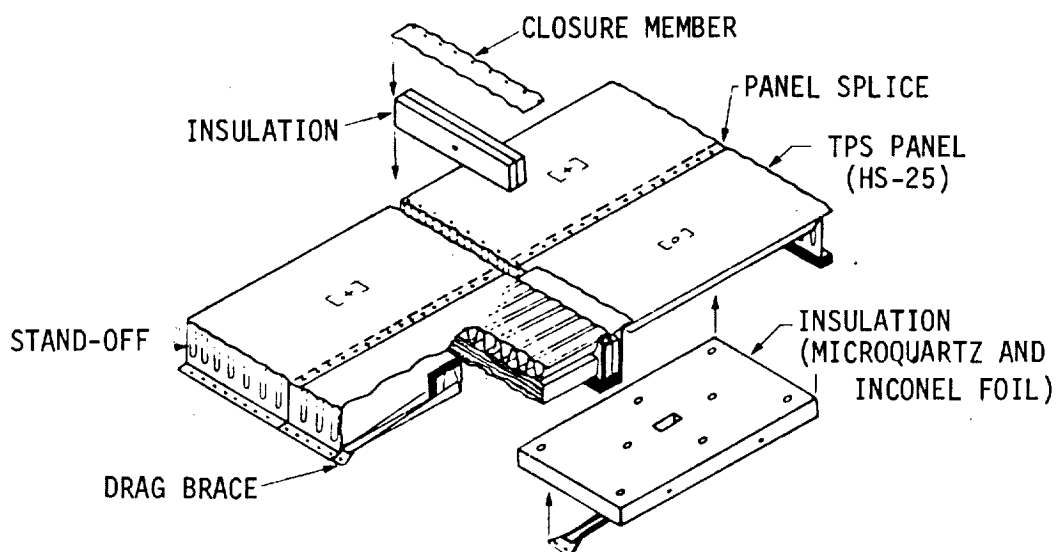
FIGURE 3-12 COMPARISON OF TD NiCr PANEL TEST DEFLECTIONS  
AND PREDICTIONS



### 3.3 GRUMMAN HAYNES 25 PANEL

A Haynes 25 (L605) panel was tested by Grumman Aerospace Corporation and results presented in References 6 and 7. The panel tested was designated as panel No. 3 in the references and was segmented, as shown in Figure 3-13, into four separate test panels. The cross section geometry was single face corrugation stiffened with a skin bead of approximately 0.25 cm (0.10 in.) depth. These panels were supported at the ends (simple support assumed for analysis) over a 47.2 cm (18.6 inch) span and subjected to a uniform pressure profile of 2.42 kPa (0.35 psi). The temperature profile used in the cyclic testing is presented in Figure 3-14. Also shown is the two step idealized temperature profile used in the analysis. Panel geometry and dimensions used in the analysis are provided in Figure 3-13. For each of the panels, analysis was conducted for two different temperature levels because of the cycle to cycle test temperature variations as indicated in Figure 3-15. The time hardening theory of creep accumulation was applied in conjunction with the L605 cycle creep empirical equation (Table 2-1) developed in Phase I.

Comparison of resulting predictions with the Reference 6 experimental data, shown for the NE and SW panels in Figure 3-16(a) and (b), respectively, show that the experimental deflections are considerably higher than predicted. No explanation of this variation between theory and test has been determined based on the data in the reference.



DIMENSIONS ARE CENTIMETERS (INCHES)

FIGURE 3-13 PANEL GEOMETRY FOR GRUMMAN HAYNES 25 PANEL TESTS

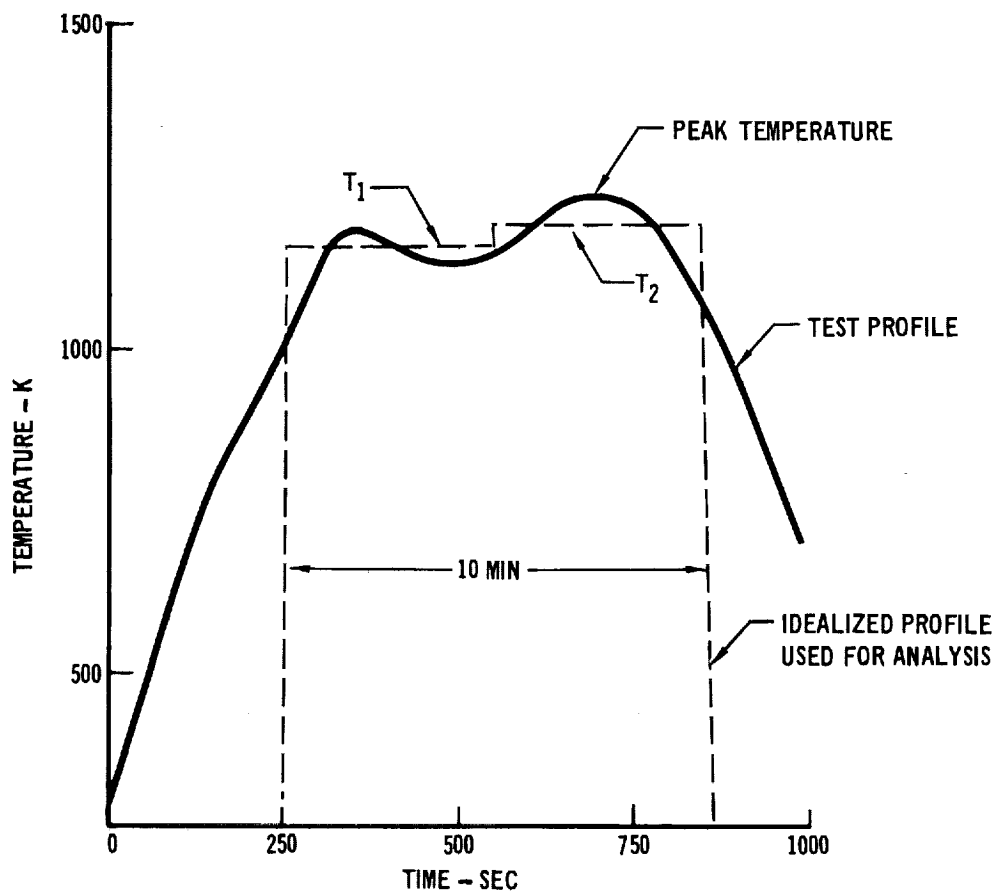


FIGURE 3-14 HAYNES 25 PANEL TEMPERATURE PROFILE

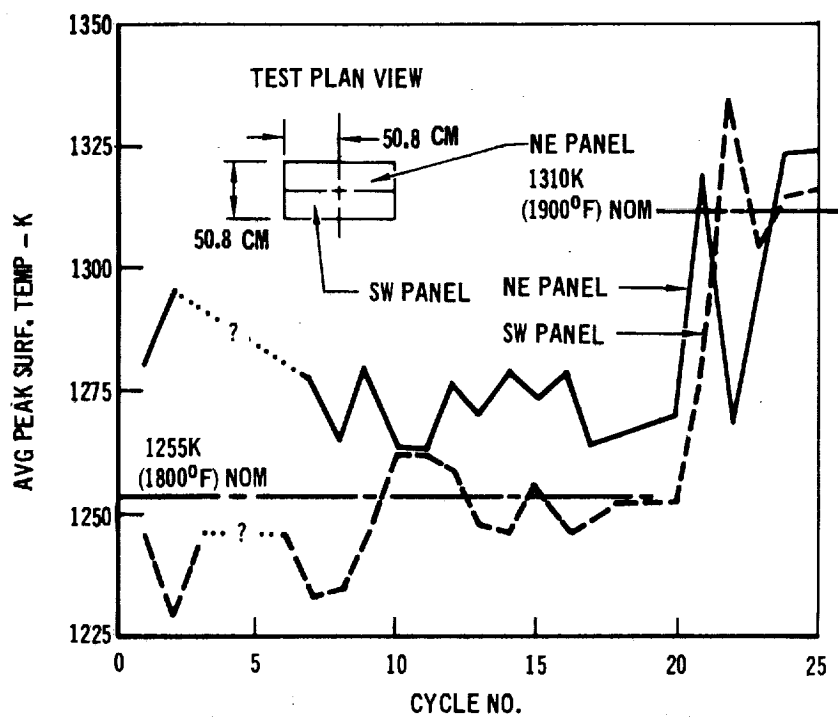
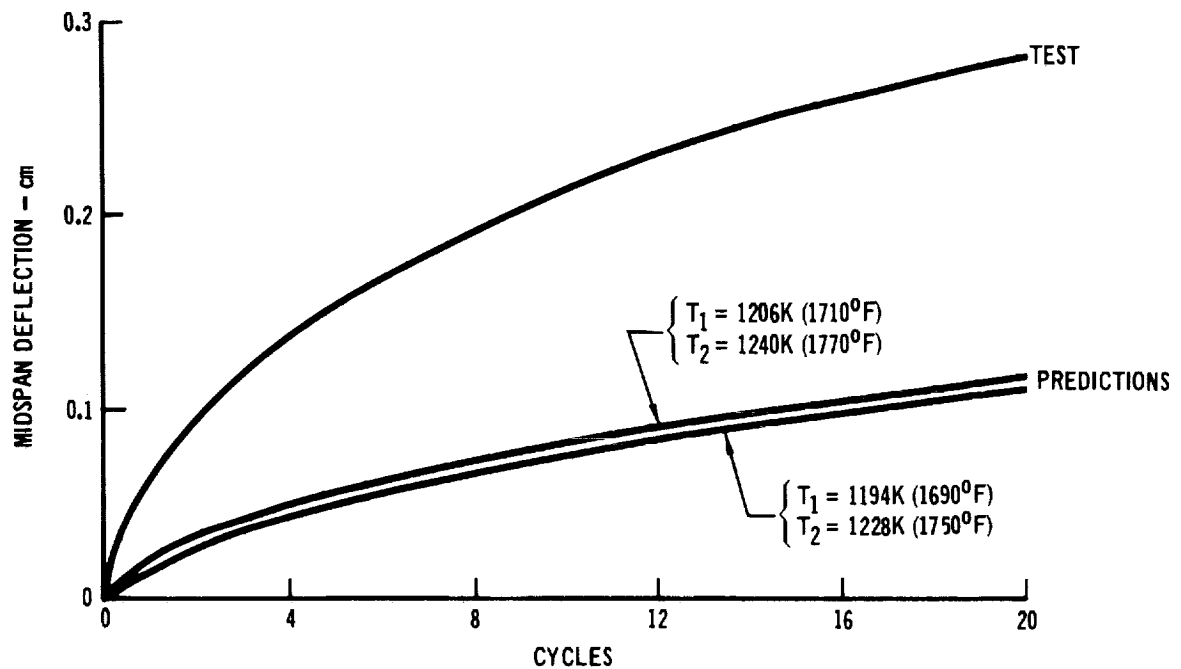
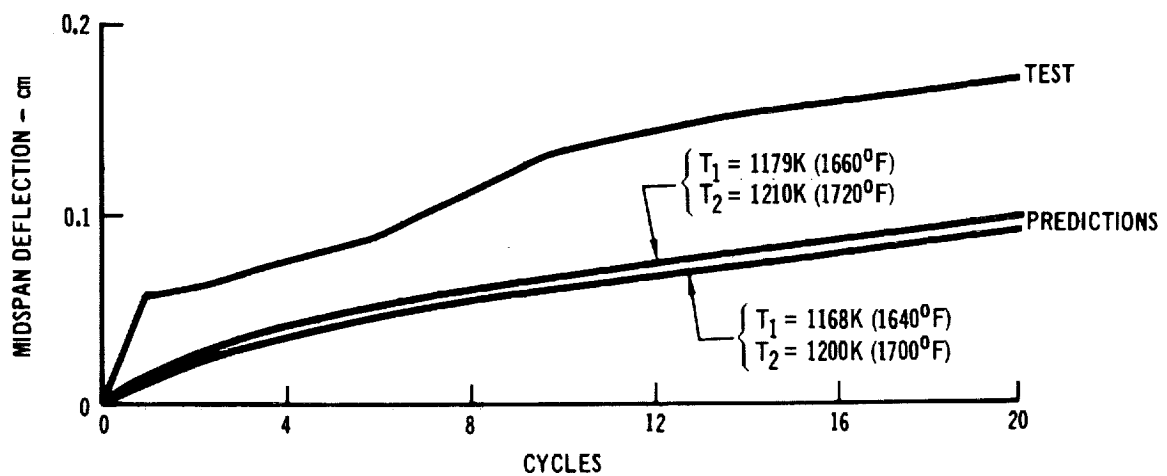


FIGURE 3-15 HAYNES 25 PANEL AVERAGE PEAK MEASURED TEMPERATURE



(a) NE Panel



(b) SW Panel

FIGURE 3-16 COMPARISON OF PREDICTED CREEP DEFLECTIONS WITH  
TEST DATA FOR HAYNES 25 PANELS

4.0 PHASE III CONCLUSIONS

Comparison of permanent cyclic creep deflections, obtained in testing of full size thermal protection system panels, with predicted values has met with varied degrees of success. Prediction capability for L605 and Rene' 41 appears to be reasonably good, although there is much variation in the test data, even for panels tested simultaneously to the same temperature and load level. Prediction capability for TDNiCr appears to be less accurate although recognition of the low creep rate of TDNiCr has led to minimization of effort on creep response definition throughout the program. Predictions for the full size panels are summarized in Table 4-1, showing that prediction accuracies cannot, in general, be expected to be better than a factor of two. These predictions were made using the time hardening theory of creep accumulation.

Resulting predictions of cyclic creep deflection have been shown to be sensitive to both stress level and temperature. This makes prediction capability more difficult since variations from cycle to cycle were known to occur but may not have been defined for each cycle. Such factors as an overshoot in temperature for only one cycle or a few cycles could significantly increase the total test deflections attained. In addition, the test panels were generally subjected to other environments such as high launch phase loading and acoustic environments, which possibly contribute to redistribution of panel relative displacement and variation in the data.

TABLE 4-1. SUMMARY OF FULL SIZE PANEL CREEP DEFLECTIONS

TEST PANEL		RANGE OF TEST DEFLECTIONS Cm	PREDICTION Cm	PREDICTION AS % OF TEST AVERAGE
L605 (Sec. 3.1.1)  30 Cycles	CENTER	.071 - .127	.064	64%
	EDGE	.036 - .066	.043	85%
Rene' 41 (Sec. 3.1.2)  100 Cycles	CENTER	.127 - .330	.033	144%
	EDGE	.076 - .239	.114	73%
TDNiCr (Sec. 3.2)  50 Cycles		.013 - .305	.064	40%
Haynes 25 (Sec. 3.3)  20 Cycles		.170 - .290	.100-.120	48%



## 5.0 THERMAL PROTECTION SYSTEM DESIGN CRITERIA

During the course of Phases I, II, and III of this program several factors affecting creep of metallic TPS and considerations in the design and analysis of metallic TPS have been identified. During Phase I (Reference 1) tensile creep testing was conducted on L605, Ti-6Al-4V, Rene' 41, and TDNiCr specimens under both steady state and cyclic loading and temperature conditions. Test matrices were established to provide maximum data throughout the temperature, stress, and strain range of interest with a minimum number of tests. Resulting data were analyzed to provide empirical equations expressing both steady state and cyclic creep strain as a function of temperature, stress, and time. Additional tests were conducted to evaluate other factors influencing cyclic creep strain such as the applicability of creep accumulation theories and effects of test time per cycle and material thickness. During Phase II (Reference 2) methods were developed for predicting creep deflections of thin gage metallic thermal protection system panels subjected to complex temperature and loading environments. Subsize panels, fabricated from the same material as used in Phase I, were tested to provide data for analysis verification. In the analysis of these data, factors such as sensitivity of the prediction to temperature variations were studied and expected accuracies were noted. Analysis of full size TPS panel test data in Phase III provided additional insight into expected analysis accuracies.

This section summarizes program results in a format which can serve as a criteria in accounting for creep in the preliminary design of metallic thermal protection systems. In addition to specific information obtained on this program, applicable experience based on results from other programs felt applicable to creep of TPS is also included.

## 5.1 GENERAL CONSIDERATIONS

### Critical Design Conditions

TPS panels must first be sized based on strength and stiffness considerations over the entire range of flight conditions. The material choice is dictated by the peak temperatures occurring during entry. Critical design conditions have generally been found to be peak pressure loads and acoustic loadings occurring at relatively low temperatures during ascent or cruise conditions. Envelopes of panel strength and flight conditions such as that demonstrated in Figure 5-1 are helpful in visualizing the critical conditions for these panels. The example shows the panel to be critical during cruise where the peak pressure is applied at low temperatures. The panel strength then exceeds requirements throughout the remainder of the mission.

### Panel Deflections

Deflections which must be considered are elastic deflections of the panel under applied differential pressure loads, thermal deflections which result from temperature gradients through the panel depth, and permanent creep deflections which accumulate throughout the life of the TPS panel. Various allowable deflections have been established such as those in References 9 and 10 which are shown in Equation (1) and (2) respectively.

$$\delta = .25 + .01L \text{ cm} \quad (1)$$

$$\delta = .25 + .04L [(B.S.-30.5)/280] \text{ cm} \quad (2)$$

where B.S. = VEHICLE BODY STATION

These equations provide for maximum deflections of .75 cm and 2.25 cm (@ B.S.=787 cm (310 in)), respectively for a 50 cm (20 inch) long panel. Allowable total deflections must be established for each system based on the thermodynamic and aerodynamic requirements.

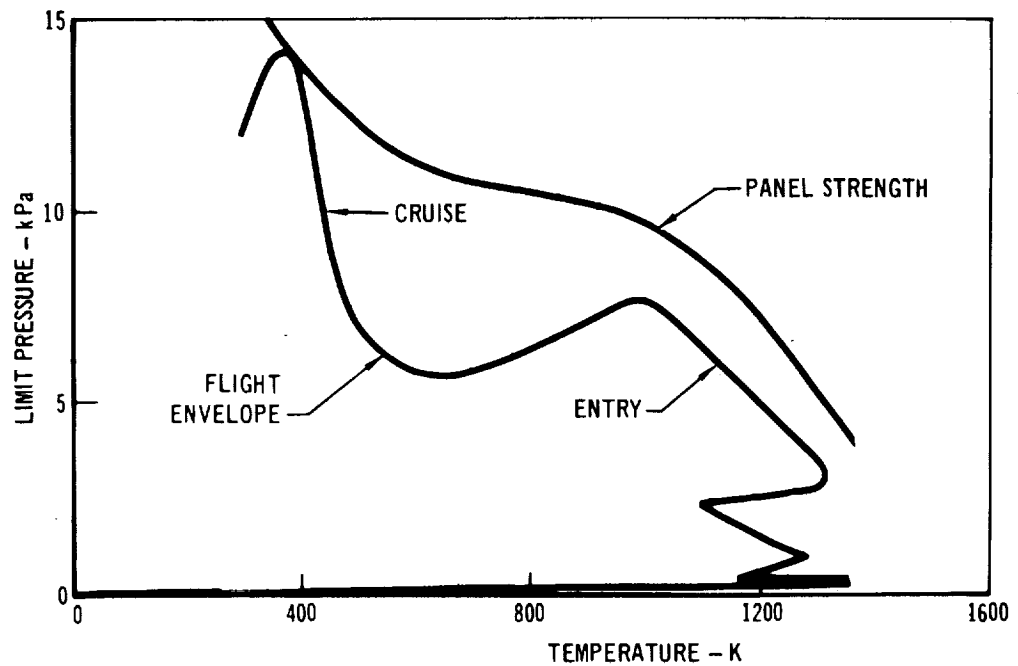


FIGURE 5-1 L 605 PANEL STRENGTH FOR FLIGHT ENVELOPE

The relative importance of thermal deflection has generally not been assessed in past studies. These deflections will be zero during steady state conditions where temperatures are uniform through the panel depth. During heating, when the maximum temperature occurs on the outer surface of the panel, the deflections will be in the opposite direction from the elastic and accumulated creep deflections.

#### Panel Replacement and Inspection

In the context of panel deflections, a failure will be an excessive deflection which requires panel replacement. Requirements for the panel design deflections will result from trading off refurbishment cost against any weight penalty which might result from the necessity to resist creep deflections.

It is expected that panels in one area of the vehicle might creep much faster than in other areas due to particular mission maneuvers, etc. Therefore, replacement of some panels may be required after each mission. It does not seem to be desirable or possible to optimize these panels from the standpoint of deflection over the entire vehicle since the mission requirements will provide considerable variation in applied loadings from one mission to another and from one location to another location on the vehicle. Visual inspection with spot centerline measurements, using a simple bar/dial gage tool would be sufficient to detect excessive deflections.

#### 5.2 DESIGN CONSIDERATIONS - THERMAL EXPANSION

One of the primary considerations in the design of TPS panels is that the panel be allowed to expand freely under thermal loadings. Allowances must be made for thermal expansion both at the panel joints and on the panel surface.

Expansion is generally accomplished at the joints by fixing the panel at one end and allowing it to slide longitudinally at the other end. Transverse deflections are accomplished by slotted holes at both attachment locations and by

providing for expansion in the longitudinal joints between adjacent panels. Typical designs can be found in References (3) and (11).

Temperature variations along the panel length due to the heat sink at the panel support cause thermal stresses in the transverse direction. During heat up of the panel, the midspan is hotter than the edges at the supports causing compressive stresses at the center and tensile stresses at the edges. These stresses are reversed during cooldown.

The presence of beads relieves the thermal stresses and prevents thermal buckling (Reference 12) of the thin skin between stiffeners by allowing the skin to flex as thermal expansion occurs. Analysis can be used to define required bead depths. Particular attention should be given to the approach for closing out the bead near the panel ends. Testing (Reference 3) has shown that cracking can occur in the skin at the tips of the beads where the beads are transitioned into a flat skin. It would be desirable from this standpoint to extend the bead to the panel ends. This, however, complicates the design at the panel joints.

### 5.3 DATA REQUIREMENTS FOR CREEP ANALYSIS

During Phase I (Reference 1) testing was conducted under both steady state and cyclic conditions to evaluate the creep response characteristics of the materials studied and to provide data for use in the analysis for panel creep deflections. During these studies considerable effort was directed at obtaining the required test data.

#### Test Matrix - Basic Data Required

One of the objectives in evaluating creep deflections should be minimizing the required testing. However, it is of interest to cover the complete range of stress, temperature, time, and strain required to provide an adequate material response definition for use in the analysis. The analyst does not want to be in

the position where extrapolation of the available data is required.

The range of strain which is required will be dependent upon the criteria for allowable deflection used. As an example of possible calculations it could be assumed that creep deflections obtained in testing will be approximately 50% of those obtained using a linear creep stress-strain assumption. This assumption tends to account for the redistribution of beam stressed due to nonlinear creep strain-stress properties. The assumption is expressed in the following equation:

$$\frac{\Delta_c}{\epsilon_c} = .5 \frac{\Delta_E}{\epsilon_E}$$

where:  $\Delta_E$  = BEAM midspan elastic deflection

$\epsilon_E$  = Maximum midspan elastic strain (extreme fiber)

$\Delta_c$  = Beam midspan creep deflection

$\epsilon_c$  = Maximum midspan creep strain (extreme fiber)

Applying this equation and assuming an elastic deflection based on a uniform pressure loading the following equation can be derived for creep strain at the beam midspan.

$$\epsilon_c = \frac{2\Delta_c}{\Delta_E} \epsilon_E = \frac{2\Delta_c}{\frac{5}{384} \frac{WL^4}{EI}} \frac{WL^2 \bar{Y}}{8 EI} = 19.2 \frac{\Delta_c \bar{Y}}{L^2}$$

Where: W = Beam pressure load

L = Panel length

E = Elastic modulus

I = Panel moment of inertial

$\bar{Y}$  = Maximum distance from neutral axis to extreme fiber

For a full size panel of 50 cm length,  $\Delta_c = .75$  cm (based on  $.25 + .01L$  cm criteria), and  $\bar{Y} = 1.5$  cm, the required creep strain would then be .86%.

It is of interest to note that if this calculation was carried out for a shorter TPS panel, the creep strain required to attain the same creep deflection is higher because the strain is inversely proportional to the square of beam length. Therefore, use of a deflection criterion with subsize panels results in requirements for greater creep strains than would be attained in a full size panel under the same criterion.

Test matrices can be established on stress-temperature charts upon which approximate constant strain lines can be drawn. In the Phase I studies (Reference 1) these were based on evaluation of steady state literature survey data. Typical designs for the test matrices are shown in Figure 5-2 based on the Reference L605 evaluation. Requirements for the designs include:

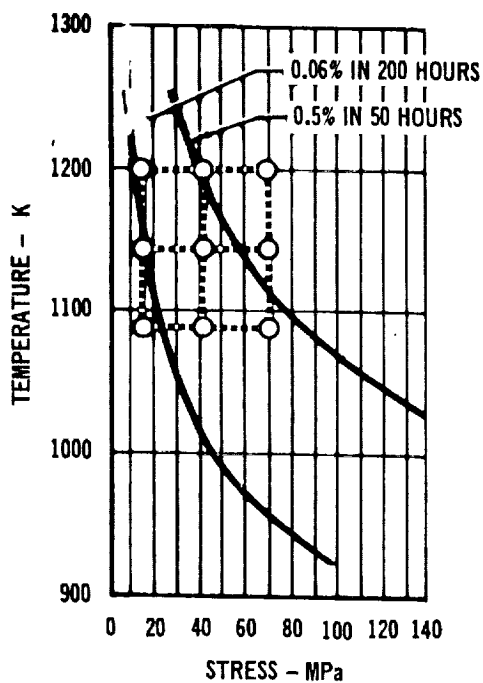
(1) Test data should be amenable to development of an empirical creep strain equation. Applicability of each design for satisfying this requirement can be checked by generating simulated creep strain data using an available equation, performing regression analysis, and evaluating the resulting prediction equation.

(2) Test temperatures should cover the range of interest for the material being tested.

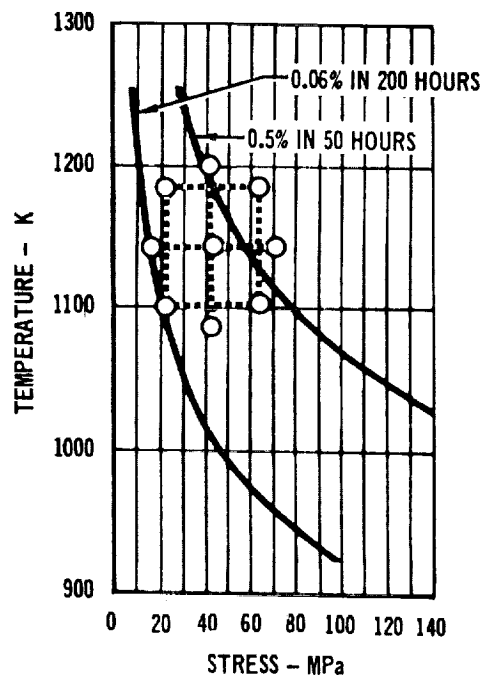
(3) Test temperatures and stress levels should produce creep strains in the range of interest.

The designs shown in Figure 5-2(a) and (b) include a simple  $3 \times 3$  factorial design and an orthogonal composite design. They are described in References 14 and 15. Although both designs satisfy the first requirement (1) above, they may satisfy the second or third requirements, in this case as indicated in the figure.

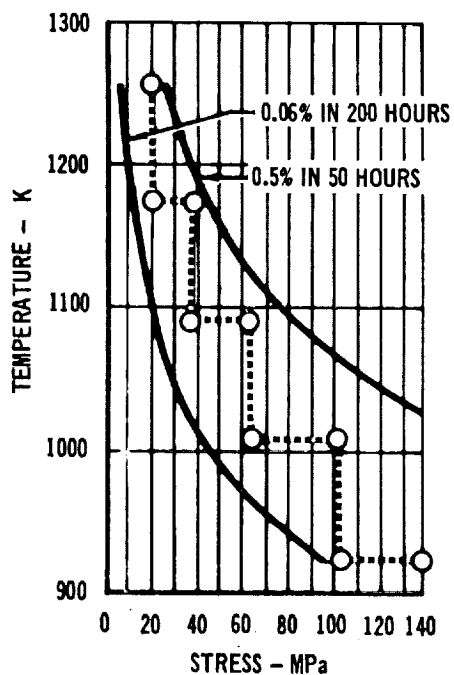
In addition to these two designs, the design shown in Figure 5-2(c) was also considered because it provides maximum coverage of the test temperature and stress range of interest. However, it was subsequently demonstrated that the resulting prediction equation, based on this design, was a function of time only.



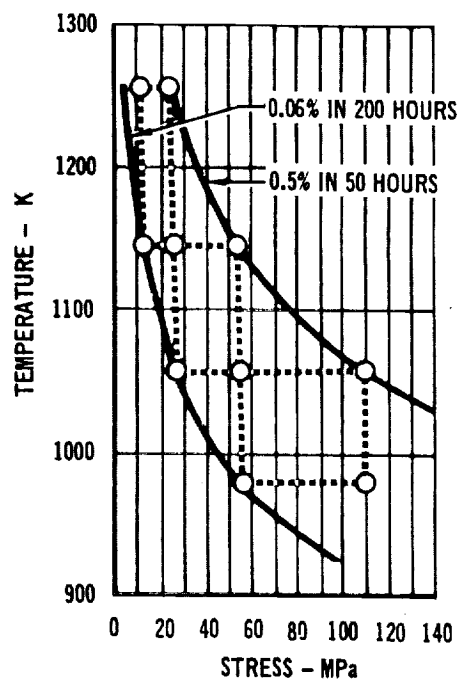
(a)



(b)



(c)



(d)

FIGURE 5-2 EXPERIMENTAL DESIGNS FOR CREEP TESTING

A fourth design considered is shown in Figure 5-2(d). This design allows testing over a wide stress and temperature range and evaluation of the design indicated that an empirical equation can be derived from the resulting data.

For the data range of interest in this program it was found that the design shown in Figure 5-2(d) was best.

The orthogonal composite design (Figure 5-2(b)) was, however, used in the Rene' 41 evaluation (Reference 1) where the lines of constant strain were found to be further apart on the stress temperature plot.

A study of proposed test designs is recommended, using applicable regression techniques, prior to conducting creep tests.

#### Determination of Empirical Equations

A very large number of equations are found in the literature which have been developed over the years to describe the complex physical process of creep. In addition, an infinite number of new relationships (or models) can be formulated.

The description of a new equation involves the determination of the relationship between the dependent variable, strain, and the independent variables, such as temperature, stress, time, thickness, and orientation. A convenient procedure for determining this relationship is the use of multiple regression techniques. Two parameters associated with this technique are (1) the multiple correlation coefficient,  $R$ , and (2) the standard error of estimate,  $S_y$ . The multiple correlation coefficient is a measure of how well the fitted equation explains the variation in the data (Reference 15). The closer the value of  $R^2$  (or  $R$ ) is to 1, the better the equation will fit the data. The standard error of estimate is an estimate of the variance about the regression line. Therefore, the precision of the estimate would be considered better the lower the value of  $S_y$ . Accordingly, in the development of the various regression equations that were examined during

the program, emphasis was placed on obtaining equations which resulted in large values of  $R$  and small values of  $S_y$ .

The development and selection of each predictive equation generally followed an interactive procedure as outlined below:

Step 1 - Select first order independent variables.

Step 2 - Using variables identified in Step 1, form new independent variables for the regression analysis consisting of higher order terms and interreaction (first and higher order) terms. Many computer programs are available to perform the regression analysis to determine the significant variables from the total identified and constructed in Steps 1 and 2.

Step 3 - Examine the residual of plots of the dependent variable vs. regressed variables. The residual is the difference between what is actually observed and what is predicted by the regression equation. If the proper variables were selected, the residual plots will have a uniform distribution with a zero mean. If the proper variables were not in the equation, then the residual plots tend to take a shape which indicates if the analysis should be weighted or a different term should have been used. An in-depth discussion of the examination of residuals and their significance is presented in Reference 15.

Step 4 - Repeat Step 2 using new variables and compare  $R$  and  $S_y$  with previously established values. Repeat Step 3 (i.e., review of plots of residuals) and form additional independent variables, if required.

Step 5 - Plot predicted creep responses and compare with experimentally observed creep curves with particular emphasis placed in identifying

discrepancies in fit and general form of the predicted surfaces.

Step 6 - If major discrepancies are observed in Step 5, modify and/or add new independent variables and repeat from Step 2.

In general, the regression analyses will be conducted using the natural logarithm of strain,  $\ln \epsilon$ , as the dependent variable. There are two primary advantages in using logarithmic strain which are: (1) the model tends to come closer to minimizing the percentage deviations which is desirable, (2) the model can be forced to satisfy initial boundary value considerations. For example, the model

$$\ln \epsilon = A_0 + A_1 \ln \sigma + A_2 \ln t$$

when transformed to strain becomes

$$\epsilon = e^{A_0} \sigma^{A_1} t^{A_2}$$

and if  $\sigma$  or  $t$  equal zero, the strain is forced to also equal zero. Boundary conditions for the equations should be carefully investigated to insure applicability to low stress and time ranges required in the TPS panel analysis.

Another factor to be considered in obtaining empirical creep equations in the exclusion of high and low strains. By excluding higher strains, a small downward bias, as shown in Figure 5-3, is introduced in the predictive equations. Likewise, a small upward bias is introduced into the predictive equation at low strains when low strains are omitted as is also shown in Figure 5-3.

The justification for removing the low values of creep data is that a significantly higher percent experimental error exists in the measurement of these very low creep strains, and that the standard error of estimate can be dominated by these large observation errors. It should be noted that a weighted least squares analysis could also be performed which would account for the large variance in the low strain regime (Reference 15). However, the complexity of such an approach is greater.

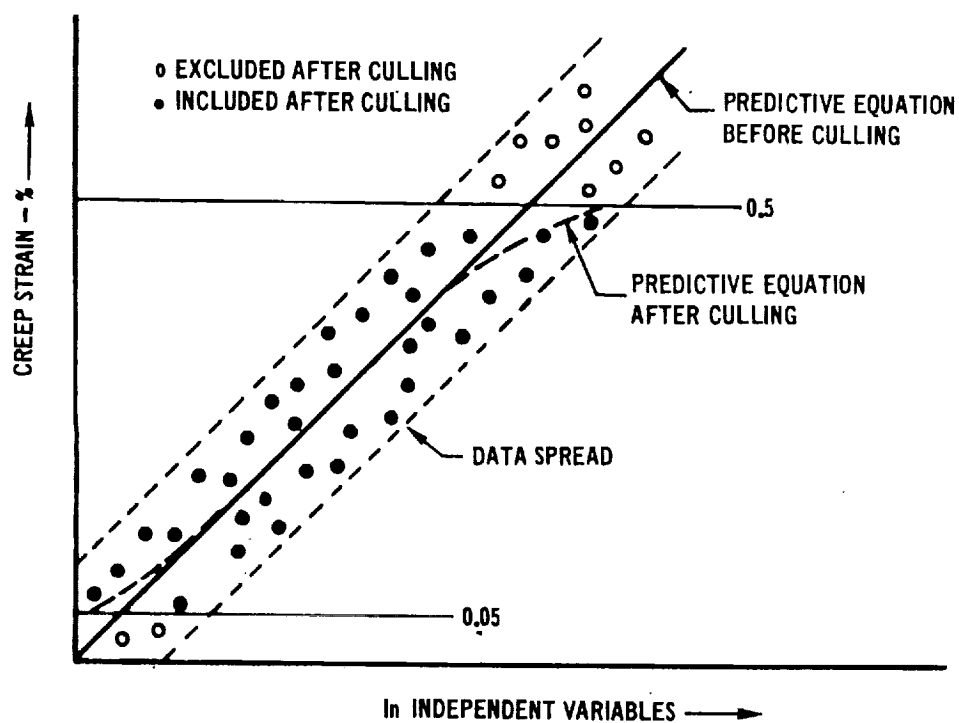


FIGURE 5-3 EFFECT OF CULLING LOW AND HIGH STRAIN DATA ON  
PREDICTIVE EQUATION DEVELOPMENT



### Creep Accumulation Theories

The applicability of creep accumulation theories appears to be the most significant limitation in the analysis for TPS panel creep deflections. During the Reference 1 studies tests were conducted to provide data for evaluation of the hardening theories. An outline of the test profiles used are shown in Figure 5-4. The objective in these tests is to vary the load as a function of cycle to simulate the increasing or decreasing stress which will occur in a panel due to stress redistribution. Additional tests could also be conducted where temperature would be changed as a function of cycle since both temperature and load level change within a cycle is the analysis due to the varying profiles as well as the stress distribution. Predictions of these tests results can be made using empirical equations developed from constant stress and temperature cycle test data, allowing assessment of the various hardening theories.

### Additional Factors Influencing Creep

Assessment of other factors affecting creep may also be important. These factors may include material gage, rolling direction, and possibility of material creep recovery. Evaluation of the effects of atmospheric pressure on creep was also investigated during the Phase I work but was found to have an insignificant effect on the materials investigated. Tensile creep tests can be conducted to assess these effects, if necessary. Steady state tests of specimens at replicate conditions should generally be sufficient to determine any significant variations due to material gage or material rolling direction. No recovery effects were determined for the materials studied based on cyclic tensile testing in Phase I (Reference (1)).

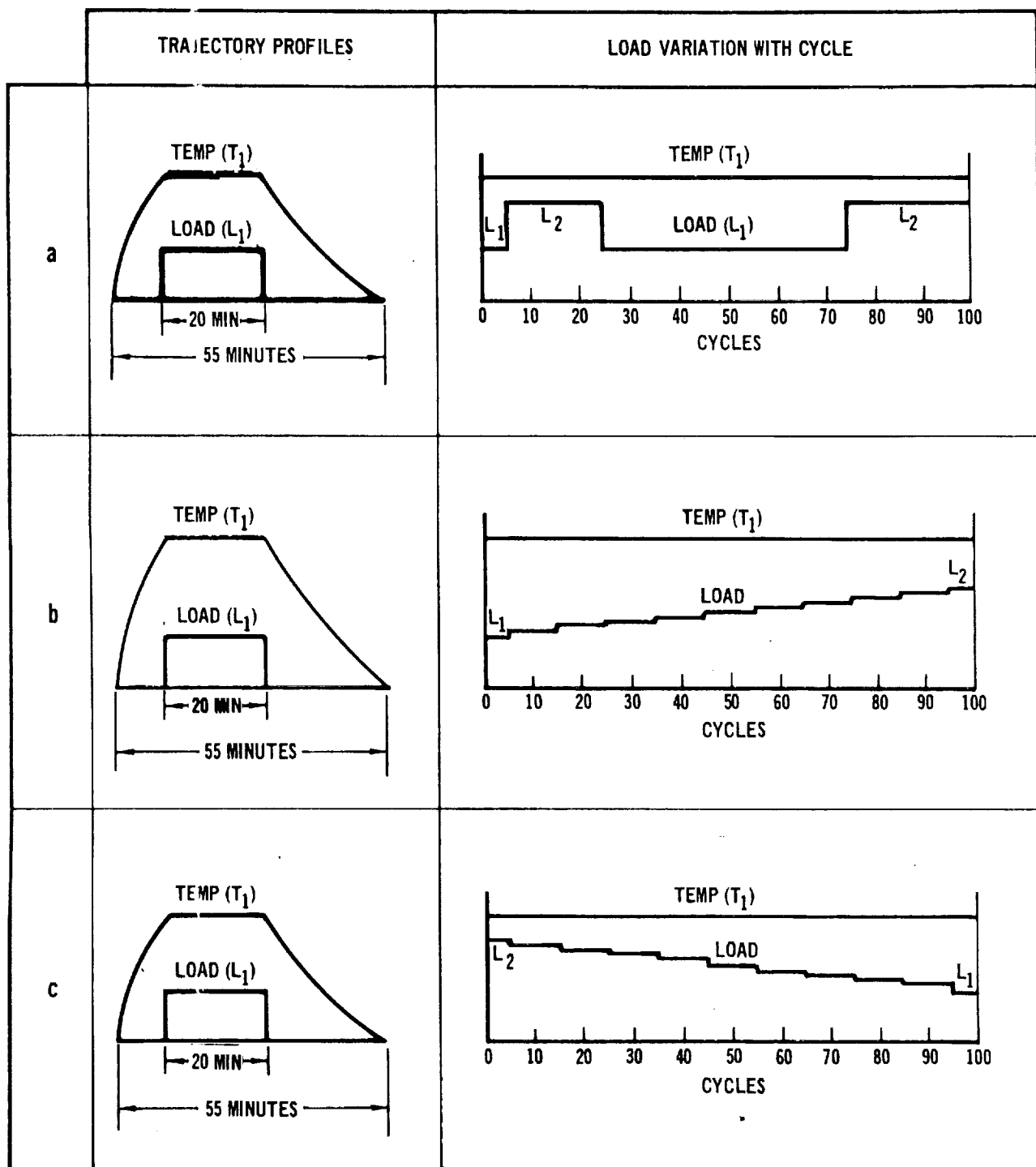


FIGURE 5-4 TESTS FOR EVALUATION OF CREEP ACCUMULATION THEORIES

Sensitivity of Creep Strains to Test Accuracy

In conducting tests to assess creep response characteristics of materials, particular attention should be given to maintaining accuracy, since predictions of creep deflections have been demonstrated to be sensitive to variations in temperature and load. Examples of these variations for the empirical equations of L605, Rene' 41, and TDNiCr are presented in Table 5-1 at a time of 20 hours. Based on results of Phase II (Reference 2) and Phase III, prediction accuracy within a factor of 2 should be attainable.

5.4 CREEP DEFLECTION ANALYSIS

Procedures developed for prediction of creep deflections in Thermal Protection System panels are presented in Reference 2 and Appendix B. Included in these references are the approaches and assumptions made in modeling the thin gage panel structures and in performing the analysis to obtain creep deflections. Steps in the analysis are (a) the development of a linear equation describing the logarithm of strain as a function of stress, temperature, and time, (b) idealize the selected loading and temperature profiles into constant steps, and (3) determine deflections using analysis capabilities of the TPSC computer program. Required input in terms of panel temperature distributions, panel geometry definition, and program control parameters are presented in Appendix B. The time hardening theory of creep accumulation has been found to provide the best predictions for subsize panel test results (Reference (2)) and is, therefore, recommended for use in the analysis.

TABLE 5-1. SENSITIVITY OF CREEP STRAINS TO TESTING ACCURACIES

MATERIAL	PARAMETERS IN STRAIN CALCULATION			CREEP STRAIN %	STRAIN VARIATION DUE TO 1% TEMP. CHANGE	STRAIN VARIATION DUE TO 1% STRESS CHANGE
	TEMP.	STRESS	TIME			
L605	1144K (1600°F)	55 MPa (8 KSI)	20 HR.	.615	14.4	3.0
RENE' 41	1089K (1500°F)	138 MPa (20 KSI)	20 HR.	.270	25.8	2.2
TDNiCr	1478K (2200°F)	35 MPa (5 KSI)	20 HR.	.076	5.8	2.0

#### Selection of "Typical" Profiles

The sensitivity of creep predictions to the temperature profile selection will be illustrated in this section. The curve of creep strain vs. temperature shown in Figure 5-5 is calculated based on the L605 empirical creep equation generated from cyclic tensile tests (Reference Table 2-1). Specific stress and time for the calculated curve are 55 MPa (8.0 ksi) and 20 hours, respectively. The table included in the figure shows resulting creep strains at several temperatures. Also included in the table are the average creep strains over 110K (200°F) temperature increments (i.e. average of strains at T+55K and T-55K) and the percentage errors in creep strain which could result from using the median temperature in each 110K temperature spread. Thus, for example, for a temperature range of 1033K (1400°F) to 1144K (1600°F) the average strain would be .362 which is 35% higher than the value .269 predicted at the median temperature of 1089K (1500°F). This example illustrates the sensitivity of creep strain to temperature and demonstrates that factors such as this should be considered in selecting typical trajectory profiles for analysis. Generally, temperatures higher than the median over the desired range will need to be used in order to arrive at average creep strains and deflections.

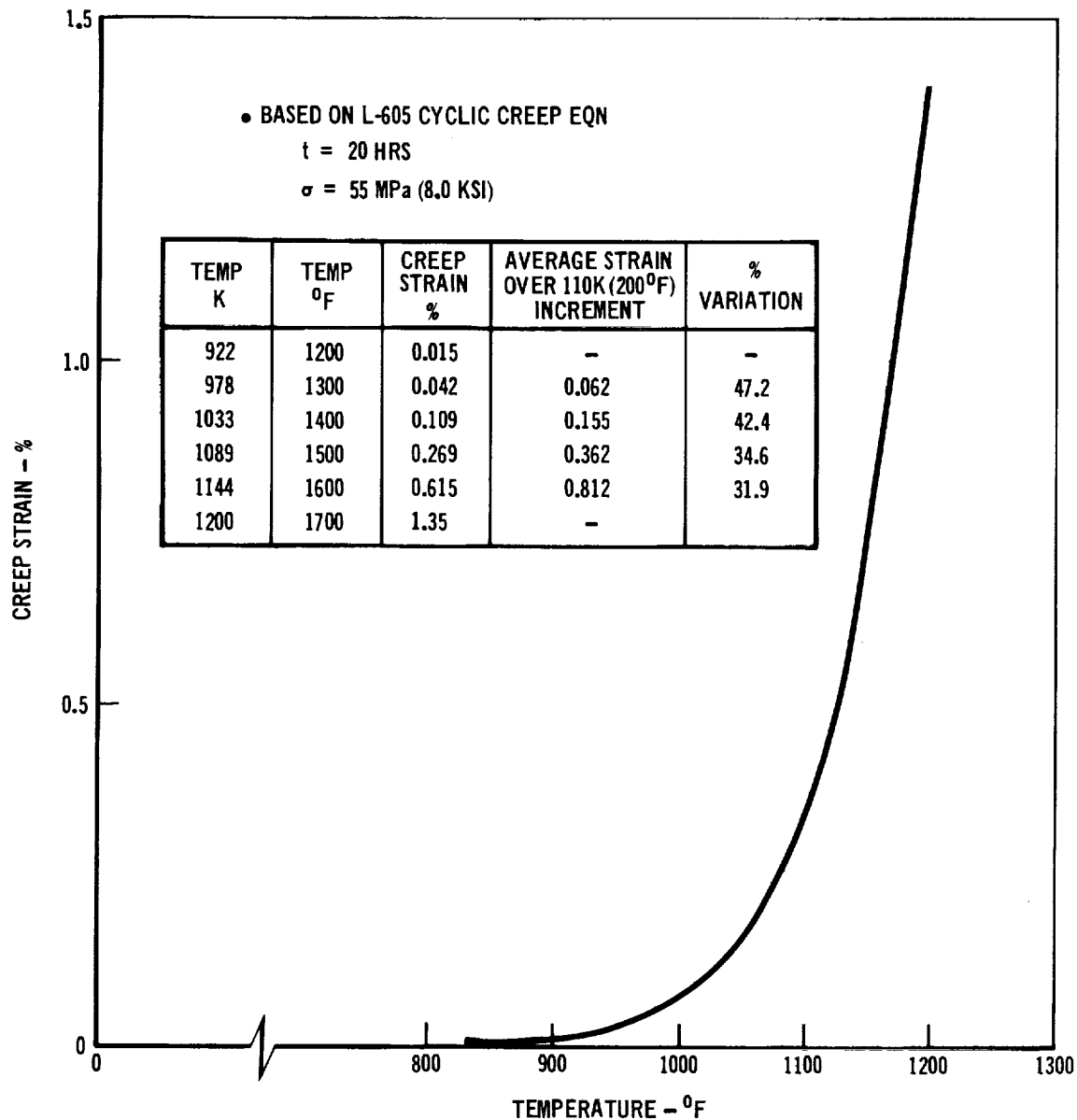


FIGURE 5-5 VARIATION OF CREEP STRAIN WITH TEMPERATURE

Idealization of Load and Temperature Profiles

Some observations related to idealization of load and temperature profiles into constant steps have been noted during the program. Observations, based on comparison of tensile test results in Phase I (Reference 1) and subsize panel data analysis in Phase II (Reference 2) have indicated that a rather simple representation of four time steps in these studies resulted in successful analysis. It was shown that a large number of time steps would not improve prediction accuracy. An example of the steps used is shown in Figure 5-6.

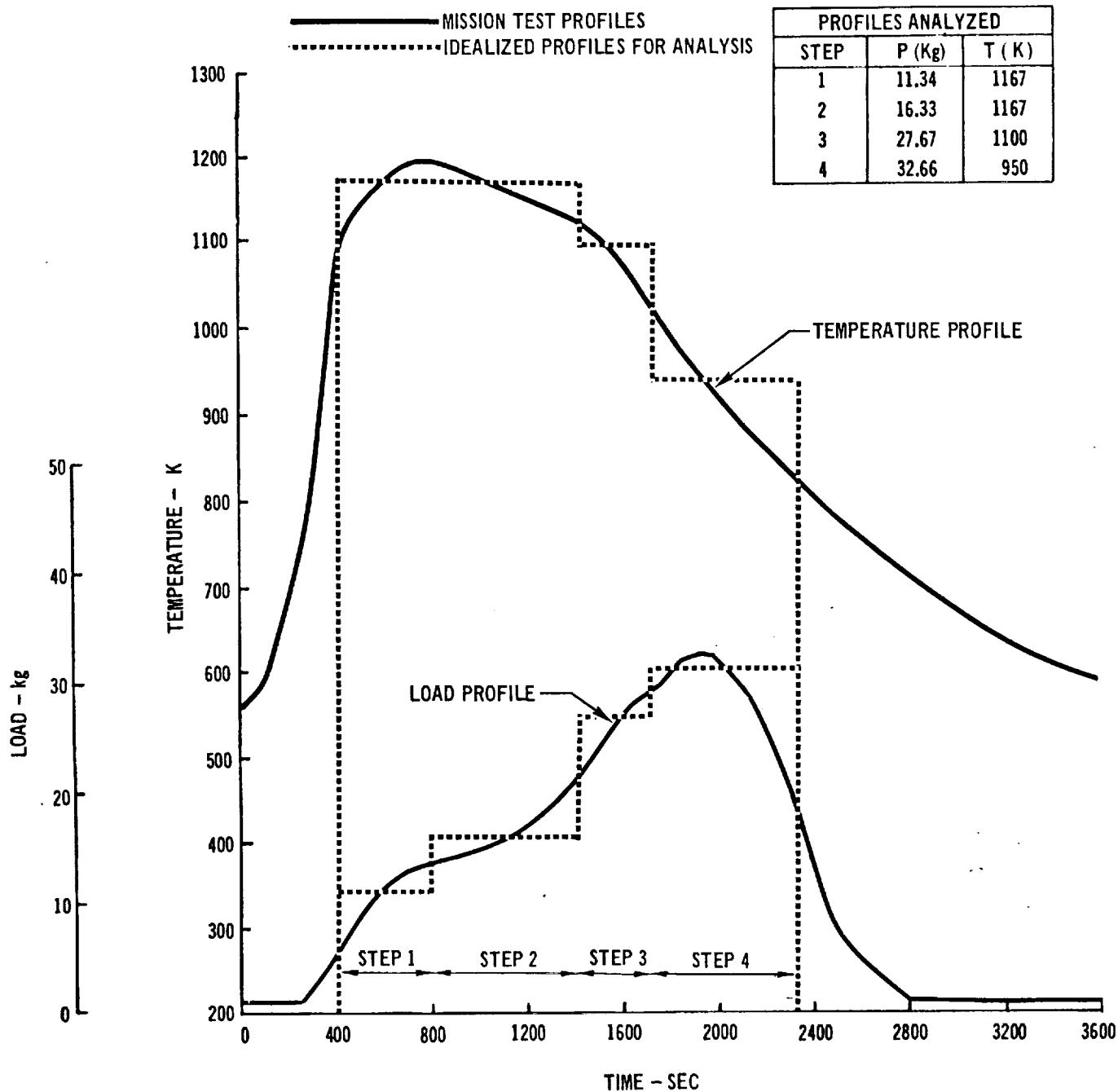


FIGURE 5-6 PROFILE IDEALIZATION FOR ANALYSIS



6.0 REFERENCES

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APPENDIX A

CONVERSION OF U.S. CUSTOMARY

UNITS TO SI UNITS

CONVERSION OF U.S. CUSTOMARY UNITS TO SI UNITS

The International System of Units (designated SI) was adopted by the Eleventh General Conference on Weights and Measures in 1960. The units and conversion factors used in this report are taken from or based on NASA SP-7012, "The International System of Units, Physical Constants and Conversion Factors - Revised, 1969".

The following table expresses the definitions of miscellaneous units of measure as exact numerical multiples of coherent SI units, and provides multiplying factors for converting numbers and miscellaneous units to corresponding new numbers of SI units.

The first two digits of each numerical entry represent a power of 10. An asterisk follows each number that expresses an exact definition. For example, the entry "-02 2.54\*" expresses the fact that 1 inch =  $2.54 \times 10^{-2}$  meter, exactly, by definition. Most of the definitions are extracted from National Bureau of Standards documents. Numbers not followed by an asterisk are only approximate representations of definitions, or are the results of physical measurements.

ALPHABETICAL LISTING

<u>To convert from</u>	<u>to</u>	<u>multiply by</u>
atmosphere (atm)	pascal (Pa)	+05 1.0133*
Fahrenheit (F)	kelvin (K)	$t_k = (5/9) (t_f + 459.67)$
foot (ft)	meter (m)	-01 3.048*
inch (in.)	meter (m)	-02 2.54*
mil	meter (m)	-05 2.54*
millimeter of mercury (mm Hg)	pascal (Pa)	+02 1.333
nautical mile, U.S. (n.mi.)	meter (m)	+03 1.852*
pound force (lb <sub>f</sub> )	newton (N)	+00 4.448*
pound mass (lb <sub>m</sub> )	kilogram (kg)	-01 4.536*
torr (°C)	pascal (Pa)	+02 1.333

# PHASE III SUMMARY REPORT

NAS-1-11774

APPENDIX A - Continued

## PHYSICAL QUANTITY LISTING

<u>To convert from</u>	<u>to</u>	<u>multiply by</u>
foot <sup>2</sup> (ft <sup>2</sup> )	meter <sup>2</sup> (m <sup>2</sup> )	-02 9.290*
inch <sup>2</sup> (in <sup>2</sup> )	meter <sup>2</sup> (m <sup>2</sup> )	-04 6.452*
inch <sup>2</sup> (in <sup>2</sup> )	centimeter <sup>2</sup> (cm <sup>2</sup> )	+00 6.452

<u>Density</u>		
pound mass/foot <sup>3</sup> (pcf, lb <sub>m</sub> /ft <sup>3</sup> )	kilogram/meter <sup>3</sup> (kg/m <sup>3</sup> )	+01 1.602
pound mass/inch <sup>3</sup> (lb <sub>m</sub> /in <sup>3</sup> )	kilogram/meter <sup>3</sup> (kg/m <sup>3</sup> )	+04 2.768
pound mass/inch <sup>3</sup> (lb <sub>m</sub> /in <sup>3</sup> )	gram/centimeter <sup>3</sup> (g/cm <sup>3</sup> )	+01 2.768

<u>Force</u>		
kilogram force (kg <sub>f</sub> )	newton (N)	+00 9.807*
pound force (lb <sub>f</sub> )	newton (N)	+00 4.448*

<u>Length</u>		
foot (ft)	meter (m)	-01 3.048*
inch (in.)	meter (m)	-02 2.54*
micron	meter (m)	-06 1.00*
mil	meter (m)	-05 2.54*
mile, U.S. nautical (n.mi.)	meter (m)	+03 1.852*

<u>Mass</u>		
pound mass (lb <sub>m</sub> )	kilogram (kg)	-01 4.536*

<u>Pressure</u>		
atmosphere (atm)	pascal (Pa)	+05 1.013*
millimeter of mercury (mm Hg)	pascal (Pa)	+02 1.333
newton/meter	pascal (Pa)	00 1.00*
pound/foot <sup>2</sup> (psf, lb <sub>f</sub> /ft <sup>2</sup> )	pascal (Pa)	+01 4.788
pound/inch <sup>2</sup> (psi, lb <sub>f</sub> /in <sup>2</sup> )	pascal (Pa)	+03 6.895

<u>Temperature</u>		
Fahrenheit (F)	Kelvin (K)	$t_k = (5/9)(t_f + 459.67)$



APPENDIX A - Continued

Volume

<u>To convert from</u>	<u>to</u>	<u>multiply by</u>
foot <sup>3</sup> (ft <sup>3</sup> )	meter <sup>3</sup> (m <sup>3</sup> )	-02 2.832*
inch <sup>3</sup> (in <sup>3</sup> )	meter <sup>3</sup> (m <sup>3</sup> )	-05 1.639*
inch <sup>3</sup> (in <sup>3</sup> )	centimeter <sup>3</sup> (cm <sup>3</sup> , cc)	-01 1.639

PREFIXES

The names of multiples and submultiples of SI units may be formed by application of the prefixes:

Multiple	Prefix
10 <sup>-6</sup>	micro (μ)
10 <sup>-3</sup>	milli (m)
10 <sup>-2</sup>	centi (c)
10 <sup>-1</sup>	deci (d)
10 <sup>3</sup>	kilo (k)
10 <sup>6</sup>	mega (M)
10 <sup>9</sup>	giga (G)

APPENDIX B

USERS INFORMATION  
THERMAL PROTECTION SYSTEM CREEP  
(TPSC)  
COMPUTER PROGRAM

B.1 INTRODUCTION

The computer program described herein, Thermal Protection System Creep (TPSC), uses iterative techniques and numerical integration to predict creep strains, residual stresses, and permanent deflections in stiffened panel structures. This program was developed jointly under internal MDAC funding and NASA Langley Research Center contractual funding. Initiated at MDAC in 1971, the program has been continually modified to increase its capability. Although the TPSC Computer Program was developed for analysis of creep deflections in thermal protection system panels, it is applicable to creep analysis in any beam or stiffened plate structure subjected to bending loads. The TPSC program is written in CDC Fortran IV and is operational on the MCAUTO/CDC 6000 series computers using KRONOS operating systems.

A flexible, user oriented input format is used. Input data include panel geometry and definition of loading and temperature profiles. Panel temperature distributions along the panel length and through the depth can be input using either polynomial equation coefficients or tabular input. Temperatures at each location in the panel are based on these distributions and the input temperature-time profile data. Also, input are equation coefficients to define material creep response as a function of time, stress, and temperature.

Program output includes a record of input data and calculated geometrical data (elastic moment of inertia), trajectory load and temperature data, and creep equation definition as well as the calculated deflections, creep strains, and residual stresses.

The program was developed specifically for analysis of thermal protection system panels. Therefore, definition of leading structural concepts, corrugation

stiffened, rib stiffened, and zee stiffened concepts, is incorporated into the TPSC program. Modeling of the specific panel structural concept for analysis is accomplished automatically based on overall section input definition. Appropriate use of input parameters also allows analysis of rectangular and I-beam sections. An option is provided for including a beaded skin into any of the cross sections since beads are frequently required in thermal protection system panel designs.

Bending moments are internally defined based on uniform pressure load input or two point load input. In addition, the moments can be calculated as a function of panel edge support stiffness and the ratio of panel stiffness in the longitudinal and transverse directions. This option is based on combining solutions for an isotropic plate with two sides simply supported and two sides elastically supported as offered by Timoshenko (Reference 4) and the solution for an orthotropic plate with four sides simply supported as offered by Lekhnitskii (Reference 5). This option provides a first order approach to account for Poisson's effects in orthotropic plate structures.

Sensitivity of predicted results to the number of elements defining panel cross section and the number of stations defining panel length has been investigated with the goal of providing guidelines for minimizing required computer time. Computer time increases almost linearly with number of analysis steps specified. The minimum number of stations along the length and elements through the depth which can be used to maintain good prediction accuracy have been defined.

The TPSC computer program provides needed capability for prediction of permanent deflections, due to creep, in entry vehicle metallic thermal protection system panels. Application is also envisioned in other structures where creep deflections may be important such as in missile structures and nuclear reactors.

B.2 METHOD OF ANALYSIS

Within the TPSC program the panel length is divided into  $i$  stations over which bending moments are assumed constant and the panel depth is divided into  $j$  elements over which stresses are assumed constant as indicated in Figure B-1.

Using the assumption of a linear total elastic plus creep strain distribution through the depth, the neutral axis and structural rotation are systematically varied at each station and time step to determine the unique stress distribution which satisfies both force and moment balance requirements. At each point in the panel the creep component of total strain is determined based on either the time hardening or strain hardening theory of creep accumulation applied in conjunction with input analytical expressions defining material tensile creep response as a function of stress, temperature, and time. Residual stresses are calculated at each time step by subtracting the elastic stress from the total calculated stress. These residual stresses are used at initiation of analysis for the next time step. Analysis proceeds through all the time steps at each designated station along the panel length, accumulating and storing structural rotations, creep strains, and residual stresses. At the completion of analysis, rotations are numerically integrated to determine creep deflections.

The following assumptions are made in the analysis:

- (a) Only bending stresses are considered in the analysis. Deflections due to shear are assumed negligible.

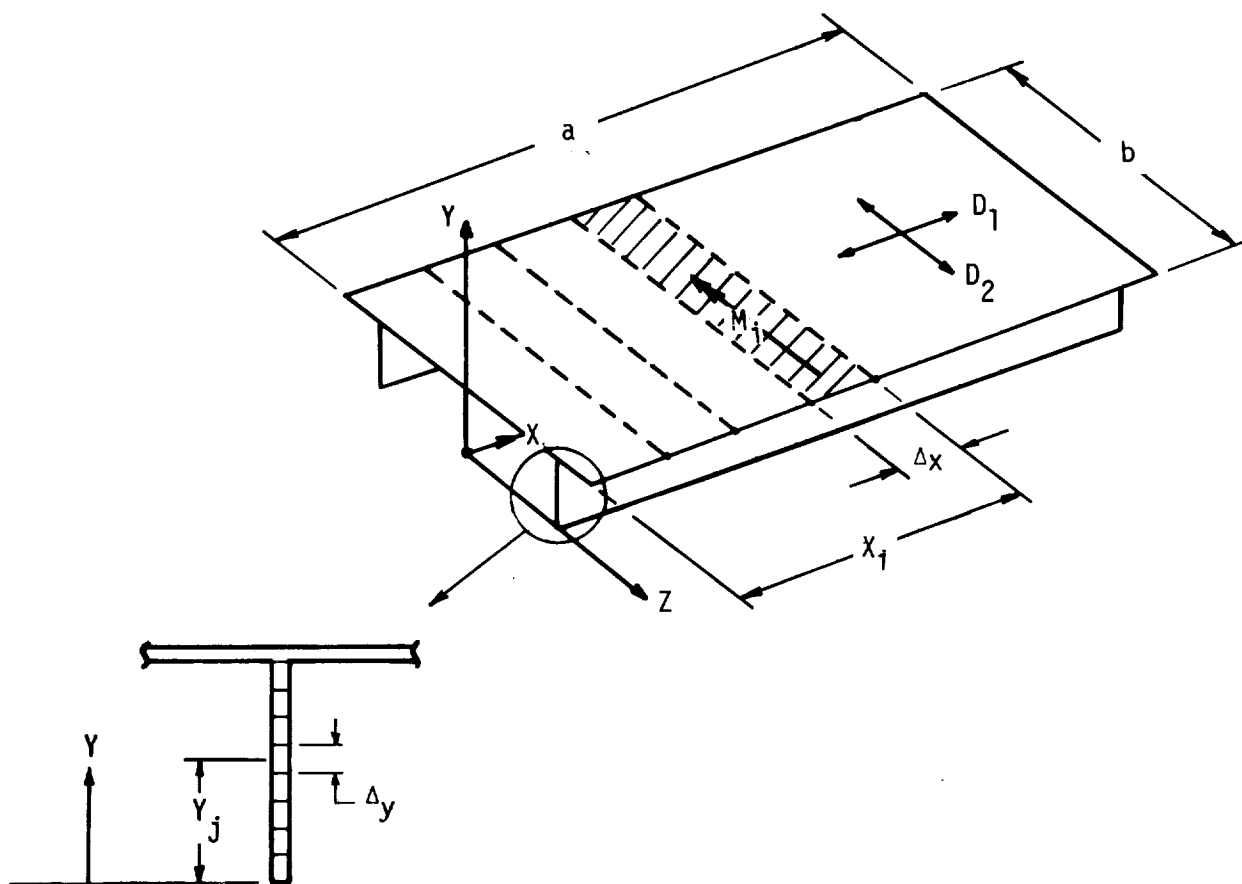


FIGURE B-1. PANEL REPRESENTATION FOR ANALYSIS



- (b) Strain distributions through the panel depth are linear.
- (c) Creep response equations, defined by the user, are assumed to be applicable for both tensile and compressive stresses.
- (d) Load and temperature distributions and calculated deflections are assumed symmetrical with respect to the panel centerline ( $X = a/2$ ).
- (e) Panels are thin gage. (Although the analysis is not restricted to thin gages, the approach for modeling specific section geometries incorporated into the program has been based on this assumption.)

Approaches used for important program calculations are presented in this section. The general analysis flow is shown in Figure B-2 for reference purposes.

#### B.2.1 Geometry Definition

Analysis capability for three thermal protection system structural concepts; rib stiffened (INDGEO = 1), corrugation stiffened (INDGEO = 2), and zee stiffened (INDGEO = 3), is incorporated into the TPSC program. The number of stiffeners across the panel width are defined by NRIB, NCOR, and NZEE for the rib, corrugation, and zee stiffened concepts, respectively. A skin bead in either the positive y or negative y direction can be included in the cross sections at the user's option (INDBD = 1). The direction of the skin bead is specified by the sign of the input bead radius (BRAD) where a + sign designates the bead in the + y direction from the skin. Geometry of the TPS cross sections and skin bead, defining program input variables, are shown in Figure B-3.

The approach for modeling the rib stiffened, corrugation stiffened, and zee stiffened subsize TPS panels is shown in Figure B-4. The number of stations in half the panel length are defined by input variable NSTAT. This is defaulted to 6 when not input by the user. The number of elements through the panel depth is defined by input variables NSECT and SEC. These are both defaulted to 10

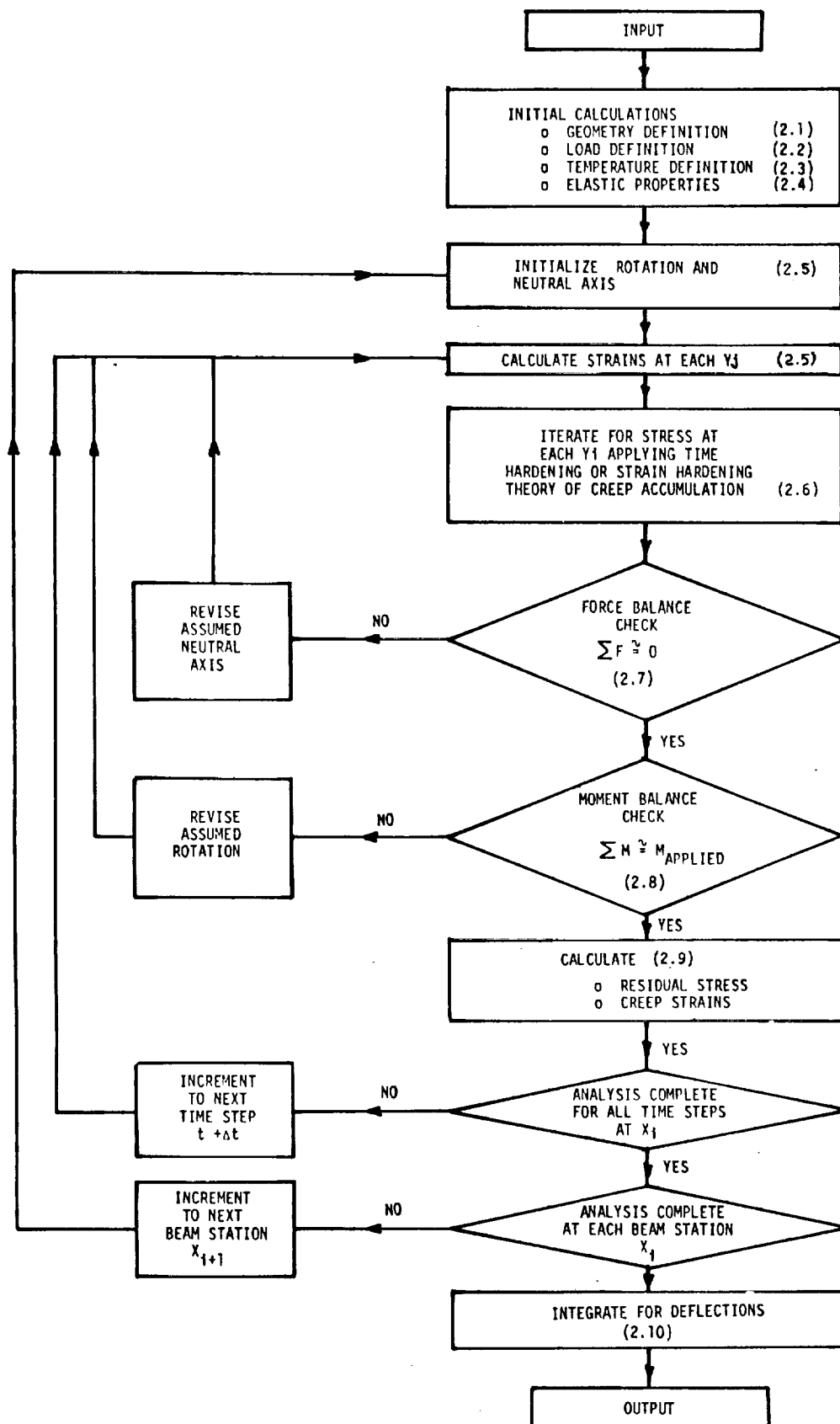


FIGURE B-2 TPSC PROGRAM ANALYSIS FLOW

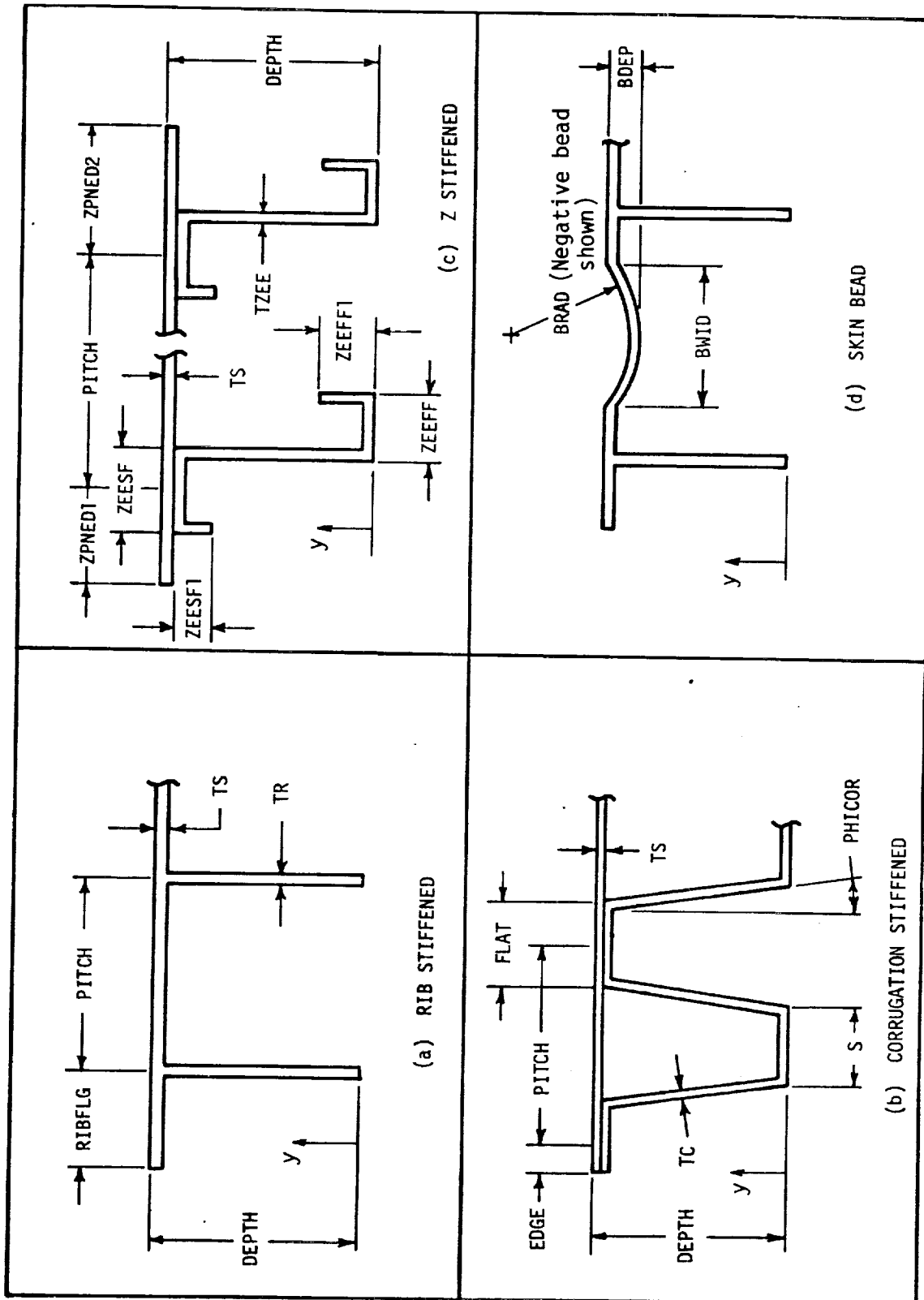


Figure B-3 PANEL GEOMETRY DEFINITION

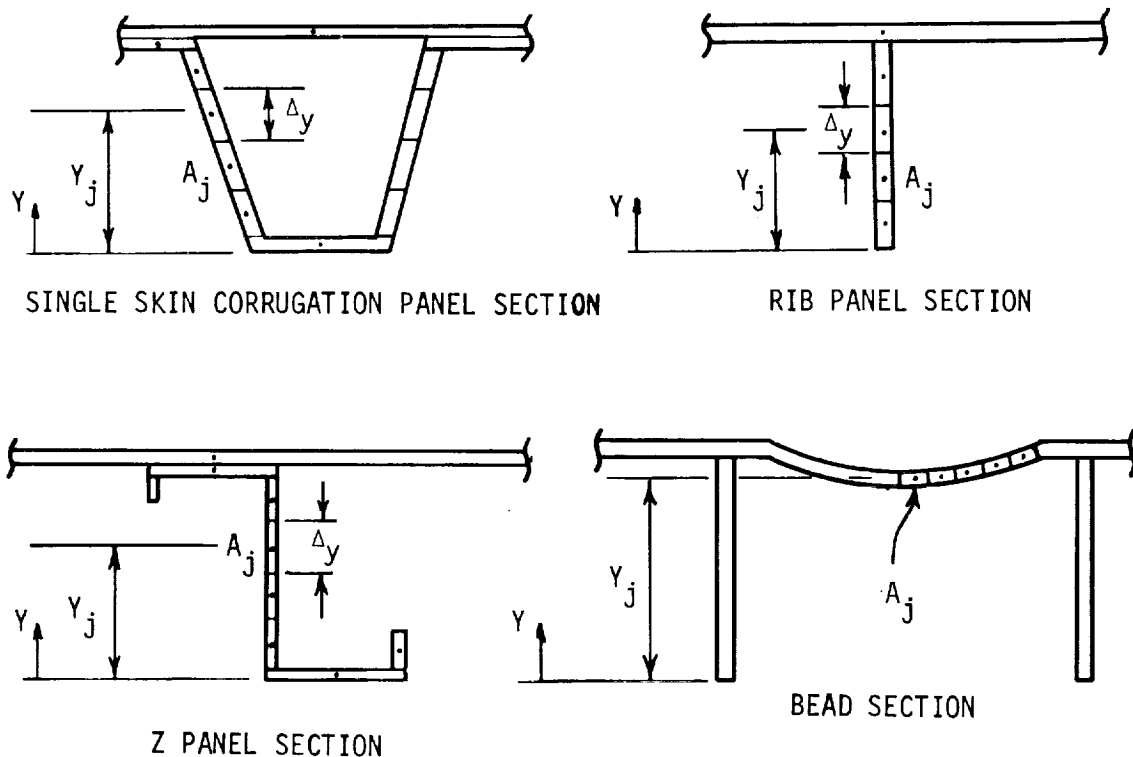


FIGURE B-4 STRUCTURAL MODELING OF PANEL AND CROSS SECTIONS

when not input by the user. Two variables (fixed point (integer) and floating point (real)) are used in order to make the program as machine independent as possible. The assumption of thin gages allows skin and horizontal stiffener sections (e.g., skin) to be defined as individual elements. Vertical portions of the stiffeners (e.g., ribs) are divided into  $\Delta Y$  elements based on the input total number of elements (NSECT, SEC) minus the number of horizontal elements. Therefore, for example, the calculation for  $\Delta Y$  for the corrugation concept is  $\Delta Y = [\text{DEPTH-TS-2(TC)}]/(\text{NSECT-3})$ . For beaded skins five additional elements are added into the cross section as shown in Figure B-4. Centroids and areas of the cross section increments are used in all subsequent program calculations.

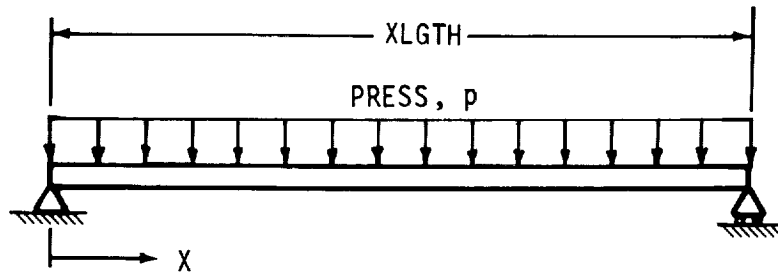
#### B.2.2 Load Definition

Panel bending loads are calculated based on input pressure (PRESS) or point loads (PLOAD) for each time step in the mission profile. These are selected by inputting INDLOD=0 for pressure load and INDLOD=1 for point loads. Calculations of beam bending moments under the pressure and point load options are shown in Figure B-5. Input XLGTH and PANWID values correspond to a and b respectively. All applied loads are assumed symmetrical about the panel centerline. Units for PLOAD are Lbs or Kg where the input value is the total panel load. Input pressure units are  $\text{Pa/cm}^2$  or  $\text{Lbs/in}^2$ . The panel width is included in the pressure loading calculations to yield the total panel bending moment at each beam station. The option for applying point loads to the panel, was specifically included to allow analysis of subsize panel test data (Reference 2). For this type of loading the distance (ALEN) from the support to the point of load application is input. Analysis for a single midspan point load can be implemented by making ALEN equal to one half of the panel length.

For panels loaded with a uniform pressure, an option is included for modifying the bending distribution (INDPLA=1) to account for plate effects. The plate option provides a first order approach to account for Poisson's effects and edge stiffness

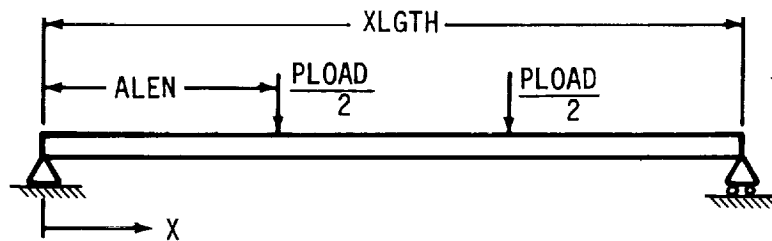


(a) PRESSURE LOADS (SIMPLE SUPPORTS)



$$M = \frac{\text{PRESS}}{2} (\text{XLGTH}(x) - x^2) (\text{PANWID})$$

(b) TWO POINT LOADING (SIMPLE SUPPORT)



$$M_{0-\text{ALEN}} = \frac{\text{PLOAD}}{2} x$$

$$M_{\text{ALEN}-\text{XLGTH}/2} = \frac{\text{PLOAD}}{2} (\text{ALEN})$$

Figure B-5. LOAD OPTIONS

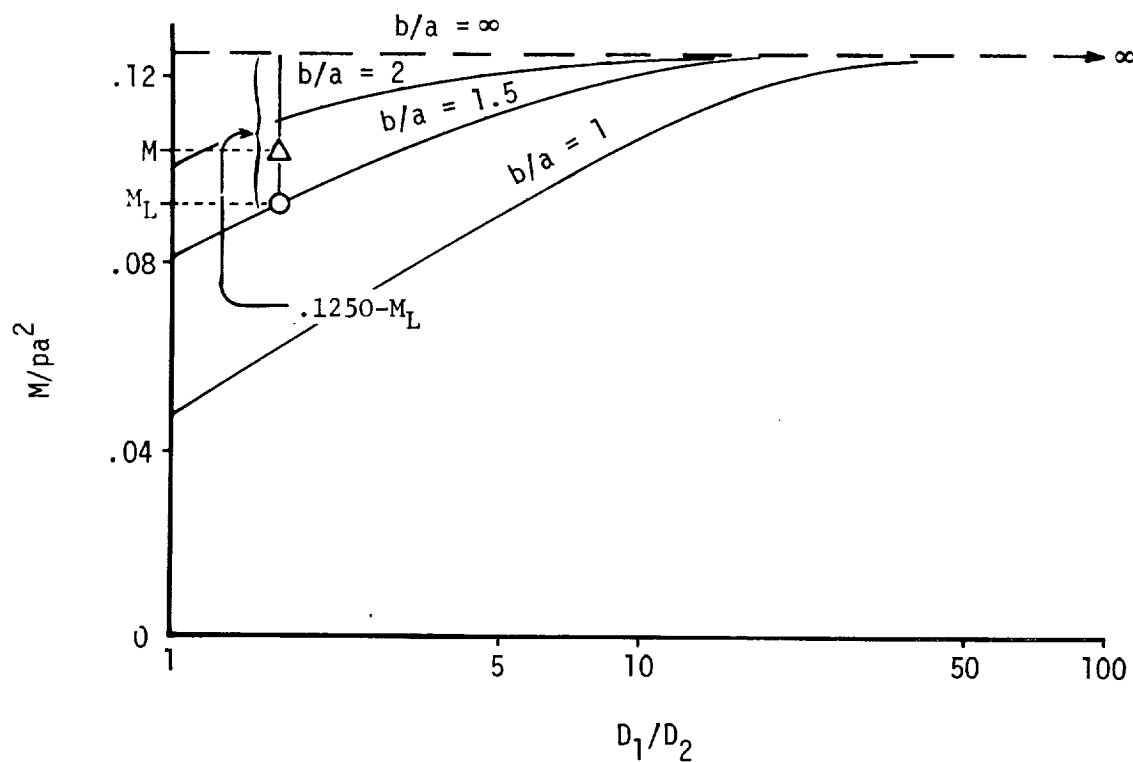
effects in orthotropic plate structures by combining solutions for an isotropic plate with two sides simply supported and two sides elastically supported as offered by Timoshenko (Reference 4) and the solution for an orthotropic plate with four sides simply supported as offered by Lekhnitskii (Reference 5). This option is restricted to panels where  $b \geq a$  and  $D_1 \geq D_2$  (Figure B-1). The value  $D_1$  is calculated in the program. Stiffness in the transverse direction ( $D_2$ ) can be input (using the variable DETWO) or calculated, based on skin thickness as  $(TS)^3/12(1-\nu)^2$ . The parameter  $\lambda$  defining the relative stiffness of the edge support and the panel is calculated as  $\lambda = ESTIFF/aD_1$  where ESTIFF is the input support stiffness along the edges  $Z = \pm b/2$ .

Panel midspan ( $Z=0$ ,  $X = a/2$ ) bending moments for typical  $b/a$  values are shown in Figure B-5(a) and B-5(b) as functions of the quantities  $\lambda$  and  $D_1/D_2$  respectively. These solutions are combined in the analysis by assuming that the variation of moment as a function of  $\lambda$ , will be applicable at all  $D_1/D_2$  values. This assumption is felt to be justified because it is exactly applicable at  $D_1/D_2 = 1$  (isotropic panel) and any variations at other  $D_1/D_2$  will be a small portion of the total moment since values are constrained in a narrow range (Figure B-5(b)).

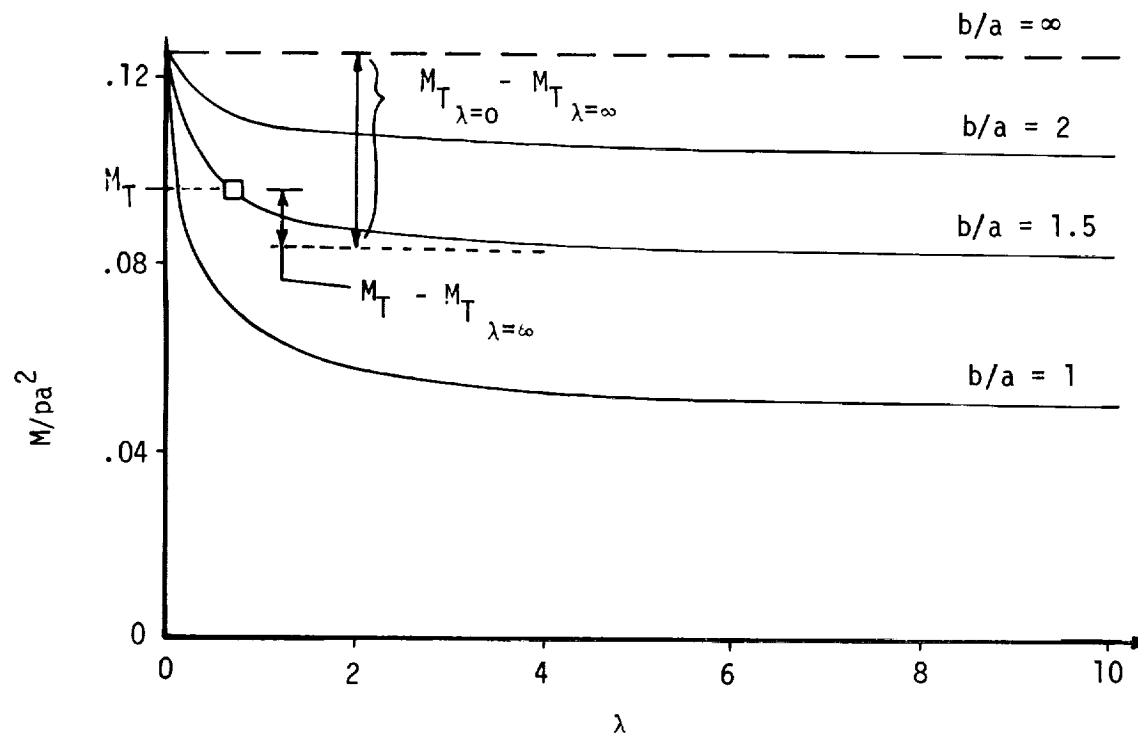
In computing the midspan moment  $M$ , the moment  $M_L$  for the orthotropic plate with four sides simply supported (Figure B-6(b)) is first calculated. This moment is then increased based on the degree of edge support using the calculated moment  $M_T$  for an isotropic plate (Figure B-6(a)). The solution for  $M$  (Figure B-6(b)) is then calculated based on the relationship that the increase in moment ( $M-M_L$ ) toward  $.125 pa^2$  will be proportional to the increase ( $M_T - M_{T_{\lambda=\infty}}$ ) due to panel edge stiffness.

This results in the equation

$$\frac{M - M_L}{.1250 - M_L} = \frac{M_T - M_{T_{\lambda=\infty}}}{M_{T_{\lambda=0}} - M_{T_{\lambda=\infty}}}$$



(b) LEKHNITSKII SOLUTION - MIDSPAN MOMENT



(a) TIMOSHENKO SOLUTION - MIDSPAN MOMENT

FIGURE B-6 PLATE BENDING MOMENT SOLUTIONS

which yields

$$M = M_L + (.1250 - M_L) \frac{M_T - M_{T_{\lambda=\infty}}}{M_{T_{\lambda=0}} - M_{T_{\lambda=\infty}}} \quad (B-1)$$

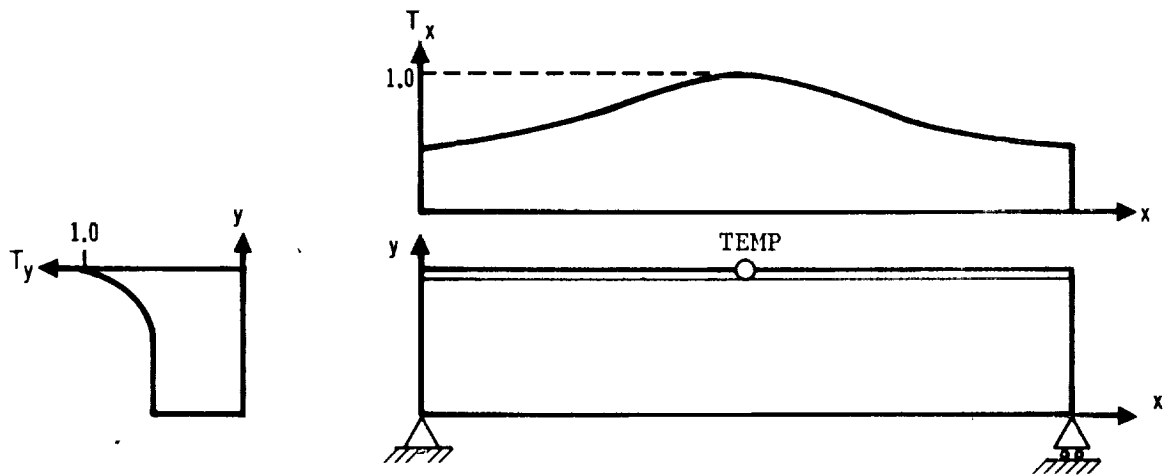
Moments along the panel length are calculated based on ratioing the beam bending moment distribution (Figure B-5(a)), using the equation B-1 value at midspan.

### B.2.3 Temperature Definition

Panel temperature distributions are defined by input of the panel midspan skin temperatures at each mission time step, the distribution of temperature through the panel depth, and the distribution of temperature along the length. Normalized temperature distributions, referenced to the midspan skin temperature, are input, as indicated in Figure B-7(a). The midspan skin temperature (TEMP) is input for each trajectory time step (DXTIME) up to the number of steps (NTIME). Distributions through the panel depth and along the length are defined by either of the following approaches.

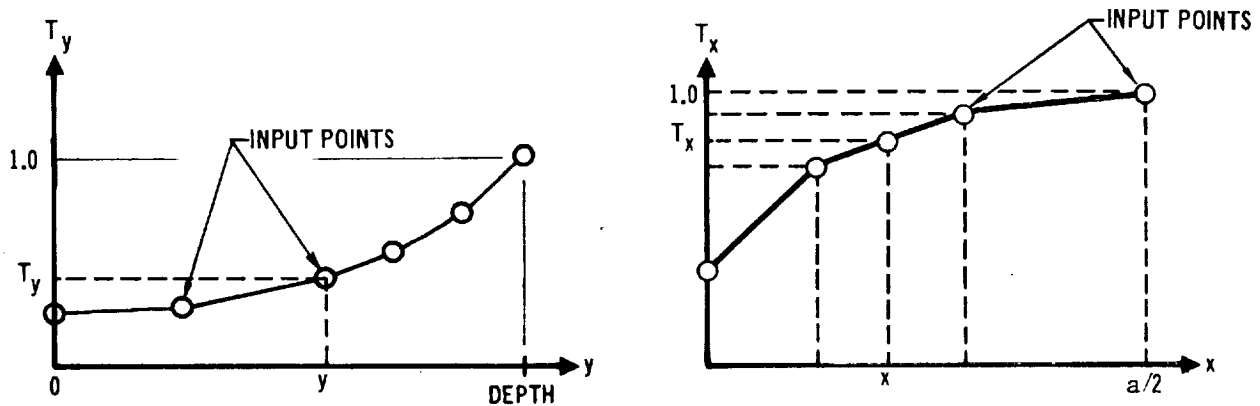
(1) A table lookup routine is included in the program to calculate  $T_x$  and  $T_y$  as functions of X and Y, respectively, based on tabular input. The distribution along the panel length (INDTFL = 1) and through the panel depth (INDTFD = 1) are input using the variables XTEMP and YTEMP, respectively. Temperatures (normalized to 1.0 at the midspan skin) are input to define these distributions as shown in Figure B-7(b). The distribution  $T_y$  is assumed to be the same at each location along the panel length. Temperature is calculated, at each point in the panel, as the product of TEMP (function of time),  $T_x$  (function of length), and  $T_y$  (function of depth).  $T_x$  and  $T_y$  are determined using linear interpolation between input points as shown in the figure.

(2) Temperature can be defined by the input of coefficients (C and D) to the following polynomial equations.

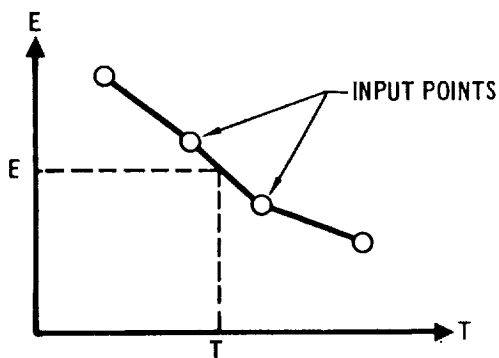


$$T = TEMP (T_x) (T_y)$$

(a) Temperature Defined as Function of Panel Length and Depth



(b) Linear Interpolation for Temperatures



(c) Linear Interpolation for Elastic Modulus

FIGURE B-7 PANEL TEMPERATURE CALCULATIONS

$$T_X = C_1 + C_2 X + C_3 X^2 + C_4 X^3 \quad (B-2)$$

$$T_Y = D_1 + D_2 Y + D_3 Y^2 + D_4 Y^3 \quad (B-3)$$

For this option the control variables INDTFI and INDTFD are defaulted to 0.

For temperatures defined by either of these options the control variable ITCON must be set equal to 0.

Elastic modulus data, for use in the analysis, are defined as a function of temperature by either of the same two approaches described for defining temperature distributions.

(1) For using the table lookup routine, modulus and temperature data are input in tabular form using the input variable ETEMP, as shown in Figure 2-7(c). For this option the control variable INDMOD is input as 1.

(2) The modulus can also be defined by equation coefficients (ECOEFF) to the equation:

$$E = ECOEFF_1 + ECOEFF_2(T) + ECOEFF_3(T^2) + ECOEFF_4(T^3) \quad (B-4)$$

#### B.2.4 Elastic Calculations

Elastic stresses ( $\sigma_e$ ), strains ( $\epsilon_e$ ), and rotations ( $\theta_e$ ) at each station as well as the section neutral axis ( $\bar{Y}_e$ ) and moment of Inertia (I) are calculated as follows, where the subscript j represents element location in the cross section (Figure B-4)

$$\bar{Y}_e = \frac{\sum_{j=1}^{NSECT} A_j Y_j}{\sum_{j=1}^{NSECT} A_j} \quad (B-5)$$

$$I = \sum_{j=1}^{N\text{SECT}} A_j Y_j^2 - \bar{Y}^2 \sum_{j=1}^{N\text{SECT}} A_j \quad (\text{B-6})$$

$$\sigma_{e_j} = \frac{M_i (\bar{Y}_e - Y_j)}{I} \quad (\text{B-7})$$

$$\epsilon_{e_j} = \sigma_{e_j} / E \quad (E = \text{elastic modulus}) \quad (\text{B-8})$$

$$\theta_{e_i} = \frac{M_i \Delta X}{E I} \quad (\text{B-9})$$

In the moment of inertia calculation, the moments of inertia of individual  $j$  elements, about their own neutral axes, have been found to be negligible and have not been included.

#### B.2.5 Iteration for Stress

At each beam station ( $X_i$ ) the incremental rotation due to creep ( $\theta_c$ ) and neutral axis ( $\bar{Y}$ ) are initialized as

$$\begin{aligned} \theta_c &= \theta_e \\ \bar{Y} &= \bar{Y}_e \end{aligned} \quad (\text{B-10})$$

Based on these values, the initial total strain assumed at each  $Y$  element ( $Y_j$ ) is calculated, using the linear total strain assumption

$$\epsilon_{T_j} = (\theta_c + \theta_e) (\bar{Y} - Y_j) / \Delta X \quad (\text{B-11})$$

For each  $j$  element through the cross section there is a unique value of stress ( $\sigma_j$ ) which satisfies the equation:

$$\epsilon_{Tj} = \epsilon_{cj} + \frac{\sigma_j + \sigma_{\text{RESIDUAL}j}}{E} \quad (\text{B-12})$$

where  $\sigma_{\text{RESIDUAL}j}$  is the residual stress based on results from calculations in the previous time step (zero for the first step) and  $E$  is the material elastic modulus at the element temperature.

The incremental creep ( $\epsilon_{cj}$ ) is a function of stress, temperature, time, and incremental time step based on the input creep strain equation (Section B.3.7) applied in conjunction with the hardening theories. Calculations of  $\epsilon_{cj}$  as a function of stress, strain, temperature, and time for the strain hardening and time hardening creep accumulation theories are discussed in Section B.2.6.

In determining the value of stress at each element which satisfies Equation B-12, assumed stresses (designated by the subscript  $\ell$ ) are systematically varied and corresponding strains ( $\epsilon_\ell$ ) are calculated. The subscript  $\ell$  has been added in this section to designate stresses and strains calculated in the iteration process. The subscript  $j$  is applied to the final stresses determined at each element. The initially assumed value of stress  $\sigma_\ell$  ( $\ell = 1$ ) is that obtained from analysis in the previous time step (elastic stress for the first time step). The assumption for the second value of  $\sigma_\ell$  ( $\ell = 2$ ) is dependent on the value of  $\sigma_1$  and the relationship between  $\epsilon_{Tj}$  and the calculated strains  $\epsilon_\ell$  ( $\epsilon_1$  for  $\ell = 1$ ) as follows:

- (a)  $\sigma_2 = -100.$  psi. for  $(\sigma_1 = 0.$  and  $\epsilon_{T_j} < \epsilon_1)$
- (b)  $\sigma_2 = +100.$  psi. for  $(\sigma_1 = 0.$  and  $\epsilon_{T_j} > \epsilon_1)$
- (c)  $\sigma_2 = 2 (\sigma_1)$  for  $(\sigma_1 < 0.$  and  $\epsilon_1 > 0.$  and  $\epsilon_{T_j} < \epsilon_1)$

or

$$(\sigma_1 > 0. \text{ and } \epsilon_1 > 0. \text{ and } \epsilon_{T_j} > \epsilon_1)$$

$$(d) \quad \sigma_2 = \sigma_1 + |\sigma_1| \text{ for } \frac{\epsilon_1}{\epsilon_{T_j}} < .1 \text{ and } \epsilon_{T_j} > 0 \quad (B-13)$$

$$(e) \quad \sigma_2 = \sigma_1 - |\sigma_1| \text{ for } \frac{\epsilon_1}{\epsilon_{T_j}} < .1 \text{ and } \epsilon_{T_j} < 0$$

$$(f) \quad \sigma_2 = \sigma_1 \frac{\epsilon_{T_j}}{\epsilon_1} \text{ for } \frac{\epsilon_1}{\epsilon_{T_j}} > .1$$

Subsequent values of  $\sigma_\ell$  are calculated by applying the equation

$$\sigma_\ell = \sigma_{\ell-1} + (\epsilon_{T_j} - \epsilon_{\ell-1}) \text{ SLOPE} \quad (B-14)$$

where  $\text{SLOPE} = (\sigma_{\ell-1} - \sigma_{\ell-2}) / (\epsilon_{\ell-1} - \epsilon_{\ell-2})$

The process which is representative in Figure B-8 is continued until the stress is determined such that

$$\frac{\epsilon_{T_j}}{\epsilon_\ell} - 1. < .001$$

An analysis proceeds through each time step the neutral axis and rotation are initialized as equal to those calculated in the previous time step.

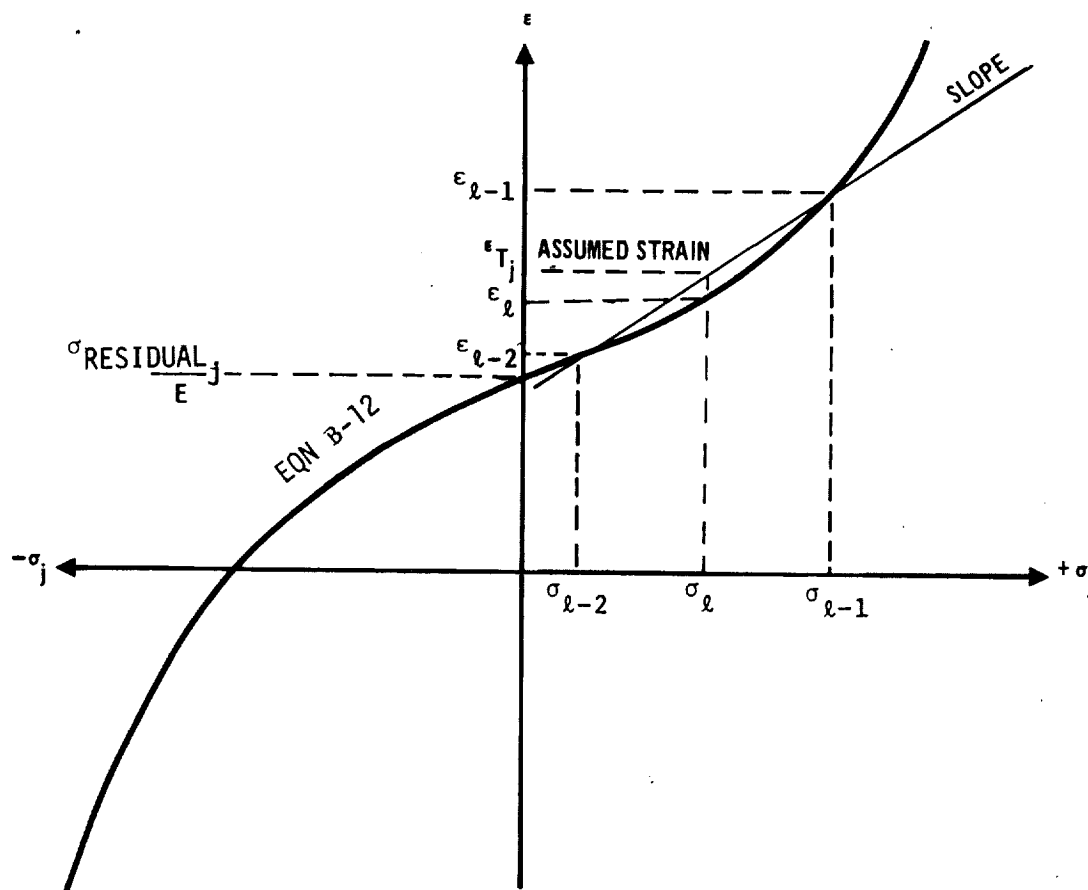


FIGURE B -8 ITERATION APPROACH FOR STRESS CALCULATION

B.2.6 Hardening Theories

The time hardening and strain hardening theories of creep accumulation are provided in the TPSC program. These are selected through input of the control variable HARDOP = 1 and HARDOP = 2 respectively.

The time hardening theory of creep accumulation is based on the assumption that the creep rate is dependent upon the total time under load. This approach for calculation of incremental creep strains is shown in Figure B-9(a). Stresses are iteratively determined at each time step based on input creep strain equations.

The strain hardening theory of creep accumulation is based on the assumption that the creep rate is dependent upon total accumulated creep strain. This approach for calculation of incremental creep strains is shown in Figure B-9(b). Additional calculations are required under this option to determine the effective time, for which the given value of strain applies, at a new stress and/or temperature. Therefore, this option requires more computer time (Reference Section B.6). To facilitate analysis of mission profiles, a maximum time cutoff (TMAX) is input to prevent application of the creep equation beyond its range of applicability. For times beyond this time cutoff, the equation creep rate is assumed constant for each stress and temperature as indicated in the figure.



### B.2.7 Force Balance Requirements

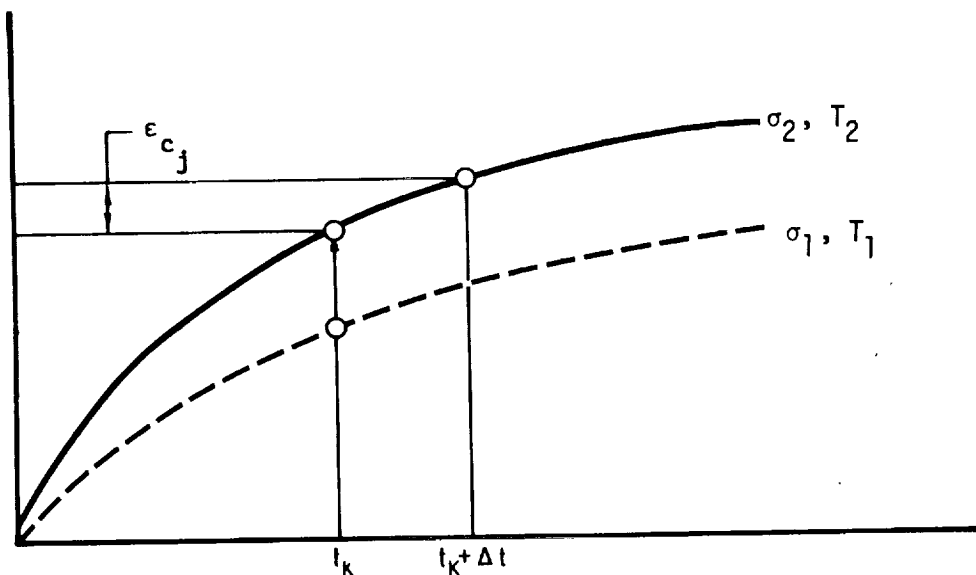
Having solved for stresses at each of the  $j$  element locations, the check for a zero force balance on the cross section is calculated as

$$F = \sum_{j=1}^{N\text{SECT}} \sigma_j A_j \quad (\text{B-15})$$

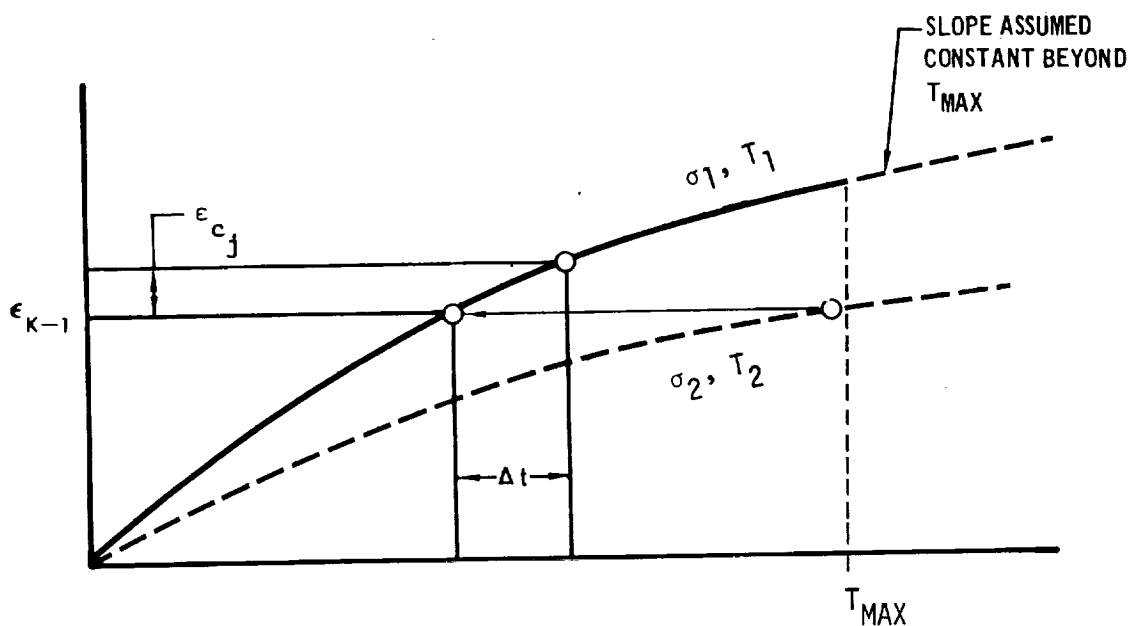
Based on the resulting sign (plus or minus) of  $F$  the neutral axis location is changed by  $\bar{Y} \pm \Delta Y$ . That is, for example, if the neutral axis is located toward the tension side of the panel, resulting in an overall net compression load ( $F = \text{negative}$ ), then the neutral axis must be moved toward the compression side ( $+Y$  direction). Strains and stresses are then recalculated. This process is continued until the sign of  $F$  changes, at which time the neutral axis location is calculated by linear interpolation using the equation

$$\bar{Y} = \bar{Y}_m - \frac{|F_m| (\bar{Y}_m - \bar{Y}_{m-1})}{|F_m| + |F_{m-1}|} \quad (\text{B-16})$$

where  $m$  is a subscript used to designate the final two axis locations and associated force summations in calculating  $\bar{Y}$ . This process is indicated in Figure B-10. It should be noted that the neutral axis location is not constrained to coincide with the  $Y_j$  element centroids.



(a) Time Hardening



(b) Strain Hardening

FIGURE B-9 HARDENING THEORIES FOR CREEP ACCUMULATION.

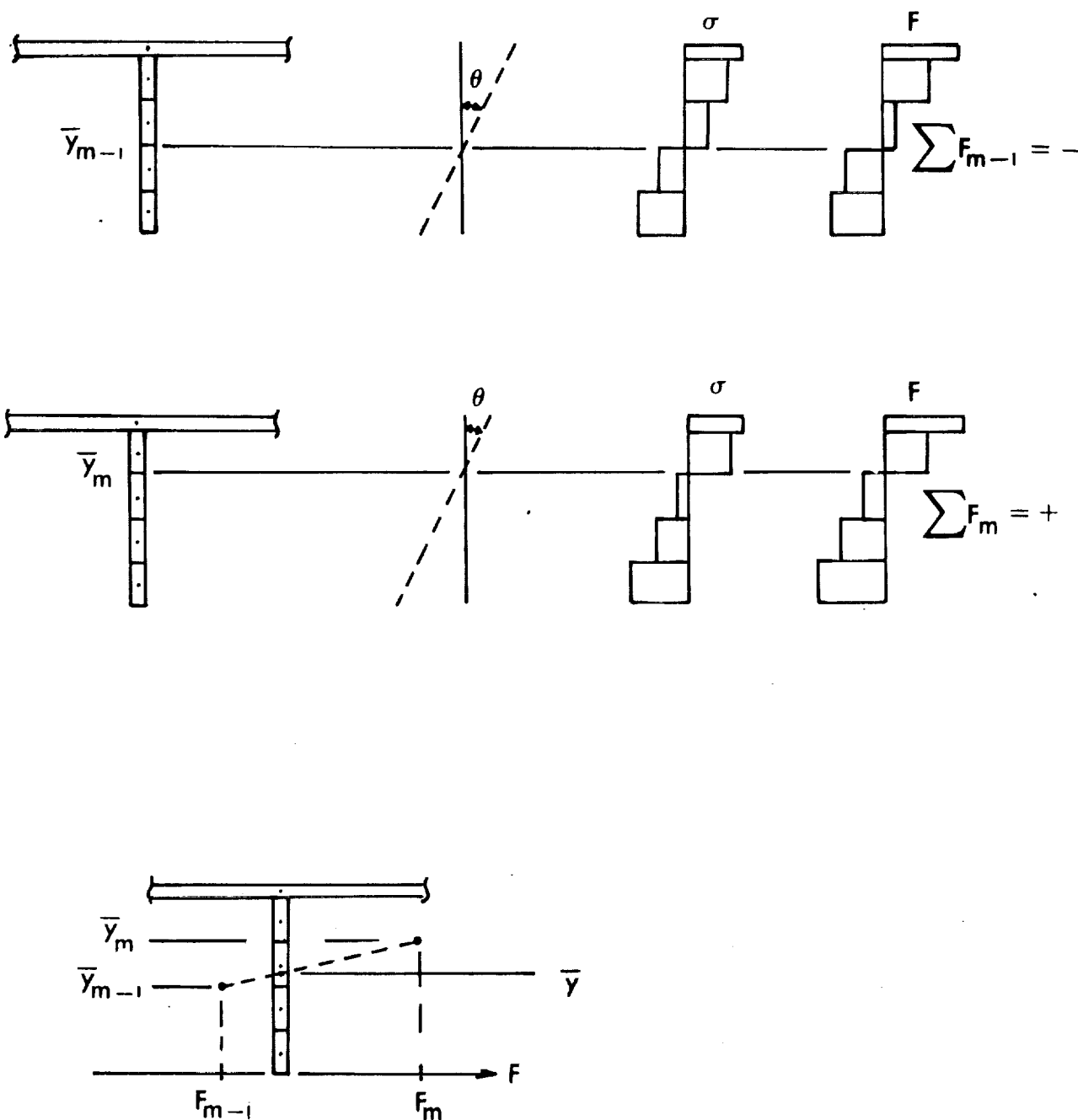


FIGURE B-10 FORCE BALANCE ITERATION APPROACH

### B.2.8 Moment Balance Requirements

After force balance is attained at the cross section the check for moment balance at the station is calculated as

$$M_1 = \sum_{j=1}^{N_{\text{SECT}}} |\sigma_j A_j (\bar{Y} - Y_j)| \quad (\text{B-17})$$

The second estimate of  $\theta$  is

$$\theta_2 = \theta_1 \left( \frac{M_{\text{applied}}}{M_1} \right) \quad (\text{B-18})$$

Subsequent assumed rotations are calculated based on the equation

$$\theta_n = \theta_{n-1} + (M_{\text{applied}} - M_{n-1}) \frac{(\theta_{n-1} - \theta_{n-2})}{(M_{n-1} - M_{n-2})} \quad (\text{B-19})$$

as depicted in Figure B-11 where  $n$  is a subscript used to designate the assumed rotations and associated moments calculated. Each  $M_n$  is compared to  $M_{\text{applied}}$ . Balance is established when

$$\frac{|M_n - M_{\text{applied}}|}{M_{\text{applied}}} < .001$$

The value of .001 for moment balance coverage was established to provide good solution accuracy within reasonable computer run times.

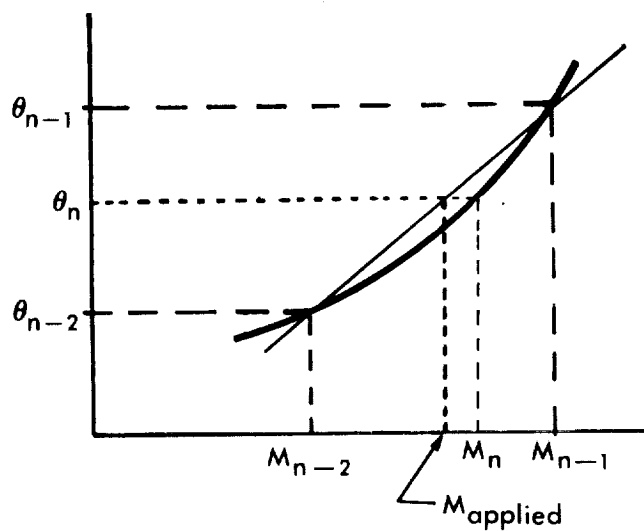
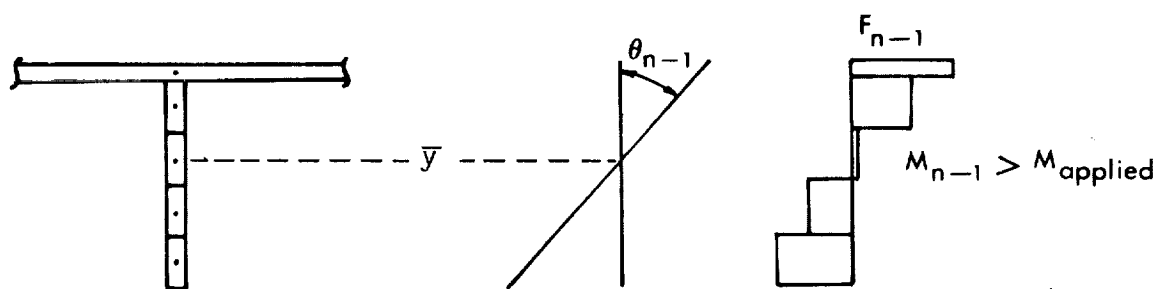
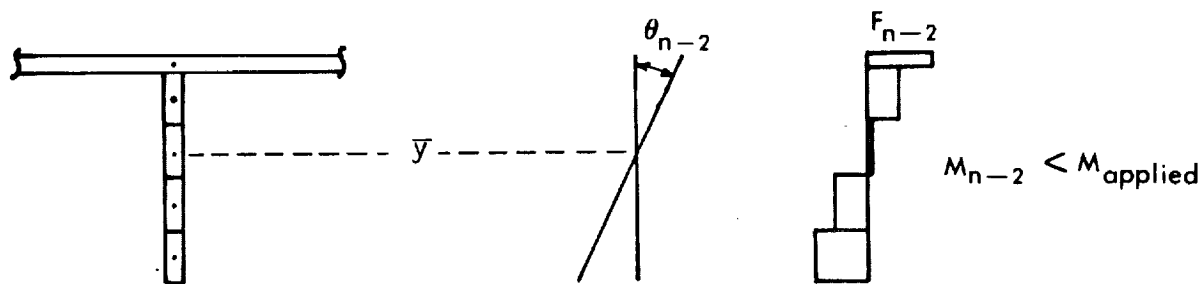


FIGURE B-11 MOMENT BALANCE ITERATION APPROACH

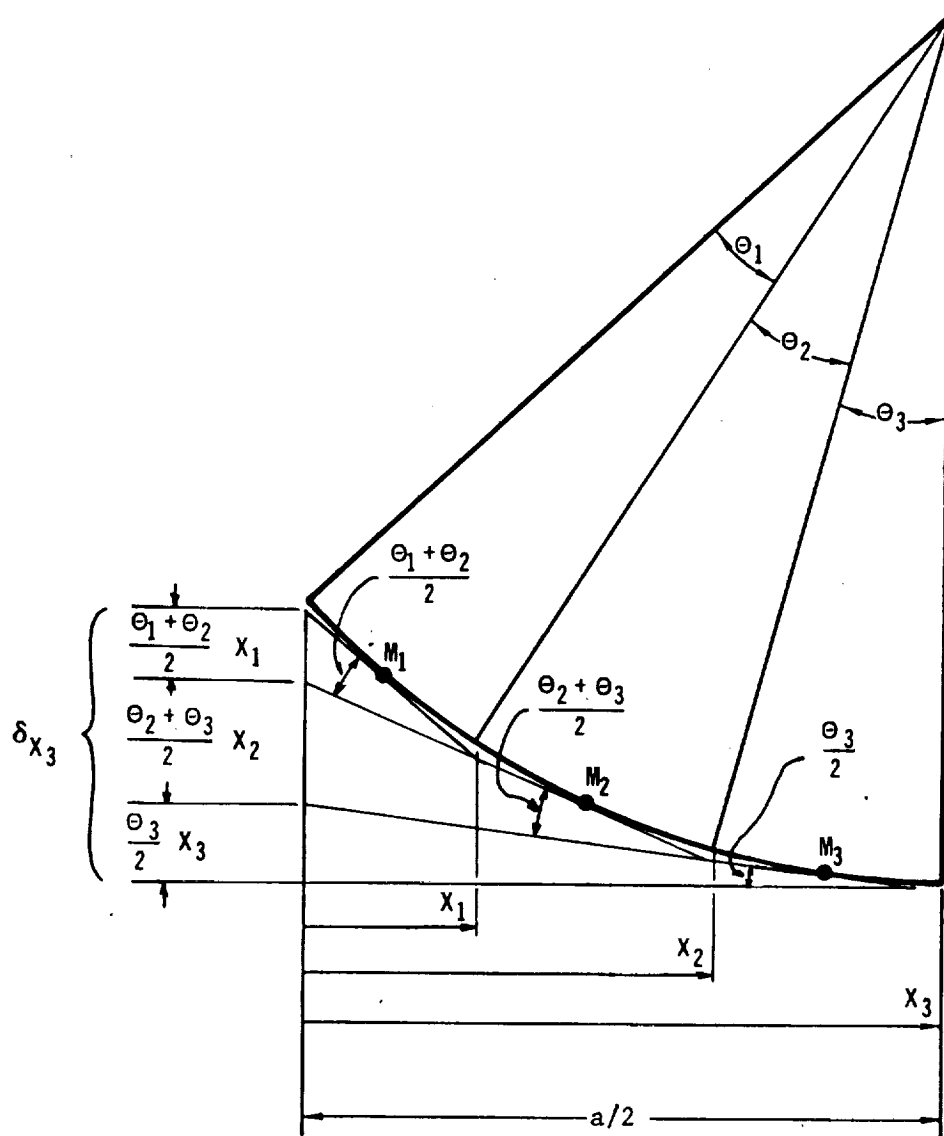


FIGURE B-12 APPROACH FOR CALCULATION OF DEFLECTIONS

B.2.9 Calculation of Creep Strains and Residual Stress

Once force balance and moment balance have been established the residual stresses and creep strains are calculated at each  $j$  element as

$$\sigma_{\text{RESIDUAL}_j} = \sigma_{e_j} - \sigma_j \quad (\text{B-20})$$

and

$$\epsilon_{c_j} = \epsilon_{T_j} - \epsilon_{e_j} = \theta_c \frac{(\bar{Y} - Y_j)}{\Delta X} - \frac{\sigma_{\text{RESIDUAL}_j}}{E} \quad (\text{B-21})$$

In addition to calculation of residual stresses and creep strains for output purposes, these values are retained for use in subsequent analysis. Creep strains are required for use in strain hardening analysis and residual stresses are added to elastic stresses for initiation of the next analysis time step.

### B.2.10 Deflection Calculations

Structural creep rotation ( $\theta_c$ ) and elastic rotations ( $\theta_e$ ) are calculated as a function of time and stored for use in numerical integration calculations for deflections.

These deflections are calculated at each station (subscript i), according to the following equations where NSTAT is the total number of beam stations in half the beam length and n is a dummy variable used to designate the beam stations

$$(i = 1 \text{ to } \text{NSTAT}-2) \quad \delta_{X_i} = \sum_{n=1}^i \frac{\theta_n + \theta_{n+1}}{2} X_n + \frac{\theta_{i+1}}{2} X_i + X_i \sum_{n=i+2}^{\text{NSTAT}} \theta_n \quad (\text{B-22})$$

$$(i = \text{NSTAT}-1) \quad \delta_{X_i} = \sum_{n=1}^i \frac{\theta_n + \theta_{n+1}}{2} X_n + \frac{\theta_{\text{NSTAT}}}{2} X_{\text{NSTAT}-1} \quad (\text{B-23})$$

$$(i = \text{NSTAT}) \quad \delta_{X_i} = \sum_{n=1}^i \frac{\theta_n + \theta_{n+1}}{2} X_n + \frac{\theta_{\text{NSTAT}}}{2} X_{\text{NSTAT}-1} \quad (\text{B-24})$$

Values of  $\theta$  are either elastic rotations or creep rotations for calculation of elastic deflections and creep deflections respectively. Shown in figure B-12 is a sketch of the rotations and midspan deflection (EQN B-24) for a beam with NSTAT = 3.

B.3 PROGRAM INPUT

The TPSC deck consists of control cards, the TPSC source deck, and input cases. End of record (EOR) cards terminate the control card deck and source deck and an end of information (EOI) card terminates the job deck. Deck setup is shown in Figure B-13. The program requires 102K of core to load and 72K to run under the KRONOS 2.1 operating system.

Input cases are stacked behind the source deck in the deck setup. The first card of each case must be a case identification card. Information listed in columns 1 through 50 on this card will be printed as a heading in the output for that case.

The remainder of data is input using a user oriented "namelist" format. This input follows the identification card for each case. The first card following the identification card must have a \$ in column 2 followed immediately by the name CREEP with no embedded blanks. Succeeding variables are read until a second \$ is encountered. Input variables are defined following the first \$. All except the last variable must be followed by a comma and the first column of each card is ignored. Constant fields may not include imbedded blanks. Blanks, however, may appear elsewhere in data records.

Examples of the input data are shown in figure B-14. Values for variable names beginning with the letters I, N and K are input without decimal points. Commas must immediately follow these values. Subscripted variables such as PRESS(N) are input as PRESS(1) = followed by the values for PRESS (1), PRESS (2), PRESS (3), etc (see example problem 2).

The general table lookup routine used in defining temperature distributions (XTEMP(N) and YTEMP(N)) and elastic modulus data (ETEMP(N)) requires special input considerations. Example input for XTEMP(N), is shown in figure B-14. This is typical also for YTEMP(N) and ETEMP(N). The values 0., 1., K, 0.,  $X_1$ ,  $X_2, \dots, X_k$ ,  $T_1$ ,

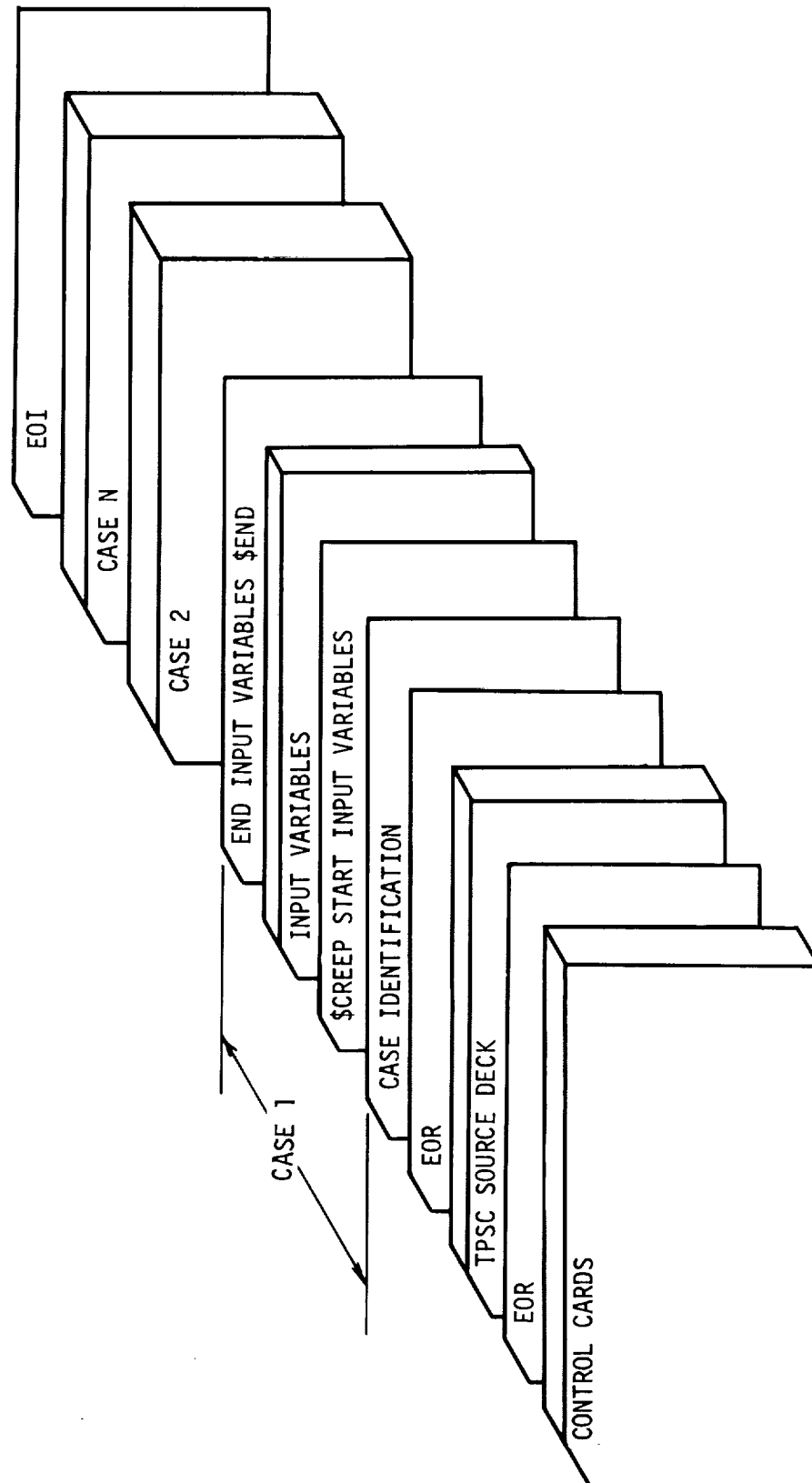


Figure B-13 TPSC PROGRAM DECK SETUP

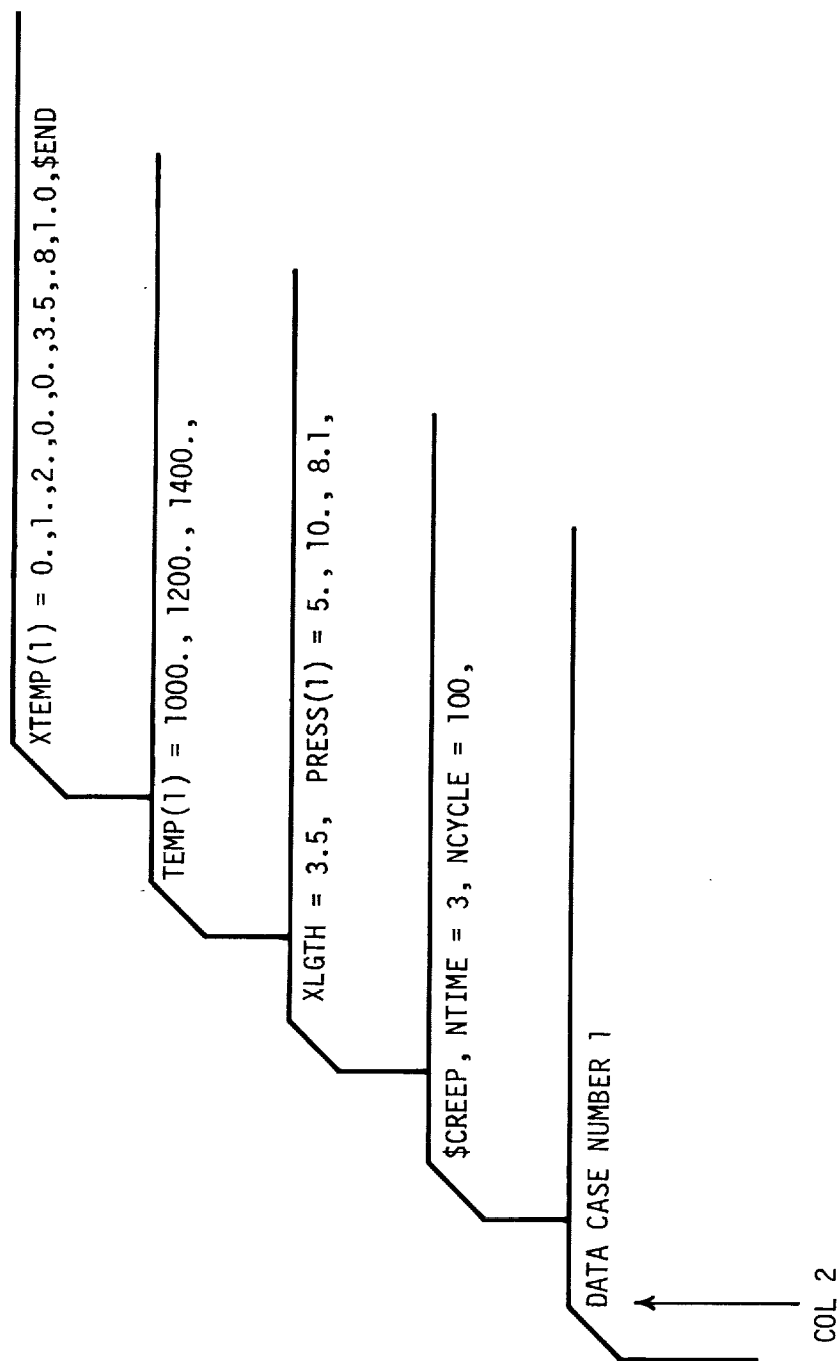


Figure B-14 EXAMPLE INPUT

$T_2, \dots, T_k$  are input following the variable name XTEMP(1) where K is the number of sets of location-temperature data (section 2.3),  $X_1, \dots, X_k$  are the corresponding locations along the panel length, and  $T_1, \dots, T_k$  are the associated normalized temperatures. The first, second, and fourth values (0., 1., 0.) are required input.

Listings of the input variables for the TPSC program are given in Tables B-1 through B-7. For variables where default values are provided, no input need be made except to use options other than those provided for by the default. Example problem inputs are given in Section B-8.

**Table B.1 INPUT FOR ANALYSIS CONTROL**

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Option for Elastic Analysis (Ref. Section B.2.5) ° Creep and Elastic Analysis ° Elastic Analysis Only	INDELA	0 (Default) 1
Number of Analysis Cases (Input in First Case Only)	NEWCAS	1 (Default)
Option for Creep Strain Accumulation Theory (Ref. Section B.2.4) ° Time Hardening ° Strain Hardening	HARDOP	1. (Default) 2.
Maximum Time (Hours) This Input Required When HARDOP = 2. (Ref. Section B.2.4)	TMAX	
Option for Linear Creep Stress- Strain Below Stress = 1.  ° Creep Equation Used ° Linear Stress-Strain EQN Override	INDSTR	0 (Default) 1
Option for TPS Panel Cross Section Geometry (Ref. Section B.2.1) ° Rib ° Corrugation ° Zee	INDGEO	1 2 3
Option for Incorporating Beaded Skin into Geometry (Ref. Section B.2.1) ° No Bead ° Bead Included	INDBD	0 (Default) 1
Number of Stations Along Panel Length Used In The Analysis (Ref. Section B.2.1)	NSTAT	6 (Default)
Number of Sections Through Panel Depth Used in The Analysis (Ref. Section B.2.1)	SEC NSECT	10. (Default) 10 (Default)

INPUT FOR ANALYSIS CONTROL (Continued)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Option for Units Of Time In Input Creep Equation <ul style="list-style-type: none"> <li>° Hours</li> <li>° Minutes</li> </ul>	ITIME	0 (Default) 1
Option for Units of Temperature In Input Creep Equation <ul style="list-style-type: none"> <li>° K/1000.</li> <li>° F/1000.</li> </ul>	IEQNTP	0 (Default) 1
Option for Units of Stress In Input Creep Equation <ul style="list-style-type: none"> <li>° MPa</li> <li>° KSI</li> </ul>	IEQNST	0 (Default) 1
Option for Temperature Input Units <ul style="list-style-type: none"> <li>° K</li> <li>° F</li> </ul>	IINTP	0 (Default) 1
Option for Pressure And Load Input Units <ul style="list-style-type: none"> <li>° Pa, KILOS</li> <li>° psi, Lbs.</li> </ul>	ILOAD	0 (Default) 1
Option for Dimension Input Units <ul style="list-style-type: none"> <li>° cm.</li> <li>° in.</li> </ul>	IDIMEN	0 (Default) 1
Option for Input of Initial Residual Stresses <ul style="list-style-type: none"> <li>° Initial Values = 0.</li> <li>° Values Input</li> </ul>	IRESID	0 (Default) 1
Initial Residual Stress Values	RESSIN (I,J)	

Table B-2 INPUT FOR OUTPUT CONTROL (Ref. Section B.1)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
<p>Option For Printing Creep Deflections at Each Trajectory Step In First Cycle</p> <ul style="list-style-type: none"> <li>° Print Not Req'd.</li> <li>° Print Req'd.</li> </ul>	INCYC	<p>0 1 (Default)</p>
Total Number of Cycles at Which Creep Deflection, Creep Strain, and Residual Stress Output Are Desired	NUMCYC	
Cycle Numbers at Which Output Is To Be Printed.	KCYCLE (N)	

Table B-3 INPUT FOR PANEL GEOMETRY DEFINITION (REF. SECTION B.2.1)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Panel Unsupported Length	XLGTH	
Panel Width	PANWID	
Panel Cross Section	DEPTH	
Stiffener Spacing	PITCH	
Panel Skin Thickness	TS	

INPUT FOR RIB-STIFFENED CROSS SECTION (REF. SECTION B.2.1)  
(INDGEO = 1)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Rib Thickness	TR	
Number of Ribs Across	NRIB	
Distance From Outer Panel Rib to Panel Edge	RIBFLG	

INPUT FOR CORRUGATION STIFFENED CROSS SECTION (REF. SECTION B.2.1)  
(INDGEO = 2)

DEFINTION	INPUT VARIABLE NAME	INPUT VALUE
Corrugation Thickness	TC	
Number of Corrugations Across Panel Width	NCOR	
Corrugation Angle	PHICOR	
Corrugation Length in Contact With Skin	FLAT	
Edge Distance in Excess of Normal Pitch Length	EDGE	

INPUT FOR ZEE STIFFENED CROSS SECTION (REF. SECTION B.2.1)  
(INDGEO = 3)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
ZEE Thickness	TZEE	
Number of Zee Stiffeners Across Panel Width	NZEE	
ZEE Stiffener Flange Dimensions	ZEESF { ZEESF1 ZEEFF ZEEFF1	
Panel Edge Distance	{ ZPNED1 ZPNED2	

INPUT FOR SKIN BEAD GEOMETRY (REF. SECTION B.2.1)  
(INDBD = 1)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Bead Width	BWID	
Bead Depth	BDEP	
Bead Radius (Sign of BRAD Indicates Positive or Negative Bead Direction)	BRAD	

Table B-4 INPUT FOR TRAJECTORY AND LOAD DEFINITION  
(REF. SECTION B.2.2)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Option For Type of Applied Load <ul style="list-style-type: none"> <li>◦ Uniform pressure</li> <li>◦ Point loads</li> </ul>	INDLOD	0 (Default) 1
Number of Time Steps in Trajectory Idealization	NTIME	
Time at End of Each Trajectory Time Step	DXTIME (N)	
Pressure Load at Each Trajectory Time Step (INDLOD = 0)	PRESS (N)	
Point Load at Each Trajectory Time Step (INDLOD = 1)	PLOAD (N)	
Distance from Beam Support to Applied Load (Input for INDLOD = 1)	ALEN	
Number of cycles to Be Analyzed	NCYCLE	
Option for Plate Bending Moment Calculations <ul style="list-style-type: none"> <li>◦ Analysis based on beam loads</li> <li>◦ Plate moments used in analysis</li> </ul>	INDPLA	0 (Default) 1
Stiffness of Panel Edge Support. (Moment of inertia). Input for INDPLA = 1.	ESTIFF	

INPUT FOR TRAJECTORY AND LOAD DEFINITION (Continued)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
<p>Option for Inputting Panel Stiffness in Transverse Panel Direction (Moment of Inertia Per Inch Of Panel Length)</p> <p>° <math>I = TS^3/12 (1-\nu)^2</math></p> <p>° I is input</p>	INDD2	<p>0 (Default)</p> <p>1</p>
Value of Input Transverse Panel Stiffness	DETWO	

TABLE B-5 INPUT FOR PANEL TEMPERATURE DISTIRBUTION  
(REF. SECTION B.2.3)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Midspan Panel Skin Temperature At Each Trajectory Time Step	TEMP (N)	
Option for Temperature As A Function of Panel Length <ul style="list-style-type: none"> <li>◦ Temperature constant along the panel length</li> <li>◦ Temperature variation along the panel length defined by equation coefficients</li> <li>◦ Temperature variation along the panel length defined by Table lookup</li> </ul>	INDTFL	0 (Default) 0 1
Equation Coefficients for Temperature Distribution along Length	C(1)	
Temperature Distribution along Length Defined by Table Lookup k = number of points in Table	XTEMP(1) = 0.,1.,k,0., X <sub>1,k</sub> , T <sub>x1,k</sub>	
Option for Temperature As A Function of Panel Depth <ul style="list-style-type: none"> <li>◦ Temperature constant through the panel depth</li> <li>◦ Temperature variation through the panel depth defined by equation coefficients</li> <li>◦ Temperature variation through the panel depth defined by Table lookup</li> </ul>	INDTFD	0 (Default) 0 1
Equation Coefficients for Temperature Distribution Through the Panel Depth	D(1)	

INPUT FOR PANEL TEMPERATURE DISTRIBUTION (Continued)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Temperature Distribution Through Depth Defined By Table Lookup k = Number of points in Table	YTEMP (1) = 0.,1.,k,0., Y <sub>1,k</sub> , T <sub>y1,k</sub>	
Temperature Variation in Panel <ul style="list-style-type: none"> <li>Constant over Panel</li> <li>Variation defined by equation or Table</li> </ul>	ITCON	1 (Default) 0
Temperature Variation in Trajectory <ul style="list-style-type: none"> <li>Constant</li> <li>Variable</li> </ul>	NTCON	1 (Default) 0



TABLE B-6 INPUT FOR MATERIAL PROPERTY DEFINITION  
(REF. SECTION B.2.3)

DEFINITION	INPUT VARIABLE NAME	INPUT VALUE
Poisson's Ratio (Req'd. if INDPLA = 1)	XNU	
Option for Definition of Elastic Modulus Data ° Constant modulus ° Modulus defined by equation ° Modulus defined by table lookup	INDMOD	0 (Default) 0 1
Equation Coefficients for Elastic Modulus as Function of Temperature	ECOEFF (1)	
Tabular Data for Elastic Modulus as A Function of Temperature	ETEMP (1) = 0., 1., k, 0., T <sub>1,k</sub> , E <sub>1,k</sub>	

TABLE B-7 INPUT FOR CREEP PROPERTY DEFINITION

Material creep properties are defined through the input of coefficients (Z) to the linear equation

$$\ln \epsilon = Z_1 X_1 + Z_2 X_2 + Z_3 X_3 + \dots + Z_N X_N$$

or

(B-25)

$$\epsilon = \exp (Z_1 X_1 + Z_2 X_2 + Z_3 X_3 + \dots + Z_N X_N)$$

In this equation, the value of  $X_1$  is defined as 1 and  $X_2$  through  $X_N$  are terms in time (t), stress ( $\sigma$ ), and temperature (T) listed in table B-7 section. Only the terms required need to be input.



TABLE B-7 INPUT FOR CREEP PROPERTY DEFINITION  
(CONTINUED)

COEFFICIENT NAME	CREEP EQUATION TERM	
	LOGARITHMIC FORM	EXPONENTIAL FORM
Z(1)	$Z_1$	$e^{Z_1}$
Z(2)	$Z_2 \sigma$	$e^{Z_2 \sigma}$
Z(3)	$Z_3 T$	$e^{Z_3 T}$
Z(4)	$Z_4 t$	$e^{Z_4 t}$
Z(5)	$Z_5 \left(\frac{1}{T}\right)$	$e^{Z_5 / T}$
Z(6)	$Z_6 \ln t$	$t^{Z_6}$
Z(7)	$Z_7 \ln \sigma$	$\sigma^{Z_7}$
Z(8)	$Z_8 \ln T$	$T^{Z_8}$
Z(9)	$Z_9 \sigma^2$	$e^{Z_9 \sigma^2}$
Z(10)	$Z_{10} \sigma^3$	$e^{Z_{10} \sigma^3}$

INPUT FOR CREEP PROPERTY DEFINITION (Continued)

COEFFICIENT NAME	CREEP EQUATION TERM	
	LOGARITHMIC FORM	EXPONENTIAL FORM
Z(11)	$Z_{11} \left(\frac{1}{T}\right)^2$	$e^{Z_{11}/T^2}$
Z(12)	$Z_{12} \left(\frac{1}{T}\right)^3$	$e^{Z_{12}/T^3}$
Z(13)	$Z_{13} \sigma T$	$e^{Z_{13} \sigma T}$
Z(14)	$Z_{14} \left(\frac{\sigma}{T}\right)$	$e^{Z_{14} (\sigma/T)}$
Z(15)	$Z_{15} (\sigma T)^2$	$e^{Z_{15} (\sigma T)^2}$
Z(16)	$Z_{16} (\sigma T)^3$	$e^{Z_{16} (\sigma T)^3}$
Z(17)	$Z_{17} \left(\frac{\sigma}{T}\right)^2$	$e^{Z_{17} (\sigma/T)^2}$
Z(18)	$Z_{18} \left(\frac{\sigma}{T}\right)^3$	$e^{Z_{18} (\sigma/T)^3}$
Z(19)	$Z_{19} (1n\sigma)^2$	$\sigma^{Z_{19} 1n\sigma}$
Z(20)	$Z_{20} (1n\sigma)^3$	$\sigma^{Z_{20} (1n\sigma)^2}$



INPUT FOR CREEP PROPERTY DEFINITION (Continued)

COEFFICIENT NAME	CREEP EQUATION TERM	
	LOGARITHMIC FORM	EXPONENTIAL FORM
Z(21)	$Z_{21} \sigma \ln T$	$T^{Z_{21} \sigma}$
Z(23)	$Z_{23} \ln \sigma \ln T$	$\sigma^{Z_{23} \ln T}$
Z(22)	$Z_{22} T \ln \sigma$	$\sigma^{Z_{22} T}$
Z(24)	$Z_{24} t \sigma T$	$e^{Z_{24} t \sigma T}$
Z(25)	$Z_{25} (t \sigma T)^2$	$e^{Z_{25} (t \sigma T)^2}$
Z(26)	$Z_{26} (t \sigma T)^3$	$e^{Z_{26} (t \sigma T)^3}$
Z(27)	$Z_{27} t^2$	$e^{Z_{27} t^2}$
Z(28)	$Z_{28} t^3$	$e^{Z_{28} t^3}$
Z(29)	$Z_{29} (\ln t)^2$	$t^{Z_{29} \ln t}$
Z(30)	$Z_{30} (\ln t)^3$	$t^{Z_{30} (\ln t)^2}$

INPUT FOR CREEP PROPERTY DEFINITION (Continued)

<u>COEFFICIENT NAME</u>	CREEP EQUATION TERM	
	LOGARITHMIC FORM	EXPONENTIAL FORM
Z(31)	$Z_{31}T^2$	$e^{Z_{31}T^2}$
Z(32)	$Z_{32}T^3$	$e^{Z_{32}T^3}$
Z(33)	$Z_{33}(\ln T)^2$	$T^{Z_{33} \ln T}$
Z(34)	$Z_{34}(\ln T)^3$	$T^{Z_{34}(\ln T)^2}$
Z(35)	$Z_{35} \sigma \ln T$	$t^{Z_{35} \sigma}$
Z(36)	$Z_{36} T \ln T$	$t^{Z_{36} T}$
Z(37)	$Z_{37} t \ln \sigma$	$\sigma^{Z_{37} t}$
Z(38)	$Z_{38} t \ln T$	$T^{Z_{38} t}$
Z(39)	$Z_{39} \ln \sigma \ln T$	$\sigma^{Z_{39} \ln T}$
Z(40)	$Z_{40} \ln T \ln T$	$t^{Z_{40} \ln T}$

INPUT FOR CREEP PROPERTY DEFINITION (Continued)

<u>COEFFICIENT NAME</u>	CREEP EQUATION TERM	
	LOGARITHMIC FORM	EXPONENTIAL FORM
Z(41)	$Z_{41} \frac{\ln \sigma}{T}$	$\sigma^{Z_{41}/T}$
Z(42)	$Z_{42} \frac{\ln t}{T}$	$t^{Z_{42}/T}$
Z(43)	$Z_{43} \frac{\ln t \ln \sigma}{T}$	$\sigma^{Z_{43} \ln t / T}$
Z(44)	$Z_{44} t \sigma$	$e^{Z_{44} t \sigma}$
Z(45)	$Z_{45} t T$	$e^{Z_{45} t T}$

## B.4 PROGRAM OUTPUT

Program output includes a listing of input variables, calculated elastic stresses at each panel station and trajectory step, panel geometry definition, trajectory load and temperature definition, creep strain equation definition, elastic deflections, creep deflections, creep strain distributions, and residual stress distributions. Creep deflections are printed at times within the first cycle and at the end of each requested cycle (KCYCLE). The example problems in Section 8 show typical program output.

Generally the output is automatic and not controlled by the user. The following items are, however, at the option of the program user.

- (a) Printout of calculated creep deflections at the end of each trajectory time step in the first cycle are controlled by the variable INCYC. These deflections are printed as a default unless INCYC = 0 is input.
- (b) The cycles at which calculated creep deflections, creep strain distributions, and residual stress distributions are printed, are controlled by the input variables NUMCYC and KCYCLE(N). The variable NUMCYC is the total number of cycles at which printout is required (maximum = 10) and KCYCLE(N) is the designated cycle number for printout.
- (c) Both analysis and output are controlled by the number of panel length increments and depth increments defined by the input variables NSECT, SEC, NSTAT, and NBSECT (ref. Section B.3). Output of stresses, deflections, and strains are calculated and printed at locations along the panel length and through the panel depth as defined by these input variables in conjunction with the panel geometry (Section B.2.1).

## B.5 PROGRAM DIAGNOSTICS

Three types of diagnostic statements are included in the TPSC program to provide user information where problems may be occurring in the analysis.

## (1) "CORRUGATION INPUT DATA YIELDS A NEGATIVE S LENGTH"

This information is printed out and the analysis terminated when the calculated corrugation flat length dimension S (Ref. Section B.2.1) is negative. The geometry definition (PHICOR, DEPTH, FLAT, PITCH) should be checked for input error.

(2) "ERROR IN TABLE-LOOKUP ROUTINE AT  $\begin{Bmatrix} XIN \\ YIN \\ TIN \end{Bmatrix} = \underline{A} ."$ 

A is the panel station X(XIN), the panel depth increment (YIN), or the temperature (TIN) which exceeds the bounds of the appropriate input table for temperature as a function of length, temperature as a function of depth, and elastic modulus as a function of temperature, respectively. The table lookup input data should be checked to insure that the range of data in the table extends over the range needed for analysis.

(3) "WARNING - TWENTY ITERATIONS ON STRESS IN THE HARDENING ROUTINE  
BEAM STATION = \_\_\_\_\_, SECTION (J) = \_\_\_\_\_, CYCLE = \_\_\_\_\_, STEP = \_\_\_\_\_  
ANALYSIS PROCEEDING WITH STRESS UNCHANGED."

This diagnostic message indicates that the iteration process for stress (Reference Section B.2.6) did not converge. The stress is defined as that at the end of the previous time step and analysis continues. When this occurs, the form of the input empirical equation should be checked.

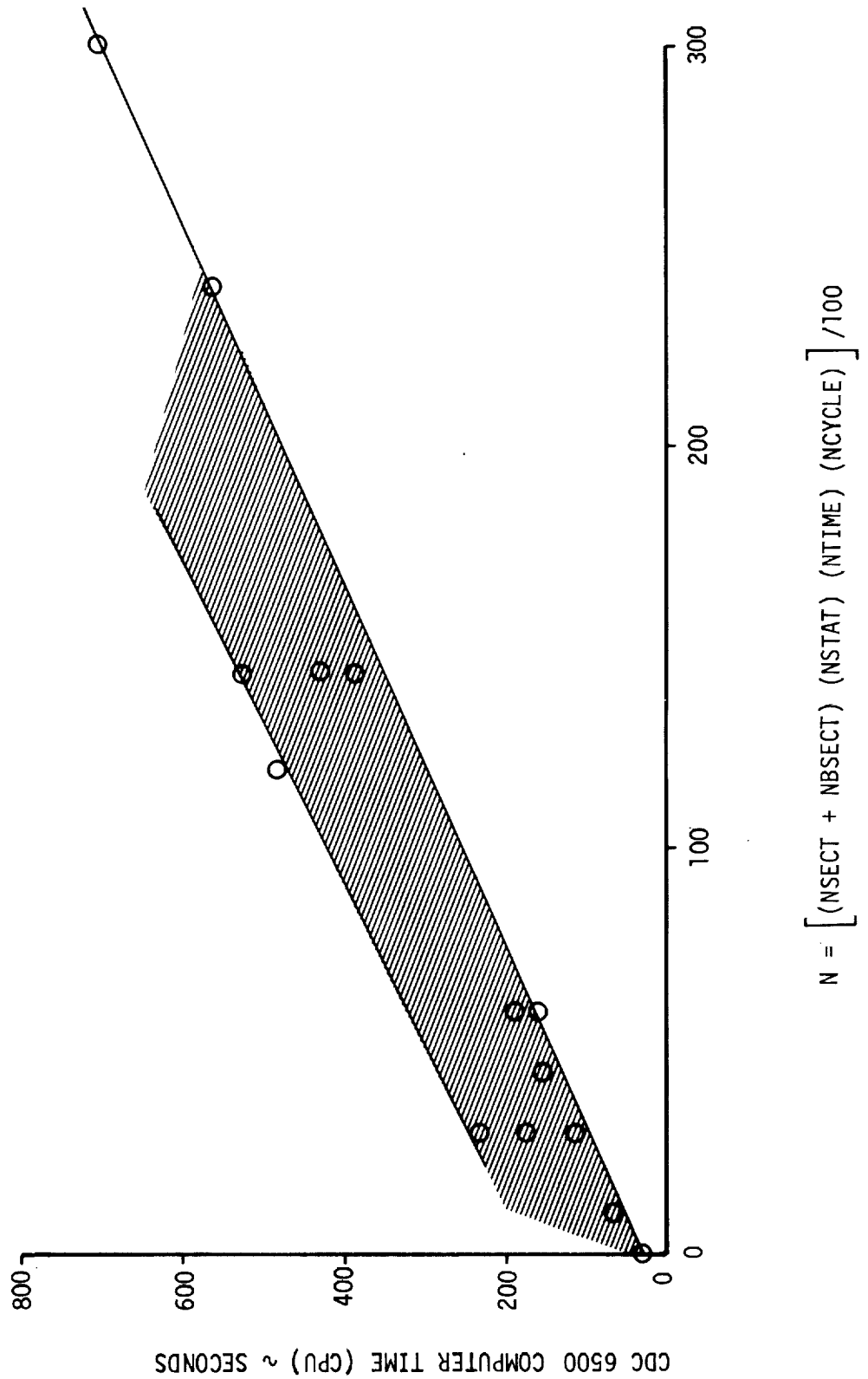
B.6 COMPUTER TIME REQUIREMENTS

Computer (CPU) time data compiled for analysis cases using the TPSC program are shown in Figure B-15. These data points are plotted against a value  $N$  which is the product of the number of increments through the panel depth ( $N_{SECT} + N_{BSECT}$ ), the number of segments along the panel length ( $N_{STAT}$ ), the number of time steps in the load-temperature trajectory ( $N_{TIME}$ ), and the number of cycles being analyzed. Data, shown in the figure are for analysis conducted using the time hardening theory of creep accumulation program option.

Because additional iterations are required in analysis using the strain hardening theory of creep accumulation, a factor of 1.8 (factors for specific cases typically range from 1.3 to 2.4) should be applied to the range shown in Figure B-15 for this option. In addition, the data plotted are for separate runs. Running of multiple cases results in a somewhat lower per case computer time.

Variations in required computer time are expected from run to run depending on times required to converge on stresses, load balances, and moment balances in the program. The data, obtained in analysis for prediction of creep deflections in subsize and full size panels, are presented to provide computer time guidelines for the TPSC program user.

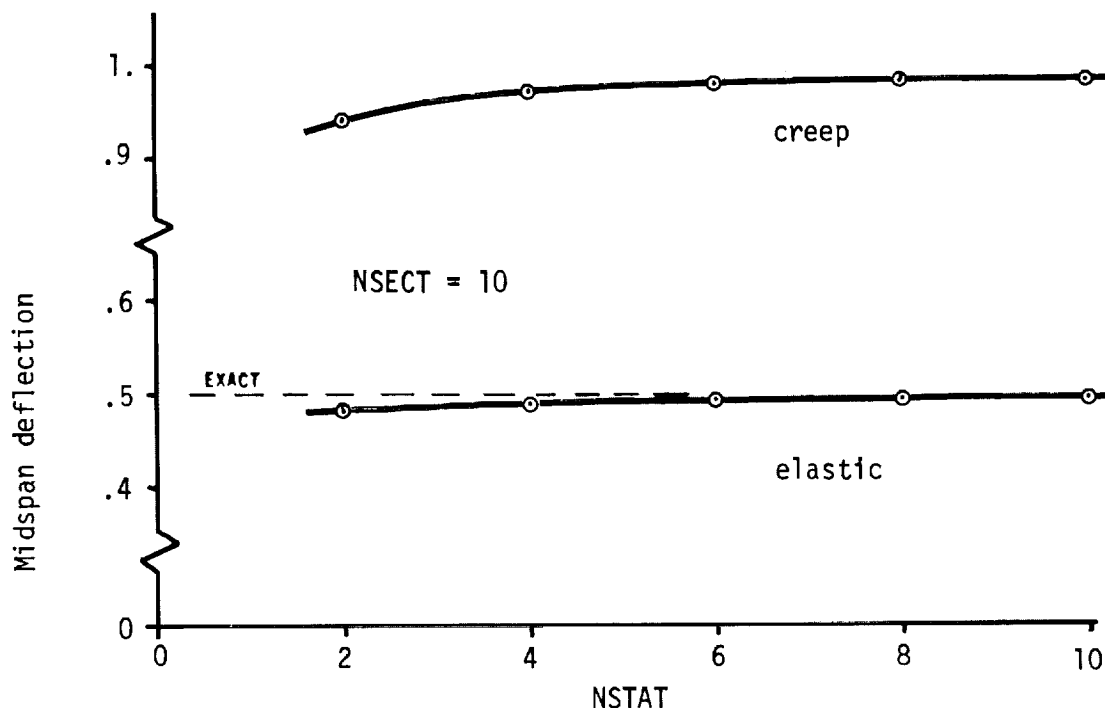
FIGURE B-15 TPSC COMPUTER TIME REQUIREMENTS



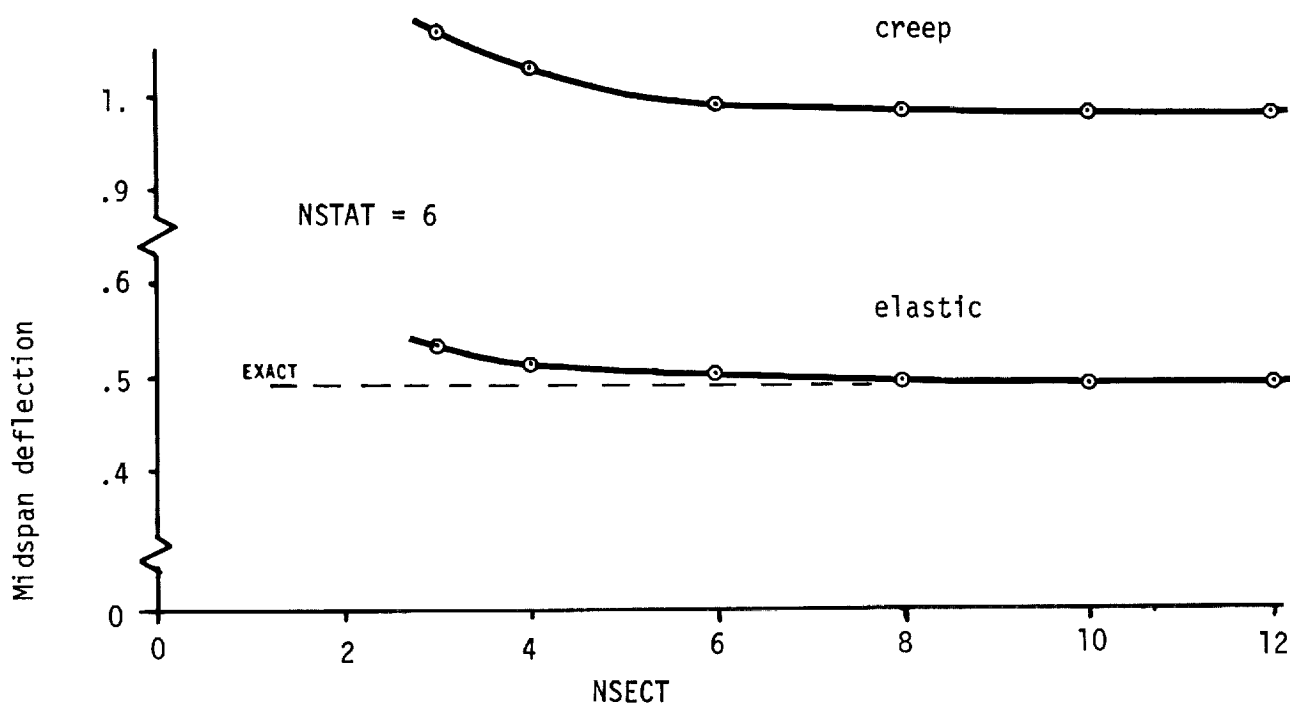
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B.7 ANALYSIS SENSITIVITY TO PANEL ELEMENT SIZE

Limited studies have been conducted to investigate the sensitivity of the TPSC program prediction capability to the number of sections along the panel length (NSTAT) and increments through the panel depth (NSECT). Shown in Figure B-16 are typical midspan elastic and creep deflection predictions as a function of these variables. This example is based on an 11-inch titanium panel subjected to a uniform pressure load. These studies have demonstrated that a minimum of these sections are needed to maintain prediction precision and, based on study results, the values NSTAT = 6 and NSECT = 10 are default values in the program. Because more sections are required to define horizontal sections and flange sections for the Z stiffened TPS and corrugation stiffened TPS than for the rib stiffened concept, a minimum value for NSECT is recommended as 8 for the rib, 10 for the corrugation, and 12 for the Z stiffened concepts. Also the effects of NSECT on creep deflection prediction capability will be somewhat dependent on the degree of nonlinearity of the input creep strain equation with respect to stress. Therefore, more sections should be included for equations which are very nonlinear in stress.



(a) SENSITIVITY TO SECTIONS ALONG PANEL LENGTH



(b) SENSITIVITY TO INCREMENTS THROUGH PANEL DEPTH

FIGURE B-16 TYPICAL SENSITIVITY OF PREDICTION TO ELEMENT SIZE

B.8 EXAMPLE PROBLEMS

The following two example problems are provided to demonstrate the input for the major programs options. Section B.3 should be consulted for default values for variable means not included in the input. These default values apply when the variable is not input.

EXAMPLE PROBLEM 1

Example problem 1 demonstrates analysis capability for the rib stiffened panel shown in Figure B-17(a). This simply supported panel is loaded with two point loads located 3.62 inches from the panel supports as shown in Figure B-17(b). In the example, the panel is subjected to ten constant load and temperature cycles of 20 minute duration each (Figure B-17(c)) and the time hardening theory of creep accumulation is used. The panel temperature is a function of panel length as shown in Figure B-17(d). Elastic modulus is defined as a function of temperature as  $12.3 \times 10^6$  psi at 725°F and  $11.8 \times 10^6$  psi at 825°F (note this covers panel temperature range of 769°F (.932 x 825, Figure B-17(d)) to 825°F). U.S. Customary units are used in the input data. The following creep equation for Titanium (Reference 2) is used:

$$\ln \epsilon = -26.22982 + 26.2485T + .000126\sigma^2 + 1.40406 \ln \sigma + .46894 \ln t$$

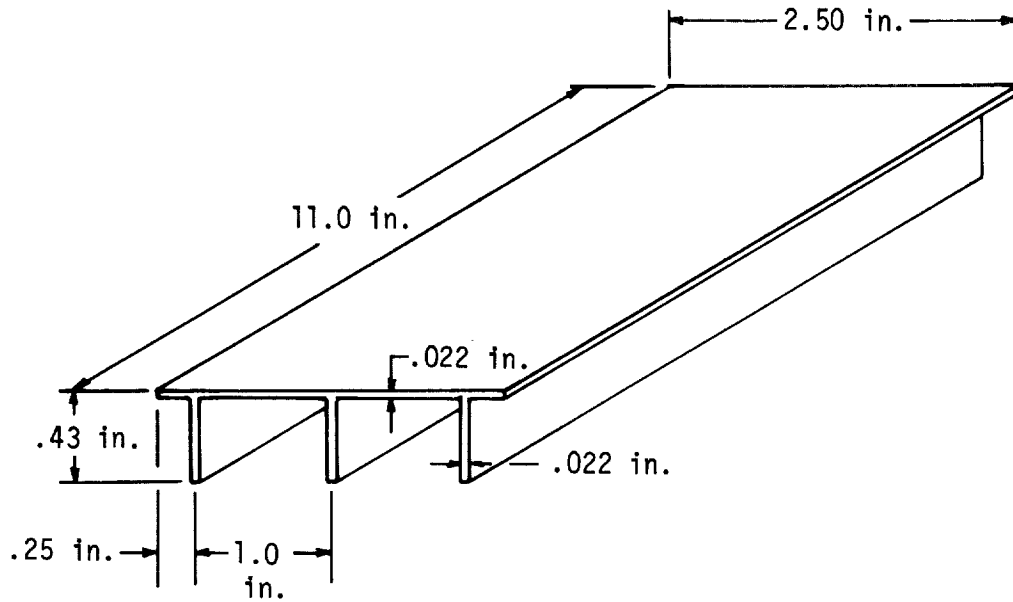
where  $t$  = time, hours

$T$  = temperature, °F/1000

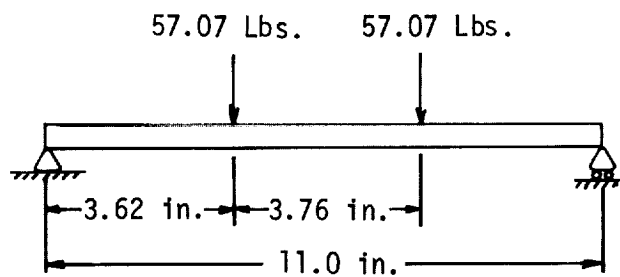
$\sigma$  = stress, ksi

Deflection, strain, and stress outputs are requested after 1, 3, and 10 cycles are data given through the panel depth and along its length are output according to the default values of NSECT (10) and NSTAT (6), respectively.

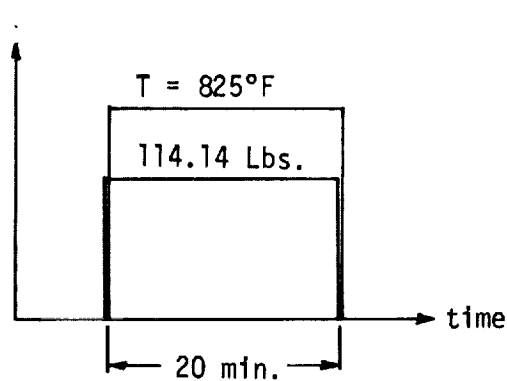
Input and output for example problem 1 follows.



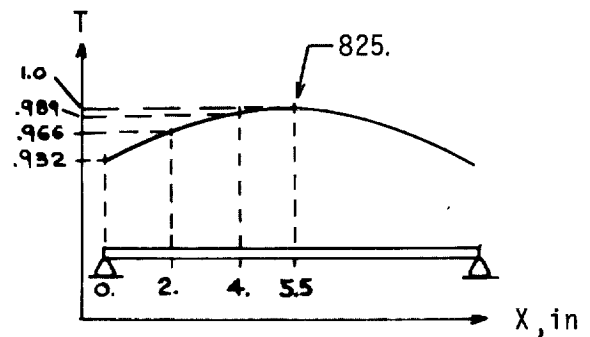
(a) PANEL GEOMETRY



(b) PANEL LOAD



(c) TRAJECTORY



(d) TEMPERATURE DISTRIBUTION

FIGURE B-17 EXAMPLE PROBLEM 1

B-60

MCDONNELL DOUGLAS ASTRONAUTICS COMPANY - EAST



### PHASE III

NAS-1-11774

TPSC EXAMPLE PROBLEM 1

### EXAMPLE PROBLEM 1 INPUT



NAS-1-11774

### EXAMPLE PROBLEM 1 OUTPUT

DEPTH = 0.43E+00,

PHICOR = 0.0.

PITCH = 0.1E+01,

FLAT = 0.0,

EDGE = 0.0,

NCOR = 3.

TS = 0.22E-01.

TC = 0.0,

XLGTH = 0.11E+02,

TR = 0.22E-01,

NRI9 = 3.

RIBFLG = 0.25E+00,

ZPNED1 = 0.0,

ZPNE02 = 0.0.

NZEE = 0.

TZEE = 0.0,

ZEESEF = 0.0,

ZEESEF1 = 0.0,

ZEEFF = 0.0.

ZEEFF1 = 0.0,

BWID = 0.0,

ВНЕР = 0.0.

BRAD = 0.07

PRESS = 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0,

TEMP = 0.825E+03, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0,

```

DXTIME   = 0.2E+02, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0,

```

PL0AD = 0.114145E+03, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0, 0.0,

ALEN = 0.3E2E+01,

XNU = D.C.

PANWID = 0.25E+01.

C = 0.1E+01, 0.0, 0.0, 0.0,

0 = 0.1E+01, 0.0, 0.0, 0.0,

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NAS-1-11774

[illegible]



NAS-1-11774

```
ESTIFF = 0.0,
INDO2  = 0,
INDSTR = 0,
DETWO  = 0.0,
$END
```

[illegible]

EXAMPLE PROBLEM 1 OUTPUT (Continued)

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= .917

J	AREA(SQ.IN)	Y(IN)	STRESS(PSI)
1	2.99201E-03	2.26667E-02	7.07220E+03
2	2.99201E-03	6.80000E-02	6.08774E+03
3	2.99201E-03	1.13333E-01	5.10328E+03
4	2.99201E-03	1.58667E-01	4.11882E+03
5	2.99201E-03	2.04000E-01	3.13436E+03
6	2.99201E-03	2.49333E-01	2.14990E+03
7	2.99201E-03	2.94667E-01	1.16544E+03
8	2.99201E-03	3.40000E-01	1.80582E+02
9	2.99201E-03	3.85333E-01	1.03478E+02
10	2.99201E-03	4.30667E-01	1.53458E+02

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 1.333

J	AREA(SQ.IN)	Y(IN)	STRESS(PSI)
1	2.99201E-03	2.26667E-02	2.12166E+04
2	2.99201E-03	6.80000E-02	1.82632E+04
3	2.99201E-03	1.13333E-01	1.53098E+04
4	2.99201E-03	1.58667E-01	1.23565E+04
5	2.99201E-03	2.04000E-01	9.40309E+03
6	2.99201E-03	2.49333E-01	6.44971E+03
7	2.99201E-03	2.94667E-01	3.49633E+03
8	2.99201E-03	3.40000E-01	5.42947E+02
9	2.99201E-03	3.85333E-01	2.41043E+02
10	2.99201E-03	4.30667E-01	4.60375E+02

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 2.750

J	AREA(SQ.IN)	Y(IN)	STRESS(PSI)
1	2.99201E-03	2.26667E-02	3.53611E+04
2	2.99201E-03	6.80000E-02	3.04388E+04
3	2.99201E-03	1.13333E-01	2.55164E+04
4	2.99201E-03	1.58667E-01	2.05941E+04
5	2.99201E-03	2.04000E-01	1.56717E+04
6	2.99201E-03	2.49333E-01	1.07493E+04
7	2.99201E-03	2.94667E-01	5.82722E+03
8	2.99201E-03	3.40000E-01	9.44911E+02
9	2.99201E-03	3.85333E-01	4.61743E+02
10	2.99201E-03	4.30667E-01	7.67292E+02

EXAMPLE PROBLEM 1 OUTPUT (Continued)

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 3.667

TRAJECTORY TIME STEP 1			
J	AREA(SQ.IN)	Y(IN)	STRESS(PSI)
1	2.99200E-03	2.26667E-02	5.58575E+04
2	2.99200E-03	6.80000E-02	4.80821E+04
3	2.99200E-03	1.13333E-01	4.03067E+04
4	2.99200E-03	1.58667E-01	3.25312E+04
5	2.99200E-03	2.04000E-01	2.47558E+04
6	2.99200E-03	2.49333E-01	1.69803E+04
7	2.99200E-03	2.94667E-01	9.20488E+03
8	2.99200E-03	3.40000E-01	1.42943E+03
9	2.99200E-03	3.85333E-01	-6.34601E+03
10	5.50000E-02	4.19000E-01	-1.21204E+04

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 4.583

TRAJECTORY TIME STEP 1			
J	AREA(SQ.IN)	Y(IN)	STRESS(PSI)
1	2.99200E-03	2.26667E-02	5.58575E+04
2	2.99200E-03	6.80000E-02	4.80821E+04
3	2.99200E-03	1.13333E-01	4.03067E+04
4	2.99200E-03	1.58667E-01	3.25312E+04
5	2.99200E-03	2.04000E-01	2.47558E+04
6	2.99200E-03	2.49333E-01	1.69803E+04
7	2.99200E-03	2.94667E-01	9.20488E+03
8	2.99200E-03	3.40000E-01	1.42943E+03
9	2.99200E-03	3.85333E-01	-6.34601E+03
10	5.50000E-02	4.19000E-01	-1.21204E+04

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 5.500

TRAJECTORY TIME STEP 1			
J	AREA(SQ.IN)	Y(IN)	STRESS(PSI)
1	2.99200E-03	2.26667E-02	5.58575E+04
2	2.99200E-03	6.80000E-02	4.80821E+04
3	2.99200E-03	1.13333E-01	4.03067E+04
4	2.99200E-03	1.58667E-01	3.25312E+04
5	2.99200E-03	2.04000E-01	2.47558E+04
6	2.99200E-03	2.49333E-01	1.69803E+04
7	2.99200E-03	2.94667E-01	9.20488E+03
8	2.99200E-03	3.40000E-01	1.42943E+03
9	2.99200E-03	3.85333E-01	-6.34601E+03
10	5.50000E-02	4.19000E-01	-1.21204E+04

EXAMPLE PROBLEM 1 OUTPUT (Continued)

CREEP PREDICTION COMPUTER PROGRAM  
TPSC EXAMPLE PROBLEM 1

RIB STIFFENED TPS PANEL

SKIN GAGE = .022 INCHES  
RIB GAGE = .022 INCHES  
NUMBER OF RIBS = 3  
PITCH LENGTH = 1.000 INCHES  
PANEL EDGE LENGTH = .250 INCHES  
PANEL DEPTH = .430 INCHES  
CALCULATED MOMENT OF INERTIA = .0012046 IN\*\*4  
ELASTIC NEUTRAL AXIS = .348 INCHES

PANEL LENGTH = 11.00 INCHES  
PANEL WIDTH = 2.50 INCHES

APPLIED LOADS

TWO POINT LOADS , DISTANCE FROM SUPPORT TO LOAD = 3.62 INCHES

TRAJECTORY DATA

TIME (SECONDS)  
LOAD (LBS)  
TEMPERATURE (DEG F)

START	TIME END	TOTAL LOAD	MIDSPAN SKIN TEMPERATURE
0.00	20.00	114.145	825.0

CREEP PREDICTION COMPUTER PROGRAM

CYCLIC CREEP EQUATION DEFINITION

LN(STRAIN) = -2.62298E+01  
2.62445E+01 \*(TEMP)  
4.68940E-01 \*LN(TIME)  
1.40466E+00 \*LN(STRESS)  
1.20000E-04 \*(STRESS)\*\*2

WHERE TIME = MINUTES  
TEMPERATURE = DEG K/1000.  
STRESS = KS1

CREEP PREDICTION COMPUTER PROGRAM  
ELASTIC DEFLECTION SUMMARY

TIME (MIN)	BEAM STATION (INCHES)	.92	1.83	2.75	3.67	4.58	5.50
20.00		.0538	.1034	.1447	.1736	.1852	.1913

CREEP PREDICTION COMPUTER PROGRAM  
FIRST CYCLE CREEP DEFLECTION SUMMARY

TIME	BEAM STATION (INCHES)	.92	1.83	2.75	3.67	4.58	5.50
20.00		.00874	.01718	.02468	.03020	.03249	.03373

EXAMPLE PROBLEM 1 OUTPUT (Continued)

CREEP PREDICTION COMPUTER PROGRAM  
CREEP DEFLECTION SUMMARY

CYCLE	BEAM STATION (INCHES)	2.75	3.67	4.58	5.50
1	.92	1.83			
3	.05874	.01718	.02468	.03020	.03249
10	.01460	.02869	.04121	.05042	.05425
	.02542	.04995	.07174	.08776	.09443
					.09801

CREEP PREDICTION COMPUTER PROGRAM  
CREEP STRAINS (PERCENT)  
CYCLE 1

HEIGHT	BEAM STATION (INCHES)	2.75	3.67	4.58	5.50
.00109	.92	1.83			
.00178	.05874	.01718	.02468	.03020	.03249
.00164	.01460	.02869	.04121	.05042	.05425
.00144	.02542	.04995	.07174	.08776	.09443
.00124					.09801
.00097					
.00065					
.00022					
.00024					
.00051					
.00017					

CREEP PREDICTION COMPUTER PROGRAM  
RESIDUAL STRESSES (PSI)  
CYCLE 1

HEIGHT	BEAM STATION (INCHES)	2.75	3.67	4.58	5.50
.00109	.92	1.83			
.00178	.05874	.01718	.02468	.03020	.03249
.00164	.01460	.02869	.04121	.05042	.05425
.00144	.02542	.04995	.07174	.08776	.09443
.00124					.09801
.00097					
.00065					
.00022					
.00024					
.00051					
.00017					

CREEP PREDICTION COMPUTER PROGRAM  
CREEP STRAINS (PERCENT)  
CYCLE 3

HEIGHT	BEAM STATION (INCHES)	2.75	3.67	4.58	5.50
.00109	.92	1.83			
.00178	.05874	.01718	.02468	.03020	.03249
.00164	.01460	.02869	.04121	.05042	.05425
.00144	.02542	.04995	.07174	.08776	.09443
.00124					.09801
.00097					
.00065					
.00022					
.00024					
.00051					
.00017					

EXAMPLE PROBLEM 1 OUTPUT (Continued)

CREEP PREDICTION COMPUTER PROGRAM  
RESIDUAL STRESSES (PSI)  
CYCLE 3

HEIGHT	BEAM STATION (INCHES)				
	0.92	1.83	2.75	3.67	4.58
.0227	6.72E+01	4.13E+02	1.23E+03	3.57E+03	4.58E+03
.0680	3.27E+01	1.89E+02	4.47E+02	1.22E+03	3.69E+03
.1133	-1.81E+01	-2.75E+01	-1.18E+02	-4.44E+02	-1.22E+03
.1587	-3.13E+01	-1.43E+02	-5.28E+02	-1.66E+03	-4.77E+03
.2040	-4.97E+01	-2.89E+02	-7.67E+02	-2.22E+03	-6.44E+03
.2493	-6.87E+01	-4.37E+02	-1.01E+03	-2.77E+03	-8.33E+03
.2947	-8.57E+01	-5.19E+02	-1.28E+03	-3.22E+03	-9.22E+03
.3400	-1.44E+02	-7.01E+02	-1.52E+03	-3.66E+03	-1.01E+04
.3853	-2.23E+02	-1.17E+03	-2.22E+03	-5.22E+03	-1.44E+04
.4190	7.31E+00	4.17E+01	1.13E+02	3.22E+02	3.66E+02

CREEP PREDICTION COMPUTER PROGRAM  
CREEP STRAINS (PERCENT)  
CYCLE 10

HEIGHT	BEAM STATION (INCHES)				
	0.92	1.83	2.75	3.67	4.58
.0227	.0057249	.0000000	.0000000	.0000000	.0000000
.0680	.0000433	.0000000	.0000000	.0000000	.0000000
.1133	.0000000	.0000000	.0000000	.0000000	.0000000
.1587	.0000000	.0000000	.0000000	.0000000	.0000000
.2040	.0000000	.0000000	.0000000	.0000000	.0000000
.2493	.0000000	.0000000	.0000000	.0000000	.0000000
.2947	.0000000	.0000000	.0000000	.0000000	.0000000
.3400	.0000000	.0000000	.0000000	.0000000	.0000000
.3853	.0000000	.0000000	.0000000	.0000000	.0000000
.4190	.0000000	.0000000	.0000000	.0000000	.0000000

CREEP PREDICTION COMPUTER PROGRAM  
RESIDUAL STRESSES (PSI)  
CYCLE 10

HEIGHT	BEAM STATION (INCHES)				
	0.92	1.83	2.75	3.67	4.58
.0227	1.18E+02	7.18E+02	1.89E+03	5.22E+03	5.58E+03
.0680	6.00E+01	3.13E+02	7.77E+02	2.22E+03	3.69E+03
.1133	4.44E+01	1.43E+02	5.28E+02	1.66E+03	4.77E+03
.1587	3.13E+01	1.43E+02	5.28E+02	1.66E+03	4.77E+03
.2040	1.81E+01	2.75E+01	1.18E+02	4.44E+02	1.22E+03
.2493	8.57E+01	5.19E+02	1.28E+03	3.22E+03	9.22E+03
.2947	1.44E+02	7.01E+02	1.52E+03	3.66E+03	1.01E+04
.3400	2.23E+02	1.17E+03	2.22E+03	5.22E+03	1.44E+04
.3853	7.31E+00	4.17E+01	1.13E+02	3.22E+02	3.66E+02
.4190	1.26E+01	7.13E+01	1.88E+02	5.44E+02	5.58E+02

EXAMPLE PROBLEM 2

Example problem 2 demonstrates analysis for the 45.7 cm. long and 51.6 cm. wide beaded single skin corrugation stiffened panel shown in figure B-18(a). This panel is loaded with a uniform pressure and temperature where the pressure and temperature vary with time in each cycle as shown in figure B-18(c). The plate option is used in this problem and therefore the panel edge stiffness and Poisson's ratio are required input. The strain hardening theory of creep accumulation is applied in this example. SI units are used and the empirical creep equation is that for L605 (obtained in Phase I) as follows.

$$\ln \epsilon = -2.89413 - .01743t + .54892 \ln t + 1.31015 \ln \sigma - 6.66548 (1/T) \\ + .19131 \ln T + .00021 (T \sigma t)$$

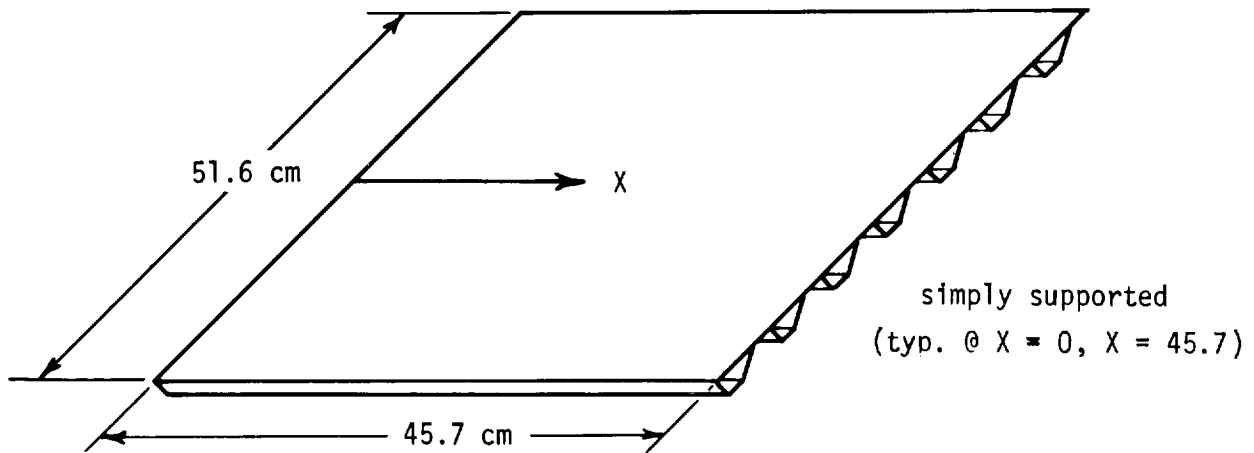
where  $t$  = time, hours

$T$  = temperature,  $^{\circ}\text{K}/1000$

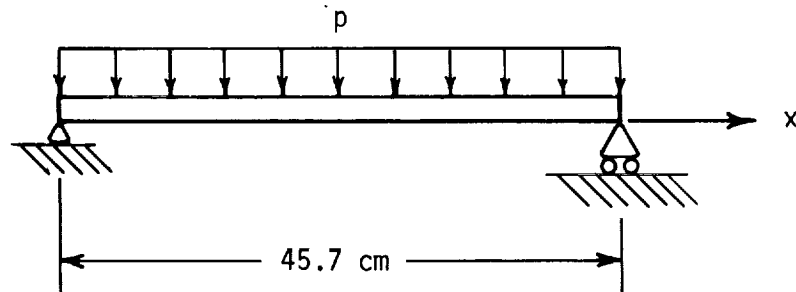
$\sigma$  = stress, MPa

Deflection, strain, and stress output are requested after 1, 2, 3, 4 cycles and output are provided at 23 locations through the panel depth (NSECT + NBSECT) and 8 locations along the length (NSTAT).

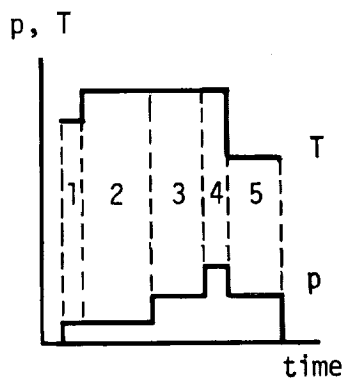
Input and output for example problem 2 follows.



(a) PANEL GEOMETRY



(b) PANEL LOAD



(c) TRAJECTORY

TRAJECTORY STEP	TIME (Min.)	TEMP ( $^{\circ}$ K)	PRESSURE (Pa)
1	0-1.33	1122	758
2	1.33-4.67	1231	758
3	4.67-7.50	1231	1655
4	7.50-8.50	1225	2482
5	8.50-11.33	1014	1655

figure B-18 EXAMPLE PROBLEM 2



## PHASE III SUMMARY REPORT

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```

NEWCAS=1,  $END
      Z(5)=-6.66546,Z(8)=.19131,Z(24)=.00021,
      Z(1)=-2.89414,Z(4)=-.01743,Z(6)=.54892,Z(7)=1.31015,
      TEMP(1)=1122.,1231.,1231.,1225.,1014.,
      DXTIME(1)=1.33,4.67,7.50,8.50,11.33,
      PRESS(1)=758.,758.,1655.,2482.,1655.,
      NTIME=5,NCYCLE=4,NUMCYC=4,KCYCLE(1)=1,2,3,4,
      INDPLA=1,ESTIFF=.5078,XNU=.3,ECOEFF(1)=13.79ELO,NTCON=0,
      INDBD=1,BRAD=-3.18,BDEP=.279,BWID=2.62,NBSECT=8,
      DEPTH=1.852,PITCH=3.63,EDGE=.36,FLAT=.711,PHICOR=12.63,
      PANWID=51.6,XLGTH=45.7,NCOR=14,TS=.0216,TC=.0140,
      $CREEP INDGEO=2,  HARDOP=1.,TMAX=20.,NSTAT=8,SEC=15.,NSECT=15,
      TPSC EXAMPLE PROBLEM 2  TIME HARD

```

**B-72**

## EXAMPLE PROBLEM 2 INPUT



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[illegible]



## PHASE III SUMMARY REPORT

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### EXAMPLE PROBLEM 2 OUTPUT

[illegible]

# PHASE III SUMMARY REPORT

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## EXAMPLE PROBLEM 2 OUTPUT

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 2.856

TRAJECTORY TIME STEP 1			
J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	7.16047E-01
2	6.03382E-02	8.90998E-02	6.66271E-01
3	6.03382E-02	2.39300E-01	5.75206E-01
4	6.03382E-02	3.89499E-01	4.84141E-01
5	6.03382E-02	5.39699E-01	3.93077E-01
6	6.03382E-02	6.89899E-01	3.02012E-01
7	6.03382E-02	8.40099E-01	2.10948E-01
8	6.03382E-02	9.90298E-01	1.19883E-01
9	6.03382E-02	1.14050E+00	2.88184E-02
10	6.03382E-02	1.29070E+00	-6.22246E-02
11	6.03382E-02	1.44090E+00	-1.53311E-01
12	6.03382E-02	1.59110E+00	-2.44375E-01
13	6.03382E-02	1.74130E+00	-3.35440E-01
14	6.03382E-02	1.89150E+00	-4.26505E-01
15	6.03382E-02	1.82340E+00	-3.85216E-01
16	6.03382E-02	1.84120E+00	-3.96008E-01
17	6.03382E-02	1.78820E+00	-3.63879E-01
18	6.03382E-02	1.69820E+00	-3.09346E-01
19	6.03382E-02	1.63020E+00	-2.68099E-01
20	6.03382E-02	1.58460E+00	-2.40435E-01
21	1.63320E-01	1.56170E+00	-2.26552E-01

TRAJECTORY TIME STEP 2			
J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	7.16047E-01
2	6.03382E-02	8.90998E-02	6.66271E-01
3	6.03382E-02	2.39300E-01	5.75206E-01
4	6.03382E-02	3.89499E-01	4.84141E-01
5	6.03382E-02	5.39699E-01	3.93077E-01
6	6.03382E-02	6.89899E-01	3.02012E-01
7	6.03382E-02	8.40099E-01	2.10948E-01
8	6.03382E-02	9.90298E-01	1.19883E-01
9	6.03382E-02	1.14050E+00	2.88184E-02
10	6.03382E-02	1.29070E+00	-6.22246E-02
11	6.03382E-02	1.44090E+00	-1.53311E-01
12	6.03382E-02	1.59110E+00	-2.44375E-01
13	6.03382E-02	1.74130E+00	-3.35440E-01
14	6.03382E-02	1.89150E+00	-4.26505E-01
15	6.03382E-02	1.82340E+00	-3.85216E-01
16	6.03382E-02	1.84120E+00	-3.96008E-01
17	6.03382E-02	1.78820E+00	-3.63879E-01
18	6.03382E-02	1.69820E+00	-3.09346E-01
19	6.03382E-02	1.63020E+00	-2.68099E-01
20	6.03382E-02	1.58460E+00	-2.40435E-01
21	1.63320E-01	1.56170E+00	-2.26552E-01

TRAJECTORY TIME STEP 3			
J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.56340E+00
2	6.03382E-02	8.90998E-02	1.45472E+00
3	6.03382E-02	2.39300E-01	1.25589E+00
4	6.03382E-02	3.89499E-01	1.05706E+00
5	6.03382E-02	5.39699E-01	8.58233E-01
6	6.03382E-02	6.89899E-01	6.59408E-01
7	6.03382E-02	8.40099E-01	4.60557E-01
8	6.03382E-02	9.90298E-01	2.61750E-01
9	6.03382E-02	1.14050E+00	6.29214E-02
10	6.03382E-02	1.29070E+00	-5.95900E-02
11	6.03382E-02	1.44090E+00	-1.37735E-01
12	6.03382E-02	1.59110E+00	-3.35664E-01
13	6.03382E-02	1.74130E+00	-5.33392E-01
14	6.03382E-02	1.89150E+00	-7.31120E-01
15	6.03382E-02	1.82340E+00	-8.41073E-01
16	6.03382E-02	1.84120E+00	-8.64635E-01
17	6.03382E-02	1.78820E+00	-7.94485E-01
18	6.03382E-02	1.69820E+00	-6.75419E-01
19	6.03382E-02	1.63020E+00	-5.85361E-01
20	6.03382E-02	1.58460E+00	-5.24959E-01
21	1.63320E-01	1.56170E+00	-4.94649E-01

TRAJECTORY TIME STEP 4			
J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	2.34463E+00
2	6.03382E-02	8.90998E-02	2.18164E+00
3	6.03382E-02	2.39300E-01	1.88346E+00
4	6.03382E-02	3.89499E-01	1.58528E+00
5	6.03382E-02	5.39699E-01	1.28709E+00
6	6.03382E-02	6.89899E-01	9.88911E-01
7	6.03382E-02	8.40099E-01	6.80726E-01
8	6.03382E-02	9.90298E-01	3.92546E-01
9	6.03382E-02	1.14050E+00	9.43633E-02
10	6.03382E-02	1.29070E+00	-2.03818E-02
11	6.03382E-02	1.44090E+00	-5.02008E-02
12	6.03382E-02	1.59110E+00	-8.00188E-02
13	6.03382E-02	1.74130E+00	-1.09833E-01
14	6.03382E-02	1.89150E+00	-1.26135E-01
15	6.03382E-02	1.82340E+00	-1.29669E-01
16	6.03382E-02	1.84120E+00	-1.19144E-01
17	6.03382E-02	1.78820E+00	-1.01293E-01
18	6.03382E-02	1.69820E+00	-8.77865E-02
19	6.03382E-02	1.63020E+00	-7.87280E-02
20	6.03382E-02	1.58460E+00	-7.41824E-02
21	1.63320E-01	1.56170E+00	-7.41824E-02



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EXAMPLE PROBLEM 2 OUTPUT

TRAJECTORY TIME STEP 5

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.56340E+00
2	6.03382E-02	8.90998E-02	1.45472E+00
3	6.03382E-02	2.39300E-01	1.25589E+00
4	6.03382E-02	3.89499E-01	1.05706E+00
5	6.03382E-02	5.39699E-01	8.58235E-01
6	6.03382E-02	6.89899E-01	6.59406E-01
7	6.03382E-02	8.40098E-01	4.60578E-01
8	6.03382E-02	9.90298E-01	2.61750E-01
9	6.03382E-02	1.14050E+00	6.29214E-02
10	6.03382E-02	1.29070E+00	-1.35907E-01
11	6.03382E-02	1.44090E+00	-3.34735E-01
12	6.03382E-02	1.59110E+00	-5.33564E-01
13	6.03382E-02	1.74130E+00	-7.32392E-01
14	6.03382E-02	1.89150E+00	-9.31220E-01
15	6.03382E-02	1.84120E+00	-8.41073E-01
16	6.03382E-02	1.84120E+00	-8.64635E-01
17	6.03382E-02	1.84120E+00	-9.44855E-01
18	6.03382E-02	1.84120E+00	-1.07541E-01
19	6.03382E-02	1.84120E+00	-1.24959E-01
20	6.03382E-02	1.84120E+00	-1.49649E-01

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 5.712

TRAJECTORY TIME STEP 1

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	2.00955E+00
2	6.03382E-02	8.90998E-02	1.86986E+00
3	6.03382E-02	2.39300E-01	1.61429E+00
4	6.03382E-02	3.89499E-01	1.35872E+00
5	6.03382E-02	5.39699E-01	1.10315E+00
6	6.03382E-02	6.89899E-01	8.47583E-01
7	6.03382E-02	8.40098E-01	5.92014E-01
8	6.03382E-02	9.90298E-01	3.36446E-01
9	6.03382E-02	1.14050E+00	8.08774E-02
10	6.03382E-02	1.29070E+00	-1.74691E-01
11	6.03382E-02	1.44090E+00	-4.30259E-01
12	6.03382E-02	1.59110E+00	-6.85828E-01
13	6.03382E-02	1.74130E+00	-9.41396E-01
14	6.03382E-02	1.89150E+00	-1.08109E+00
15	6.03382E-02	1.84120E+00	-1.11138E+00
16	6.03382E-02	1.84120E+00	-1.02121E+00
17	6.03382E-02	1.84120E+00	-8.68165E-01
18	6.03382E-02	1.84120E+00	-7.52407E-01
19	6.03382E-02	1.84120E+00	-6.74768E-01
20	6.03382E-02	1.84120E+00	-6.35808E-01

TRAJECTORY TIME STEP 2

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	2.00955E+00
2	6.03382E-02	8.90998E-02	1.86986E+00
3	6.03382E-02	2.39300E-01	1.61429E+00
4	6.03382E-02	3.89499E-01	1.35872E+00
5	6.03382E-02	5.39699E-01	1.10315E+00
6	6.03382E-02	6.89899E-01	8.47583E-01
7	6.03382E-02	8.40098E-01	5.92014E-01
8	6.03382E-02	9.90298E-01	3.36446E-01
9	6.03382E-02	1.14050E+00	8.08774E-02
10	6.03382E-02	1.29070E+00	-1.74691E-01
11	6.03382E-02	1.44090E+00	-4.30259E-01
12	6.03382E-02	1.59110E+00	-6.85828E-01
13	6.03382E-02	1.74130E+00	-9.41396E-01
14	6.03382E-02	1.89150E+00	-1.08109E+00
15	6.03382E-02	1.84120E+00	-1.11138E+00
16	6.03382E-02	1.84120E+00	-1.02121E+00
17	6.03382E-02	1.84120E+00	-8.68165E-01
18	6.03382E-02	1.84120E+00	-7.52407E-01
19	6.03382E-02	1.84120E+00	-6.74768E-01
20	6.03382E-02	1.84120E+00	-6.35808E-01

TRAJECTORY TIME STEP 3

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	4.34761E+00
2	6.03382E-02	8.90998E-02	4.08260E+00
3	6.03382E-02	2.39300E-01	3.52460E+00
4	6.03382E-02	3.89499E-01	2.96660E+00
5	6.03382E-02	5.39699E-01	2.40859E+00
6	6.03382E-02	6.89899E-01	1.85059E+00
7	6.03382E-02	8.40098E-01	1.29259E+00
8	6.03382E-02	9.90298E-01	7.34588E-01
9	6.03382E-02	1.14050E+00	1.76586E-01
10	6.03382E-02	1.29070E+00	-3.81441E-01
11	6.03382E-02	1.44090E+00	-9.34119E-01
12	6.03382E-02	1.59110E+00	-1.49742E+00
13	6.03382E-02	1.74130E+00	-2.05542E+00
14	6.03382E-02	1.89150E+00	-2.61350E+00
15	6.03382E-02	1.84120E+00	-3.17158E+00
16	6.03382E-02	1.84120E+00	-3.72966E+00
17	6.03382E-02	1.84120E+00	-4.28774E+00
18	6.03382E-02	1.84120E+00	-4.84582E+00
19	6.03382E-02	1.84120E+00	-5.40390E+00
20	6.03382E-02	1.84120E+00	-5.96198E+00

# PHASE III SUMMARY REPORT

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## EXAMPLE PROBLEM 2 OUTPUT

TRAJECTORY TIME STEP 4			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	6.58000E+00
2	6.03382E-02	8.90998E-02	5.12267E+00
3	6.03382E-02	2.39300E-01	5.28583E+00
4	6.03382E-02	3.89499E-01	5.44900E+00
5	6.03382E-02	5.39699E-01	5.61216E+00
6	6.03382E-02	6.89899E-01	5.77533E+00
7	6.03382E-02	8.40098E-01	5.93849E+00
8	6.03382E-02	9.90298E-01	6.10166E+00
9	6.03382E-02	1.14050E+00	6.26482E+00
10	6.03382E-02	1.29070E+00	6.42799E+00
11	6.03382E-02	1.44090E+00	6.59116E+00
12	6.03382E-02	1.59110E+00	6.75433E+00
13	6.03382E-02	1.74130E+00	6.91749E+00
14	6.03382E-02	1.89150E+00	7.08066E+00
15	6.03382E-02	1.84120E+00	7.24382E+00
16	6.03382E-02	1.79090E+00	7.40699E+00
17	6.03382E-02	1.74060E+00	7.57016E+00
18	6.03382E-02	1.69030E+00	7.73333E+00
19	6.03382E-02	1.64000E+00	7.89649E+00
20	6.03382E-02	1.58970E+00	8.05966E+00

TRAJECTORY TIME STEP 5			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	4.38761E+00
2	6.03382E-02	8.90998E-02	4.08260E+00
3	6.03382E-02	2.39300E-01	4.52460E+00
4	6.03382E-02	3.89499E-01	4.96660E+00
5	6.03382E-02	5.39699E-01	5.40859E+00
6	6.03382E-02	6.89899E-01	5.85059E+00
7	6.03382E-02	8.40098E-01	6.29259E+00
8	6.03382E-02	9.90298E-01	6.73459E+00
9	6.03382E-02	1.14050E+00	7.17659E+00
10	6.03382E-02	1.29070E+00	7.61859E+00
11	6.03382E-02	1.44090E+00	8.06059E+00
12	6.03382E-02	1.59110E+00	8.50259E+00
13	6.03382E-02	1.74130E+00	8.94459E+00
14	6.03382E-02	1.89150E+00	9.38659E+00
15	6.03382E-02	1.84120E+00	9.82859E+00
16	6.03382E-02	1.79090E+00	10.27059E+00
17	6.03382E-02	1.74060E+00	10.71259E+00
18	6.03382E-02	1.69030E+00	11.15459E+00
19	6.03382E-02	1.64000E+00	11.59659E+00
20	6.03382E-02	1.58970E+00	12.03859E+00

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 8.569

TRAJECTORY TIME STEP 1			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	3.11827E+00
2	6.03382E-02	8.90998E-02	3.00150E+00
3	6.03382E-02	2.39300E-01	3.50433E+00
4	6.03382E-02	3.89499E-01	3.90833E+00
5	6.03382E-02	5.39699E-01	4.31233E+00
6	6.03382E-02	6.89899E-01	4.71633E+00
7	6.03382E-02	8.40098E-01	5.12033E+00
8	6.03382E-02	9.90298E-01	5.52433E+00
9	6.03382E-02	1.14050E+00	5.92833E+00
10	6.03382E-02	1.29070E+00	6.33233E+00
11	6.03382E-02	1.44090E+00	6.73633E+00
12	6.03382E-02	1.59110E+00	7.14033E+00
13	6.03382E-02	1.74130E+00	7.54433E+00
14	6.03382E-02	1.89150E+00	7.94833E+00
15	6.03382E-02	1.84120E+00	8.35233E+00
16	6.03382E-02	1.79090E+00	8.75633E+00
17	6.03382E-02	1.74060E+00	9.16033E+00
18	6.03382E-02	1.69030E+00	9.56433E+00
19	6.03382E-02	1.64000E+00	9.96833E+00
20	6.03382E-02	1.58970E+00	10.37233E+00

TRAJECTORY TIME STEP 2			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	3.11827E+00
2	6.03382E-02	8.90998E-02	3.00150E+00
3	6.03382E-02	2.39300E-01	3.50433E+00
4	6.03382E-02	3.89499E-01	3.90833E+00
5	6.03382E-02	5.39699E-01	4.31233E+00
6	6.03382E-02	6.89899E-01	4.71633E+00
7	6.03382E-02	8.40098E-01	5.12033E+00
8	6.03382E-02	9.90298E-01	5.52433E+00
9	6.03382E-02	1.14050E+00	5.92833E+00
10	6.03382E-02	1.29070E+00	6.33233E+00
11	6.03382E-02	1.44090E+00	6.73633E+00
12	6.03382E-02	1.59110E+00	7.14033E+00
13	6.03382E-02	1.74130E+00	7.54433E+00
14	6.03382E-02	1.89150E+00	7.94833E+00
15	6.03382E-02	1.84120E+00	8.35233E+00
16	6.03382E-02	1.79090E+00	8.75633E+00
17	6.03382E-02	1.74060E+00	9.16033E+00
18	6.03382E-02	1.69030E+00	9.56433E+00
19	6.03382E-02	1.64000E+00	9.96833E+00
20	6.03382E-02	1.58970E+00	10.37233E+00

EXAMPLE PROBLEM 2 OUTPUT  
TRAJECTORY TIME STEP 3

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	6.80836E+00
2	6.03382E-02	8.90999E-02	3.35077E+00
3	6.03382E-02	2.39300E-01	5.64220E+00
4	6.03382E-02	3.89499E-01	4.60334E+00
5	6.03382E-02	5.39699E-01	3.73747E+00
6	6.03382E-02	6.89899E-01	2.87151E+00
7	6.03382E-02	8.40099E-01	2.08574E+00
8	6.03382E-02	9.90299E-01	1.39883E+00
9	6.03382E-02	1.14050E+00	7.40133E-01
10	6.03382E-02	1.29070E+00	9.18533E-01
11	6.03382E-02	1.44090E+00	1.45772E+00
12	6.03382E-02	1.59110E+00	2.32358E+00
13	6.03382E-02	1.74130E+00	3.18945E+00
14	1.49435E-01	1.82340E+00	3.66274E+00
15	3.22271E-01	1.84120E+00	3.76535E+00
16	1.63320E-01	1.78820E+00	3.45985E+00
17	1.63320E-01	1.69826E+00	2.94134E+00
18	1.63320E-01	1.63023E+00	2.54115E+00
19	1.63320E-01	1.58460E+00	2.28611E+00
20	1.63320E-01	1.56170E+00	2.15412E+00

TRAJECTORY TIME STEP 4

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.02105E+01
2	6.03382E-02	8.90999E-02	9.50086E+00
3	6.03382E-02	2.39300E-01	8.20216E+00
4	6.03382E-02	3.89499E-01	6.90335E+00
5	6.03382E-02	5.39699E-01	5.60508E+00
6	6.03382E-02	6.89899E-01	4.30655E+00
7	6.03382E-02	8.40099E-01	3.00801E+00
8	6.03382E-02	9.90299E-01	1.70947E+00
9	6.03382E-02	1.14050E+00	4.16933E-01
10	6.03382E-02	1.29070E+00	3.81614E-01
11	6.03382E-02	1.44090E+00	4.48467E+00
12	6.03382E-02	1.59110E+00	4.78321E+00
13	6.03382E-02	1.74130E+00	5.43100E+00
14	1.49435E-01	1.82340E+00	5.64682E+00
15	3.22271E-01	1.84120E+00	5.18874E+00
16	1.63320E-01	1.78820E+00	4.41113E+00
17	1.63320E-01	1.69826E+00	3.82296E+00
18	1.63320E-01	1.63023E+00	3.42848E+00
19	1.63320E-01	1.58460E+00	3.23053E+00
20	1.63320E-01	1.56170E+00	

TRAJECTORY TIME STEP 5

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	6.80836E+00
2	6.03382E-02	8.90999E-02	6.33507E+00
3	6.03382E-02	2.39300E-01	5.64220E+00
4	6.03382E-02	3.89499E-01	4.60334E+00
5	6.03382E-02	5.39699E-01	3.73747E+00
6	6.03382E-02	6.89899E-01	2.87151E+00
7	6.03382E-02	8.40099E-01	2.08574E+00
8	6.03382E-02	9.90299E-01	1.39883E+00
9	6.03382E-02	1.14050E+00	7.40133E-01
10	6.03382E-02	1.29070E+00	9.18533E-01
11	6.03382E-02	1.44090E+00	1.45772E+00
12	6.03382E-02	1.59110E+00	2.32358E+00
13	6.03382E-02	1.74130E+00	3.18945E+00
14	1.49435E-01	1.82340E+00	3.66274E+00
15	3.22271E-01	1.84120E+00	3.76535E+00
16	1.63320E-01	1.78820E+00	3.45985E+00
17	1.63320E-01	1.69826E+00	2.94134E+00
18	1.63320E-01	1.63023E+00	2.54115E+00
19	1.63320E-01	1.58460E+00	2.28611E+00
20	1.63320E-01	1.56170E+00	2.15412E+00

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X = 11.425

TRAJECTORY TIME STEP 1

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	4.04220E+00
2	6.03382E-02	8.90999E-02	3.76120E+00
3	6.03382E-02	2.39300E-01	3.24713E+00
4	6.03382E-02	3.89499E-01	2.73306E+00
5	6.03382E-02	5.39699E-01	2.21898E+00
6	6.03382E-02	6.89899E-01	1.70491E+00
7	6.03382E-02	8.40099E-01	1.19083E+00
8	6.03382E-02	9.90299E-01	6.76759E-01
9	6.03382E-02	1.14050E+00	1.62644E-01
10	6.03382E-02	1.29070E+00	3.51300E-01
11	6.03382E-02	1.44090E+00	8.65464E-01
12	6.03382E-02	1.59110E+00	1.37954E+00
13	6.03382E-02	1.74130E+00	1.89361E+00
14	1.49435E-01	1.82340E+00	2.17461E+00
15	3.22271E-01	1.84120E+00	2.23553E+00
16	1.63320E-01	1.78820E+00	2.05416E+00
17	1.63320E-01	1.69826E+00	1.74631E+00
18	1.63320E-01	1.63023E+00	1.51146E+00
19	1.63320E-01	1.58460E+00	1.35729E+00
20	1.63320E-01	1.56170E+00	1.27892E+00

EXAMPLE PROBLEM 2 OUTPUT

TRAJECTORY TIME STEP 2			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	4.04222E+00
2	6.03382E-02	8.40099E-02	3.76112E+00
3	6.03382E-02	2.39300E-01	3.24713E+00
4	6.03382E-02	3.89499E-01	2.73336E+00
5	6.03382E-02	5.39699E-01	2.21893E+00
6	6.03382E-02	6.89899E-01	1.70493E+00
7	6.03382E-02	8.40099E-01	1.19093E+00
8	6.03382E-02	9.90299E-01	6.76783E-01
9	6.03382E-02	1.14050E+00	1.62268E-01
10	6.03382E-02	1.29070E+00	-1.62268E-01
11	6.03382E-02	1.44090E+00	-1.62268E-01
12	6.03382E-02	1.59110E+00	-1.62268E-01
13	6.03382E-02	1.74130E+00	-1.62268E-01
14	1.49433E-01	1.82340E+00	-1.62268E-01
15	3.22271E-01	1.84120E+00	-1.62268E-01
16	1.63320E-01	1.78820E+00	-1.62268E-01
17	1.63320E-01	1.69820E+00	-1.62268E-01
18	1.63320E-01	1.63320E+00	-1.62268E-01
19	1.63320E-01	1.58460E+00	-1.62268E-01
20	1.63320E-01	1.56170E+00	-1.62268E-01

TRAJECTORY TIME STEP 3			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	8.82565E+00
2	6.03382E-02	8.40099E-02	8.21213E+00
3	6.03382E-02	2.39300E-01	7.08971E+00
4	6.03382E-02	3.89499E-01	5.96729E+00
5	6.03382E-02	5.39699E-01	4.84487E+00
6	6.03382E-02	6.89899E-01	3.72246E+00
7	6.03382E-02	8.40099E-01	2.60004E+00
8	6.03382E-02	9.90299E-01	1.47762E+00
9	6.03382E-02	1.14050E+00	3.55201E-01
10	6.03382E-02	1.29070E+00	-7.67217E-01
11	6.03382E-02	1.44090E+00	-1.88964E-01
12	6.03382E-02	1.59110E+00	-3.01205E-01
13	6.03382E-02	1.74130E+00	-4.13447E-01
14	1.49433E-01	1.82340E+00	-4.74799E-01
15	3.22271E-01	1.84120E+00	-4.88101E-01
16	1.63320E-01	1.78820E+00	-4.48500E-01
17	1.63320E-01	1.69820E+00	-3.81288E-01
18	1.63320E-01	1.63320E+00	-3.30446E-01
19	1.63320E-01	1.58460E+00	-2.96348E-01
20	1.63320E-01	1.56170E+00	-2.79237E-01

TRAJECTORY TIME STEP 4			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.32355E+01
2	6.03382E-02	8.40099E-02	1.23157E+01
3	6.03382E-02	2.39300E-01	1.16322E+01
4	6.03382E-02	3.89499E-01	1.04911E+01
5	6.03382E-02	5.39699E-01	9.26588E+00
6	6.03382E-02	6.89899E-01	8.08255E+00
7	6.03382E-02	8.40099E-01	6.89922E+00
8	6.03382E-02	9.90299E-01	5.72255E+00
9	6.03382E-02	1.14050E+00	4.53269E+00
10	6.03382E-02	1.29070E+00	3.32695E+00
11	6.03382E-02	1.44090E+00	2.15055E+00
12	6.03382E-02	1.59110E+00	1.03338E+00
13	6.03382E-02	1.74130E+00	-4.51717E+00
14	1.49433E-01	1.82340E+00	-6.20046E+00
15	3.22271E-01	1.84120E+00	-7.12059E+00
16	1.63320E-01	1.78820E+00	-7.32004E+00
17	1.63320E-01	1.69820E+00	-6.72614E+00
18	1.63320E-01	1.63320E+00	-5.71813E+00
19	1.63320E-01	1.58460E+00	-4.95569E+00
20	1.63320E-01	1.56170E+00	-4.44432E+00

TRAJECTORY TIME STEP 5			
J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	8.82565E+00
2	6.03382E-02	8.40099E-02	8.21213E+00
3	6.03382E-02	2.39300E-01	7.08971E+00
4	6.03382E-02	3.89499E-01	5.96729E+00
5	6.03382E-02	5.39699E-01	4.84487E+00
6	6.03382E-02	6.89899E-01	3.72246E+00
7	6.03382E-02	8.40099E-01	2.60004E+00
8	6.03382E-02	9.90299E-01	1.47762E+00
9	6.03382E-02	1.14050E+00	3.55201E-01
10	6.03382E-02	1.29070E+00	-7.67217E-01
11	6.03382E-02	1.44090E+00	-1.88964E-01
12	6.03382E-02	1.59110E+00	-3.01205E-01
13	6.03382E-02	1.74130E+00	-4.13447E-01
14	1.49433E-01	1.82340E+00	-4.74799E-01
15	3.22271E-01	1.84120E+00	-4.88101E-01
16	1.63320E-01	1.78820E+00	-4.48500E-01
17	1.63320E-01	1.69820E+00	-3.81288E-01
18	1.63320E-01	1.63320E+00	-3.30446E-01
19	1.63320E-01	1.58460E+00	-2.96348E-01
20	1.63320E-01	1.56170E+00	-2.79237E-01

EXAMPLE PROBLEM 2 OUTPUT

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 14.281

TRAJECTORY TIME STEP 1			
J	AREA(SQ.CM)	Y(CM)	STRESS(MPA)
1	4.12574E-01	6.99999E-03	4.78135E+00
2	6.03382E-02	8.90998E-02	4.44897E+00
3	6.03382E-02	2.39300E-01	3.84889E+00
4	6.03382E-02	3.89499E-01	3.23281E+00
5	6.03382E-02	5.39699E-01	2.62474E+00
6	6.03382E-02	6.89899E-01	2.01666E+00
7	6.03382E-02	8.40098E-01	1.40859E+00
8	6.03382E-02	9.90298E-01	8.00509E-01
9	6.03382E-02	1.14050E+00	1.92432E-01
10	6.03382E-02	1.29070E+00	-1.15644E-01
11	6.03382E-02	1.44090E+00	-1.02372E-01
12	6.03382E-02	1.59110E+00	-1.63180E-01
13	6.03382E-02	1.74130E+00	-2.23877E-01
14	3.22277E-01	1.82340E+00	-5.72255E-01
15	1.63320E-01	1.84120E+00	-6.64431E-01
16	1.63320E-01	1.78820E+00	-4.29777E-01
17	1.63320E-01	1.69826E+00	-3.05511E-01
18	1.63320E-01	1.63023E+00	-1.79021E-01
19	1.63320E-01	1.58460E+00	-1.60954E-01
20	1.63320E-01	1.56170E+00	-1.51278E-01

TRAJECTORY TIME STEP 2			
J	AREA(SQ.CM)	Y(CM)	STRESS(MPA)
1	4.12574E-01	6.99999E-03	4.78135E+00
2	6.03382E-02	8.90998E-02	4.44897E+00
3	6.03382E-02	2.39300E-01	3.84889E+00
4	6.03382E-02	3.89499E-01	3.23281E+00
5	6.03382E-02	5.39699E-01	2.62474E+00
6	6.03382E-02	6.89899E-01	2.01666E+00
7	6.03382E-02	8.40098E-01	1.40859E+00
8	6.03382E-02	9.90298E-01	8.00509E-01
9	6.03382E-02	1.14050E+00	1.92432E-01
10	6.03382E-02	1.29070E+00	-1.15644E-01
11	6.03382E-02	1.44090E+00	-1.02372E-01
12	6.03382E-02	1.59110E+00	-1.63180E-01
13	6.03382E-02	1.74130E+00	-2.23877E-01
14	3.22277E-01	1.82340E+00	-5.72255E-01
15	1.63320E-01	1.84120E+00	-6.64431E-01
16	1.63320E-01	1.78820E+00	-4.29777E-01
17	1.63320E-01	1.69826E+00	-3.05511E-01
18	1.63320E-01	1.63023E+00	-1.79021E-01
19	1.63320E-01	1.58460E+00	-1.60954E-01
20	1.63320E-01	1.56170E+00	-1.51278E-01

TRAJECTORY TIME STEP 3			
J	AREA(SQ.CM)	Y(CM)	STRESS(MPA)
1	4.12574E-01	6.99999E-03	1.04395E+01
2	6.03382E-02	8.90998E-02	9.71378E+00
3	6.03382E-02	2.39300E-01	8.38611E+00
4	6.03382E-02	3.89499E-01	7.05845E+00
5	6.03382E-02	5.39699E-01	5.73079E+00
6	6.03382E-02	6.89899E-01	4.40313E+00
7	6.03382E-02	8.40098E-01	3.07547E+00
8	6.03382E-02	9.90298E-01	1.74781E+00
9	6.03382E-02	1.14050E+00	4.20155E-01
10	6.03382E-02	1.29070E+00	-9.07508E-01
11	6.03382E-02	1.44090E+00	-2.23517E+00
12	6.03382E-02	1.59110E+00	-3.56283E+00
13	6.03382E-02	1.74130E+00	-4.88900E+00
14	3.22277E-01	1.82340E+00	-5.61614E+00
15	1.63320E-01	1.84120E+00	-5.77353E+00
16	1.63320E-01	1.78820E+00	-5.30511E+00
17	1.63320E-01	1.69826E+00	-4.51006E+00
18	1.63320E-01	1.63023E+00	-3.90877E+00
19	1.63320E-01	1.58460E+00	-3.50537E+00
20	1.63320E-01	1.56170E+00	-3.30298E+00

TRAJECTORY TIME STEP 4			
J	AREA(SQ.CM)	Y(CM)	STRESS(MPA)
1	4.12574E-01	6.99999E-03	1.56561E+01
2	6.03382E-02	8.90998E-02	1.42597E+01
3	6.03382E-02	2.39300E-01	1.25956E+01
4	6.03382E-02	3.89499E-01	1.05855E+01
5	6.03382E-02	5.39699E-01	8.59445E+00
6	6.03382E-02	6.89899E-01	6.60337E+00
7	6.03382E-02	8.40098E-01	4.61228E+00
8	6.03382E-02	9.90298E-01	2.62119E+00
9	6.03382E-02	1.14050E+00	6.40102E-01
10	6.03382E-02	1.29070E+00	-1.35609E+00
11	6.03382E-02	1.44090E+00	-3.35208E+00
12	6.03382E-02	1.59110E+00	-5.34317E+00
13	6.03382E-02	1.74130E+00	-7.33466E+00
14	3.22277E-01	1.82340E+00	-8.42229E+00
15	1.63320E-01	1.84120E+00	-8.65586E+00
16	1.63320E-01	1.78820E+00	-7.95566E+00
17	1.63320E-01	1.69826E+00	-6.76373E+00
18	1.63320E-01	1.63023E+00	-5.86147E+00
19	1.63320E-01	1.58460E+00	-5.25700E+00
20	1.63320E-01	1.56170E+00	-4.95347E+00

# PHASE III SUMMARY REPORT

NAS-1-11774

## EXAMPLE PROBLEM 2 OUTPUT

### TRAJECTORY TIME STEP 5

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.04395E+01
2	6.03382E-02	8.90999E-02	9.71337E+00
3	6.03382E-02	2.39300E-01	9.58818E+00
4	6.03382E-02	3.89499E-01	9.58818E+00
5	6.03382E-02	5.39699E-01	9.58818E+00
6	6.03382E-02	6.89899E-01	9.58818E+00
7	6.03382E-02	8.40099E-01	9.58818E+00
8	6.03382E-02	9.90299E-01	9.58818E+00
9	6.03382E-02	1.14050E+00	9.58818E+00
10	6.03382E-02	1.29070E+00	9.58818E+00
11	6.03382E-02	1.44090E+00	9.58818E+00
12	6.03382E-02	1.59110E+00	9.58818E+00
13	6.03382E-02	1.74130E+00	9.58818E+00
14	6.03382E-02	1.89150E+00	9.58818E+00
15	6.03382E-02	1.78820E+00	9.58818E+00
16	6.03382E-02	1.69826E+00	9.58818E+00
17	6.03382E-02	1.63023E+00	9.58818E+00
18	6.03382E-02	1.58460E+00	9.58818E+00
19	6.03382E-02	1.56170E+00	9.58818E+00
20	6.03382E-02	1.56170E+00	9.58818E+00

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 17.137

### TRAJECTORY TIME STEP 1

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	5.33570E+00
2	6.03382E-02	8.90999E-02	4.96479E+00
3	6.03382E-02	2.39300E-01	4.28621E+00
4	6.03382E-02	3.89499E-01	3.60763E+00
5	6.03382E-02	5.39699E-01	2.92906E+00
6	6.03382E-02	6.89899E-01	2.25048E+00
7	6.03382E-02	8.40099E-01	1.57190E+00
8	6.03382E-02	9.90299E-01	8.93322E-01
9	6.03382E-02	1.14050E+00	2.14743E-01
10	6.03382E-02	1.29070E+00	-4.63835E-01
11	6.03382E-02	1.44090E+00	-1.14241E+00
12	6.03382E-02	1.59110E+00	-1.82099E+00
13	6.03382E-02	1.74130E+00	-2.49957E+00
14	6.03382E-02	1.89150E+00	-2.87044E+00
15	6.03382E-02	1.78820E+00	-2.95090E+00
16	6.03382E-02	1.69826E+00	-2.71149E+00
17	6.03382E-02	1.63023E+00	-2.30513E+00
18	6.03382E-02	1.58460E+00	-1.99777E+00
19	6.03382E-02	1.56170E+00	-1.79163E+00
20	6.03382E-02	1.56170E+00	-1.68818E+00

### TRAJECTORY TIME STEP 2

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	5.33570E+00
2	6.03382E-02	8.90999E-02	4.96479E+00
3	6.03382E-02	2.39300E-01	4.28621E+00
4	6.03382E-02	3.89499E-01	3.60763E+00
5	6.03382E-02	5.39699E-01	2.92906E+00
6	6.03382E-02	6.89899E-01	2.25048E+00
7	6.03382E-02	8.40099E-01	1.57190E+00
8	6.03382E-02	9.90299E-01	8.93322E-01
9	6.03382E-02	1.14050E+00	2.14743E-01
10	6.03382E-02	1.29070E+00	-4.63835E-01
11	6.03382E-02	1.44090E+00	-1.14241E+00
12	6.03382E-02	1.59110E+00	-1.82099E+00
13	6.03382E-02	1.74130E+00	-2.49957E+00
14	6.03382E-02	1.89150E+00	-2.87044E+00
15	6.03382E-02	1.78820E+00	-2.95090E+00
16	6.03382E-02	1.69826E+00	-2.71149E+00
17	6.03382E-02	1.63023E+00	-2.30513E+00
18	6.03382E-02	1.58460E+00	-1.99777E+00
19	6.03382E-02	1.56170E+00	-1.79163E+00
20	6.03382E-02	1.56170E+00	-1.68818E+00

### TRAJECTORY TIME STEP 3

J	AREA (SQ.CH)	Y (CH)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.16499E+01
2	6.03382E-02	8.90999E-02	1.08400E+01
3	6.03382E-02	2.39300E-01	9.55942E+00
4	6.03382E-02	3.89499E-01	7.87683E+00
5	6.03382E-02	5.39699E-01	6.39523E+00
6	6.03382E-02	6.89899E-01	4.91364E+00
7	6.03382E-02	8.40099E-01	3.43205E+00
8	6.03382E-02	9.90299E-01	1.95044E+00
9	6.03382E-02	1.14050E+00	4.62866E-01
10	6.03382E-02	1.29070E+00	-1.01273E+00
11	6.03382E-02	1.44090E+00	-4.94332E+00
12	6.03382E-02	1.59110E+00	-9.75991E+00
13	6.03382E-02	1.74130E+00	-1.45759E+00
14	6.03382E-02	1.89150E+00	-6.26733E+00
15	6.03382E-02	1.78820E+00	-6.44293E+00
16	6.03382E-02	1.69826E+00	-5.92019E+00
17	6.03382E-02	1.63023E+00	-5.03296E+00
18	6.03382E-02	1.58460E+00	-4.36188E+00
19	6.03382E-02	1.56170E+00	-3.91179E+00
20	6.03382E-02	1.56170E+00	-3.68593E+00



PHASE III  
SUMMARY REPORT

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EXAMPLE PROBLEM 2 OUTPUT  
TRAJECTORY TIME STEP 4

J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.74713E+01
2	6.03382E-02	8.40098E-02	1.62567E+01
3	6.03382E-02	2.39300E-01	1.40348E+01
4	6.03382E-02	3.89499E-01	1.18129E+01
5	6.03382E-02	5.39699E-01	9.59092E+00
6	6.03382E-02	6.89899E-01	7.36938E+00
7	6.03382E-02	8.40098E-01	5.14704E+00
8	6.03382E-02	9.90298E-01	2.92513E+00
9	6.03382E-02	1.14050E+00	7.03157E-01
10	6.03382E-02	1.29070E+00	-1.51877E+00
11	6.03382E-02	1.44090E+00	-3.74072E+00
12	6.03382E-02	1.59110E+00	-5.96266E+00
13	6.03382E-02	1.74130E+00	-8.18460E+00
14	1.44943E-01	1.82340E+00	-9.33931E+00
15	3.22271E-01	1.84120E+00	-9.65245E+00
16	1.63320E-01	1.78820E+00	-8.87850E+00
17	1.63320E-01	1.69826E+00	-7.54793E+00
18	1.63320E-01	1.63023E+00	-6.54151E+00
19	1.63320E-01	1.58460E+00	-5.86651E+00
20	1.63320E-01	1.56170E+00	-5.52779E+00

TRAJECTORY TIME STEP 5

J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.16499E+01
2	6.03382E-02	8.40098E-02	1.08400E+01
3	6.03382E-02	2.39300E-01	9.58442E+00
4	6.03382E-02	3.89499E-01	7.87633E+00
5	6.03382E-02	5.39699E-01	6.39523E+00
6	6.03382E-02	6.89899E-01	4.91364E+00
7	6.03382E-02	8.40098E-01	3.43205E+00
8	6.03382E-02	9.90298E-01	1.95046E+00
9	6.03382E-02	1.14050E+00	4.68836E-01
10	6.03382E-02	1.29070E+00	-1.01273E+00
11	6.03382E-02	1.44090E+00	-2.49432E+00
12	6.03382E-02	1.59110E+00	-3.97594E+00
13	6.03382E-02	1.74130E+00	-5.45756E+00
14	1.44943E-01	1.82340E+00	-6.26735E+00
15	3.22271E-01	1.84120E+00	-6.44293E+00
16	1.63320E-01	1.78820E+00	-5.92019E+00
17	1.63320E-01	1.69826E+00	-5.03296E+00
18	1.63320E-01	1.63023E+00	-4.36188E+00
19	1.63320E-01	1.58460E+00	-3.91179E+00
20	1.63320E-01	1.56170E+00	-3.68593E+00

CASE 1 ANALYSTS  
ELASTIC STRESSES AT X= 19.994

TRAJECTORY TIME STEP 1

J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	5.70528E+00
2	6.03382E-02	8.40098E-02	5.70867E+00
3	6.03382E-02	2.39300E-01	4.58309E+00
4	6.03382E-02	3.89499E-01	3.85751E+00
5	6.03382E-02	5.39699E-01	3.13193E+00
6	6.03382E-02	6.89899E-01	2.40635E+00
7	6.03382E-02	8.40098E-01	1.68078E+00
8	6.03382E-02	9.90298E-01	9.55197E-01
9	6.03382E-02	1.14050E+00	2.29617E-01
10	6.03382E-02	1.29070E+00	-4.95962E-01
11	6.03382E-02	1.44090E+00	-1.94712E+00
12	6.03382E-02	1.59110E+00	-2.67270E+00
13	6.03382E-02	1.74130E+00	-3.06930E+00
14	1.44943E-01	1.82340E+00	-3.15520E+00
15	3.22271E-01	1.84120E+00	-2.89929E+00
16	1.63320E-01	1.78820E+00	-2.46479E+00
17	1.63320E-01	1.69826E+00	-2.13614E+00
18	1.63320E-01	1.63023E+00	-1.91572E+00
19	1.63320E-01	1.58460E+00	-1.80511E+00
20	1.63320E-01	1.56170E+00	-1.80511E+00

TRAJECTORY TIME STEP 2

J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	5.70528E+00
2	6.03382E-02	8.40098E-02	5.30867E+00
3	6.03382E-02	2.39300E-01	4.58309E+00
4	6.03382E-02	3.89499E-01	3.85751E+00
5	6.03382E-02	5.39699E-01	3.13193E+00
6	6.03382E-02	6.89899E-01	2.40635E+00
7	6.03382E-02	8.40098E-01	1.68078E+00
8	6.03382E-02	9.90298E-01	9.55197E-01
9	6.03382E-02	1.14050E+00	2.29617E-01
10	6.03382E-02	1.29070E+00	-4.95962E-01
11	6.03382E-02	1.44090E+00	-1.94712E+00
12	6.03382E-02	1.59110E+00	-2.67270E+00
13	6.03382E-02	1.74130E+00	-3.06930E+00
14	1.44943E-01	1.82340E+00	-3.15520E+00
15	3.22271E-01	1.84120E+00	-2.89929E+00
16	1.63320E-01	1.78820E+00	-2.46479E+00
17	1.63320E-01	1.69826E+00	-2.13614E+00
18	1.63320E-01	1.63023E+00	-1.91572E+00
19	1.63320E-01	1.58460E+00	-1.80511E+00
20	1.63320E-01	1.56170E+00	-1.80511E+00



PHASE III  
SUMMARY REPORT

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EXAMPLE PROBLEM 2 OUTPUT  
TRAJECTORY TIME STEP 3

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.24568E+01
2	6.03382E-02	8.90998E-02	1.15908E+01
3	6.03382E-02	2.39300E-01	1.00066E+01
4	6.03382E-02	3.89499E-01	8.42241E+00
5	6.03382E-02	5.39699E-01	6.83819E+00
6	6.03382E-02	6.89899E-01	5.25398E+00
7	6.03382E-02	8.40098E-01	3.66977E+00
8	6.03382E-02	9.90298E-01	2.08555E+00
9	6.03382E-02	1.14050E+00	5.01341E-01
10	6.03382E-02	1.29070E+00	-1.08287E+00
11	6.03382E-02	1.44090E+00	-2.66708E+00
12	6.03382E-02	1.59110E+00	-5.83551E+00
13	6.03382E-02	1.74130E+00	-6.70145E+00
14	1.49435E-01	1.82340E+00	-6.88919E+00
15	3.22271E-01	1.84120E+00	-6.33025E+00
16	1.63320E-01	1.78820E+00	-5.38157E+00
17	1.63320E-01	1.69826E+00	-4.66401E+00
18	1.63320E-01	1.63023E+00	-4.18274E+00
19	1.63320E-01	1.58460E+00	-3.94124E+00
20	1.63320E-01	1.56170E+00	

TRAJECTORY TIME STEP 4

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.86814E+01
2	6.03382E-02	8.90998E-02	1.73327E+01
3	6.03382E-02	2.39300E-01	1.50069E+01
4	6.03382E-02	3.89499E-01	1.26311E+01
5	6.03382E-02	5.39699E-01	1.02552E+01
6	6.03382E-02	6.89899E-01	7.87938E+00
7	6.03382E-02	8.40098E-01	5.50354E+00
8	6.03382E-02	9.90298E-01	3.12770E+00
9	6.03382E-02	1.14050E+00	5.31861E-01
10	6.03382E-02	1.29070E+00	-1.66733E+00
11	6.03382E-02	1.44090E+00	-4.65988E+00
12	6.03382E-02	1.59110E+00	-8.33756E+00
13	6.03382E-02	1.74130E+00	-8.75104E+00
14	1.49435E-01	1.82340E+00	-8.08050E+00
15	3.22271E-01	1.84120E+00	-6.83347E+00
16	1.63320E-01	1.78820E+00	-5.40773E+00
17	1.63320E-01	1.69826E+00	-4.90460E+00
18	1.63320E-01	1.63023E+00	-6.27285E+00
19	1.63320E-01	1.58460E+00	-5.91066E+00
20	1.63320E-01	1.56170E+00	

TRAJECTORY TIME STEP 5

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.24568E+01
2	6.03382E-02	8.90998E-02	1.15908E+01
3	6.03382E-02	2.39300E-01	1.00066E+01
4	6.03382E-02	3.89499E-01	8.42241E+00
5	6.03382E-02	5.39699E-01	6.83819E+00
6	6.03382E-02	6.89899E-01	5.25398E+00
7	6.03382E-02	8.40098E-01	3.66977E+00
8	6.03382E-02	9.90298E-01	2.08555E+00
9	6.03382E-02	1.14050E+00	5.01341E-01
10	6.03382E-02	1.29070E+00	-1.08287E+00
11	6.03382E-02	1.44090E+00	-2.66708E+00
12	6.03382E-02	1.59110E+00	-5.83551E+00
13	6.03382E-02	1.74130E+00	-6.70145E+00
14	1.49435E-01	1.82340E+00	-6.88919E+00
15	3.22271E-01	1.84120E+00	-6.33025E+00
16	1.63320E-01	1.78820E+00	-5.38157E+00
17	1.63320E-01	1.69826E+00	-4.66401E+00
18	1.63320E-01	1.63023E+00	-4.18274E+00
19	1.63320E-01	1.58460E+00	-3.94124E+00
20	1.63320E-01	1.56170E+00	

CASE 1 ANALYSIS  
ELASTIC STRESSES AT X= 22.850

TRAJECTORY TIME STEP 1

J	AREA (SQ.CM)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	5.89006E+00
2	6.03382E-02	8.90998E-02	5.48061E+00
3	6.03382E-02	2.39300E-01	4.73153E+00
4	6.03382E-02	3.89499E-01	3.98245E+00
5	6.03382E-02	5.39699E-01	3.23337E+00
6	6.03382E-02	6.89899E-01	2.48429E+00
7	6.03382E-02	8.40098E-01	1.73521E+00
8	6.03382E-02	9.90298E-01	9.88613E-01
9	6.03382E-02	1.14050E+00	3.37054E-01
10	6.03382E-02	1.29070E+00	-1.20225E-01
11	6.03382E-02	1.44090E+00	-1.26111E+00
12	6.03382E-02	1.59110E+00	-1.01018E+00
13	6.03382E-02	1.74130E+00	-7.59226E+00
14	1.49435E-01	1.82340E+00	-1.66771E+00
15	3.22271E-01	1.84120E+00	-1.57749E+00
16	1.63320E-01	1.78820E+00	-1.39320E+00
17	1.63320E-01	1.69826E+00	-1.34462E+00
18	1.63320E-01	1.63023E+00	-1.20533E+00
19	1.63320E-01	1.58460E+00	-1.09777E+00
20	1.63320E-01	1.56170E+00	-1.86358E+00



EXAMPLE PROBLEM 2 OUTPUT

TRAJECTORY TIME STEP 2			
J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	5.89000E+00
2	6.03382E-02	8.40099E-02	5.48000E+00
3	6.03382E-02	2.39300E-01	5.73150E+00
4	6.03382E-02	3.89499E-01	5.98240E+00
5	6.03382E-02	5.39699E-01	6.23330E+00
6	6.03382E-02	6.89899E-01	6.48420E+00
7	6.03382E-02	8.40099E-01	6.73510E+00
8	6.03382E-02	9.90299E-01	6.98600E+00
9	6.03382E-02	1.14050E+00	7.23690E+00
10	6.03382E-02	1.29070E+00	7.48780E+00
11	6.03382E-02	1.44090E+00	7.73870E+00
12	6.03382E-02	1.59110E+00	7.98960E+00
13	6.03382E-02	1.74130E+00	8.24050E+00
14	1.49435E-01	1.89150E+00	8.49140E+00
15	3.22271E-01	1.84120E+00	8.74230E+00
16	1.63320E-01	1.78820E+00	8.99320E+00
17	1.63320E-01	1.69826E+00	9.24410E+00
18	1.63320E-01	1.63023E+00	9.49500E+00
19	1.63320E-01	1.58460E+00	9.74590E+00
20	1.63320E-01	1.56170E+00	9.99680E+00

TRAJECTORY TIME STEP 3			
J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.28602E+01
2	6.03382E-02	8.90099E-02	1.19662E+01
3	6.03382E-02	2.39300E-01	1.03107E+01
4	6.03382E-02	3.89499E-01	8.69920E+00
5	6.03382E-02	5.39699E-01	7.05967E+00
6	6.03382E-02	6.89899E-01	5.42415E+00
7	6.03382E-02	8.40099E-01	3.78863E+00
8	6.03382E-02	9.90299E-01	2.15310E+00
9	6.03382E-02	1.14050E+00	5.17579E-01
10	6.03382E-02	1.29070E+00	1.11794E+00
11	6.03382E-02	1.44090E+00	2.75347E+00
12	6.03382E-02	1.59110E+00	3.88999E+00
13	6.03382E-02	1.74130E+00	5.02452E+00
14	1.49435E-01	1.89150E+00	6.15905E+00
15	3.22271E-01	1.84120E+00	7.29358E+00
16	1.63320E-01	1.78820E+00	8.42811E+00
17	1.63320E-01	1.69826E+00	9.56264E+00
18	1.63320E-01	1.63023E+00	1.06971E+01
19	1.63320E-01	1.58460E+00	1.18318E+01
20	1.63320E-01	1.56170E+00	1.29665E+01

TRAJECTORY TIME STEP 4			
J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.92865E+01
2	6.03382E-02	8.40099E-02	1.79059E+01
3	6.03382E-02	2.39300E-01	1.54030E+01
4	6.03382E-02	3.89499E-01	1.30402E+01
5	6.03382E-02	5.39699E-01	1.05374E+01
6	6.03382E-02	6.89899E-01	8.13500E+00
7	6.03382E-02	8.40099E-01	6.81800E+00
8	6.03382E-02	9.90299E-01	5.49200E+00
9	6.03382E-02	1.14050E+00	4.16600E+00
10	6.03382E-02	1.29070E+00	2.84000E+00
11	6.03382E-02	1.44090E+00	1.51400E+00
12	6.03382E-02	1.59110E+00	2.18800E+00
13	6.03382E-02	1.74130E+00	3.29200E+00
14	1.49435E-01	1.89150E+00	4.39600E+00
15	3.22271E-01	1.84120E+00	5.49900E+00
16	1.63320E-01	1.78820E+00	6.60200E+00
17	1.63320E-01	1.69826E+00	7.70500E+00
18	1.63320E-01	1.63023E+00	8.80800E+00
19	1.63320E-01	1.58460E+00	9.91100E+00
20	1.63320E-01	1.56170E+00	1.10140E+01

TRAJECTORY TIME STEP 5			
J	AREA (SQ.CH)	Y (CM)	STRESS (MPA)
1	4.12574E-01	6.99999E-03	1.28602E+01
2	6.03382E-02	8.90099E-02	1.19662E+01
3	6.03382E-02	2.39300E-01	1.03107E+01
4	6.03382E-02	3.89499E-01	8.69920E+00
5	6.03382E-02	5.39699E-01	7.05967E+00
6	6.03382E-02	6.89899E-01	5.42415E+00
7	6.03382E-02	8.40099E-01	3.78863E+00
8	6.03382E-02	9.90299E-01	2.15310E+00
9	6.03382E-02	1.14050E+00	5.17579E-01
10	6.03382E-02	1.29070E+00	1.11794E+00
11	6.03382E-02	1.44090E+00	2.75347E+00
12	6.03382E-02	1.59110E+00	3.88999E+00
13	6.03382E-02	1.74130E+00	5.02452E+00
14	1.49435E-01	1.89150E+00	6.15905E+00
15	3.22271E-01	1.84120E+00	7.29358E+00
16	1.63320E-01	1.78820E+00	8.42811E+00
17	1.63320E-01	1.69826E+00	9.56264E+00
18	1.63320E-01	1.63023E+00	1.06971E+01
19	1.63320E-01	1.58460E+00	1.18318E+01
20	1.63320E-01	1.56170E+00	1.29665E+01



EXAMPLE PROBLEM 2 OUTPUT  
CREEP PREDICTION COMPUTER PROGRAM  
TPSC EXAMPLE PROBLEM 2 TIME HARD

SINGLE FACED CORRUGATION TPS

SKIN GAGE = .0216 CM  
CORRUGATION GAGE = .0140 CM  
NUMBER OF CORRUGATIONS = 14  
PITCH LENGTH = 3.630 CM  
FLAT LENGTH = .711 CM  
PANEL EDGE LENGTH = .360 CM  
CORRUGATION ANGLE = 12.830 DEGREES  
PANEL DEPTH = 1.852 CM  
CALCULATED MOMENT OF INERTIA = 1.2036087 CM\*\*4  
ELASTIC NEUTRAL AXIS = 1.188 CM

PANEL LENGTH = 45.70 CM  
PANEL WIDTH = 51.60 CM

NEGATIVE BEAD  
RADIUS = -3.180 CM  
WIDTH = 2.620 CM

APPLIED LOADS  
UNIFORM PRESSURE (PLATE OPTION)

BENDING MOMENT DISTRIBUTION

TIME	2.86	5.71	8.57	11.42	14.28	17.14	19.99	22.85
1.33	7.44	20.88	32.41	42.01	49.69	55.45	59.29	61.21
4.67	7.44	20.88	32.41	42.01	49.69	55.45	59.29	61.21
7.50	16.25	45.60	70.75	91.72	108.49	121.07	129.45	133.65
8.50	24.37	68.38	106.11	137.55	162.70	181.57	194.14	200.43
11.33	16.25	45.60	70.75	91.72	108.49	121.07	129.45	133.65

TRAJECTORY DATA

TIME (MINUTES)  
PRESSURE (PA)  
TEMPERATURE (DEG K)

TIME START	TIME END	PRESSURE	MIDSPAN SKIN TEMPERATURE
0.00	1.33	758.000	1122.0
1.33	4.67	758.000	1231.0
4.67	7.50	1655.000	1231.0
7.50	8.50	2482.000	1225.0
8.50	11.33	1655.000	1014.0

CYCLIC CREEP EQUATION DEFINITION

LN(STRAIN) = -2.89414E+00  
-1.74300E-02 \*(TIME)  
-8.66518E-00 \*(1/TMP)  
-4.88220E-01 \*LN(TIME)  
1.31015E+00 \*LN(STRESS)  
1.91310E-01 \*LN(TEMP)  
2.10000E-04 \*TIME\*STRESS\*TEMP

WHERE  
TIME = HOURS  
TEMPERATURE = DEG K/1000.  
STRESS = MPa



PREDICTION OF CREEP IN  
METALLIC TPS PANELS

PHASE III  
SUMMARY REPORT

NAS-1-11774

EXAMPLE PROBLEM 2 OUTPUT  
ELASTIC DEFLECTION SUMMARY

TIME (HR)	BEAM STATION (CM)	2.86	5.71	8.57	11.42	14.28	17.14	19.99	22.85
1.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
4.67	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
7.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
8.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
11.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030

FIRST CYCLE CREEP DEFLECTION SUMMARY

TIME	BEAM STATION (CM)	2.86	5.71	8.57	11.42	14.28	17.14	19.99	22.85
1.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
4.67	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
7.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
8.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
11.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030

CREEP DEFLECTION SUMMARY

CYCLE	BEAM STATION (CM)	2.86	5.71	8.57	11.42	14.28	17.14	19.99	22.85
1	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
2	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
3	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
4	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030

CREEP PREDICTION COMPUTER PROGRAM

CREEP STRAINS (PERCENT)

CYCLE 1

HEIGHT	BEAM STATION (CM)	2.86	5.71	8.57	11.42	14.28	17.14	19.99	22.85
1.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
4.67	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
7.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
8.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
11.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030

CREEP PREDICTION COMPUTER PROGRAM

RESIDUAL STRESSES (MPA)

CYCLE 1

HEIGHT	BEAM STATION (CM)	2.86	5.71	8.57	11.42	14.28	17.14	19.99	22.85
1.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
4.67	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
7.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
8.50	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030
11.33	.000006	.000011	.000017	.000021	.000025	.000028	.000030	.000030	.000030



NAS-1-11774



## PHASE III SUMMARY REPORT

NAS-1-11774

### EXAMPLE PROBLEM 2 OUTPUT

CREEP PREDICTION COMPUTER PROGRAM

RESIDUAL STRESSES (MPa)

CYCLE 3

[illegible]

CREEP PREDICTION COMPUTER PROGRAM

CREEP STRAINS (PERCENT)

CYCLE 4

HEIGHT	BEAM STATION (CM)									
0.0070	2.86	5.71	8.57	11.42	14.28	17.14	19.99	22.85		
0.0091	0.000188	0.00072	0.00128	0.00180	0.00222	0.00260	0.00299	0.00336	0.00374	0.00411
0.0093	0.000189	0.000724	0.001287	0.001809	0.002227	0.002606	0.002994	0.003372	0.003750	0.004128
0.2333	0.000146	0.000551	0.000901	0.001306	0.001674	0.002006	0.002301	0.002559	0.002781	0.002966
0.3835	0.000143	0.000550	0.000900	0.001305	0.001673	0.002005	0.002299	0.002557	0.002779	0.002964
0.5337	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
0.6839	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
0.8341	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
0.9843	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
1.1345	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
1.2847	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
1.4349	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
1.5851	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
1.7353	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
1.8855	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
2.0357	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
2.1859	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
2.3361	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
2.4863	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
2.6365	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
2.7867	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
2.9369	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
3.0871	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
3.2373	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
3.3875	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
3.5377	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
3.6879	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
3.8381	0.000123	0.000478	0.000770	0.001063	0.001356	0.001648	0.001930	0.002201	0.002462	0.002713
3.9883	0									

CREEP PREDICTION COMPUTER PROGRAM

RESIDUAL STRESSES (MPa)

CYCLE 4

[illegible]



APPENDIX C

TPSC

PROGRAM LISTING

C-1



```
PROGRAM TPSC(INPUT,OUTPUT,TAPES=INPUT,TAPE6=OUTPUT)
DIMENSION PRESS (10) , TEMP (10) , DXTIME (10) , XTP (20) , IPSC 10
1 XTEMP (24) , YTP (60) , YTEMP (24) , FORC (60) , IPSC 20
2 STRAT (60) , REFP (24) , C (4) , D (4) , IPSC 30
3 STRESS (60) , A (60) , Y (60) , BETA (10) , IPSC 40
4 XMOD (10) , ECODEFF (4) , THETAT (10) , XPRINT (10) , IPSC 50
5 X (20) , IPRINT (10) , KCYCLE (10) , YNA (10) , IPSC 60
6 Z (75) , PLDAD (10) , RESSIR (60) , ESTIRAN (60) , IPSC 70
7 STRES (21) , IPSC 80
DIMENSION E (10,60) , F (20,60) , IPSC 90
1 THETAE (20,10) , STRANE (10,60) , STRESE (10,60) , IPSC 100
2 DEFL (10,10) , DEPLE (10,10) , THETAP (20,10) , IPSC 110
3 DEFLIN (10,10) , THET (20,10) , RESSIN (10,60) , IPSC 120
4 PLADNM (10,10) , PRIMUM (10,10) , IPSC 130
DIMENSION RESP (10,60,10) , PSTRAN (10,60,10) , IPSC 140
COMMON PLADNM,PRIMUM,INDPLA , IPSC 150
NAMELIST /CREEP/ DEPTH , PHICOR , PITCH , FLAT , EDGE , IPSC 160
INCOR , TS , TC , XLGTH , F1 , NRIB , XIBFLG , IPSC 170
2ZPNE01 , ZPNE02 , NZEE , TZEE , ZEESF , ZEESF1 , ZEEFF , IPSC 180
3ZEEFF1 , BWID , BDEJ , BRAD , IPSC 190
4PRESS , TEMP , DXTIME , PLDAD , IPSC 200
5ALEN , XMOD , PANAD , C , D , E , NTIME , IPSC 210
6XTEMP , NEWCAS , YTEMP , EMOD , ETEMP , INDGEO , IPSC 220
7INDAD , INDLOD , INDSUP , INDPLA , INDIFL , INDIFD , KCYCLE , IPSC 230
8INDMID , INDELA , IPSC 240
9NCYCLE , NUMCYC , ITCON , NTCON , ECODEFF , Z , IPSC 250
AHARDOP , TMAX , INCYC , ITIME , IEQNTP , INTPT , IDIMEN , IPSC 260
HILDAD , IEONST , NSTAT , SEC , NSECT , NASECT , IPSC 270
CIRESID , RESSIN , ESTIFF , INDD2 , INDIR , DETWO , IPSC 280
C LOOP = 0 , IPSC 290
NEWCAS=1 , IPSC 300
10 DO 20 II=1,75 , IPSC 310
20 Z(II)=0. , IPSC 320
C C C C C , IPSC 330
C C C C C , IPSC 340
C C C C C , IPSC 350
C C C C C , IPSC 360
C C C C C , IPSC 370
C C C C C , IPSC 380
C C C C C , IPSC 390
C C C C C , IPSC 400
C C C C C , IPSC 410
C C C C C , IPSC 420
C C C C C , IPSC 430
C C C C C , IPSC 440
C C C C C , IPSC 450
C C C C C , IPSC 460
C C C C C , IPSC 470
C C C C C , IPSC 480
C C C C C , IPSC 490
C C C C C , IPSC 500
C C C C C , IPSC 510
C C C C C , IPSC 520
C C C C C , IPSC 530
C C C C C , IPSC 540
C C C C C , IPSC 550
C C C C C , IPSC 560
C C C C C , IPSC 570
C C C C C , IPSC 580
C C C C C , IPSC 590
C C C C C , IPSC 600
C C C C C , IPSC 610
C C C C C , IPSC 620
C C C C C , IPSC 630
C C C C C , IPSC 640
C C C C C , IPSC 650
C C C C C , IPSC 660
C C C C C , IPSC 670
C C C C C , IPSC 680
C C C C C , IPSC 690
C C C C C , IPSC 700
C C C C C , IPSC 710
C C C C C , IPSC 720
C C C C C , IPSC 730
C C C C C , IPSC 740
C C C C C , IPSC 750
C C C C C , IPSC 760
C C C C C , IPSC 770
C C C C C , IPSC 780
C C C C C , IPSC 790
C C C C C , IPSC 800
C C C C C , IPSC 810
C C C C C , IPSC 820
C C C C C , IPSC 830
C C C C C , IPSC 840
C C C C C , IPSC 850
C C C C C , IPSC 860
C C C C C , IPSC 870
C C C C C , IPSC 880
C C C C C , IPSC 890
C C C C C , IPSC 900
C C C C C , IPSC 910
C C C C C , IPSC 920
C C C C C , IPSC 930
C C C C C , IPSC 940
C C C C C , IPSC 950
C C C C C , IPSC 960
C C C C C , IPSC 970
C C C C C , IPSC 980
C C C C C , IPSC 990
C C C C C , IPSC 1000
```



C(1)=1.  
INDTEL=0  
INDLID=0  
INDSUP=0  
INDYC=1  
INDFLA=0  
HARDOP=1  
INDSTR=0  
IND4ID=0  
S=1.  
INDRD=0  
INDPLA=0  
DEPTH=0.  
PITCH=0.  
FLAT=0.  
EDGE=0.  
IS=0.  
TC=0.  
XLGTH=0.  
TR=0.  
RIBFLG=0.  
ZPNEID1=0.  
ZPNEID2=0.  
TZEE=0.  
ZEESE=0.  
ZEESE1=0.  
ZEEEF=0.  
ZEEEF1=0.  
HWID=0.  
RDEP=0.  
BRAD=0.  
ALEN=0.  
PANWID=0.  
INDID2=0  
INDPLA=0  
D(2)=0.  
D(3)=0.  
D(4)=0.

C  
C

DUMLGTH = 0.0  
NSFAT=6  
NSECT=10  
SEC=10.  
NSECT=6  
READ (5,1760) AM1,AM2,AM3,AM4,AM5  
READ (5,CREEP)  
WRITE(6,CREEP)  
NSECT=NSECT+5  
NSECT1=NSECT+1  
IF(LOAD.NE.0)GO TO 6)  
DO 50 LL=1,NTIME  
PLOAD(LL)=PLOAD(LL)/.45359  
PRESS(LL)=PRESS(LL)/6894.8  
50 CONTINUE  
60 CONTINUE  
IF(DIMEN.NE.0)GO TO 70  
DEPTH=DEPTH /2.540005  
PITCH=PITCH /2.540005

IPSC 570  
IPSC 580  
IPSC 590  
IPSC 600  
IPSC 610  
IPSC 620  
IPSC 630  
IPSC 640  
IPSC 650  
IPSC 660  
IPSC 670  
IPSC 680  
IPSC 690  
IPSC 700  
IPSC 710  
IPSC 720  
IPSC 730  
IPSC 740  
IPSC 750  
IPSC 760  
IPSC 770  
IPSC 780  
IPSC 790  
IPSC 800  
IPSC 810  
IPSC 820  
IPSC 830  
IPSC 840  
IPSC 850  
IPSC 860  
IPSC 870  
IPSC 880  
IPSC 890  
IPSC 900  
IPSC 910  
IPSC 920  
IPSC 930  
IPSC 940  
IPSC 950  
IPSC 960  
IPSC 970  
IPSC 980  
IPSC 990  
IPSC1000  
IPSC1010  
IPSC1020  
IPSC1030  
IPSC1040  
IPSC1050  
IPSC1060  
IPSC1070  
IPSC1080  
IPSC1090  
IPSC1100  
IPSC1110  
IPSC1120  
IPSC1130  
IPSC1140  
IPSC1150



```

FLAT=FLAT /2.540005
EDGE=EDGE /2.540005
FS=FS /2.540005
TC=TC /2.540005
TR=TR /2.540005
XLGTH=XLGTH /2.540005
RIBFLG=RIBFLG /2.540005
ZPNED1=ZPNED1 /2.540005
ZPNED2=ZPNED2 /2.540005
ZEE=ZEE /2.540005
ZEESF1=ZEESF1 /2.540005
ZEEFF=ZEEFF /2.540005
ZEEFF1=ZEEFF1 /2.540005
BWID=BWID /2.540005
RDEP=RDEP /2.540005
BRAD=BRAD /2.540005
ALEN=ALEN /2.540005
PANWID=PANWID /2.540005
ZEESF=ZEESF /2.540005
ESTIFF=ESTIFF/(2.540005**4.)
70 CONTINUE
IF(XLGTH.EQ.DUMLG) GO TO 100
DX=XLGTH/(2.*NSFAT)
DO 80 I=1,NSFAT
80 X(I)=I*DX
DUMLG=XLGTH
DO 90 II=1,NSFAT
IPRINT(II)=II
90 XPRINT(II)=IPRINT(II)*DX
100 CALL GEOM(INDGEO,DEPTH,TR,NRIB,PITCH,
1 RIBFLG,FS,TC,NCUR,EDGE,PHICUR,FLAT,
2 ZEE,ZPNED1,ZPNED2,ZEEFF1,ZEEFF,
3 ZEESF,A,Y,DY,PANWID,
4 NSECT,SEC,S)
IF(S.LT.0.)GO TO 1740
C---- CALCULATION OF GEOMETRY OF BEAD; INDBD = 1
IF(INDBD.EQ.0) GO TO 220
H1=SQRT(BRAD**2-(.5*BWID)**2)
ANGLE=BWID/(2.*H1)
ALPHA=ATAN(ANGLE)
AREABD=ABS(.2*ALPHA*BRAD*FS)
C---- RIB SUPPORT GEOMETRY; INDGEO = 1
C---- CORRUGATION SUPPORT GEOMETRY; INDGEO = 2
C---- ZEE SUPPORT GEOMETRY; INDGEO = 3
IF(INDGEO=2) 110,130,150
110 A(NSECT)=A(NSECT)-IS*BWID*(NRIB-1)
DO 120 I=NSECT1,NSECT
120 A(I)=AREABD*(NRIB-1)*2.
GO TO 170
130 A(NSECT)=A(NSECT)-IS*BWID*NCUR
DO 140 I=NSECT1,NSECT
140 A(I)=AREABD*NCUR*2.
GO TO 170
150 A(NSECT)=A(NSECT)-IS*BWID*(ZEE-1)
DO 160 I=NSECT1,NSECT
160 A(I)=AREABD*(ZEE-1)*2.
170 DO 180 J=NSECT1,NSECT
180 BETA(J)=(1.10-(J-NSECT)*.70)*ALPHA
IF(BRAD.GT.0.0) GO TO 200

```

IPSC1160  
 IPSC1170  
 IPSC1180  
 IPSC1190  
 IPSC1200  
 IPSC1210  
 IPSC1220  
 IPSC1230  
 IPSC1240  
 IPSC1250  
 IPSC1260  
 IPSC1270  
 IPSC1280  
 IPSC1290  
 IPSC1300  
 IPSC1310  
 IPSC1320  
 IPSC1330  
 IPSC1340  
 IPSC1350  
 IPSC1360  
 IPSC1370  
 IPSC1380  
 IPSC1390  
 IPSC1400  
 IPSC1410  
 IPSC1420  
 IPSC1430  
 IPSC1440  
 IPSC1450  
 IPSC1460  
 IPSC1470  
 IPSC1480  
 IPSC1490  
 IPSC1500  
 IPSC1510  
 IPSC1520  
 IPSC1530  
 IPSC1540  
 IPSC1550  
 IPSC1560  
 IPSC1570  
 IPSC1580  
 IPSC1590  
 IPSC1600  
 IPSC1610  
 IPSC1620  
 IPSC1630  
 IPSC1640  
 IPSC1650  
 IPSC1660  
 IPSC1670  
 IPSC1680  
 IPSC1690  
 IPSC1700  
 IPSC1710  
 IPSC1720  
 IPSC1730  
 IPSC1740

```

      XX2=ABS(ARA0)
      XX1=Y(NSFC1)+H1
      DO 190 K=NSC11,NSCCT
190   Y(K)=XX1-XX2*COS(BETA(K))
      GO TO 220
200   XX1=Y(NSFC1)-H1
      DO 210 L=NSC11,NSCCT
210   Y(L)=XX1+BRAD*COS(BETA(L))
C---- CALCULATION OF SECTION PROPERTIES OF BEAM
220   NAREA=NSECT
      IF(INDHD.NE.0) NAREA=NSECTT
      ATOT = 0.0
      AYTOT = 0.0
      AYYTOT = 0.0
      DO 230 J=1,NAREA
      AYTOT = AYTOT + A(J)*Y(J)
      ATOT = ATOT + A(J)
230   AYYTOT = AYYTOT + A(J)*(Y(J)**2)
      YBAR = AYTOT/ATOT
      XI = AYYTOT - ATOT*(YBAR**2)
      DO 1550 I=1,NSFAT
C
CC --- BEAM STATION LOOP
C
      LLOOPCA=LLOOP+1
      IF(INDMEM.EQ.0)GO TO 250
      WRITE(6,1770)LLOOPCA,X(I)
      GO TO 260
250   SUBSTX=X(I)*2.54
      WRITE(6,1770)LLOOPCA,SUBSTX
260   CONTINUE
      IF(IRESID.NE.0)GO TO 280
      DO 270 J=1,NAREA
270   RESSTR(J)=RESSIN(I,J)
280   CONTINUE
      TIME1 = 0.0
      TIME=0.
C---- CALCULATION OF TEMPERATURE AS A FUNCTION OF BEAM LENGTH
      IF(INDTFL.EQ.1) GO TO 300
C---- TEMPERATURE CALCULATED VIA EQUATION; INDTFL = 0
C---- EQUATION METHOD IS ALSO USED IF TEMPERATURE IS CONSTANT
      XTP(I) = C(1) + C(2)*X(I) + C(3)*X(I)**2 + C(4)*X(I)**3
      GO TO 310
C---- TEMPERATURE CALCULATED VIA TABLE-LOOKUP; INDTFL = 1
300   XIN = X(I)
      CALL TRKLP(XTEMP,1,XIN,0.0,XTMP,IE)
      XTP(I) = XTMP
      IF(IE.NE.0) WRITE ( 6,1780 ) XIN
310   CONTINUE
C---- CALCULATION OF TEMPERATURE AS A FUNCTION OF BEAM DEPTH
      IF(INDTFD.EQ.1) GO TO 330
C---- TEMPERATURE CALCULATED VIA EQUATION; INDTFD = 0
C---- EQUATION METHOD IS ALSO USED IF TEMPERATURE IS CONSTANT
      DO 320 J=1,NAREA
320   YTP(J) = D(1) + D(2)*Y(J) + D(3)*Y(J)**2 + D(4)*Y(J)**3
      GO TO 350
C---- TEMPERATURE CALCULATED VIA TABLE-LOOKUP; INDTFD = 1
330   DO 340 J=1,NAREA
      XIN = Y(J)

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CALL TBLKP(YTEMP,1,XIN,0.0,YEMP,IE)
YIP(J) = YTMP
IF(IE.NE.0) WRITE ( 5,1790 ) XIN
340 CONTINUE
350 CONTINUE
C---- CALCULATION OF TOTAL TEMPERATURES AT X AND AT ANY TIME IN THE CYCLE
DO 370 K=1,NTIME
DO 360 L=1,NAREA
T(K,L) = TEMP(K)*XIP(L)*YIP(L)
360 CONTINUE
370 CONTINUE
C---- CALCULATION OF MODULUS OF ELASTICITY AS A FUNCTION OF TEMPERATURE
IF(INDMOD.EQ.1) GO TO 400
C---- MODULUS OF ELASTICITY CALCULATED VIA EQUATION; INDMOD = 0
C---- EQUATION METHOD IS ALSO USED IF MODULUS IS CONSTANT
DO 390 K=1,NTIME
DO 380 L=1,NAREA
E(K,L) = ECOEFF(1) + ECOEFF(2)*T(K,L) + ECOEFF(3)*T(K,L)**2 +
1 ECOEFF(4)*T(K,L)**3
390 CONTINUE
GO TO 430
C---- MODULUS OF ELASTICITY CALCULATED VIA TABLE-LOOKUP; INDMOD = 1
400 DO 420 K=1,NTIME
DO 410 L=1,NAREA
TIN = T(K,L)
CALL TBLKP(TEMP,1,TIN,0.0,EMOD,IE)
E(K,L) = EMOD
410 IF(IE.NE.0) WRITE ( 6,1800 ) TIN
420 CONTINUE
C---- CALCULATION OF MOMENTS AT X AND TIME IN THE CYCLE; INCLUD = 1 (POI
430 CONTINUE
IF(ILOAD.EQ.1) GO TO 450
DO 440 K=1,NTIME
DO 440 L=1,NAREA
E(K,L) = E(K,L)/6894.8
440 CONTINUE
450 CONTINUE
C
C
C PLATE SOLUTION
IF(INCLUD.EQ.1) GO TO 630
IF(INOPLA.EQ.0) GO TO 560
DEONE=XI/PANWID
IF(INDO2.EQ.1) GO TO 450
DETWO= TS*TS*TS/(12.*(1.-XNU*XNU))
460 CONTINUE
IF(XLGTH.GT.PANWID) GO TO 560
IF(DETWO.GT.DEONE) GO TO 560
C
C
C SOLUTION FOR LEHNITSKII MAX MOMENT
DRAF=DEONE/DETWO
IF(DRAF.GT.26.) GO TO 520
ELEK=(PANWID/XLGTH)*(DEONE/DETWO)**.25
IF(ELEK.GE.1.AND.ELEK.LT.1.5) GO TO 470
IF(ELEK.GE.1.5.AND.ELEK.LT.2.0) GO TO 480
IF(ELEK.GE.2.0.AND.ELEK.LT.2.5) GO TO 490
IF(ELEK.GE.2.5.AND.ELEK.LT.3.0) GO TO 500
IF(ELEK.GE.3.0.AND.ELEK.LT.5.0) GO TO 510
IF(ELEK.GE.5.0) GO TO 520

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470 XMU11=.0368-(ELEK-1.0)\*(.0368-.0230)/.5  
XMU22=.0728-(1.5-ELEK)\*(.0728-.0368)/.5  
GO TO 530  
480 XMU11=.0280-(ELEK-1.5)\*(.0280-.0174)/.5  
XMU22=.0964-(2.0-ELEK)\*(.0964-.0728)/.5  
GO TO 530  
490 XMU11=.0174-(ELEK-2.0)\*(.0174-.0099)/.5  
XMU22=.1100-(2.5-ELEK)\*(.1100-.0964)/.5  
GO TO 530  
500 XMU11=.0099-(ELEK-2.5)\*(.0099-.0055)/.5  
XMU22=.1172-(3.0-ELEK)\*(.1172-.1100)/.5  
GO TO 530  
510 XMU11=.0055-(ELEK-3.0)\*(.0055-.0004)/2.0  
XMU22=.1245-(5.0-ELEK)\*(.1245-.1172)/2.0  
GO TO 530  
520 XMU11=.0  
XMU22=.1250  
530 CONTINUE  
XLEKH=(XMU22+XMU11\*XNU)\*(DEUNE/DEFWO)\*\*.5)\*XLGTH\*XLGTH  
SUMPRX=0.  
SUMPRY=0.  
SUMXIN=0.  
SUMYIN=0.  
SUMXZR=0.  
SUMYZR=0.  
XMM=1.  
540 CONTINUE  
ALSUBM=XMM\*3.14159\*PANWID\*.5/XLGTH  
XLAMDA=ESFIF/(XLGTH\*XLGTH)  
HSIN=TANH(ALSUBM)/(SQR(1.-TANH(ALSUBM)\*TANH(ALSUBM)))  
HCOS=1./SQR(1.-TANH(ALSUBM)\*TANH(ALSUBM))  
PT1=XMU\*(1.+XNU)\*HSIN  
PT2=XMU\*(1.-XNU)\*HCOS\*ALSUBM  
PT3=2.\*HCOS+ALSUBM\*HSIN  
PT4=(3.+XNU)\*(1.-XNU)\*HSIN\*HCOS  
PT5=(1.-XNU)\*\*2.\*ALSUBM  
PT6=2.\*XMM\*3.14159\*XLAMDA\*HCOS\*\*2.  
PT7=XMU\*(1.-XNU)\*HSIN  
PT8=XMM\*3.14159\*XLAMDA\*HCOS  
PT9=4./(XMM\*3.14159)\*\*5.  
AM=PT9\*(PT1-PT2-XMM\*3.14159\*XLAMDA\*PT3)/(PT4-PT5+PT6)  
BM=PT9\*(PT7+PT8)/(PT4-PT5+PT6)  
PT10=(XMM\*3.14159/XLGTH)  
PT11=PT10\*PT10  
PARFX=PT11\*(PT9+AM)\*SIN(PT10\*XLGTH/2.)  
PARTY=(PT11\*AM+2.\*BM\*PT11)\*SIN(PT10\*XLGTH/2.)  
SUMPRX=SUMPRX+PARFX  
SUMPRY=SUMPRY+PARTY  
AMZERU=PT9\*(PT1-PT2)/(PT4-PT5)  
BMZERU=PT9\*PT7/(PT4-PT5)  
AMINFN=-PT9\*PT3/(2.\*HCOS\*HCOS)  
HMINFN=PT9/(2.\*HCOS)  
PARXIN=PT11\*(PT9+AMINFN)\*SIN(PT10\*XLGTH/2.)  
PARYIN=(PT11\*AMINFN+2.\*HMINFN\*PT11)\*SIN(PT10\*XLGTH/2.)  
PARXZR=PT11\*(PT9+AMZERU)\*SIN(PT10\*XLGTH/2.)  
PARYZR=(PT11\*AMZERU+2.\*BMZERU\*PT11)\*SIN(PT10\*XLGTH/2.)  
SUMXIN=SUMXIN+PARXIN  
SUMYIN=SUMYIN+PARYIN  
SUMXZR=SUMXZR+PARXZR  
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SUMYZR=SUMYZR+PARYZR
IF(XMM.EQ.7.)GO TO 550
XMM=XMM+2.
GO TO 540
550 XTIMO=(XLGTH**4.)*(F5JAPX-XNU*SUMPYR)
TIMOIN=(XLGTH**4.)*(+SUMXIN-XNU*SUMYIN)
TIMOZR=(XLGTH**4.)*(+SUMXZ-XNU*SUMYZR)
PLAM=XLEKH+(.1250-XLEKH)*(XTIMO-TIMOIN)/(TIMOZR-TIMOIN)
560 CONTINUE
C---- MOMENTS FOR PRESSURE LOADS
C---- BEAM WITH SIMPLE SUPPORTS
DO 580 M=1,NTIME
580 XMOM(M)=(PRESS(M)*XLGTH*(X(I)-DX/2.0)/2.0-PRESS(M)*
1((X(I)-DX/2.0)**2)/2.0)*PANWID
IF(INDELA.EQ.0)GO TO 560
IF(XLGTH.GT.PANWID)GO TO 660
IF(DEFNO.GT.DONE)GO TO 660
DO 590 M=1,NTIME
590 XMOM(M)=PLAM*PRESS(M)*XLGTH*PANWID/8.
DO 600 M=1,NTIME
600 PRMOM(I,M)=XMOM(M)
GO TO 660
C---- BEAM WITH FIXED SUPPORTS
XX1=(XLGTH/2.)*(X(I)-(X(I)**2)/XLGTH-XLGTH/6.0)*PANWID
DO 620 M=1,NTIME
620 XMOM(M)=PRESS(M)*XX1
GO TO 660
C---- MOMENTS FOR POINT LOADS
630 DO 650 K=1,NTIME
IF(X(I).LT.ALEN)GO TO 640
XMOM(K)=PLOAD(K)*ALEN/2.0
GO TO 650
640 XMOM(K)=PLOAD(K)*(X(I)-DX/2.)/2.
650 CONTINUE
660 CONTINUE
KOUNTX=0
CALL IFIMOM( C , D , NAREA , XMOM , YBAR ,
1Y , XI , F , NTIME , A , DX , DY ,
2STRANE , THEIAE , STRESE , YNA , I , INDTED , NTCUN ,
3NSECT , IICON )
IF(INDELA.EQ.1)GO TO 1550
THEIAC=0.
DO 670 J=1,NAREA
670 CSTRAN(J)=0.0
IF(ILLAD.EQ.1)GO TO 700
DO 690 K1=1,NTIME
WRITE(6,1820)K1
WRITE(6,1830)
DO 680 J=1,NAREA
SUBA=A(J)*2.54*2.54
SUBY=Y(J)*2.54
SUBSTR=STRESE(K1,J)*6.8948/1000.
WRITE(6,1840)J,SUBA,SUBY,SUBSTR
680 CONTINUE
690 CONTINUE
GO TO 730
700 CONTINUE
DO 720 K1=1,NTIME

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 IPSC 4100  
 IPSC 4110



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      WRITE(6,1820)K1
      WRITE(6,1850)
      DO 710 J=1,NAREA
710  WRITE(6,1840)J,A(J),Y(J),STRESE(K1,J)
720  CONTINUE
730  CONTINUE
      DO 1540 KK=1,NCYCLE
C    --- CYCLE LOOP
C
      DO 1530 K1=1,NTIME
C    --- TRAJECTORY STEP LOOP
C
        TIME1=TIME
        TIME=((KK-1)*DXTIME(NTIME)+DXTIME(K1))/60.
        IF(KK.EQ.1.AND.K1.EQ.1)GO TO 750
        IF(INDEQ.EQ.1)GO TO 740
        THETAT(K1)=SAVTHE*PRESS(K1)*T(K1,1)*(TIME-TIME1)
        1/(SAVPRS*SAVIMP*TIMBER)+THETAE(I,K1)
        GO TO 750
740  THETAT(K1)=SAVTHE*PLUAD(K1)*T(K1,1)*(TIME-TIME1)
        1/(SAVIMP*SAVIMP*TIMBER)+THETAE(I,K1)
750  CONTINUE
        GO TO 780
760  CONTINUE
        AYNAC=YNA(K1)
        THETAT(K1)=2.*THETAE(I,K1)
        DO 770 J=1,NAREA
770  STRESS(J)=STRESE(K1,J)
780  CONTINUE
        TIMBER=TIME-TIME1
        FIRST=0.
790  F2=0.
800  CONTINUE
        XM1=0.0
        FBAL=0.
        TOTFOR=0.
        DO 1340 J=1,NAREA
        ASIRAT(J)=THETAT(K1)*(AYNAC-Y(J))/DX
        IF(FBAL.EQ.0.)GO TO 820
C    8011
C    820 CONTINUE
C
C    ITERATION FOR STRESS
C
        SIGN=0.
        L=1
        STRES(L)=STRESS(J)
        IF(IINTP.EQ.IEQNTP)GO TO 850
        IF(IEQNTP.EQ.0)GO TO 840
        X3=((9./5.)*T(K1,J)-459.67)/1000.
        GO TO 860
840  X3=((5./9.)*(T(K1,J)+459.67)/1000.)
        GO TO 860.
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      IPSC4130
      IPSC4140
      IPSC4150
      IPSC4160
      IPSC4170
      IPSC4180
      IPSC4190
      IPSC4200
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      IPSC4220
      IPSC4230
      IPSC4240
      IPSC4250
      IPSC4260
      IPSC4270
      IPSC4280
      IPSC4290
      IPSC4300
      IPSC4310
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      IPSC4330
      IPSC4340
      IPSC4350
      IPSC4360
      IPSC4370
      IPSC4380
      IPSC4390
      IPSC4400
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      IPSC4660
      IPSC4670
      IPSC4680
      IPSC4690
      IPSC4700
      IPSC4710

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850 X3=F(KL,1)/1000.  
860 CONTINUE  
X5=1./X3  
X8=ALOG(X3)  
X11=X5\*X5  
X12=X5\*X11  
X31=X3\*X3  
X32=X3\*X31  
X33=X8\*X8  
X34=X8\*X33  
870 IF(HARDENP.EQ.2.)GO TO 1000  
C  
C  
C  
C  
TIME HARDENING THEORY OF CREEP ACCUMULATION  
IF(LINE=1)GO TO 900  
X4=TIME  
X4X=TIME1  
X6=ALOG(X4)  
IF(TIME1.EQ.0.)GO TO 880  
X6X=ALOG(X4X)  
GO TO 890  
880 X6X=0.  
890 CONTINUE  
X27=X4\*X4  
X27X=X4X\*X4X  
X2H=X4\*X27  
X2HX=X4X\*X27X  
X29=X6\*X6  
X29X=X6X\*X6X  
X30=X6\*X29  
X30X=X6X\*X29X  
X36=X3\*X6  
X36X=X3X\*X6X  
X3H=X4\*XH  
X3HX=X4X\*XH  
X40=X6\*X8  
X40X=X6X\*X8  
X42=X5\*X6  
X42X=X5X\*X6X  
X45=X3\*X4  
X45X=X3X\*X4X  
900 CONTINUE  
IF(STRESS(L).GT.0.)GO TO 910  
IF(STRESS(L).EQ.0.)GO TO 970  
STRESS(L)=ABS(STRESS(L))  
SIGN=1.0  
910 CONTINUE  
IF(IFORMST.EQ.0.)GO TO 920  
X2=STRESS(L)/1000.  
GO TO 930  
920 X2=STRESS(L)\*6.8948/1000.  
930 CONTINUE  
LINEAR=0  
IF(X2.GT.1.)GO TO 940  
IF(INDSTR.EQ.0.)GO TO 940  
X2=1.  
LINEAR=1  
IPSC4720  
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IPSC5180  
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IPSC5200  
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IPSC5250  
IPSC5260  
IPSC5270  
IPSC5280  
IPSC5290  
IPSC5300

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940 CONTINUE
X7=ALOG(X2)
X9=X2*X2
X10=X9*X2
X13=X2*X3
X14=X2/X3
X15=X13*X13
X16=X15*X13
X17=X14*X14
X18=X14*X17
X19=X7*X7
X20=X7*X19
X21=X2*X8
X22=X3*X7
X23=X7*X4
X24=X2*X3*X4
X24X=X2*X3*X4X
X25=X24*X24
X25X=X24X*X24X
X26=X24*X25
X26X=X24X*X25X
X35=X2*X6
X35X=X2*X6X
X37=X4*X7
X37X=X4X*X7
X39=X6*X7
X39X=X6X*X7
X41=X5*X7
X43=X42*X7
X43X=X42X*X7
X44=X2*X4
X44X=X2*X4X
TERM3=Z(2)*X2+Z(7)*X7+Z(9)*X9+Z(10)*X10+Z(13)*X13+Z(14)*X14
1+Z(15)*X15+Z(16)*X16+Z(17)*X17+Z(18)*X18+Z(19)*X19+Z(20)*X20
2+Z(21)*X21+Z(22)*X22+Z(23)*X23+Z(41)*X41
IF(TIME1.EQ.0.)GO TO 950
TERM4=Z(1)+Z(3)*X3+Z(4)*X4+Z(5)*X5+Z(6)*X6+Z(8)*X8+Z(11)*X11
1+Z(12)*X12+Z(27)*X27+Z(28)*X28+Z(29)*X29+Z(30)*X30X
2+Z(31)*X31+Z(32)*X32+Z(33)*X33+Z(34)*X34+Z(35)*X35+Z(36)*X36+Z(38)*X38X
3+Z(40)*X40X+Z(42)*X42X+Z(45)*X45X
FUNC1=TERM4+TERM3+Z(24)*X24X+Z(25)*X25X+Z(26)*X26X+Z(35)*X35X
1+Z(37)*X37X+Z(39)*X39X+Z(43)*X43X+Z(44)*X44X
FUNC1=EXP(FUNC1)/100.
GO TO 960
950 CONTINUE
FUNC1=0.
960 CONTINUE
TERM1=Z(1)+Z(3)*X3+Z(4)*X4+Z(5)*X5+Z(6)*X6+Z(8)*X8+Z(11)*X11
1+Z(12)*X12+Z(27)*X27+Z(28)*X28+Z(29)*X29+Z(30)*X30+Z(31)*X31
2+Z(32)*X32+Z(33)*X33+Z(34)*X34+Z(36)*X36+Z(38)*X38+Z(40)*X40
3+Z(42)*X42+Z(45)*X45
FUNC=TERM1+TERM3+Z(24)*X24+Z(25)*X25+Z(26)*X26+Z(35)*X35
1+Z(37)*X37+Z(39)*X39+Z(43)*X43+Z(44)*X44
FUNC=EXP(FUNC)/100.
IF(FUNC.GT.FUNC1)GO TO 980
970 EPSCR=0.
GO TO 990
980 EPSCR=FUNC-FUNC1
IF(LINEAR.EQ.0)GO TO 990
IPSC5310
IPSC5320
IPSC5330
IPSC5340
IPSC5350
IPSC5360
IPSC5370
IPSC5380
IPSC5390
IPSC5400
IPSC5410
IPSC5420
IPSC5430
IPSC5440
IPSC5450
IPSC5460
IPSC5470
IPSC5480
IPSC5490
IPSC5500
IPSC5510
IPSC5520
IPSC5530
IPSC5540
IPSC5550
IPSC5560
IPSC5570
IPSC5580
IPSC5590
IPSC5600
IPSC5610
IPSC5620
IPSC5630
IPSC5640
IPSC5650
IPSC5660
IPSC5670
IPSC5680
IPSC5690
IPSC5700
IPSC5710
IPSC5720
IPSC5730
IPSC5740
IPSC5750
IPSC5760
IPSC5770
IPSC5780
IPSC5790
IPSC5800
IPSC5810
IPSC5820
IPSC5830
IPSC5840
IPSC5850
IPSC5860
IPSC5870
IPSC5880
IPSC5890

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	EPSCRP=EPSCRP*(STRES(L)/1000.)	IPSC5900
990	CONTINUE	IPSC5910
1000	IF(HARDEN.EQ.1)GO TO 1150	IPSC5920
	IF(L.EQ.6.AND.KK.EQ.50)GO TO 1010	IPSC5930
	GO TO 1020	IPSC5940
1010	WR1 IF(6,1860)J,L,STRES(L)	IPSC5950
1020	CONTINUE	IPSC5960
C		IPSC5970
C		IPSC5980
C	STRAIN HARDENING THEORY OF CREEP ACCUMULATION	IPSC5990
		IPSC6000
		IPSC6010
	IF(STRES(L).GT.0.)GO TO 1030	IPSC6020
	STRES(L)=ABS(STRES(L))	IPSC6030
	SIGN=1.0	IPSC6040
1030	CONTINUE	IPSC6050
	IF(STRES(L).GT.10.)GO TO 1040	IPSC6060
	DSIR2=0.	IPSC6070
	GO TO 1150	IPSC6080
1040	CONTINUE	IPSC6090
	FEFF1=0.	IPSC6100
	FEFF2=0.	IPSC6110
	ACONST=0.	IPSC6120
	BCONST=0.	IPSC6130
	CCONST=0.	IPSC6140
	FCONST=0.	IPSC6150
	IF(LEQVST.EQ.0.)GO TO 1050	IPSC6160
	X2=STRES(L)/1000.	IPSC6170
	GO TO 1060	IPSC6180
1050	X2=STRES(L)*6.8948/1000.	IPSC6190
1060	CONTINUE	IPSC6200
	X7=ALOG(X2)	IPSC6210
	X9=X2*X2	IPSC6220
	X10=X9*X2	IPSC6230
	X13=X2*X3	IPSC6240
	X14=X2/X3	IPSC6250
	X15=X13*X13	IPSC6260
	X16=X15*X13	IPSC6270
	X17=X14*X14	IPSC6280
	X18=X14*X17	IPSC6290
	X19=X7*X7	IPSC6300
	X20=X7*X19	IPSC6310
	X21=X2*X4	IPSC6320
	X22=X3*X7	IPSC6330
	X23=X7*X8	IPSC6340
	X41=X5*X7	IPSC6350
	IFR45=Z(1)+Z(2)*X2+Z(3)*X3+Z(5)*X5+Z(7)*X7+	IPSC6360
	17(8)*X8+Z(9)*X9+Z(10)*X10+Z(11)*X11+Z(12)*X12+	IPSC6370
	2Z(13)*X13+Z(14)*X14+Z(15)*X15+Z(16)*X16+Z(17)*X17+Z(18)*X18+	IPSC6380
	3Z(19)*X19+Z(20)*X20+Z(21)*X21+Z(22)*X22+Z(23)*X23+Z(31)*X31+	IPSC6390
	4Z(32)*X32+Z(33)*X33+Z(34)*X34+Z(41)*X41	IPSC6400
	IF(CSTRAN(J).GT..0001)GO TO 1070	IPSC6410
	X4=(TIME-TIME1)	IPSC6420
	X6=ALOG(X4)	IPSC6430
	X24=X2*X3*X4	IPSC6440
	X25=X24*X24	IPSC6450
	X26=X24*X25	IPSC6460
	X27=X4*X4	IPSC6470
	X28=X4*X27	IPSC6480

X29=X6*X6	IPSC6490
X30=X6*X29	IPSC6500
X35=X2*X6	IPSC6510
X36=X3*X6	IPSC6520
X37=X4*X7	IPSC6530
X38=X4*X8	IPSC6540
X39=X6*X7	IPSC6550
X40=X6*X8	IPSC6560
X42=X5*X6	IPSC6570
X43=X42*X7	IPSC6580
X44=X2*X4	IPSC6590
X45=X3*X4	IPSC6600
TERM6=Z(4)*X4+Z(6)*X6+Z(24)*X24+Z(25)*X25+Z(26)*X26+Z(27)*X27+	IPSC6610
1Z(28)*X28+Z(29)*X29+Z(30)*X30+Z(35)*X35+Z(36)*X36+Z(37)*X37+	IPSC6620
2Z(38)*X38+Z(39)*X39+Z(40)*X40+Z(42)*X42+Z(43)*X43+Z(44)*X44+	IPSC6630
3Z(45)*X45	IPSC6640
DSTR2=TERM5+TERM6	IPSC6650
DSTR2=EXP(DSTR2)/100.	IPSC6660
GO TO 1150	IPSC6670
1070 CONTINUE	IPSC6680
TEFF1=EMAX	IPSC6690
X4=TEFF1	IPSC6700
X6=ALOG(X4)	IPSC6710
X24=X2*X3*X4	IPSC6720
X25=X24*X24	IPSC6730
X26=X24*X25	IPSC6740
X27=X4*X4	IPSC6750
X28=X4*X27	IPSC6760
X29=X6*X6	IPSC6770
X30=X6*X29	IPSC6780
X35=X2*X6	IPSC6790
X36=X3*X6	IPSC6800
X37=X4*X7	IPSC6810
X38=X4*X8	IPSC6820
X39=X6*X7	IPSC6830
X40=X6*X8	IPSC6840
X42=X5*X6	IPSC6850
X43=X42*X7	IPSC6860
X44=X2*X4	IPSC6870
X45=X3*X4	IPSC6880
TERM6=Z(4)*X4+Z(6)*X6+Z(24)*X24+Z(25)*X25+Z(26)*X26+Z(27)*X27+	IPSC6890
1Z(28)*X28+Z(29)*X29+Z(30)*X30+Z(35)*X35+Z(36)*X36+Z(37)*X37+	IPSC6900
2Z(38)*X38+Z(39)*X39+Z(40)*X40+Z(42)*X42+Z(43)*X43+Z(44)*X44+	IPSC6910
3Z(45)*X45	IPSC6920
EPS1=TERM5+TERM6	IPSC6930
EPS1=EXP(EPS1)/100.	IPSC6940
IF(EPS1.GT.CSTRAN(J)) GO TO 1090	IPSC6950
TEFF2=TEFF1+1./60.	IPSC6960
X4=TEFF2	IPSC6970
X6=ALOG(X4)	IPSC6980
X24=X2*X3*X4	IPSC6990
X25=X24*X24	IPSC7000
X26=X24*X25	IPSC7010
X27=X4*X4	IPSC7020
X28=X4*X27	IPSC7030
X29=X6*X6	IPSC7040
X30=X6*X29	IPSC7050
X35=X2*X6	IPSC7060
X36=X3*X6	IPSC7070



PREDICTION OF CREEP IN  
METALLIC TPS PANELS

PHASE III  
SUMMARY REPORT

NAS-1-11774

```
X37=X4*X7
X38=X4*X8
X39=X6*X7
X40=X6*X8
X42=X5*X6
X43=X42*X7
X44=X2*X4
X45=X3*X4
FERM6=Z(4)*X4+Z(6)*X6+Z(24)*X24+Z(25)*X25+Z(26)*X26+Z(27)*X27+
1Z(28)*X28+Z(29)*X29+Z(30)*X30+Z(35)*X35+Z(36)*X36+Z(37)*X37+
2Z(38)*X38+Z(39)*X39+Z(40)*X40+Z(42)*X42+Z(43)*X43+Z(44)*X44+
3Z(45)*X45
EPS2=FERM6+FERM6
EPS2=EXP(EPS2)/100.
IF(EPS2.LE.EPS1)GO TO 1080
DSTR2=(EPS2-EPS1)*(TIME-TIME1)*60.
BCONST=2.
GO TO 1150
1080 DSTR2=0.
BCONST=3.
GO TO 1150
1090 TEFF2=IMAX*(CSTRAN(J)/EPS1)
1100 CONFINUE
X4=TEFF2
X6=ALOG(X4)
X24=X2*X3*X4
X25=X24*X24
X26=X24*X25
X27=X4*X4
X28=X4*X27
X29=X6*X6
X30=X6*X29
X35=X2*X6
X36=X3*X6
X37=X4*X7
X38=X4*X8
X39=X6*X7
X40=X6*X8
X42=X5*X6
X43=X42*X7
X44=X2*X4
X45=X3*X4
FERM6=Z(4)*X4+Z(6)*X6+Z(24)*X24+Z(25)*X25+Z(26)*X26+Z(27)*X27+
1Z(28)*X28+Z(29)*X29+Z(30)*X30+Z(35)*X35+Z(36)*X36+Z(37)*X37+
2Z(38)*X38+Z(39)*X39+Z(40)*X40+Z(42)*X42+Z(43)*X43+Z(44)*X44+
3Z(45)*X45
EPS2=FERM6+FERM6
EPS2=EXP(EPS2)/100.
IF(EPS2.LE.EPS1)GO TO 1110
TEFF2=TEFF2*(CSTRAN(J)/EPS2)
GO TO 1100
1110 IF(EPS2.GT.CSTRAN(J).AND.EPS1.GT.CSTRAN(J))GO TO 1120
SLOPE=(TEFF1-TEFF2)/(EPS1-EPS2)
TEFF1=TEFF2
EPS1=EPS2
TEFF2=SLOPE*(CSTRAN(J)-EPS2)+TEFF2
GO TO 1130
1120 TEFF1=TEFF2
EPS1=EPS2
TPSC7080
TPSC7090
TPSC7100
TPSC7110
TPSC7120
TPSC7130
TPSC7140
TPSC7150
TPSC7160
TPSC7170
TPSC7180
TPSC7190
TPSC7200
TPSC7210
TPSC7220
TPSC7230
TPSC7240
TPSC7250
TPSC7260
TPSC7270
TPSC7280
TPSC7290
TPSC7300
TPSC7310
TPSC7320
TPSC7330
TPSC7340
TPSC7350
TPSC7360
TPSC7370
TPSC7380
TPSC7390
TPSC7400
TPSC7410
TPSC7420
TPSC7430
TPSC7440
TPSC7450
TPSC7460
TPSC7470
TPSC7480
TPSC7490
TPSC7500
TPSC7510
TPSC7520
TPSC7530
TPSC7540
TPSC7550
TPSC7560
TPSC7570
TPSC7580
TPSC7590
TPSC7600
TPSC7610
TPSC7620
TPSC7630
TPSC7640
TPSC7650
TPSC7660
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1130  TFFF2=TFFF2*(CSTRAN(J)/EPS2)
      CDW TIME
      X4=TFFF2
      X6=ALOG(X4)
      X24=X2*X3*X4
      X25=X24*X24
      X26=X24*X25
      X27=X4*X4
      X28=X4*X27
      X29=X6*X6
      X30=X6*X29
      X35=X2*X6
      X36=X3*X6
      X37=X4*X7
      X38=X4*X8
      X39=X6*X7
      X40=X6*X8
      X42=X5*X6
      X43=X42*X7
      X44=X2*X4
      X45=X3*X4
      TERM5=Z(4)*X4+Z(6)*X6+Z(24)*X24+Z(25)*X25+Z(26)*X26+Z(27)*X27+
      1Z(28)*X28+Z(29)*X29+Z(30)*X30+Z(35)*X35+Z(36)*X36+Z(37)*X37+
      2Z(38)*X38+Z(39)*X39+Z(40)*X40+Z(42)*X42+Z(43)*X43+Z(44)*X44+
      3Z(45)*X45
      EPS2=TERM5+TERM6
      EPS2=EXP(EPS2)/100.
      IF(ABS((EPS2-CSTRAN(J))/CSTRAN(J)).LT..0001)GO TO 1140
      GO TO 1110
1140  TFFF2=TFFF2+TIME-TIME1
      X4=TFFF2
      X6=ALOG(X4)
      X24=X2*X3*X4
      X25=X24*X24
      X26=X24*X25
      X27=X4*X4
      X28=X4*X27
      X29=X6*X6
      X30=X6*X29
      X35=X2*X6
      X36=X3*X6
      X37=X4*X7
      X38=X4*X8
      X39=X6*X7
      X40=X6*X8
      X42=X5*X6
      X43=X42*X7
      X44=X2*X4
      X45=X3*X4
      TERM5=Z(4)*X4+Z(6)*X6+Z(24)*X24+Z(25)*X25+Z(26)*X26+Z(27)*X27+
      1Z(28)*X28+Z(29)*X29+Z(30)*X30+Z(35)*X35+Z(36)*X36+Z(37)*X37+
      2Z(38)*X38+Z(39)*X39+Z(40)*X40+Z(42)*X42+Z(43)*X43+Z(44)*X44+
      3Z(45)*X45
      C3=TERM5+TERM6
      C3=EXP(C3)/100.
      DSTR2=C3-EPS2
1150  CONTINUE
      EPSCR=DSTR2
1160  CONTINUE

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IPSC1670  
 IPSC1680  
 IPSC1690  
 IPSC1700  
 IPSC1710  
 IPSC1720  
 IPSC1730  
 IPSC1740  
 IPSC1750  
 IPSC1760  
 IPSC1770  
 IPSC1780  
 IPSC1790  
 IPSC1800  
 IPSC1810  
 IPSC1820  
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 IPSC1880  
 IPSC1890  
 IPSC1900  
 IPSC1910  
 IPSC1920  
 IPSC1930  
 IPSC1940  
 IPSC1950  
 IPSC1960  
 IPSC1970  
 IPSC1980  
 IPSC1990  
 IPSC8000  
 IPSC8010  
 IPSC8020  
 IPSC8030  
 IPSC8040  
 IPSC8050  
 IPSC8060  
 IPSC8070  
 IPSC8080  
 IPSC8090  
 IPSC8100  
 IPSC8110  
 IPSC8120  
 IPSC8130  
 IPSC8140  
 IPSC8150  
 IPSC8160  
 IPSC8170  
 IPSC8180  
 IPSC8190  
 IPSC8200  
 IPSC8210  
 IPSC8220  
 IPSC8230  
 IPSC8240  
 IPSC8250

	IF(SIGN.EQ.0.)GO TO 1170		TPSC8260
	STRES(L)=-STRES(L)		TPSC8270
	SIGN=0.		TPSC8280
	EPSCRP=-EPSCRP		TPSC8290
1170	CONTINUE		TPSC8300
	SSSTART=STRES(1)		TPSC8310
	TOTCRP=EPSCRP+(STRES(L)+SSSTR(J))/E(KL,J)		TPSC8320
C			TPSC8330
C	8015		TPSC8340
C			TPSC8350
	IF(ASRAT(J).EQ.TOTCRP)GO TO 1320		TPSC8360
	CRATIO=TOTCRP/ASRAT(J)		TPSC8370
	IF(ABS(CRATIO-1.000).LT..001)GO TO 1320		TPSC8380
	IF(L.NE.1)GO TO 1270		TPSC8390
	L=2		TPSC8400
	ISTR1=TOTCRP		TPSC8410
	IF(STRES(1).NE.0.)GO TO 1200		TPSC8420
	IF(ASRAT(J).GT.TOTCRP)GO TO 1190		TPSC8430
1180	STRES(2)=STRES(1)-100.		TPSC8440
	GO TO 870		TPSC8450
1190	STRES(2)=STRES(1)+100.		TPSC8460
	GO TO 870		TPSC8470
1200	IF(ASRAT(J).NE.0.)GO TO 1210		TPSC8480
	IF(TOTCRP.LT.0.)GO TO 1190		TPSC8490
	GO TO 1180		TPSC8500
1210	IF(STRES(1).LT.0..AND.TOTCRP.GT.0..AND.ASRAT(J).LT.TOTCRP)		TPSC8510
	GO TO 1220		TPSC8520
	IF(STRES(1).GT.0..AND.TOTCRP.LT.0..AND.ASRAT(J).GT.TOTCRP)		TPSC8530
	GO TO 1220		TPSC8540
	GO TO 1230		TPSC8550
1220	STRES(2)=STRES(1)*2.		TPSC8560
	GO TO 870		TPSC8570
1230	CONTINUE		TPSC8580
	IF(ABS(CRATIO).GT..1)GO TO 1250		TPSC8590
	IF(ASRAT(J).LT.0.)GO TO 1240		TPSC8600
	STRES(2)=STRES(1)+ABS(STRES(1))		TPSC8610
	GO TO 1260		TPSC8620
1240	STRES(2)=STRES(1)-ABS(STRES(1))		TPSC8630
	GO TO 1260		TPSC8640
1250	CONTINUE		TPSC8650
	STRES(2)=STRES(1)/CRATIO		TPSC8660
1260	CONTINUE		TPSC8670
	GO TO 870		TPSC8680
1270	L=L+1		TPSC8690
	IF(L.NE.20)GO TO 1280		TPSC8700
	WRITE(6,1870)I,J,KK,K1		TPSC8710
	GO TO 1320		TPSC8720
1280	CONTINUE		TPSC8730
	ISTR2=TOTCRP		TPSC8740
	IF(ASRAT(J).EQ.0.)GO TO 1310		TPSC8750
	CRATIO=TOTCRP/ASRAT(J)		TPSC8760
	IF(ABS(CRATIO-1.000).LT..001)GO TO 1290		TPSC8770
	GO TO 1300		TPSC8780
1290	L=L-1		TPSC8790
	GO TO 1320		TPSC8800
1300	CONTINUE		TPSC8810
	SLOPE=(STRES(L-1)-STRES(L-2))/(ISTR2-ISTR1)		TPSC8820
	STRES(L)=STRES(L-1)+(ASRAT(J)-ISTR2)*SLOPE		TPSC8830
	ISTR1=ISTR2		TPSC8840

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      GO TO 870
1310 SLOPE=(STRES(L-1)-STRES(L-2))/(TS TR2-TS TR1)
      STRES(L)=STRES(L-1)-TS TR2*SLOPE
      IF(ABS(STRES(L)/STRES(L-1)-1.00).LT..001)GO TO 1320
      IF(ABS(STRES(L)/STRES(L-2)-1.00).LT..001)GO TO 1320
      TS TR1=TS TR2
      GO TO 870
1320 CONTINUE
      STRESS(J)=STRES(L)
      FORC(J) = STRESS(J)*A(J)
1340 TOTFOR = TOTFOR + FORC(J)
C
C      8010
C
      IF(FHAL.NE.0.)GO TO 1360
      IF(TOTFOR.LT.0.0.AND.F2.GT.0.0) GO TO 1350
      IF(TOTFOR.GT.0.0.AND.F2.LT.0.0) GO TO 1350
      IF(ABS(TOTFOR).LT..001) GO TO 1380
      Y2 = AYNAC
      IF(TOTFOR.LT.0.0) AYNAC = AYNAC + DY
      IF(TOTFOR.GT.0.0) AYNAC = AYNAC - DY
      F2 = TOTFOR
      GO TO 800
1350 CONTINUE
      AYNAC = AYNAC - (ABS(TOTFOR)*(AYNAC - Y2))/(ABS(TOTFOR) + ABS(F2))
      FHAL=1.
      TOTFOR=0.
      GO TO 810
1360 CONTINUE
      DO 1370 J=1,NAREA
      ASTRA(J) = THETAT(K1)*(AYNAC - Y(J))/DX
1370 CONTINUE
      TOTFOR = 0.0
      DO 1380 J=1,NAREA
      FORC(J) = STRESS(J)*A(J)
      TOTFOR = TOTFOR + FORC(J)
      XM1 = XM1 + ABS(FORC(J))*(AYNAC - Y(J))
C
C      8013
C
1390 CONTINUE
      TOL1 = ABS(XM1 - XMOM(K1))/XMOM(K1)
C
C      8006
C
      IF(TOL1.LT..001) GO TO 1410
      IF(FIRST4.NE.0.)GO TO 1400
      FIRST4=1.
      XM10=XM1
      THAT1=THETAT(K1)
      THEAT(K1)=THEAT(K1)*(XMOM(K1)/X41)
      GO TO 790
1400 CONTINUE
      THAT2=THETAT(K1)
      TOL1=ABS(XM1-XMOM(K1))/XMOM(K1)
      IF(TOL1.LT..001)GO TO 1410
      SLOPEM=(THAT2-THAT1)/(XM1-XM10)
      THETAT(K1)=SLOPEM*(XMOM(K1)-XM1)+THAT2
C

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IPSC8850  
 IPSC8860  
 IPSC8870  
 IPSC8880  
 IPSC8890  
 IPSC8900  
 IPSC8910  
 IPSC8920  
 IPSC8930  
 IPSC8940  
 IPSC8950  
 IPSC8960  
 IPSC8970  
 IPSC8980  
 IPSC8990  
 IPSC9000  
 IPSC9010  
 IPSC9020  
 IPSC9030  
 IPSC9040  
 IPSC9050  
 IPSC9060  
 IPSC9070  
 IPSC9080  
 IPSC9090  
 IPSC9100  
 IPSC9110  
 IPSC9120  
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 IPSC9320  
 IPSC9330  
 IPSC9340  
 IPSC9350  
 IPSC9360  
 IPSC9370  
 IPSC9380  
 IPSC9390  
 IPSC9400  
 IPSC9410  
 IPSC9420  
 IPSC9430



```
C 3007
C
  THA11=THA12
  X410=X41
  GO TO 1420
1410 CONTINUE
  SAVTHE=THE TAT(K1)-THE TAE(1,K1)
  SAVTMP=T(K1,1)
  IF(INDLNO.EQ.1) GO TO 1420
  SAVPRS=PRESS(K1)
  GO TO 1430
1420 SAVLUD=LOAD(K1)
1430 CONTINUE
  DO 1440 J=1,NAREA
  RESSTR(J)=STRESS(K1,J)-STRESS(J)
1440 CONTINUE
  THETAC=THETAC+THE TAT(K1)-THE TAE(1,K1)
  DO 1450 J=1,NAREA
1450 CSFRAN(J)=THETAC*(AYNAC-Y(J))/DX
  RESSTR(J)=RESSTR(J)/E(K1,J)
  IF(INCYC.EQ.0) GO TO 1460
  IF(KK.NE.1) GO TO 1460
  THE T(I,K1)=THETAC
1460 IF(K1.NE.NTIME) GO TO 1530
  DO 1470 N=1,NUMCYC
  NUMC1 = N
  IF(KK.EQ.KCYCLE(N)) GO TO 1480
1470 CONTINUE
  GO TO 1500
1480 DO 1490 J=1,NAREA
  PSFRAN(I,J,NUMC1)=CSFRAN(J)*100.
  RESP(I,J,NUMC1)=RESSTR(J)
1490 CONTINUE
1500 CONTINUE
  DO 1510 N=1,NUMCYC
  NUMC2 = N
1510 IF(KK.EQ.KCYCLE(N)) GO TO 1520
  GO TO 1530
1520 THE TAP(I,NUMC2) = THETAC
1530 CONTINUE
1540 CONTINUE
1550 CONTINUE
  IF(INDLA.EQ.1) GO TO 1610
  DO 1600 L1=1,NUMCYC
  NMIN2=NSIAT-2
  DO 1580 L2=1,NMIN2
  SUM1=0.0
  SUM2=0.0
  SUM3=0.0
  IPRIL2=IPRINT(L2)
  DO 1560 N1=L,IPRIL2
  THETAX=(THE TAP(N1,L1)+THE TAP(N1+1,L1))*X(N1)/2.0
  SUM1=SUM1+THETAX
1560 CONTINUE
  IPRIL2=IPRIL2+2
  DO 1570 N2=IPRIL2,NSIAT
  SUM2=SUM2+THE TAP(N2,L1)
  DEFL(L1,L2)=SUM1+SUM2* X(L2)+ X(L2)*THE TAP(L2+1,L1)/2.
1580 CONTINUE
```

```
TPSC9440
TPSC9450
TPSC9460
TPSC9470
TPSC9480
TPSC9490
TPSC9500
TPSC9510
TPSC9520
TPSC9530
TPSC9540
TPSC9550
TPSC9560
TPSC9570
TPSC9580
TPSC9590
TPSC9600
TPSC9610
TPSC9620
TPSC9630
TPSC9640
TPSC9650
TPSC9660
TPSC9670
TPSC9680
TPSC9690
TPSC9700
TPSC9710
TPSC9720
TPSC9730
TPSC9740
TPSC9750
TPSC9760
TPSC9770
TPSC9780
TPSC9790
TPSC9800
TPSC9810
TPSC9820
TPSC9830
TPSC9840
TPSC9850
TPSC9860
TPSC9870
TPSC9880
TPSC9890
TPSC9900
TPSC9910
TPSC9920
TPSC9930
TPSC9940
TPSC9950
TPSC9960
TPSC9970
TPSC9980
TPSC9990
TPSC 10
TPSC 20
TPSC 30
```



```

NMIN1=NSTAT-1
DO 1590 N1=1,NMIN1
THE TAY=(THE TAP(N1,L1)+THE TAP(N1+1,L1))*X(N1)/2.0
SUM3=SUM3+THE TAY
1590 CONTINUE
DEFL(L1,NSTAT-1)=SUM3+THE TAP(NSTAT,L1)*X(NSTAT-1)/2.0
DEFL(L1,NSTAT)=SUM3+THE TAP(NSTAT,L1)*X(NSTAT)/2.0
1600 CONTINUE
1610 CONTINUE
DO 1660 L1=1,NTIME
NMIN2=NSTAT-2
DO 1640 L2=1,NMIN2
SUM4=0.
SUM5=0.
SUM6=0.
IPRIL2=IPRINT(L2)
DO 1620 N1=1,IPRIL2
ETHETA=(THE TAE(N1,L1)+THE TAE(N1+1,L1))*X(N1)/2.0
SUM4=SUM4+ETHETA
1620 CONTINUE
IPRIL2=IPRIL2+2
DO 1630 N2=IPRIL2,NSTAT
SUM5=SUM5+THE TAE(N2,L1)
DEFL(L1,L2)=SUM4+SUM5* X(L2)+ X(L2)*THE TAE(L2+1,L1)/2.
1640 CONTINUE
NMIN1=NSTAT-1
DO 1650 N1=1,NMIN1
THE TAY=(THE TAE(N1,L1)+THE TAE(N1+1,L1))*X(N1)/2.
SUM6=SUM6+THE TAY
1650 CONTINUE
DEFL(L1,NSTAT-1)=SUM4+THE TAE(NSTAT,L1)*X(NSTAT-1)/2.
DEFL(L1,NSTAT)=SUM6+THE TAE(NSTAT,L1)*X(NSTAT)/2.
1660 CONTINUE
IF(INCYC.EQ.0)GO TO 1720
DO 1710 L1=1,NTIME
NMIN2=NSTAT-2
DO 1690 L2=1,NMIN2
SUM7=0.0
SUM8=0.0
SUM9=0.0
IPRIL2=IPRINT(L2)
DO 1670 N1=1,IPRIL2
THE TAY=(THE T(N1,L1)+THE T(N1+1,L1))*X(N1)/2.0
SUM7=SUM7+THE TAY
1670 CONTINUE
IPRIL2=IPRIL2+2
DO 1680 N2=IPRIL2,NSTAT
SUM8=SUM8+THE T(N2,L1)
DEFLIN(L1,L2)=SUM7+SUM8* X(L2)+ X(L2)*THE T(L2+1,L1)/2.
1690 CONTINUE
NMIN1=NSTAT-1
DO 1700 N1=1,NMIN1
THE TAY=(THE T(N1,L1)+THE T(N1+1,L1))*X(N1)/2.0
SUM9=SUM9+THE TAY
1700 CONTINUE
DEFLIN(L1,NSTAT-1)=SUM9+THE T(NSTAT,L1)*X(NSTAT-1)/2.0
DEFLIN(L1,NSTAT)=SUM9+THE T(NSTAT,L1)*X(NSTAT)/2.0
1710 CONTINUE
1720 CONTINUE
```

IPSC 40  
IPSC 50  
IPSC 60  
IPSC 70  
IPSC 80  
IPSC 90  
IPSC 100  
IPSC 110  
IPSC 120  
IPSC 130  
IPSC 140  
IPSC 150  
IPSC 160  
IPSC 170  
IPSC 180  
IPSC 190  
IPSC 200  
IPSC 210  
IPSC 220  
IPSC 230  
IPSC 240  
IPSC 250  
IPSC 260  
IPSC 270  
IPSC 280  
IPSC 290  
IPSC 300  
IPSC 310  
IPSC 320  
IPSC 330  
IPSC 340  
IPSC 350  
IPSC 360  
IPSC 370  
IPSC 380  
IPSC 390  
IPSC 400  
IPSC 410  
IPSC 420  
IPSC 430  
IPSC 440  
IPSC 450  
IPSC 460  
IPSC 470  
IPSC 480  
IPSC 490  
IPSC 500  
IPSC 510  
IPSC 520  
IPSC 530  
IPSC 540  
IPSC 550  
IPSC 560  
IPSC 570  
IPSC 580  
IPSC 590  
IPSC 600  
IPSC 610  
IPSC 620



```
IF (IDIMEN.EQ.1) GO TO 1730
DEPTH=DEPTH *2.540005
PITCH=PITCH *2.540005
FLAT=FLAT *2.540005
EDGE=EDGE *2.540005
IS=IS *2.540005
IC=IC *2.540005
IR=IR *2.540005
XLGTH=XLGTH *2.540005
RIBFLG=RIBFLG *2.540005
ZPNED1=ZPNED1 *2.540005
ZPNED2=ZPNED2 *2.540005
ZEEF=ZEEF *2.540005
ZEEF1=ZEEF1 *2.540005
ZEEFF=ZEEFF *2.540005
ZEEF1=ZEEF1 *2.540005
BWID=BWID *2.540005
BDEP=BDEP *2.540005
BRAD=BRAD *2.540005
ALEN=ALEN *2.540005
PANWID=PANWID *2.540005
ZEEF=ZEEF *2.540005
ESTIFF=ESTIFF*(2.540005**4.)
1730 CONTINUE
CALL OUTPUT( INDEG, IS, IR, NRIB, PITCH,
1RIBFLG, DEPTH, IC, NCOR, FLAT, EDGE, PHICOR,
2TZEE, NZEE, ZPNED1, ZPNED2, ZEEF, ZEEF1, ZEEFF,
3ZEEF1, XLGTH, INDRD, BRAD, BWID, INLOD, INDSUP,
4ALEN, DXTIME, PRESS, TEMP, XPRINT, NSTAT,
5DEPLE, NUMCYC, KCYCLE, DEFL, NAREA, Y, RESP,
6A1, A2, A3, A4, A5, INDELA, PSTAN, PLOAD,
7 XI, YBAR, Z, PANWID, DEFLIN, INCYC,
HITIME, LEONIP, IINIP, IDIMEN, ILOAD, IENNST)
1740 LOOP=LOOP+1
IF (LOOP.LT.NENCAS) GO TO 10
1760 FORMAT(5A10)
1770 FORMAT(1H,777,50X,4HCASE,14,9H ANALYSIS,/,
148X,22HELASTIC STRESSES AT X=,F7.3,/)
1780 FORMAT(40H ERROR IN TABLE-LOOKUP ROUTINE AT XIN = ,E15.3)
1790 FORMAT(40H ERROR IN TABLE-LOOKUP ROUTINE AT XIN = ,E15.4)
1800 FORMAT(40H ERROR IN TABLE-LOOKUP ROUTINE AT XIN = ,E15.5)
1820 FORMAT(77,48X,21HTRAJECTORY TIME STEP,12,/)
1830 FORMAT(30X,1HJ,15X,11HAREA(SQ.CM),13X,5HY(CM),14X,11HSTRESS(MPA)/)
1840 FORMAT(28X,15,5X,3E20.5)
1850 FORMAT(30X,1HJ,15X,11HAREA(SQ.IN),13X,5HY(IN),14X,11HSTRESS(PSI)/)
1860 FORMAT(5H 9003,10X,215,E15.5)
1870 FORMAT(65H WARNING - TWENTY ITERATIONS ON STRESS IN THE HARDENING
1 ROUTINE.,/14H BEAM STATION,12,7X,11H SECTION(J),
212,7X,7H CYCLE,13,7X,6H STEP,12/42H ANALYSIS PROCEEDING WITH STR
3ESS UNCHANGED,/)
END
SUBROUTINE GEOM( INDEG, DEPTH, TR, NRIB, PITCH,
1RIBFLG, IS, IC, NCOR, EDGE, PHICOR, FLAT,
2TZEE, NZEE, ZPNED1, ZPNED2, ZEEF1, ZEEF,
3ZEEF1, A, Y, DY, PANWID,
4 NSECT, SEC, S )
C
C
C----- THIS SUBROUTINE DIVIDES THE TPS CROSS SECTION INTO DESCRETE
GEOM 10
GEOM 20
GEOM 30
GEOM 40
GEOM 50
GEOM 60
GEOM 70
GEOM 80
```



```
C      ELEMENTS AND ASSIGNS Y(J) CONTROLLED LOCATIONS AND AREAS TO
C      EACH ELEMENT.
C
C      DIMENSION A(50) , Y(50)
C      COMMON PLAM00,PR0000,INDGEO
C      DATA RAD/57.2957795/
C      NSECT2=NSECT1-1
C      NSECT3=NSECT2-2
C      NSECT4=NSECT3-4
C      NSECT5=NSECT3-3
C      GO TO (10,40,80,230),INDGEO
C
C      --- CALCULATION OF RIB SECTION AREAS AND Y LOCATIONS; INDGEO = 1
C
C      10 DY= (DEPTH-TS)/(SEC-1.)
C      A(1)=((DEPTH-TS)/(SEC-1.))*CR*NRIB
C      DO 20 J=2,NSECT2
C      20 A(J)= A(J-1)
C      A(NSECT)=TS*((NRIB-1)*PITCH +2.0*RIBFLG)
C      DO 30 J=1,NSECT2
C      30 Y(J)= (2.0*J - 1.0)*DY/2.0
C      Y(NSECT)=DEPTH-TS/2.0
C      RETURN
C
C      --- CALCULATION OF CORRUGATED SECTION AREAS AND Y LOCATIONS; INDGEO =
C
C      40 DY= (DEPTH-TS-2.*TC)/(SEC-3.)
C      Y(1)= TC/2.0
C      DO 50 J=2,NSECT3
C      50 Y(J)= TC + (J-2)*DY + DY/2.0
C      Y(NSECT2)=Y(NSECT3)+TC/2.0+DY/2.0
C      Y(NSECT)=DEPTH-TS/2.0
C      S=PITCH - 2.0*((Y(NSECT2)-Y(1))*(TAN(PHICOR/RAD)))-FLAT
C      IF(S.GE.0.0) GO TO 60
C      WRITE (6,240)
C      RETURN
C      60 A(1)= S*NCOR*TC
C      DO 70 J=2,NSECT3
C      70 A(J)=(DY/COS(PHICOR/RAD))*TC*NCOR*2.0
C      A(NSECT2)= (NCOR*FLAT + 2.0*EDGE)*TC
C      A(NSECT)= PANWID*TS
C      RETURN
C
C      --- CALCULATION OF Z SECTION AREAS AND Y LOCATIONS; INDGEO = 3
C
C      80 DY= (DEPTH - TS - 2.0*TZEE)/(SEC-5.)
C      Y(1)= TZEE/2.0
C      DO 90 I=2,NSECT4
C      90 Y(I)= TZEE + (I-2)*DY + DY/2.0
C      IF(ZEEFF1.LE.TZEE) GO TO 120
C      DO 100 I=2,NSECT
C      100 IF((Y(I) + DY/2.0).GT.ZEEFF1) GO TO 110
C      CONTINUE
C      110 INGT1= I - 1
C      H47= ZEEFF1 - Y(I-1) - DY/2.0
C      Y(NSECT5)= ZEEFF1-H47/2.0
C      A(NSECT5)= H47*NZEE*TZEE
C      GO TO 130
C
C      GEOM 90
C      GEOM 100
C      GEOM 110
C      GEOM 120
C      GEOM 130
C      GEOM 140
C      GEOM 150
C      GEOM 160
C      GEOM 170
C      GEOM 180
C      GEOM 190
C      GEOM 200
C      GEOM 210
C      GEOM 220
C      GEOM 230
C      GEOM 240
C      GEOM 250
C      GEOM 260
C      GEOM 270
C      GEOM 280
C      GEOM 290
C      GEOM 300
C      GEOM 310
C      GEOM 320
C      GEOM 330
C      GEOM 340
C      GEOM 350
C      GEOM 360
C      GEOM 370
C      GEOM 380
C      GEOM 390
C      GEOM 400
C      GEOM 410
C      GEOM 420
C      GEOM 430
C      GEOM 440
C      GEOM 450
C      GEOM 460
C      GEOM 470
C      GEOM 480
C      GEOM 490
C      GEOM 500
C      GEOM 510
C      GEOM 520
C      GEOM 530
C      GEOM 540
C      GEOM 550
C      GEOM 560
C      GEOM 570
C      GEOM 580
C      GEOM 590
C      GEOM 600
C      GEOM 610
C      GEOM 620
C      GEOM 630
C      GEOM 640
C      GEOM 650
C      GEOM 660
C      GEOM 670
```

```

120 Y(NSECT5) = 0.0
A(NSECT5) = 0.0
130 IF(ZEESF1.LE.TZEE) GO TO 150
DO 140 J=2,NSECT
DEPTH1 = DEPTH - TS - ZEESF1
IF((Y(J) + DY/2.0).GT.DEPTH1) GO TO 150
140 CONTINUE
150 INGT2 = J + 1
H48 = Y(J) + DY/2.0 - DEPTH1
Y(NSECT3) = DEPTH1 + H48/2.0
A(NSECT3) = H48*NZEE*TZEE
GO TO 170
160 Y(NSECT3) = 0.0
A(NSECT3) = 0.0
170 Y(NSECT2) = DEPTH - TS - TZEE/2.0
Y(NSECT) = DEPTH - TS/2.0
A(1) = TZEE*ZEESF*VZEE
DO 180 K=2,NSECT4
180 A(K) = DY*NZEE*TZEE
IF(ZEESF1.LE.TZEE) GO TO 200
DO 190 L=2,INGT1
190 A(L) = A(L) + DY*NZEE*TZEE
200 CONTINUE
IF(ZEESF1.EQ.0.0.OR.ZEESF1.LE.TZEE) GO TO 220
DO 210 M=INGT2,NSECT4
210 A(M) = A(M) + DY*NZEE*TZEE
220 CONTINUE
A(NSECT2) = NZEE*ZEESF*TZEE
A(NSECT) = PANWID * TS
230 CONTINUE
RETURN
240 FORMAT(50H CORRUGATION INPUT DATA YIELDS A NEGATIVE S LENGTH)
END
SUBROUTINE TBLKP(T, IEX, XIN, YIN, Z, IE)
DIMENSION T(1)
COMMON PLAMOM,PRTMOM,INDPLA
I = IABS(IEX)
IE = 0
NX = T(1+2)
NY = T(1+3)
C----- INDEPENDENT VARIABLE, X TABLE LOOK-UP
LL = 5
LU = NX + 4
LLS = T(1)
VAR = XIN
NRTN = 0
GO TO 50
10 T(1) = LL
IF (NY.GT.0) GO TO 20
LZ = LL + NX
Z = FRAC*(T(LZ+1) - T(LZ)) + T(LZ)
GO TO 190
C----- INDEPENDENT VARIABLE, Y TABLE LOOK-UP -- BIVARIATE
20 LX = LL - 4
FRX = FRAC
LL = NX + 5
LU = NX + NY + 4
LLS = T(1+1)
VAR = YIN

```

GEOM 680  
 GEOM 690  
 GEOM 700  
 GEOM 710  
 GEOM 720  
 GEOM 730  
 GEOM 740  
 GEOM 750  
 GEOM 760  
 GEOM 770  
 GEOM 780  
 GEOM 790  
 GEOM 800  
 GEOM 810  
 GEOM 820  
 GEOM 830  
 GEOM 840  
 GEOM 850  
 GEOM 860  
 GEOM 870  
 GEOM 880  
 GEOM 890  
 GEOM 900  
 GEOM 910  
 GEOM 920  
 GEOM 930  
 GEOM 940  
 GEOM 950  
 GEOM 960  
 GEOM 970  
 GEOM 980  
 GEOM 990  
 -  
 TBLK 10  
 TBLK 20  
 TBLK 30  
 TBLK 40  
 TBLK 50  
 TBLK 60  
 TBLK 70  
 TBLK 80  
 TBLK 90  
 TBLK 100  
 TBLK 110  
 TBLK 120  
 TBLK 130  
 TBLK 140  
 TBLK 150  
 TBLK 160  
 TBLK 170  
 TBLK 180  
 TBLK 190  
 TBLK 200  
 TBLK 210  
 TBLK 220  
 TBLK 230  
 TBLK 240  
 TBLK 250  
 TBLK 260



```
NRIN = 1
GO TO 50
30 I(I+1) = LL
C---- INTERPOLATION
LZ = LL + NY * LX
IF (FRX.EQ.1.) GO TO 40
Z1 = FRX * (T(LZ+NY) - T(LL)) + T(LL)
LZ = LZ + 1
Z2 = FRX * (T(LZ+NY) - T(LL)) + T(LL)
Z = FRAC * (Z2 - Z1) + Z1
GO TO 190
40 Z = FRAC * (T(LZ+NY+1) - T(LZ+NY)) + T(LZ+NY)
GO TO 190
C---- BASIC SEARCH ROUTINE USING HALVING TECHNIQUE
50 CONTINUE
C---- TEST IF INDEPENDENT VARIABLE EXCEEDS BOUNDS
IF (VAR.LE.T(LU)) GO TO 60
LL = LU - 1
IF (LEX.GT.0) VAR = T(LL)
GO TO 70
60 IF (VAR.GE.T(LL)) GO TO 80
IF (LEX.GT.0) VAR = T(LL)
70 IE = 1
GO TO 180
C---- TEST IF SAVED INDEX IS WITHIN TABLE BOUNDS
80 LLS = MAXO(MINO(LLS,LU),LL)
IF (LLS.GT.LL) LLS = LLS - 1
LK = MINO(LLS+2,LU)
IF (VAR-T(LLS)) 120,110,90
90 IF (VAR-T(LLS+1)) 110,110,100
100 IF (LLS.EQ.LK) GO TO 130
LLS = LLS + 1
GO TO 90
110 LL = LLS
GO TO 180
120 LU = LLS
GO TO 140
130 LL = LLS + 1
140 IF (LU-LL.EQ.1) GO TO 180
MID = (LU + LL) / 2.
D = VAR - T(MID)
IF (D) 150,170,160
150 LU = MID
GO TO 140
160 LL = MID
GO TO 140
170 LL = MID
180 CONTINUE
FRAC = (VAR - T(LL)) / (T(LL+1) - T(LL))
IF (NRIN) 10,10,30
190 RETURN
END
SUBROUTINE ITMOM(C, D, NAREA, XMOM, YBAR,
1Y, XI, E, NTIME, A, DX, DY,
2STRANE, THETA, STRES, YNA, I, INDFD, NCON,
3NSECT, ITCON)
DIMENSION C(4), D(4), XMOM(10), Y(50)
1 DIMENSION YNA(10), A(60), FORCE(60)
DIMENSION STRES(10,60), STRANE(10,60), E(10,60)
```

IBLK 270  
IBLK 280  
IBLK 290  
IBLK 300  
IBLK 310  
IBLK 320  
IBLK 330  
IBLK 340  
IBLK 350  
IBLK 360  
IBLK 370  
IBLK 380  
IBLK 390  
IBLK 400  
IBLK 410  
IBLK 420  
IBLK 430  
IBLK 440  
IBLK 450  
IBLK 460  
IBLK 470  
IBLK 480  
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IBLK 500  
IBLK 510  
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IBLK 580  
IBLK 590  
IBLK 600  
IBLK 610  
IBLK 620  
IBLK 630  
IBLK 640  
IBLK 650  
IBLK 660  
IBLK 670  
IBLK 680  
IBLK 690  
IBLK 700  
IBLK 710  
IBLK 720  
IBLK 730  
IBLK 740  
IBLK 750  
IBLK 760  
IBLK 770  
-  
ITIM 10  
ITIM 20  
ITIM 30  
ITIM 40  
ITIM 50  
ITIM 60  
ITIM 70



```
1          THETA(20,10)
COMMON PLAMUM,PRIMUM,INPLA
C----- THIS SUBROUTINE SOLVES FOR ELASTIC STRESS AND STRAIN DISTRIBUTIONS
IF(ITCON.EQ.1)GO TO 10
IF(1-ITCON.EQ.0.AND.0(2).EQ.0..AND.0(3).EQ.0..AND.0(4).EQ.0.)
10  GO TO 10
GO TO 60
10  DO 30 M=1,NTIME
    DO 20 J=1,NAREA
        STRESE(M,J) = XMOM(M)*(YBAR - Y(J))/XI
        STRANE(M,J) = STRESE(M,J)/E(M,J)
    20  CONTINUE
    DO 50 K=1,NTIME
        YNA(K)=YBAR
        THETA(1,K) = (XMOM(K)*DX)/(E(K,1)*XI)
    50  GO TO 220
    DO 70 K=1,NTIME
        THETA(1,K)=(XMOM(K)*DX)/(E(K,NSECT)*XI)
    70  DO 180 M=1,NTIME
        YNA(M)=YBAR
        F1 = 0.0
    90  TOTF = 0.0
        XM = 0.0
        DO 120 J=1,NAREA
            STRANE(M,J) = (YNA(M) - Y(J))*THETA(1,M)/DX
            STRESE(M,J) = STRANE(M,J)*E(M,J)
            FORCEE(J) = STRESE(M,J)*A(J)
            IF(1.EQ.3) GO TO 110
        110 CONTINUE
        TOTF = TOTF + FORCEE(J)
        120 IF(TOTF.LT.0.0.AND.F1.GT.0.0) GO TO 140
            IF(TOTF.GT.0.0.AND.F1.LT.0.0) GO TO 140
            Y1 = YNA(M)
            IF(TOTF.LT.0.0) YNA(M) = YNA(M) + DY
            IF(TOTF.GT.0.0) YNA(M) = YNA(M) - DY
            IF(TOTF.EQ.0.0) GO TO 150
            F1 = TOTF
            GO TO 90
        140 YNA(M) = YNA(M) - (ABS(TOTF)*(YNA(M) - Y1))/(ABS(TOTF) + ABS(F1))
        150 DO 160 K=1,NAREA
            STRANE(M,K) = (YNA(M) - Y(K))*THETA(1,M)/DX
            STRESE(M,K) = STRANE(M,K)*E(M,K)
            FORCEE(K) = STRESE(M,K)*A(K)
        160 XM = XM + ABS(FORCEE(K)*(YNA(M) - Y(K)))
            TOTL = ABS(XM - XMOM(M))/XMOM(M)
            IF(ABS(TOTL).LT..01) GO TO 170
            THETA(1,4) = THETA(1,4)*(XMOM(M)/XM)
            GO TO 80
        170 IF(NITCON.EQ.1) GO TO 190
        180 CONTINUE
        GO TO 220
    190 DO 210 M=2,NTIME
        DO 200 N=1,NAREA
            YNA(M) = YNA(1)
            STRANE(M,N) = STRANE(1,N)*XMOM(M)/XMOM(1)
            STRESE(M,N) = STRESE(1,N)*XMOM(M)/XMOM(1)
        200 THETA(1,M) = THETA(1,1)*XMOM(M)/XMOM(1)
        210 CONTINUE
        220 RETURN
```



```

END
SUBROUTINE OUTPUT (INDGED, TS, TR, NRIB, PITCH, OUTPUT, 10
1 RIBFLG, DEPTH, TC, NCOR, FLAT, EDGE, PHICOR, OUTPUT, 20
2 TZE, NZEE, ZPNED1, ZPNED2, ZEESF, ZEESF1, ZEEFF, OUTPUT, 30
3 ZEEFF1, XLGTH, INDO, RAD, INDO, INDSUP, OUTPUT, 40
4 ALEN, DXTIME, PRESS, TEMP, XPRINT, NTIME, NSTAT, OUTPUT, 50
5 DEFL, NUACYC, KCYLE, DEFL, NAREA, Y, RESP, OUTPUT, 60
6 AM1, AM2, AM3, AM4, AM5, INDELA, PSTAN, PLOAD, OUTPUT, 70
7 XI, YBAR, Z, PANWID, DEFLIN, INEYC, OUTPUT, 80
8 ITIME, IEONTP, IINTP, IDIMEN, ILOAD, IEONST, OUTPUT, 90
9 DIMENSION OXTIME (10), PRESS (10), TEMP (10), XPRINT (10), OUTPUT, 100
10 KCYCLE (10), Y (60), Z (75), PLOAD (10), DEFLIN (10,10), OUTPUT, 110
11 DIMENSION DEFL (10,10), DEFL (10,10), DEFLIN (10,10), OUTPUT, 120
12 DIMENSION RESP (10,60,10), PSTAN (10,60,10), OUTPUT, 130
13 PLAMOM (10,10), PRIMOM (10,10), OUTPUT, 140
14 COMMON PLAMOM, PRIMOM, INDOPLA, OUTPUT, 150
15 NUAPRT = NSTAT, OUTPUT, 160
16 WRITE (6,2190) , OUTPUT, 170
17 WRITE (6,1300) AM1, AM2, AM3, AM4, AM5, OUTPUT, 180
18 GO TO (10,30,50,70), INDGED, OUTPUT, 190
19 IF (IDIMEN.EQ.0) GO TO 20, OUTPUT, 200
20 WRITE (6,2200) TS, TR, NRIB, PITCH, RIBFLG, DEPTH, OUTPUT, 210
21 XI, YBAR, OUTPUT, 220
22 GO TO 80, OUTPUT, 230
23 XI = XI * 2.54 * 2.54 * 2.54 * 2.54, OUTPUT, 240
24 YBAR = YBAR * 2.54, OUTPUT, 250
25 WRITE (6,1310) TS, TR, NRIB, PITCH, RIBFLG, DEPTH, OUTPUT, 260
26 XI, YBAR, OUTPUT, 270
27 GO TO 80, OUTPUT, 280
28 IF (IDIMEN.EQ.0) GO TO 40, OUTPUT, 290
29 WRITE (6,2210) TS, TC, NCOR, PITCH, FLAT, EDGE, PHICOR, DEPTH, OUTPUT, 300
30 XI, YBAR, OUTPUT, 310
31 GO TO 80, OUTPUT, 320
32 XI = XI * 2.54 * 2.54 * 2.54 * 2.54, OUTPUT, 330
33 YBAR = YBAR * 2.54, OUTPUT, 340
34 WRITE (6,1320) TS, TC, NCOR, PITCH, FLAT, EDGE, PHICOR, DEPTH, OUTPUT, 350
35 XI, YBAR, OUTPUT, 360
36 GO TO 80, OUTPUT, 370
37 IF (IDIMEN.EQ.0) GO TO 60, OUTPUT, 380
38 WRITE (6,2220) TS, TZE, NZEE, PITCH, ZPNED1, ZPNED2, DEPTH, ZEESF, OUTPUT, 390
39 ZEESF1, ZEEFF, ZEEFF1, OUTPUT, 400
40 XI, YBAR, OUTPUT, 410
41 GO TO 80, OUTPUT, 420
42 XI = XI * 2.54 * 2.54 * 2.54 * 2.54, OUTPUT, 430
43 YBAR = YBAR * 2.54, OUTPUT, 440
44 WRITE (6,1330) TS, TZE, NZEE, PITCH, ZPNED1, ZPNED2, DEPTH, ZEESF, OUTPUT, 450
45 ZEESF1, ZEEFF, ZEEFF1, OUTPUT, 460
46 XI, YBAR, OUTPUT, 470
47 GO TO 80, OUTPUT, 480
48 CONTINUE, OUTPUT, 490
49 IF (IDIMEN.EQ.0) GO TO 90, OUTPUT, 500
50 WRITE (6,2230) XLGTH, PANWID, OUTPUT, 510
51 GO TO 100, OUTPUT, 520
52 WRITE (6,1340) XLGTH, PANWID, OUTPUT, 530
53 CONTINUE, OUTPUT, 540
54 IF (INDO.EQ.0) GO TO 150, OUTPUT, 550
55 IF (RAD.EQ.0) GO TO 110, OUTPUT, 560
56 WRITE (6,2240) , OUTPUT, 570
57 GO TO 120, OUTPUT, 580

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110 WRITE ( 6,2250 )
120 IF(IDIMEN.EQ.0)GO TO 130
    WRITE(6,2260) BRAD,BWID
    GO TO 140
130 WRITE(6,1350) BRAD,BWID
140 CONTINUE
150 WRITE ( 6,2270 )
    IF(INDPLA.EQ.0)GO TO 220
    WRITE(6,1360)
    WRITE(6,2190)
    WRITE(6,1370)
    IF(IDIMEN.EQ.0)GO TO 170
    WRITE(6,1380)
    WRITE(6,1390) (XPRINT(N),N=1,NSFAT)
    WRITE(6,1400)
    DO 160 M=1,NTIME
        WRITE(6,1410)DXTIME(M),(PRTMOM(I,M),I=1,NSFAT)
160 CONTINUE
    GO TO 210
170 CONTINUE
    DO 180 N=1,NSFAT
180 XPRINT(N)=XPRINT(N)*2.54
        WRITE(6,1420)
        WRITE(6,1390) (XPRINT(N),N=1,NSFAT)
        WRITE(6,1430)
        DO 200 M=1,NTIME
            DO 190 I=1,NSFAT
190 PRTMOM(I,M)=PRTMOM(I,M)*.4536*2.54
200 WRITE(6,1410)DXTIME(M),(PRTMOM(I,M),I=1,NSFAT)
210 CONTINUE
    GO TO 350
220 CONTINUE
    IF(INDLOD.EQ.1) GO TO 240
    WRITE ( 6,2280 )
    IF(INDSUP.EQ.1) GO TO 230
    WRITE ( 6,2290 )
    GO TO 350
230 WRITE ( 6,2300 )
    GO TO 350
240 IF(IDIMEN.EQ.0)GO TO 250
    WRITE(6,1440)ALEN
    GO TO 260
250 WRITE(6,1450)ALEN
260 CONTINUE
    WRITE(6,1460)
    IF(ITIME.EQ.0) GO TO 270
    WRITE(6,1470)
    GO TO 280
270 WRITE(6,1480)
280 CONTINUE
    IF(ILLIAD.EQ.0.)GO TO 290
    WRITE(6,1490)
    GO TO 310
290 WRITE(6,1500)
    DO 300 LL=1,NTIME
300 PLOAD(LL)=PLOAD(LL)*.45359
310 CONTINUE
    IF(IINTP.EQ.0)GO TO 320
    WRITE(6,1510)
```

00TP 590  
00TP 600  
00TP 610  
00TP 620  
00TP 630  
00TP 640  
00TP 650  
00TP 660  
00TP 670  
00TP 680  
00TP 690  
00TP 700  
00TP 710  
00TP 720  
00TP 730  
00TP 740  
00TP 750  
00TP 760  
00TP 770  
00TP 780  
00TP 790  
00TP 800  
00TP 810  
00TP 820  
00TP 830  
00TP 840  
00TP 850  
00TP 860  
00TP 870  
00TP 880  
00TP 890  
00TP 900  
00TP 910  
00TP 920  
00TP 930  
00TP 940  
00TP 950  
00TP 960  
00TP 970  
00TP 980  
00TP 990  
00TP 1000  
00TP 1010  
00TP 1020  
00TP 1030  
00TP 1040  
00TP 1050  
00TP 1060  
00TP 1070  
00TP 1080  
00TP 1090  
00TP 1100  
00TP 1110  
00TP 1120  
00TP 1130  
00TP 1140  
00TP 1150  
00TP 1160  
00TP 1170



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```
GO TO 330
320 WRITE(6,1520)
330 CONTINUE
WRITE(6,1530)
WRITE(6,1560)DXTIME(1),PLOAD(1),TEMP(1)
IF(NTIME.EQ.1) GO TO 340
WRITE(6,2330)(DXTIME(N-1),DXTIME(N),PLOAD(N),TEMP(N),N=2,NTIME)
340 CONTINUE
GO TO 430
350 CONTINUE
WRITE(6,1460)
IF(ITEMP.EQ.0) GO TO 360
WRITE(6,1470)
GO TO 370
360 WRITE(6,1480)
370 CONTINUE
IF(LOAD.EQ.0) GO TO 380
WRITE(6,1550)
GO TO 400
380 WRITE(6,1560)
DO 390 LL=1,NTIME
390 PRESS(LL)=PRESS(LL)*6894.8
400 CONTINUE
IF(IINTP.EQ.0) GO TO 410
WRITE(6,1510)
GO TO 420
410 WRITE(6,1520)
420 CONTINUE
WRITE(6,1570)
WRITE(6,1540)DXTIME(1),PRESS(1),TEMP(1)
IF(NTIME.EQ.1)GO TO 430
WRITE(6,2330)(DXTIME(N-1),DXTIME(N),PRESS(N),TEMP(N),N=2,NTIME)
430 CONTINUE
WRITE(6,2190)
WRITE(6,1580)
WRITE(6,1590)Z(1)
IF(Z(2).EQ.0.)GO TO 440
WRITE(6,1600)Z(2)
440 CONTINUE
IF(Z(3).EQ.0.)GO TO 450
WRITE(6,1610)Z(3)
450 CONTINUE
IF(Z(4).EQ.0.)GO TO 460
WRITE(6,1620)Z(4)
460 CONTINUE
IF(Z(5).EQ.0.)GO TO 470
WRITE(6,1630)Z(5)
470 CONTINUE
IF(Z(6).EQ.0.)GO TO 480
WRITE(6,1640)Z(6)
480 CONTINUE
IF(Z(7).EQ.0.)GO TO 490
WRITE(6,1650)Z(7)
490 CONTINUE
IF(Z(8).EQ.0.)GO TO 500
WRITE(6,1660)Z(8)
500 CONTINUE
IF(Z(9).EQ.0.)GO TO 510
WRITE(6,1670)Z(9)
```

001P1130  
001P1140  
001P1190  
001P1210  
001P1220  
001P1230  
001P1240  
001P1250  
001P1260  
001P1270  
001P1280  
001P1290  
001P1300  
001P1310  
001P1320  
001P1330  
001P1340  
001P1350  
001P1360  
001P1370  
001P1380  
001P1390  
001P1400  
001P1410  
001P1420  
001P1430  
001P1440  
001P1450  
001P1460  
001P1470  
001P1480  
001P1490  
001P1500  
001P1510  
001P1520  
001P1530  
001P1540  
001P1550  
001P1560  
001P1570  
001P1580  
001P1590  
001P1600  
001P1610  
001P1620  
001P1630  
001P1640  
001P1650  
001P1660  
001P1670  
001P1680  
001P1690  
001P1700  
001P1710  
001P1720  
001P1730  
001P1740  
001P1750  
001P1760

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510 CONTINUE
    IF(Z(10).EQ.0.)GO TO 520
    WRITE(6,1680)Z(10)
520 CONTINUE
    IF(Z(11).EQ.0.)GO TO 530
    WRITE(6,1690)Z(11)
530 CONTINUE
    IF(Z(12).EQ.0.)GO TO 540
    WRITE(6,1700)Z(12)
540 CONTINUE
    IF(Z(13).EQ.0.)GO TO 550
    WRITE(6,1710)Z(13)
550 CONTINUE
    IF(Z(14).EQ.0.)GO TO 560
    WRITE(6,1720)Z(14)
560 CONTINUE
    IF(Z(15).EQ.0.)GO TO 570
    WRITE(6,1730)Z(15)
570 CONTINUE
    IF(Z(16).EQ.0.)GO TO 580
    WRITE(6,1740)Z(16)
580 CONTINUE
    IF(Z(17).EQ.0.)GO TO 590
    WRITE(6,1750)Z(17)
590 CONTINUE
    IF(Z(18).EQ.0.)GO TO 600
    WRITE(6,1760)Z(18)
600 CONTINUE
    IF(Z(19).EQ.0.)GO TO 610
    WRITE(6,1770)Z(19)
610 CONTINUE
    IF(Z(20).EQ.0.)GO TO 620
    WRITE(6,1780)Z(20)
620 CONTINUE
    IF(Z(21).EQ.0.)GO TO 630
    WRITE(6,1790)Z(21)
630 CONTINUE
    IF(Z(22).EQ.0.)GO TO 640
    WRITE(6,1800)Z(22)
640 CONTINUE
    IF(Z(23).EQ.0.)GO TO 650
    WRITE(6,1810)Z(23)
650 CONTINUE
    IF(Z(24).EQ.0.)GO TO 660
    WRITE(6,1820)Z(24)
660 CONTINUE
    IF(Z(25).EQ.0.)GO TO 670
    WRITE(6,1830)Z(25)
670 CONTINUE
    IF(Z(26).EQ.0.)GO TO 680
    WRITE(6,1840)Z(26)
680 CONTINUE
    IF(Z(27).EQ.0.)GO TO 690
    WRITE(6,1850)Z(27)
690 CONTINUE
    IF(Z(28).EQ.0.)GO TO 700
    WRITE(6,1860)Z(28)
700 CONTINUE
    IF(Z(29).EQ.0.)GO TO 710

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001P1770
001P1780
001P1790
001P1800
001P1810
001P1820
001P1830
001P1840
001P1850
001P1860
001P1870
001P1880
001P1890
001P1900
001P1910
001P1920
001P1930
001P1940
001P1950
001P1960
001P1970
001P1980
001P1990
001P2000
001P2010
001P2020
001P2030
001P2040
001P2050
001P2060
001P2070
001P2080
001P2090
001P2100
001P2110
001P2120
001P2130
001P2140
001P2150
001P2160
001P2170
001P2180
001P2190
001P2200
001P2210
001P2220
001P2230
001P2240
001P2250
001P2260
001P2270
001P2280
001P2290
001P2300
001P2310
001P2320
001P2330
001P2340
001P2350

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710 WRITE(6,1870)Z(29)  
CONTINUE  
IF(Z(31),EQ,0.)GO TO 720  
WRITE(6,1880)Z(30)  
720 CONTINUE  
IF(Z(31),EQ,0.)GO TO 730  
WRITE(6,1890)Z(31)  
730 CONTINUE  
IF(Z(32),EQ,0.)GO TO 740  
WRITE(6,1900)Z(32)  
740 CONTINUE  
IF(Z(33),EQ,0.)GO TO 750  
WRITE(6,1910)Z(33)  
750 CONTINUE  
IF(Z(34),EQ,0.)GO TO 760  
WRITE(6,1920)Z(34)  
760 CONTINUE  
IF(Z(35),EQ,0.)GO TO 770  
WRITE(6,1930)Z(35)  
770 CONTINUE  
IF(Z(36),EQ,0.)GO TO 780  
WRITE(6,1940)Z(36)  
780 CONTINUE  
IF(Z(37),EQ,0.)GO TO 790  
WRITE(6,1950)Z(37)  
790 CONTINUE  
IF(Z(38),EQ,0.)GO TO 800  
WRITE(6,1960)Z(38)  
800 CONTINUE  
IF(Z(39),EQ,0.)GO TO 810  
WRITE(6,1970)Z(39)  
810 CONTINUE  
IF(Z(40),EQ,0.)GO TO 820  
WRITE(6,1980)Z(40)  
820 CONTINUE  
IF(Z(41),EQ,0.)GO TO 830  
WRITE(6,1990)Z(41)  
830 CONTINUE  
IF(Z(42),EQ,0.)GO TO 840  
WRITE(6,2000)Z(42)  
840 CONTINUE  
IF(Z(43),EQ,0.)GO TO 850  
WRITE(6,2010)Z(43)  
850 CONTINUE  
IF(Z(44),EQ,0.)GO TO 860  
WRITE(6,2020)Z(44)  
860 CONTINUE  
IF(Z(45),EQ,0.)GO TO 870  
WRITE(6,2030)Z(45)  
870 CONTINUE  
IF(111111,EQ,0.)GO TO 880  
WRITE(6,2040)  
GO TO 890  
880 WRITE(6,2050)  
890 CONTINUE  
IF(111111,EQ,0.)GO TO 900  
WRITE(6,2060)  
GO TO 910  
900 WRITE(6,2070)

001P2350  
001P2370  
001P2380  
001P2390  
001P2400  
001P2410  
001P2420  
001P2430  
001P2440  
001P2450  
001P2460  
001P2470  
001P2480  
001P2490  
001P2500  
001P2510  
001P2520  
001P2530  
001P2540  
001P2550  
001P2560  
001P2570  
001P2580  
001P2590  
001P2600  
001P2610  
001P2620  
001P2630  
001P2640  
001P2650  
001P2660  
001P2670  
001P2680  
001P2690  
001P2700  
001P2710  
001P2720  
001P2730  
001P2740  
001P2750  
001P2760  
001P2770  
001P2780  
001P2790  
001P2800  
001P2810  
001P2820  
001P2830  
001P2840  
001P2850  
001P2860  
001P2870  
001P2880  
001P2890  
001P2900  
001P2910  
001P2920  
001P2930  
001P2940

910	CONTINUE	001P2450
	IF(IEONST.EQ.0)GO TO 920	001P2460
	WRITE(6,2080)	001P2470
	GO TO 930	001P2480
920	WRITE(6,2090)	001P2490
930	CONTINUE	001P2500
	IF(IDI4=4.EQ.0)GO TO 940	001P2510
	WRITE(6,2360)	001P2520
	IF(IEI4E.EQ.0)GO TO 940	001P2530
	WRITE(6,2350) (XPRINT(N),N=1,NSSTAT)	001P2540
	GO TO 950	001P2550
940	WRITE(6,2100) (XPRINT(N),N=1,NSSTAT)	001P2560
950	CONTINUE	001P2570
	DO 960 M=1,NTIME	001P2580
	WRITE(6,2360) DXTIME(M),(DEFLE(M,N),N=1,NSSTAT)	001P2590
960	CONTINUE	001P2600
	GO TO 1030	001P2610
970	CONTINUE	001P2620
	WRITE(6,2110)	001P2630
	IF(ETIME.EQ.0)GO TO 980	001P2640
	WRITE(6,2350) (XPRINT(N),N=1,NSSTAT)	001P2650
	GO TO 990	001P2660
980	WRITE(6,2100) (XPRINT(N),N=1,NSSTAT)	001P2670
990	CONTINUE	001P2680
	DO 1010 M=1,NTIME	001P2690
	DO 1000 N=1,NSSTAT	001P2700
	DEFLE(M,N)=DEFLE(M,N)*2.54	001P2710
1000	CONTINUE	001P2720
1010	CONTINUE	001P2730
	DO 1020 M=1,NTIME	001P2740
	WRITE(6,2360) DXTIME(M),(DEFLE(M,N),N=1,NSSTAT)	001P2750
1020	CONTINUE	001P2760
1030	CONTINUE	001P2770
	IF(INDELA.EQ.1) RETURN	001P2780
	IF(INCYC.EQ.0) GO TO 1100	001P2790
	IF(IDIMEN.EQ.0)GO TO 1050	001P2800
	WRITE(6,2120)	001P2810
	WRITE(6,2130) (XPRINT(N),N=1,NSSTAT)	001P2820
	DO 1040 M=1,NTIME	001P2830
	WRITE(6,2140) DXTIME(M),(DEFLIN(M,N),N=1,NSSTAT)	001P2840
1040	CONTINUE	001P2850
	GO TO 1090	001P2860
1050	WRITE(6,2150)	001P2870
	WRITE(6,2130) (XPRINT(N),N=1,NSSTAT)	001P2880
	DO 1070 M=1,NTIME	001P2890
	DO 1060 N=1,NSSTAT	001P2900
	DEFLIN(M,N)=DEFLIN(M,N)*2.54	001P2910
1060	CONTINUE	001P2920
1070	CONTINUE	001P2930
	DO 1080 M=1,NTIME	001P2940
	WRITE(6,2140) DXTIME(M),(DEFLIN(M,N),N=1,NSSTAT)	001P2950
1080	CONTINUE	001P2960
1090	CONTINUE	001P2970
1100	CONTINUE	001P2980
	IF(IDIMEN.EQ.0)GO TO 1120	001P2990
	WRITE(6,2370)	001P3000
	WRITE(6,2380) (XPRINT(N),N=1,NSSTAT)	001P3010
	DO 1110 M=1,NUMCYC	001P3020
	WRITE(6,2390) KCYCLE(M),(DEFL(M,N),N=1,NSSTAT)	001P3030



```
1110 CONTINUE
1120 GO TO 1160
1120 WRITE(6,2160)
      WRITE(6,2380) (XPRINT(N),N=1,NSSTAT)
      DO 1140 N=1,NUMCYC
      DO 1130 M=1,NSFAT
      DEFL(M,N)=DEFL(M,N)*2.54
1130 CONTINUE
1140 CONTINUE
      DO 1150 M=1,NUMCYC
      WRITE(6,2390) KCYCLE(M),(DEFL(M,N),N=1,NSFAT)
1150 CONTINUE
1160 CONTINUE
      IF(101MEN.EQ.1)GO TO 1180
      DO 1170 N=1,NAREA
      Y(N)=Y(N)*2.54
1170 CONTINUE
      DO 1190 N1=1,NUMCYC
      IF(101MEN.EQ.0) GO TO 1190
      GO TO 1200
1190 WRITE(6,2170) KCYCLE(N1)
      GO TO 1210
1200 WRITE(6,2430) KCYCLE(N1)
1210 CONTINUE
      WRITE(6,2440) (XPRINT(N),N=1,NSFAT)
      DO 1220 N2=1,NAREA
      WRITE(6,2450) Y(N2), (PSFRAN(N3,N2,N1),N3=1,NSFAT)
1220 CONTINUE
      IF(101MEN.EQ.0)GO TO 1230
      GO TO 1260
1230 WRITE(6,2180) KCYCLE(N1)
      DO 1250 L2=1,NAREA
      DO 1240 L3=1,NSFAT
      RESP(L3,L2,N1)=RESP(L3,L2,N1)*.0068948
1240 CONTINUE
1250 CONTINUE
      GO TO 1270
1260 WRITE(6,2400) KCYCLE(N1)
1270 CONTINUE
      WRITE(6,2410) (XPRINT(N),N=1,NSFAT)
      DO 1280 N2=1,NAREA
      WRITE(6,2420) Y(N2), (RESP(N3,N2,N1),N3=1,NSFAT)
1280 CONTINUE
1290 CONTINUE
      RETURN
1300 FORMAT(50X,5A10,/)
1310 FORMAT(43X,23HRIR STIFFENED TPS PANEL//53X,12HSKIN GAGE = ,F5.4,
17H CM /53X,11HRIR GAGE = ,F5.4,7H CM /53X,17HNUMBER OF RIBS
2= ,I3/53X,15HPITCH LENGTH = ,F6.3,7H CM /53X,20HPANEL EDGE LENGTH
3TH = ,F6.3,7H CM /53X,14HPANEL DEPTH = ,F5.3,7H CM /,
453X,31HCALCULATED MOMENT OF INERTIA = ,F10.7,6H CM**4/
553X,23HELASTIC NEUTRAL AXIS = ,F5.3,7H CM /)
1320 FORMAT(43X,28HSINGLE FACED CORRUGATION TPS//53X,12HSKIN GAGE = ,
1F5.4,7H CM /53X,19HCORRUGATION GAGE = ,F5.4,7H CM /53X,
225HNUMBER OF CORRUGATIONS = ,I3/53X,15HPITCH LENGTH = ,F6.3,
37H CM /53X,14HFLAT LENGTH = ,F6.3,7H CM /53X,
420HPANEL EDGE LENGTH = ,F6.3,7H CM /53X,20HCORRUGATION ANGLE =
5,F6.3,8H DEGREES/53X,14HPANEL DEPTH = ,F5.3,7H CM /,
653X,31HCALCULATED MOMENT OF INERTIA = ,F10.7,6H CM**4/
      UUTP 3540
      UUTP 3550
      UUTP 3560
      UUTP 3570
      UUTP 3580
      UUTP 3590
      UUTP 3600
      UUTP 3610
      UUTP 3620
      UUTP 3630
      UUTP 3640
      UUTP 3650
      UUTP 3660
      UUTP 3670
      UUTP 3680
      UUTP 3690
      UUTP 3700
      UUTP 3710
      UUTP 3720
      UUTP 3730
      UUTP 3740
      UUTP 3750
      UUTP 3760
      UUTP 3770
      UUTP 3780
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      UUTP 3920
      UUTP 3930
      UUTP 3940
      UUTP 3950
      UUTP 3960
      UUTP 3970
      UUTP 3980
      UUTP 3990
      UUTP 4000
      UUTP 4010
      UUTP 4020
      UUTP 4030
      UUTP 4040
      UUTP 4050
      UUTP 4060
      UUTP 4070
      UUTP 4080
      UUTP 4090
      UUTP 4100
      UUTP 4110
      UUTP 4120
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753X,23HELASTIC NEUTRAL AXIS = ,F5.3,7H CM /) UU TP4130
1330 FORMAT(43X,21HZ STIFFENED TPS PANEL//53X,12H SKIN GAGE = ,F5.4, UU TP4140
17H CM /53X,9HZ GAGE = ,F5.4,7H CM /53X,25H NUMBER OF Z STIFFENERS UU TP4150
2NERS = ,I3/53X,15H PLATE LENGTH = ,F6.3,7H CM /53X, UU TP4160
32HPANEL EDGE LENGTHS = ,F5.3,5H AND ,F6.3,7H CM /53X, UU TP4170
41HPANEL DEPTH = ,F5.3,7H CM /53X,27HZ FLANGE LENGTHS AT SKIN = UU TP4180
5 ,F5.3,5H AND ,F5.3,7H CM /53X,24HZ FREE FLANGE LENGTHS = , UU TP4190
6F5.3,5H AND ,F5.3,7H CM /, UU TP4200
753X,31H CALCULATED MOMENT OF INERTIA = ,F10.7,6H CM**4/ UU TP4210
853X,23HELASTIC NEUTRAL AXIS = ,F5.3,7H CM /) UU TP4220
1340 FORMAT(43X,15HPANEL LENGTH = ,F6.2,7H CM /) UU TP4230
1,43X,14HPANEL WIDTH = ,F6.2,7H CM /) UU TP4240
1350 FORMAT(53X,9H RADIUS = ,F6.3,7H CM /53X,9H WIDTH = ,F5.3, UU TP4250
17H CM /) UU TP4260
1360 FORMAT(53X,31H UNIFORM PRESSURE (PLATE OPTION)) UU TP4270
1370 FORMAT(52X,27H BENDING MOMENT DISTRIBUTION) UU TP4280
1380 FORMAT(77/25X,21H BEAM STATION (INCHES)) UU TP4290
1390 FORMAT(711X,4H TIME,9X,10(F5.2,5X)/) UU TP4300
1400 FORMAT(62X,7H (IN LBS)) UU TP4310
1410 FORMAT(10X,F6.2,4X,10(F10.2)) UU TP4320
1420 FORMAT(77/25X,17H BEAM STATION (CM)) UU TP4330
1430 FORMAT(60X,10H (CM KILOS)) UU TP4340
1440 FORMAT(53X,50H TWO POINT LOADS , DISTANCE FROM SUPPORT TO LOAD = , UU TP4350
1F5.2,7H INCHES//) UU TP4360
1450 FORMAT(53X,50H TWO POINT LOADS , DISTANCE FROM SUPPORT TO LOAD = , UU TP4370
1F5.2,7H CM//) UU TP4380
1460 FORMAT(42X,77/77,16H TRAJECTORY DATA//) UU TP4390
1470 FORMAT(45X,15H TIME (SECONDS)) UU TP4400
1480 FORMAT(45X,15H TIME (MINUTES)) UU TP4410
1490 FORMAT(45X,11H LOAD (LBS)) UU TP4420
1500 FORMAT(45X,13H LOAD (KILOS)) UU TP4430
1510 FORMAT(45X,20H TEMPERATURE (DEG F)//) UU TP4440
1520 FORMAT(45X,20H TEMPERATURE (DEG C)//) UU TP4450
1530 FORMAT(55X,4H TIME,16X,10H LOAD ,6X,12H MIDSPAN SKIN// UU TP4460
151X,5H START,4X,3H END,24X,11H TEMPERATURE//) UU TP4470
1540 FORMAT(52X,4H0.00,3X,F6.2,11X,F8.3,13X,F6.1) UU TP4480
1550 FORMAT(46X,14H PRESSURE (PSI)) UU TP4490
1560 FORMAT(46X,13H PRESSURE (PA)) UU TP4500
1570 FORMAT(55X,4H TIME,16X,10H PRESSURE ,6X,12H MIDSPAN SKIN// UU TP4510
151X,5H START,4X,3H END,24X,11H TEMPERATURE//) UU TP4520
1580 FORMAT(77/77,49X,32H CYCLIC CREEP EQUATION DEFINITION //) UU TP4530
1590 FORMAT(40X,12H LN(STRAIN) = ,F12.5) UU TP4540
1600 FORMAT(52X,F12.5,10H *(STRESS)) UU TP4550
1610 FORMAT(52X,F12.5,8H *(TEMP)) UU TP4560
1620 FORMAT(52X,F12.5,8H *(TIME)) UU TP4570
1630 FORMAT(52X,F12.5,11H *(1./TEMP)) UU TP4580
1640 FORMAT(52X,F12.5,11H *(LN(TIME)) UU TP4590
1650 FORMAT(52X,F12.5,13H *(LN(STRESS)) UU TP4600
1660 FORMAT(52X,F12.5,11H *(LN(TEMP)) UU TP4610
1670 FORMAT(52X,F12.5,13H *(STRESS)**2) UU TP4620
1680 FORMAT(52X,F12.5,13H *(STRESS)**3) UU TP4630
1690 FORMAT(52X,F12.5,14H *(1./TEMP)**2) UU TP4640
1700 FORMAT(52X,F12.5,14H *(1./TEMP)**3) UU TP4650
1710 FORMAT(52X,F12.5,16H *(STRESS)(TEMP)) UU TP4660
1720 FORMAT(52X,F12.5,17H *(STRESS)/(TEMP)) UU TP4670
1730 FORMAT(52X,F12.5,18H *(STRESS*TEMP)**2) UU TP4680
1740 FORMAT(52X,F12.5,18H *(STRESS*TEMP)**3) UU TP4690
1750 FORMAT(52X,F12.5,18H *(STRESS/TEMP)**2) UU TP4700
1760 FORMAT(52X,F12.5,18H *(STRESS/TEMP)**3) UU TP4710
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1770 FORMAT(52X,E12.5,17H *(LN(STRESS))**2)
1780 FORMAT(52X,E12.5,17H *(LN(STRESS))**3)
1790 FORMAT(52X,E12.5,17H *STRESS*LN(TEMP))
1800 FORMAT(52X,E12.5,17H *TEMP*LN(STRESS))
1810 FORMAT(52X,E12.5,21H *LN(STRESS)*LN(TEMP))
1820 FORMAT(52X,E12.5,18H *TIME*STRESS*TEMP)
1830 FORMAT(52X,E12.5,23H *(TIME*STRESS*(TEMP)**2))
1840 FORMAT(52X,E12.5,23H *(TIME*STRESS*TEMP)**3)
1850 FORMAT(52X,E12.5,11H *(TIME)**2)
1860 FORMAT(52X,E12.5,11H *(TIME)**3)
1870 FORMAT(52X,E12.5,15H *(LN(TIME))**2)
1880 FORMAT(52X,E12.5,15H *(LN(TIME))**3)
1890 FORMAT(52X,E12.5,11H *(TEMP)**2)
1900 FORMAT(52X,E12.5,11H *(TEMP)**3)
1910 FORMAT(52X,E12.5,15H *(LN(TEMP))**2)
1920 FORMAT(52X,E12.5,15H *(LN(TEMP))**3)
1930 FORMAT(52X,E12.5,17H *STRESS*LN(TIME))
1940 FORMAT(52X,E12.5,15H *TEMP*LN(TIME))
1950 FORMAT(52X,E12.5,17H *TIME*LN(STRESS))
1960 FORMAT(52X,E12.5,15H *TIME*LN(TEMP))
1970 FORMAT(52X,E12.5,21H *LN(STRESS)*LN(TIME))
1980 FORMAT(52X,E12.5,19H *LN(TIME)*LN(TEMP))
1990 FORMAT(52X,E12.5,19H *(LN(STRESS))/(TEMP))
2000 FORMAT(52X,E12.5,17H *(LN(TIME))/(TEMP))
2010 FORMAT(52X,E12.5,30H *(LN(TIME))*(LN(STRESS))/(TEMP))
2020 FORMAT(52X,E12.5,13H *TIME*STRESS)
2030 FORMAT(52X,E12.5,11H *TIME*TEMP)
2040 FORMAT(77,50X,24HWHERE TIME = MINUTES)
2050 FORMAT(77,50X,24HWHERE TIME = HOURS)
2060 FORMAT(60X,25HTEMPERATURE = DEG F/1000.)
2070 FORMAT(60X,25HTEMPERATURE = DEG C/1000.)
2080 FORMAT(60X,12HSTRESS = KSI)
2090 FORMAT(60X,12HSTRESS = MPA)
2100 FORMAT(13X,9HTIME(HR),3X,10(F5.2,5X))
2110 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//51X,
126HELASTIC DEFLECTION SUMMARY////23X,21HBEAM STATION (CM)
2120 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//47X,
136HFIRST CYCLE CREEP DEFLECTION SUMMARY////23X,
221HBEAM STATION (INCHES))
2130 FORMAT(13X,4HTIME,9X,10(F5.2,6X))
2140 FORMAT(11X,F7.2,6X,10(F10.5,1X))
2150 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//47X,
136HFIRST CYCLE CREEP DEFLECTION SUMMARY////23X,
221HBEAM STATION (CM))
2160 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//53X,
124HCREEP DEFLECTION SUMMARY////23X,21HBEAM STATION (CM))
2170 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//57X,
123HCREEP STRAINS (PERCENT)//64X,6HCYCLE,13//25X,
221HBEAM STATION (C4))
2180 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//57X,
123HRESIDUAL STRESSES (MPA)//64X,6HCYCLE,13//25X,
221HBEAM STATION (CM))
2190 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//)
2200 FORMAT(43X,23HRIE STIFFENED TPS PANEL//53X,12HSKIN GAGE = ,F5.4,
17H INCHES/53X,11HRIE GAGE = ,F5.4,7H INCHES/53X,17HNUMBER OF RIBS
2= ,13/53X,15HPITCH LENGTH = ,F6.3,7H INCHES/53X,20HPANEL EDGE LENGTH
3TH = ,F6.3,7H INCHES/53X,14HPANEL DEPTH = ,F5.3,7H INCHES/,
453X,31HCALCULATED MOMENT OF INERTIA = ,F10.7,6H IN**4/,
553X,23HELASTIC NEUTRAL AXIS = ,F5.3,7H INCHES/)
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001P4720  
001P4730  
001P4740  
001P4750  
001P4760  
001P4770  
001P4780  
001P4790  
001P4800  
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001P4980  
001P4990  
001P5000  
001P5010  
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001P5200  
001P5210  
001P5220  
001P5230  
001P5240  
001P5250  
001P5260  
001P5270  
001P5280  
001P5290  
001P5300



2210 FORMAT(43X,2HBSINGLE FACED CORRUGATION TPS//53X,12HSKIN GAGE = , OUTP5310  
F5.4,7H INCHES/53X,19H CORRUGATION GAGE = ,F5.4,7H INCHES/53X, OUTP5320  
225HNUMBER OF CORRUGATIONS = ,13/53X,15HPITCH LENGTH = ,F6.3, OUTP5330  
37H INCHES/53X,14HFLAT LENGTH = ,F6.3,7H INCHES/53X, OUTP5340  
420HPANEL EDGE LENGTH = ,F5.3,7H INCHES/53X,20H CORRUGATION ANGLE = OUTP5350  
5,F6.3,8H DEGREES/53X,14HPANEL DEPTH = ,F5.3,7H INCHES/, OUTP5360  
653X,31H CALCULATED MOMENT OF INERTIA = ,F10.7,6H IN\*\*4/, OUTP5370  
753X,23HELASTIC NEUTRAL AXIS = ,F5.3,7H INCHES/, OUTP5380  
2220 FORMAT(43X,21HZ STIFFENED TPS PANEL//53X,12HSKIN GAGE = ,F5.4, OUTP5390  
17H INCHES/53X,9HZ GAGE = ,F5.4,7H INCHES/53X,25HNUMBER OF Z STIFFENERS = ,13/53X,15HPITCH LENGTH = ,F6.3,7H INCHES/53X, OUTP5400  
321HPANEL EDGE LENGTHS = ,F6.3,5H AND ,F6.3,7H INCHES/53X, OUTP5410  
414HPANEL DEPTH = ,F5.3,7H INCHES/53X,27HZ FLANGE LENGTHS AT SKIN = OUTP5420  
5 ,F5.3,5H AND ,F5.3,7H INCHES/53X,24HZ FREE FLANGE LENGTHS = , OUTP5430  
6F5.3,5H AND ,F5.3,7H INCHES/, OUTP5440  
753X,31H CALCULATED MOMENT OF INERTIA = ,F10.7,6H IN\*\*4/, OUTP5450  
853X,23HELASTIC NEUTRAL AXIS = ,F5.3,7H INCHES/, OUTP5460  
2230 FORMAT(43X,15HPANEL LENGTH = ,F6.2,7H INCHES/, OUTP5470  
1,43X,14HPANEL WIDTH = ,F5.2,7H INCHES/, OUTP5480  
2240 FORMAT(43X,13HNEGATIVE BEAD) OUTP5490  
2250 FORMAT(43X,13HPOSITIVE BEAD) OUTP5500  
2260 FORMAT(53X,9HRADIUS = ,F6.3,7H INCHES/53X,9HWIDTH = ,F6.3, OUTP5510  
17H INCHES/, OUTP5520  
2270 FORMAT(43X,13HAPPLIED LOADS) OUTP5530  
2280 FORMAT(53X,15HUNIFORM PRESSURE) OUTP5540  
2290 FORMAT(53X,15HSIMPLE SUPPORTS//) OUTP5550  
2300 FORMAT(53X,14HFIXED SUPPORTS//) OUTP5560  
2330 FORMAT(50X,F6.2,3X,F6.2,11X,F8.3,13X,F6.1) OUTP5570  
2340 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//51X, OUTP5580  
126HELASTIC DEFLECTION SUMMARY///23X,21HBEAM STATION (INCHES)) OUTP5590  
2350 FORMAT(13X,9H14E(MIN),3X,10(F5.2,5X)) OUTP5600  
2360 FORMAT(13X,F7.2,3X,10(F7.4,3X)) OUTP5610  
2370 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//53X, OUTP5620  
124HCREEP DEFLECTION SUMMARY///23X,21HBEAM STATION (INCHES)) OUTP5630  
2380 FORMAT(13X,5HCYCLE,8X,10(F5.2,6X)) OUTP5640  
2390 FORMAT(13X,13,8X,10(F10.5,1X)) OUTP5650  
2400 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//57X, OUTP5660  
123HRESIDUAL STRESSES (PSI)//64X,6HCYCLE ,13///25X, OUTP5670  
221HBEAM STATION (INCHES)) OUTP5680  
2410 FORMAT(3X,6HHEIGHT,6X,10(F6.2,5X)) OUTP5690  
2420 FORMAT(3X,F6.4,5X,10(F10.2,1X)) OUTP5700  
2430 FORMAT(1H1,49X,33HCREEP PREDICTION COMPUTER PROGRAM//57X, OUTP5710  
123HCREEP STRAINS (PERCENT)//64X,6HCYCLE ,13///25X, OUTP5720  
221HBEAM STATION (INCHES)) OUTP5730  
2440 FORMAT(13X,6HHEIGHT,6X,10(F6.2,5X)) OUTP5740  
2450 FORMAT(3X,F6.4,5X,10(F10.7,1X)) OUTP5750  
END